



TRUCK DESIGN OPTIMIZATION PROJECT PHASE II

PHASE I DATA EVALUATION AND ANALYSIS REPORT

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AUGUST 1979

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Springfield, Virginia 22161

Prepared for
U.S. DEPARTMENT OF TRANSPORTATION

Federal Railroad Administration
Office of Research and Development
Washington, D.C. 20590

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1. Report No. FRA/ORD-78/52	2. Government Accession No.	3. Recipient's Catalog No.	
4. Title and Subtitle TRUCK DESIGN OPTIMIZATION PROJECT (TDOP) PHASE II Phase I Data Evaluation and Analysis Report		5. Report Date August 1979	
		6. Performing Organization Code	
7. Author(s) David W. Gibson and Robert J. Glaser		8. Performing Organization Report No. TDOP TR-02	
9. Performing Organization Name and Address Wyle Laboratories Scientific Services & Systems Group 4620 Edison Avenue Colorado Springs, CO 80915		10. Work Unit No. (TRAIS)	
		11. Contract or Grant No. DOT-FR-742-4277	
12. Sponsoring Agency Name and Address Department of Transportation Federal Railroad Administration (FRA) Office of Research and Development Washington, DC 20590		13. Type of Report and Period Covered Technical Report Dec. 1977 - April 1979	
		14. Sponsoring Agency Code FRA/RRD-12	
15. Supplementary Notes See FRA Report No. FRA/ORD-70/34, Phase I Data Evaluation and Analysis Plan, September 1978			
16. Abstract As part of the TDOP Phase II project, the FRA directed Wyle Laboratories to evaluate and analyze the test data acquired during TDOP Phase I for use in Phase II model validation and specification of performance indices. These data were contained on 204 magnetic tapes and computer printouts. The applicability of the Phase I test data to Phase II was evaluated from three points of view. The first was completeness of the test matrix. Most Phase I tests were conducted using the 70 ton refrigerator car on an ASF ride control truck with new wheel profiles. Although this over-emphasis on one configuration will necessitate additional testing of the Type I truck, it was possible to derive useful information from the Phase I test data. The second was measurement accuracy. The quality of measurements was acceptable except for measurements of lateral wheel force at the wheel/rail interface and in the detection of ALD targets. The third point of view was the Phase I data's adequacy to perform the Type I truck model validation and specification of performance indices. The data in the regimes of ride quality and lateral stability appear to be adequate. In the regimes of curve negotiation and trackability, the lack of adequate measurements of wheel/rail forces makes it difficult to extract meaningful information from the data.			
17. Key Words Truck Design Optimization Project (TDOP), Phase II, Phase I Data Evaluation and Analysis, Freight Car Trucks (Type I), Railroad Technology		18. Distribution Statement Document is available to the public through the National Technical Infor- mation Service, Springfield, VA 22161	
19. Security Classif. (of this report) Unclassified	20. Security Classif. (of this page) Unclassified	21. No. of Pages 82	22. Price

METRIC CONVERSION FACTORS

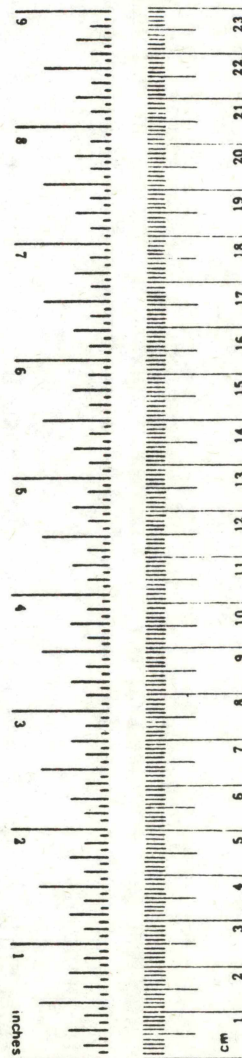
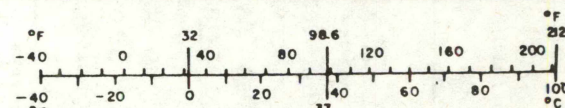
Approximate Conversions to Metric Measures

Symbol	When You Know	Multiply by	To Find	Symbol
LENGTH				
in	inches	2.5	centimeters	cm
ft	feet	30	centimeters	cm
yd	yards	0.9	meters	m
mi	miles	1.6	kilometers	km
AREA				
in ²	square inches	6.5	square centimeters	cm ²
ft ²	square feet	0.09	square meters	m ²
yd ²	square yards	0.8	square meters	m ²
mi ²	square miles	2.6	square kilometers	km ²
	acres	0.4	hectares	ha
MASS (weight)				
oz	ounces	28	grams	g
lb	pounds	0.45	kilograms	kg
	short tons (2000 lb)	0.9	tonnes	t
VOLUME				
tsp	teaspoons	5	milliliters	ml
Tbsp	tablespoons	15	milliliters	ml
fl oz	fluid ounces	30	milliliters	ml
c	cups	0.24	liters	l
pt	pints	0.47	liters	l
qt	quarts	0.95	liters	l
gal	gallons	3.8	liters	l
ft ³	cubic feet	0.03	cubic meters	m ³
yd ³	cubic yards	0.76	cubic meters	m ³
TEMPERATURE (exact)				
°F	Fahrenheit temperature	5/9 (after subtracting 32)	Celsius temperature	°C

* 1 in = 2.54 (exactly). For other exact conversions and more detailed tables, see NBS Misc. Publ. 286, Units of Weights and Measures, Price \$2.25, SD Catalog No. C13.10-286.

Approximate Conversions from Metric Measures

Symbol	When You Know	Multiply by	To Find	Symbol
LENGTH				
mm	millimeters	0.04	inches	in
cm	centimeters	0.4	inches	in
m	meters	3.3	feet	ft
m	meters	1.1	yards	yd
km	kilometers	0.6	miles	mi
AREA				
cm ²	square centimeters	0.16	square inches	in ²
m ²	square meters	1.2	square yards	yd ²
km ²	square kilometers	0.4	square miles	mi ²
ha	hectares (10,000 m ²)	2.5	acres	
MASS (weight)				
g	grams	0.035	ounces	oz
kg	kilograms	2.2	pounds	lb
t	tonnes (1000 kg)	1.1	short tons	
VOLUME				
ml	milliliters	0.03	fluid ounces	fl oz
l	liters	2.1	pints	pt
l	liters	1.06	quarts	qt
l	liters	0.26	gallons	gal
m ³	cubic meters	35	cubic feet	ft ³
m ³	cubic meters	1.3	cubic yards	yd ³
TEMPERATURE (exact)				
°C	Celsius temperature	9/5 (then add 32)	Fahrenheit temperature	°F



EXECUTIVE SUMMARY

The purpose of this report is to evaluate and analyze the TDOP Phase I test data for its applicability to the TDOP Phase II project. Specifically, the evaluation will determine if the Phase I data can be used in Phase II model validation and performance indices specification.

The report discusses the three approaches used to determine the usefulness of the Phase I data. First, the quantity and scope of the data was evaluated. Using a data sorting routine, a series of matrices was developed. This analysis showed that the preponderance of Phase I testing was conducted on the 70-ton refrigerator car with the ASF truck and new wheels. Since the refrigerator car is not typical of most cars in service, reliance on the data may well bias the results of the Phase II analytical work.

Secondly, the evaluation determined if the Phase I measurements accurately represent the quantity measured. For example, did the vertical accelerometers on the carbody give an accurate representation of car bounce? The conclusion was that the measurements was satisfactory except in two areas: the measurement of lateral wheel forces at the wheel/rail interface, and the detection of automatic location detector (ALD) devices. The first deficiency is of major significance. Without the data on the lateral wheel forces, the Phase I data cannot be used in validating the various curving models or in assessing the curve negotiation performance indices of the Type I truck. The lack of precise ALD target locations limits the usefulness of the Phase I data for trackability regime analysis and, to a lesser degree, ride quality analysis.

Finally, the Phase I data were evaluated for its adequacy in performing the Type I truck model validation and specification of performance indices. In other words, what data are required versus what data are available from TDOP Phase I. For the lateral stability and ride quality regimes, the data appear to be adequate; however, the lack of accurate measurements on the lateral forces at the wheel/rail interface will make it difficult to extract from the data meaningful information for the curve negotiation and trackability regimes.

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SECTION 1 - INTRODUCTION

1.1 BACKGROUND

As part of the Truck Design Optimization Project (TDOP) Phase II study, the Federal Railroad Administration (FRA) directed Wyle Laboratories to evaluate and analyze the test data acquired during Phase I of TDOP for use in Phase II model validation and specification of performance indices.

Prior to commencing this work, Wyle Laboratories prepared a TDOP Phase I Data Evaluation and Analysis Plan which the FRA approved (Reference 1). The plan describes what will be accomplished during the Phase I data evaluation and analysis and how the task will be implemented. It contains a description of the hardware and software to be used, the specific analytical techniques to be employed, and the selection and format of the data to be reduced. The plan also defines the expected results of this effort and the format for this report.

1.2 REPORT ORGANIZATION

The remainder of the report is divided into these sections: Section 2 summarizes what data were available for this effort and the computer programs used in the study. Section 3 provides an evaluation of the data and their applicability to Phase II. Section 4 provides a sample of data usage in the form of a pilot program of data analysis for the ride quality regime. Section 5 summarizes the results of the data evaluation and analysis and provides recommendations for future testing.

SECTION 2 - IMPLEMENTATION

2.1 PHASE I DATA

The TDOP Phase I data, in the form of data tapes and computer printouts of analyzed data, were provided to Wyle Laboratories by the FRA. The data were categorized by a computer-based inventory and stored in boxes. The boxes contain 204 magnetic data tapes from the five test series in Phase I, analyzed data from the car response measurements, and the track geometry. The analyzed data included the ENSCO Track Geometry Data Report, and reduced data from various test runs consisting of power spectral densities (PSDs), time histories, and statistical summaries. The complete catalog of the FRA-supplied Phase I data is contained in Appendix A.

Wyle initially explored the idea of reformatting the Phase I data to permit selection by the railroad industry of a particular phenomenon, characteristic, or parameter variation. However, a survey of the railroad industry revealed little appeal for reformatting. Furthermore, the need for a summary of TDOP Phase I data has been met by these FRA documents: the Freight Car Truck Design Optimization Phase I Executive Summary, the Test Results Reports, and the FRA Critique of the Test Results Reports (Reference 4).

2.2 DESCRIPTION OF DATA ANALYSIS SOFTWARE

2.2.1 TDOP Data Sorting Routine

Because of the vast amount of Phase I data generated, a TDOP data sorting routine was developed by Wyle which provides ready access to these data. The sorting routine allows for the specification of a given set of test conditions; the routine then lists all test runs which meet that set of requirements. Details of this sorting routine and the parameters on which it sorts are contained in Appendix B. This sorting routine was used extensively in the evaluation of the Phase I data as discussed in Section 3.2.

2.2.2 Post Processing Program

The analysis of the Phase I data was accomplished by utilizing the Post Processing Program developed by the Southern Pacific Transportation Company (SPTCo.). The Post Processing Program was received from the FRA on magnetic tape and converted for use on Wyle's Interdata 8/32 computer system. Documentation on the program was provided by the Post Processing Program manual (Reference 2). The effort required to convert this program to the Interdata computer system proved to be considerably more difficult than originally anticipated (see Appendix B). The program, as revised and implemented for Phase II analytical work, is described in Reference 3.

2.2.2.1 Program Validation. To assure the accuracy of analyzed data using the Post Processing Program, a series of steps was executed to validate the operation of the program on the Interdata computer. The first step involved running test cases on the Interdata computer and comparing results with those obtained by the SPTCo. The second step involved evaluation of the equations used in the program to determine their accuracy. The only problems experienced were in the PSD calculation. The results of the validation effort are also described in Appendix B.

2.2.2.2 Enhancements. The only modifications made to the Post Processing Program were those associated with the PSD package to enable it to give the correct results. These consisted of removing the mean from the signal before any PSD calculation, calculating the area under the PSD curve, removing an erroneous factor of two, and printing the gravity root-mean-square (g_{rms}) level on the plot.

The Post Processing Program from Phase I provided plots of up to a maximum of 20 seconds in duration. A need was identified in connection with the automatic location detector (ALD) problem (see Section 3.3.2) to provide time history plots of greater than 20 seconds. This capability was implemented by writing a new program which takes the Phase I tapes and produces a reformatted tape compatible with the Wyle library of analysis routines. These routines provide the capability to produce a time history plot for one channel at a time for any duration.

SECTION 3 - DATA EVALUATION

3.1 INTRODUCTION

The data evaluation task first determined the quantity and scope of test data provided by Phase I. The trucks, the carbody types, and track conditions were identified. Secondly, the evaluation determined which measurements taken during Phase I provided useful and accurate representations of the quantity measured. For example, did the vertical accelerometers on the carbody give an accurate representation of car bounce; did the pins on which the strain gages were mounted in the adapter give an accurate representation of the lateral load at the wheel/rail interface, etc.? If the measured data did give a valid quantification of the desired parameter, they are considered acceptable for the model validation and specification of performance indices.

Thirdly, the completeness of the Phase I measurements in providing the required data was evaluated. It is not the purpose of this evaluation to judge if the data will perform the actual model validation or specification of performance indices. This determination will be made part of the analytical and engineering task areas.

The original plan for this report called for an appendix which would catalog all reduced data. However, at the completion of this task, the volume of the reduced data would have resulted in an appendix of several thousand pages. No useful purpose would have been served by publishing a report of this size. However, header sheets describing the test conditions for each run which was reduced are contained in Appendix C. All the data has been cataloged and stored at Wyle in a manner which permits ready access.

3.2 DATA SORTING ANALYSIS

The data sorting routine was used to assess the number of test runs made during the Type I truck testing conducted during the TDOP Phase I test. The parameters used during this sort sequence were car type, truck type, percent load, wheel profile, and track type. The first sort is shown in Table 3-1 and shows the number of runs by car, truck and wheel type. Note that a test run in this discussion includes a number of different speeds and thus may encompass several entries in the data sorting catalog.

Table 3-1 shows that the preponderance of test runs was made with a refrigerator car on ASF 70-ton ride control trucks with new wheels. This emphasis made the data more difficult to use because the refrigerator car is not considered a typical freight car; its uneven weight distribution and very high empty weight tends to bias the data and give misleading answers. The empty weight of the 70-ton capacity refrigerator car is 89,600 pounds compared with 61,200 pounds for the empty 70-ton box car. This is approximately a 46% greater empty weight. The A-end of the empty refrigerator car weighs 49,300 pounds compared to 40,200 pounds on the B-end. This is approximately a 10% difference in the weights of the two ends. Because of these two factors, most of the analysis described in this report was accomplished using test data for the other carbody types shown.

Table 3-1. Number of Test Runs by Body, Truck and Wheel Type

By car type:	Refrigerator Car	234	(86%)
	70-ton Box	9	(3%)
	100-ton Box	12	(4.5%)
	89-ft. Flat	10	(3.5%)
	100-ton Hopper	8	(3%)
	Total Test Runs:	273	
By truck type:	ASF 70-ton Ride Control	225	(82%)
	ASF 100-ton Ride Control	6	(2%)
	Barber 70-ton	18	(7%)
	Barber 100-ton	14	(5%)
	ASF 70-ton Low Level	10	(4%)
	Total Test Runs:	273	
By wheel type:	1/20 (new)	195	(72%)
	1/40 (new)	11	(4%)
	Cylindrical	34	(12%)
	Half Worn	5	(2%)
	Worn	28	(10%)
	Total Test Runs:	273	

While the test data acquired on the test runs using the refrigerator car are considered valid, with the exceptions noted for the other data (see Section 3.3), there is a concern that using the test data from the refrigerator car may tend to skew the analytical results. As previously noted, the car's uneven weight distribution and the high empty weight can give analytical results which will not be typical for other freight cars. For this reason, Wyle decided not to include the refrigerator car in this analysis of the Phase I data. However, the data acquired from these tests are of good quality and can be used in the analytical and engineering effort should it be required.

A more detailed breakdown by track type was conducted as shown in Table 3-2 for the refrigerator car and Table 3-3 for the other four carbody types. Again, this shows the heavy emphasis on the refrigerator car tests. On the other carbody types, only one test run was conducted for each track type. This makes any assessment of repeatability difficult.

The test data sorting information is summarized in Tables 3-4 through 3-8 which show a matrix of test combinations with a dot noting those which were tested during Phase I. Each table refers to one kind of track condition and shows the tests run by the SPTCo according to carbody type, loading condition, truck type, and wheel type and condition.

Table 3-2. Number of Runs of 70-ton Mechanical Refrigerator Car

	CURVED 25, 35 MPH	Hi-Speed Jointed 30, 40, 50, 60, 70, 79 MPH	Hi-Speed CWR 30, 40, 50, 60, 70, 79 MPH	Med-Speed Jointed	Shimmed
<u>ASF 70-ton Refrigerator Car Trucks</u>					
Empty, 1/20 (new) wheels	10	29	28	23	
Empty, 1/40 (new) wheels	1	1	1	2	1
Empty, cylindrical	3	4	3	2	3
Empty, half worn		1	2	2	
Empty, worn	1	4	4	4	
Half Full, 1/20 (new) wheels	1	2	2	2	
Fully loaded, 1/20 (new) wheels	11	16	16	14	
Fully loaded, 1/40 (new) wheels	1	1	1	1	1
Fully loaded, cylindrical	3	3	3	3	3
	1	4	4	4	
<u>Barber 70-ton trucks</u>					
Empty, 1/20 (new) wheels	1	1	1	1	
Fully loaded, 1/20 (new) wheels	1	1	1	2	

Table 3-3. Number of Runs by Car Configuration
(All Car Body Types Except
Mechanical Refrigerator)

	CURVED 25, 35 MPH	Hi-Speed Jointed 30, 40, 50, 60, 70, 79 MPH	Hi-Speed CMR 30, 40, 50, 60, 70, 79 MPH	Med-Speed Jointed 30, 40, 50, 60, 70, 79 MPH	Shimmed
100-ton box, Barber trucks, new wheels, empty	1	1	1	1	
100-ton box, Barber trucks, new wheels, loaded	1	1	1	1	
100-ton box, Barber trucks, cylindrical, empty				1	1
100-ton box, Barber trucks, cylindrical, loaded				1	1
70-ton box, Barber trucks, new wheels, empty	1	1	1	1	
70-ton box, Barber trucks, new wheels, loaded	2	1	1	1	
89' flat, ASF trucks, new wheels, empty	1	1	1	1	
89' flat, ASF trucks, new wheels, loaded	1	1	1	1	
89' flat, ASF trucks, worn wheels, empty		1	1		
100-ton hopper, Barber truck, new wheels, empty	1				
100-ton hopper, Barber truck, new wheels, loaded	1				
100-ton hopper, ASF truck, new wheels, empty		1	1	1	
100-ton hopper, ASF truck, new wheels, loaded		1	1	1	

Table 3-4. Curved Track Test Matrix

TDOP PHASE I TEST MATRIX

☒ TEST DATA AVAILABLE
☐ NO TEST CONDUCTED
☒ NON-APPLICABLE TEST CONDITION

TRACK TYPE : CURVED

CAR BODY	TRUCK	ASF 70-TON RIDE CONTROL					ASF 100-TON RIDE CONTROL					BARBER 70-TON					BARBER 100-TON					ASF 70-TON LOW LEVEL				
	LOAD COND.	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E
REFRIGERATOR	EMPTY	•	•	•		•						•														
	HALF FULL																									
	LOADED	•	•	•		•						•														
70-TON BOXCAR	EMPTY											•														
	HALF FULL																									
	LOADED											•														
100-TON BOXCAR	EMPTY																•									
	HALF FULL																									
	LOADED																•									
89-FT FLATCAR	EMPTY																					•				
	HALF FULL																									
	LOADED																					•				
100-TON HOPPERCAR	EMPTY																•									
	HALF FULL																									
	LOADED																•									

• A - NEW 1/20 B - NEW 1/10 C - CYLINDRICAL D - HALF WORN E - WORN

* A - NEW 1/20 B - NEW 1/10 C - CYLINDRICAL D - HALF WORN E - WORN

Table 3-7. Medium-Speed Jointed Track Test Matrix

TDOP PHASE I TEST MATRIX

TRACK TYPE : MEDIUM-SPEED JOINTED

☐ TEST DATA AVAILABLE
☐ NO TEST CONDUCTED
☒ NON-APPLICABLE TEST CONDITION

CAR BODY	TRUCK	ASF 70-TON RIDE CONTROL					ASF 100-TON RIDE CONTROL					BARBER 70-TON					BARBER 100-TON					ASF 70-TON LOW LEVEL				
	LOAD COND.	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E
REFRIGERATOR	EMPTY	•	•	•	•	•						•														
	HALF FULL	•										•														
	LOADED	•	•	•		•						•														
70-TON BOXCAR	EMPTY											•														
	HALF FULL											•														
	LOADED											•														
100-TON BOXCAR	EMPTY																•		•							
	HALF FULL																•		•							
	LOADED																•		•							
89-FT FLATCAR	EMPTY																					•				
	HALF FULL																					•				
	LOADED																					•				
100-TON HOPPERCAR	EMPTY						•																			
	HALF FULL						•																			
	LOADED						•																			

* A - NEW 1/20 B - NEW 1/40 C - CYLINDRICAL D - HALF WORN E - WORN

Table 3-8. Shimmed Track Test Matrix

TDOP PHASE I TEST MATRIX

☒ TEST DATA AVAILABLE
☐ NO TEST CONDUCTED
☒ NON-APPLICABLE TEST CONDITION

TRACK TYPE : SHIMMED

CAR BODY	TRUCK	ASF 70-TON RIDE CONTROL					ASF 100-TON RIDE CONTROL					BARBER 70 TON					BARBER 100 TON					ASF 70-TON LOW LEVEL					
	LOAD COND.	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	A	B	C	D	E	
REFRIGERATOR	EMPTY	•	•																								
	HALF FULL																										
	LOADED	•	•																								
70-TON BOXCAR	EMPTY																										
	HALF FULL																										
	LOADED																										
100-TON BOXCAR	EMPTY																	•									
	HALF FULL																										
	LOADED																	•									
89-FT FLATCAR	EMPTY																										
	HALF FULL																										
	LOADED																										
100-TON HOPPERCAR	EMPTY																										
	HALF FULL																										
	LOADED																										

* A - NEW 1/20 B - NEW 1/40 C - CYLINDRICAL D - HALF WORN E - WORN

3.2.1 Test Matrix Omissions

While it is not claimed that the test matrix need to be completely filled in for the purposes of Phase II analysis, the following omissions are considered most significant:

- a. No curving tests were run on 100-ton box cars and hopper cars with the ASF ride control truck. Since there are significant differences between the ASF and Barber trucks related to warp stiffness, a curving test should have been run with both trucks.
- b. No curving tests were run with worn wheels on any car except the refrigerator car. Wheel wear has some effect on curving performance.
- c. The curving test runs and the conditions omitted have no significance because of the improper measurement of lateral wheel loads.
- d. No high-speed CWR tests were run with the 100-ton box car on an ASF truck, or the 100-ton hopper car with the Barber truck.
- e. No tangent track tests were run with worn wheels except for the refrigerator car, and the empty 89-foot flat car. Thus, data on lateral stability appear to be inadequate.
- f. There were no medium-speed jointed rail test runs on a 100-ton box car on an ASF truck, or the 100-ton hopper car with the Barber truck. Since this type of track exercises the friction snubber, this omission makes it difficult to compare the two types of snubbing systems.
- g. Shimmed track tests with other than cylindrical wheels were run only with the refrigerator car. This abbreviated test does not reflect the variety of devices present in the suspension system. An evaluation of the shimmed track tests thus requires more detailed scrutiny.

3.2.2 Ride Quality Data

In terms of ride quality, the only deficiency of the Phase I data is the lack of correlation between measured track geometry and response data as discussed in Section 3.3. This should not significantly hinder the ride quality analysis as shown in the pilot program. When test data become available during the Phase II testing of Type I trucks, it can be used to further validate the results from the Phase I data.

3.2.3 Impact on TDOP Phase II

The Phase I data omissions discussed in Sections 3.2.1 and 3.2.2 will necessitate additional testing during Phase II of the Type I truck. The extent of this testing will be directly related to the amount of data required by the model validation and engineering task requirements. After each of these tasks has been reviewed, a preliminary matrix of tests for the Type I truck will be prepared. These matrices will be reviewed and consolidated by the testing group and an integrated test plan developed to perform the desired tests.

3.3 MEASUREMENT EVALUATION

3.3.1 Lateral Wheel Load Measurements

In Phase I, lateral wheel loads due to creep and flange forces were improperly measured. During Phase I, lateral forces between the side frame pedestals and the roller bearing adapters were measured by strain gages on pins that were located on both sides of the roller bearing adapters.

As generally known, lateral forces applied to truck components are of two types:

- a. The first consists of external and inertial forces, such as those applied by angled couplers, centrifugal forces during curve negotiation at other than balance speed, and forces due to periodic car body accelerations having lateral components. These lateral forces are eventually reacted between the wheel and rail, and the load path passes through the bearing adapter and side frame, which justifies the method of measurement used in Phase I.
- b. The second comprises creep and flange forces which are partly reacted between wheels of the same wheelset, and partly between wheelsets through the track structure; only the lateral components of the latter can be measured by the adapter pins on which strain gages were mounted. However, during curving with flange contact, a large part of the lateral load on the outer leading wheel is due to the creep forces on the forward wheelset; the load path is confined to the wheels and axle, thus it bypasses the adapter which transmits only the lateral creep forces from the rear wheelset (if it is not in flange contact). Therefore, lateral wheel loads measured by this method during curving at equilibrium speed are bound to be low, and the contribution of dynamic loads as coupler forces cannot be separated from those of the creep forces.

The lack of lateral wheel/rail force measurements particularly affects the curving data where the most important parameter is the lateral force at the wheel/rail interface, since this directly relates to the amount of rail and wheel wear which occurs during curving. Thus, in Phase II these missing curving tests may have to be run (hopper car with ASF truck, trucks with worn wheels), and some Phase I tests repeated to provide an adequate matrix of data to characterize the Type I truck in curve negotiation. The data available from Phase I are not sufficient for validation of any meaningful curve negotiation model since the primary quantity to be derived from the model would be the lateral force. Also, the data do not provide sufficient information to quantify any performance indices relative to curving. However, some preliminary work can be done in the area of truck motions related to degree of curvature and super-elevation.

The measurement of the lateral wheel/rail force is also of importance in lateral stability (hunting), since again it relates directly to wear. The importance of lateral force/vertical force (L/V) ratio is related primarily to dynamic regimes involving contact between the throat or flange of the wheel and the rail, either when the lateral force is high (such as occurs during hunting), or when the vertical force is low (which occurs during harmonic roll). Both situations produce a high L/V ratio and thus pose the risk of derailment.

There is no question that a better method of measuring lateral wheel loads would have been preferable than that used in Phase I. However, in the case of hunting, some very useful information may be extracted from the Phase I data by combining the vertical forces measured by the strain gages mounted in the bearing adapters with the known wheel contours, the inertial properties of the wheels and side frames, and the vertical accelerations of the pedestals to calculate the lateral loads on the wheels with a level of accuracy acceptable for engineering purposes.

Also, much model validation may be done from the Phase I data in relating critical speed to the model parameters. As several test configurations are being instrumented for curving tests, it is planned that hunting tests will be run at the same time as the curving tests with the same test configurations. Thus, some additional lateral force data will also be provided for the lateral stability regime.

3.3.2 Track Geometry Correlation

A problem area discussed in the TDOP Phase I Data Evaluation and Analysis Plan (Reference 1) is the difficulty of correlating response measurements with the track geometry location. The automatic location detector (ALD) used by the SPTCo. during Phase I picked up numerous extraneous signals which made determining the exact location of the test car difficult. The technique used during Phase I for determining the exact milepost location of the test car, so that the car response data may be correlated to the track geometry measurements, was to place metal targets at known locations along the track. A detector on the train sensed the targets as the train passed over them. In theory, this method would then identify the exact location of the train; the milepost location between targets could then be obtained by integration of train speed. In practice, however, this technique did not work because the target detector also picked up extraneous signals in addition to detecting the targets.

This problem is illustrated in Figure 3-1 from test run 030201TWA001 which shows the ALD channel (solid line) versus milepost. The milepost location was obtained by integrating train speed from the known starting milepost. A positive voltage signal indicates the ALD located a target. The dashed lines in Figure 3-1 were overlaid on the plot of ALD at those known locations at which the metal targets were placed. If the dashed lines (target location) were close to agreement with the ALD detection signal (solid lines), then it would be a relatively simple matter to put some small adjustments into the speed integration to get the dashed and solid lines to match exactly. However, the discrepancy between the two signals is so great that it is not possible to determine what corrections should be made to line them up.

The problem of knowing the exact track input which corresponds to a given response is particularly critical in time-domain analysis. In this type of analysis, the model must be given exactly the same input as the test car if the response data are to be compared.

During the Phase I Data Evaluation and Analysis, the ALD signals from several runs spanning the duration of Phase I testing were plotted. Figure 3-2 is a plot of one of the first tests; the ALD system, was not operating satisfactorily at that time. After the first few test runs, the ALD signal was improved. The remaining plots show a great degree of similarity. Figures 3-3 through 3-5 all show almost exactly the same pattern. This probably indicates that some fixed object (such as a switch or crossing) causes the ALD to register and the problem that now remains is to sort these occurrences out from the actual ALD target detections. One approach to correlating the signals could be to try to relate each ALD signal with a known object and then to determine the actual ALD signals. If the ALD signal can be made to line up with the car response data, then it will be possible to use Phase I data in conjunction with the time-domain models.

At present, no additional effort is planned in attempting to correlate the track geometry and milepost location because it is not critical to perform an analysis of the data. If in the future, the track geometry/response data correlation is required, additional effort may be expended on the task.

The problem associated with the ALD was caused by the detector sensing any metal object including the desired target. This problem will be corrected during Phase II by using an alternate technique. The two techniques currently under evaluation consist of either a tuned coil or magnet buried in the ballast and an appropriate detection circuit attached to the instrumentation car. This approach should eliminate the problem of spurious signals.

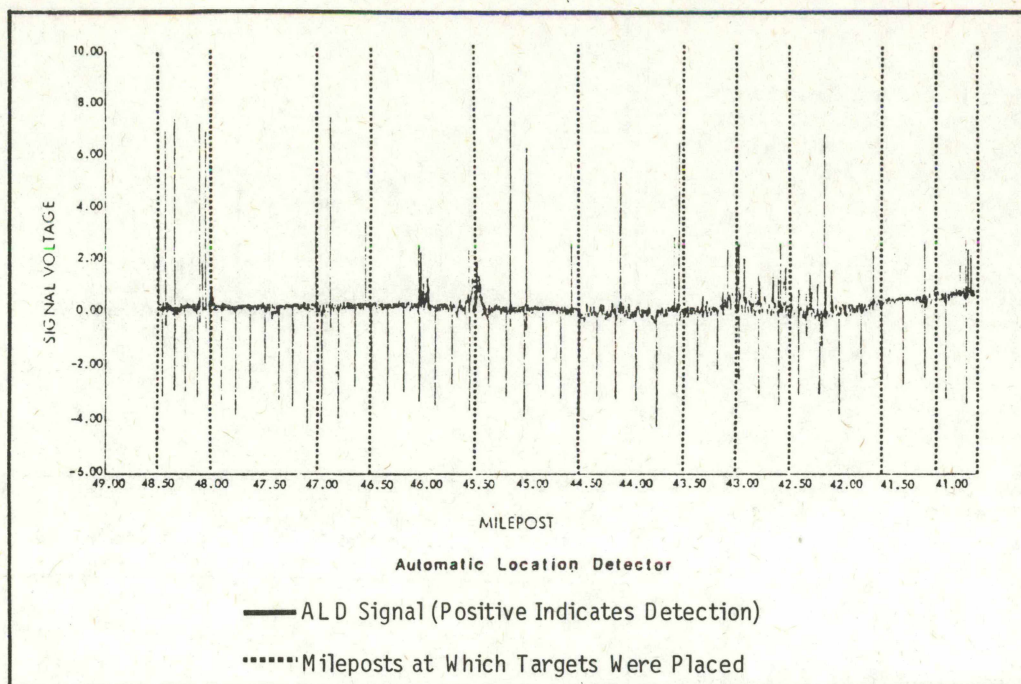


Figure 3-1. ALD Location Plot

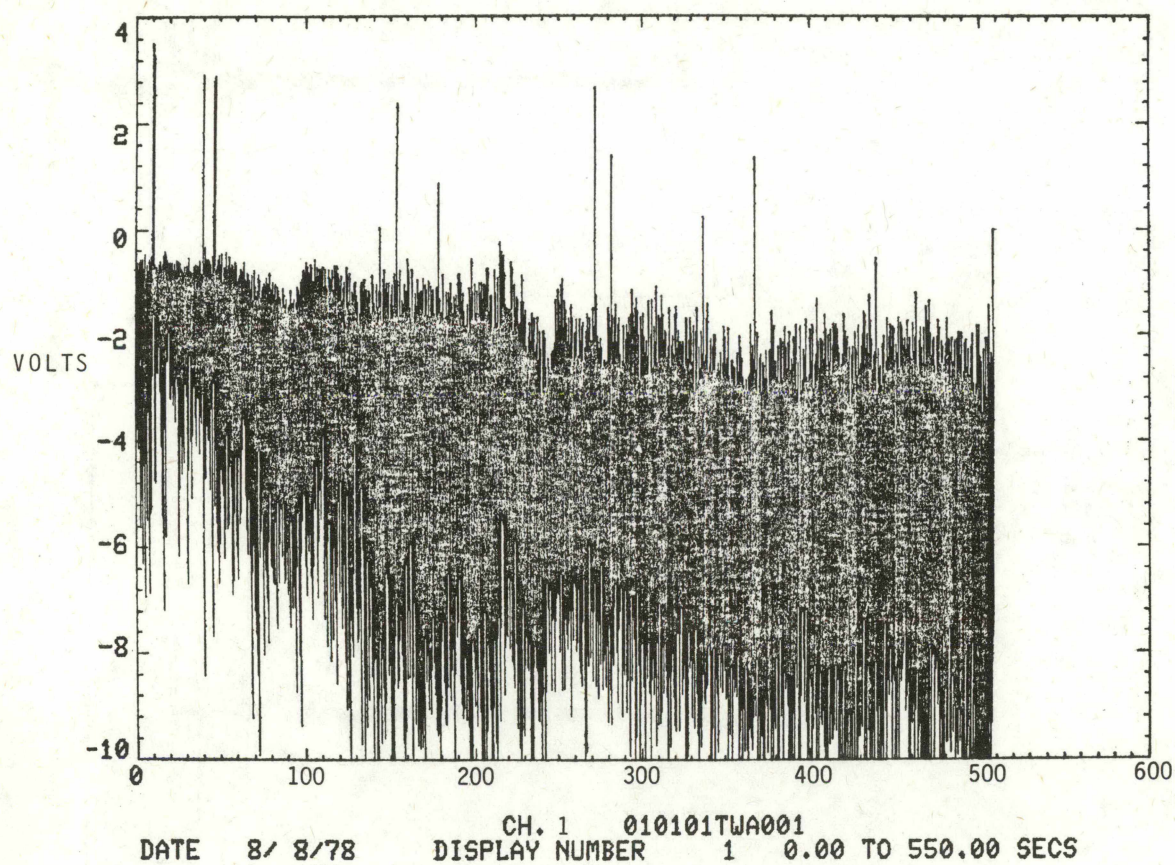


Figure 3-2. Plot of One of the First Tests, ALD System Inoperative

Figure 3-3. Plots from First Test Series, ALD System Operative (Note similarity of patterns in Figures 3-4 and 3-5.)

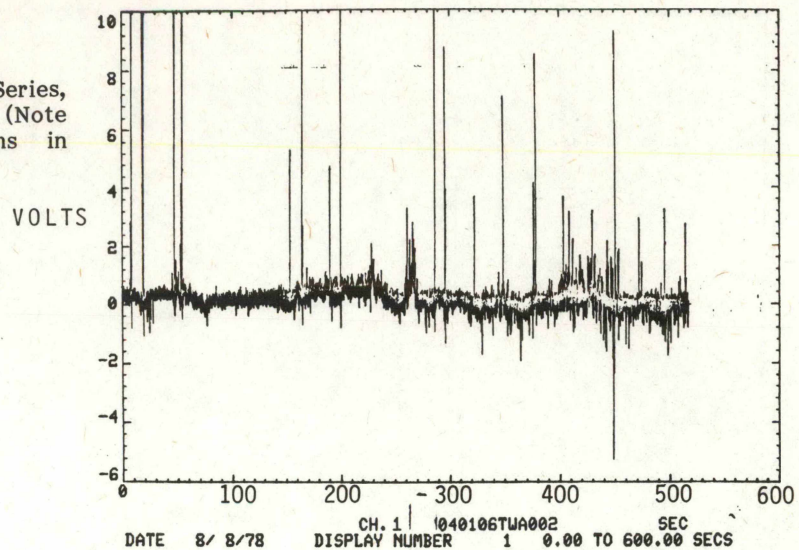


Figure 3-4. Plots from First Test Series, ALD System Operative (Note similarity of patterns in Figures 3-3 and 3-5.)

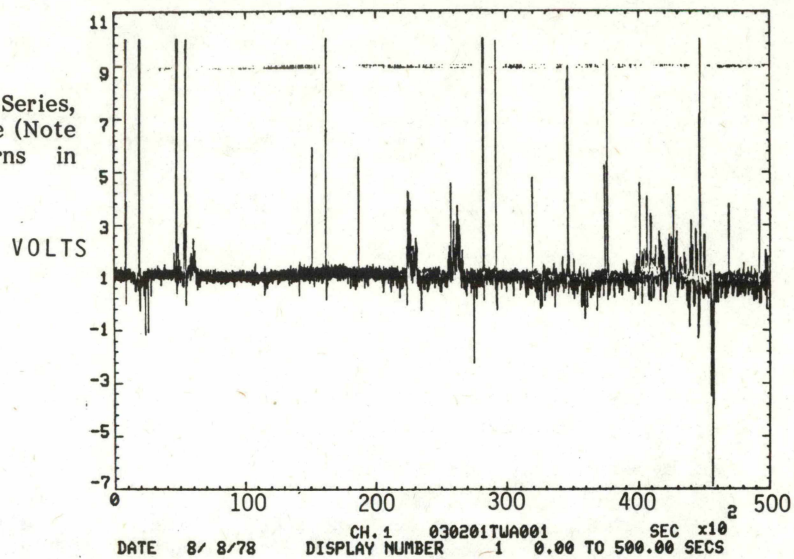
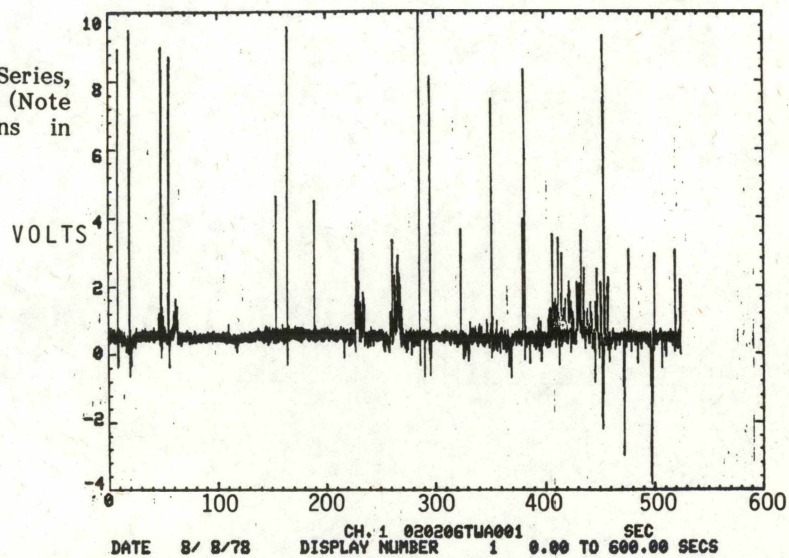


Figure 3-5. Plots from First Test Series, ALD System Operative (Note similarity of patterns in Figures 3-3 and 3-4.)



3.3.3 Quality of Measurements

The measured data from Phase I proved to be of acceptable quality in the evaluation of performance parameters, with the exception of the inability of the ALD measurement to correlate to track location, the improper technique used to measure lateral wheel loads, and the lack of friction snubber force measurements.

3.3.3.1 Measurements Made Incorrectly. During TDOP Phase I, a strain gage mounted on a pin was used in the adapter to measure lateral force. The data acquired were accurate measurements of the lateral force at the adapter but cannot be correlated to lateral forces at the wheel/rail interface as previously discussed in Section 3.3.1. Thus, the data cannot be used in calculating L/V values. Several alternate techniques for measuring wheel lateral force are being investigated during Phase II and the most promising approach will be adopted.

The signal conditioning during the Phase I testing used a calibration technique which introduces a small error in the data depending upon the length of the cable from the signal conditioner to the transducer and the bridge resistance of the transducer. Electron signal conditioning was used which has the excitation voltage sensing at the signal conditioner. Thus, the voltage drop it senses includes not only the transducer bridge, but also the line drop in the cable. For a 300-foot cable, this results in a two to five percent error in the calibration voltage, depending upon the transducer type. This amount of error does not significantly affect the data; however, it does introduce a slight bias on the low side to all data acquired during Phase I. Without knowing the length of cable used for each transducer on each run, it is not possible to correct for it and thus the bias is left in the data. During TDOP Phase II, the voltage sensing is being moved to the junction box on the test car. This decreases the maximum cable length to about 30 feet and the resultant error will be of a lesser order of magnitude.

3.3.3.2 Measurements Not Made. The lack of friction snubber force measurements was identified early in Phase I and plans were developed independently by Wyle Laboratories to design a device which will measure these forces. However, development of the device was not completed until the end of Phase I, and no over-the-road data were ever acquired. Hence, a test program using the Friction Snubber Force Measurement System (FSFMS) was conducted in Phase II to obtain the desired characterization of friction snubber forces (reference 5).

3.4 APPLICABILITY OF DATA TO PHASE II

3.4.1 Analysis

The Phase I data were evaluated to determine their applicability to the validation of analytical tools. This evaluation consisted of a listing, by regime, of the planned models for the Phase II analysis work, the test data requirements for each model, and the quality of the Phase I data. This survey is contained in Tables 3-9 through 3-12 for the four performance regimes planned for Phase II.

Significant data are available in the lateral stability regime for model validation as shown in Table 3-9. The primary shortcomings were the lack of wheel/rail force measurements and the lack of tests on wheels with worn profiles other than for the refrigerator car. The data required for the ride quality regime are generally complete. A few minor exceptions shown in Table 3-10 were some carbody and truck motion measurements. However, these deficiencies are not considered critical.

In the curve negotiation regime, the two most critical measurements (lateral force and angle of attack) were not measured (see Table 3-11). This makes extraction of meaningful information from the other data difficult. The data available for the trackability regime is shown in Table 3-12. There are sufficient data for the linear models; however, the nonlinear models lack adequate measurement of wheel/rail forces and of correlatable track geometry.

As previously discussed in paragraph 3.2, even when adequate data channels were acquired during Phase I, the matrix of configurations which were tested is often inadequate. Thus, additional data may be required on other carbodies and wheel profiles.

3.4.2 Engineering

The Phase I data were evaluated to determine their applicability to the specification of performance indices. This evaluation is shown in Table 3-13 which lists the performance index for each of the four regimes and the necessary test data required to specify the performance index. The right-hand column in Table 3-13 defines the availability of test data from Phase I for the given performance index. The data available in the ride quality and lateral stability are sufficiently complete and adequate. Limited data are available to handle portions of the trackability regime. In the curve negotiation regime, the measurements are completely lacking.

Table 3-9. Lateral Stability Validation Requirements

Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Engineering Models	<ul style="list-style-type: none"> Kinematic frequency versus speed 	Measured	Satisfactory
Simple Kinematic Models	<ul style="list-style-type: none"> Car body rigid body modes 	Five rigid body modes available.	Longitudinal mode not measured
Eigenvalue Analysis Models	<ul style="list-style-type: none"> Hunting at some critical speed for various wheel profiles. (Linearized models of the car/truck combination stability will be compared to the predicted critical speeds.) 	Hunting tests with various profiles limited to refrigerator car.	Satisfactory
Nonlinear time-domain models	<ul style="list-style-type: none"> Truck kinematics vs. speed 	Measured	Satisfactory
	<ul style="list-style-type: none"> Car body dynamics vs. speed 	Measured	Satisfactory
	<ul style="list-style-type: none"> Time histories of the vertical and lateral forces at the wheel/rail interface. 	Not Measured	N/A
	<ul style="list-style-type: none"> Mode shapes of the car/truck during limit cycle hunting motions for primary (body hunting). 	Measured	Limited to rigid body modes

Table 3-10. Ride Quality Validation Requirements

Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Linear Frequency Domain Models	<ul style="list-style-type: none"> Nominal truck and car body vibration response data while running over tangent track on both continuous welded and jointed rail. 	Measured	Satisfactory
	<ul style="list-style-type: none"> Duration of recorded data should be at least 100 seconds at a given constant speed in order to obtain sufficient statistical confidence in the measured PSD and transmissibilities. 	Runs at most speeds averaged 60 sec. of data	Satisfactory
	<ul style="list-style-type: none"> Track geometry shall be correlatable over the test section to within ± 100 feet. (Since the work will be in the frequency domain only, the position accuracy is not as stringent as it is for the trackability regime.) 	Track geometry measured.	Correlation difficult. Providing estimate of accuracy also difficult, ± 100 ft. may not be possible.
	<ul style="list-style-type: none"> Required truck response measurements shall include: <ul style="list-style-type: none"> - Vertical and lateral accelerations at each of the four bearing adapters. 	Vertical accel. each end of both axles, lateral accel. on each axle.	Satisfactory
	<ul style="list-style-type: none"> - Two vertical acceleration measurements sufficient to determine vertical and roll motion. 	Not measured (displacement data can be used to derive roll motion).	N/A
	<ul style="list-style-type: none"> - Lateral acceleration measurement 	Measured	Satisfactory
	<ul style="list-style-type: none"> Car body <ul style="list-style-type: none"> - Vertical and lateral 	Measured	Satisfactory
	<ul style="list-style-type: none"> - Center A end and B end 	Measured	Satisfactory
	<ul style="list-style-type: none"> - Both corners A and B end top and bottom 	Not Measured (measurement is required to locate center of roll)	N/A
	<ul style="list-style-type: none"> - Lateral and vertical at car body center 	Not Measured (required for flexible car bodies).	N/A

Table 3-11. Curve Negotiation Validation Requirements

Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Simple Engineering Models	Required data should provide the set up and wheel/rail forces during curving.	Not Measured	N/A
Kinematic Models	Measurements to include: <ul style="list-style-type: none"> • Truck to car body yaw • Truck tram angle • Angle of attack at each wheelset 	Measured	Satisfactory
		Measured	Satisfactory
		Not Measured (critical deficiency)	N/A
	• Wheel/rail forces, particularly during flanging	Not Measured (critical deficiency)	N/A
Steady-State Curving Models*	• Wheel/rail lateral force	Not Measured	N/A
	• Wheel/rail lateral displacement	Not Measured	N/A
Dynamic Curving Models	Data during curve entry and exit in addition to the above measurements should include time history responses of: <ul style="list-style-type: none"> • Car body dynamics in the form of acceleration measurements sufficient to determine car body roll, roll center, car body yaw, sway and pitch. • Accelerometers on the truck sufficient to determine the truck component motions for: <ul style="list-style-type: none"> - Truck bolster - Side frame - Wheelsets 	Measured	Satisfactory
		Lateral only measured Measured Measured	Satisfactory

*Data acquired during the steady-state portion of the curve. (Filtering or averaging of the data will be required to extract the steady-state forces and positions.)

Table 3-12. Trackability Validation Requirements

Type of Model	Test Data Requirements	Phase I Data Availability	Quality of Data
Engineering Models	Data required should be sufficient to extract truck/car resonances and mode shapes	Data available for rigid body resonances and modes.	Satisfactory
Linear Spring Mass Models	Small vehicle responses (which minimize the nonlinear reactions) over both regular and perturbed track.	Tests run on both regular and perturbed track.	Satisfactory
	Harmonic roll critical speed on perturbed track (linear models will be used as a means of estimating the critical speed before the more costly nonlinear simulations are run).	Harmonic roll available on shimmed track	Available only in two mph increments
Nonlinear Time-Domain Models	In addition to the above data, validation of the nonlinear time-domain models will require: <ul style="list-style-type: none"> • Wheel/rail vertical and lateral forces. • Measurements prior to and during wheel lift off. • Extreme center plate dynamics during <ul style="list-style-type: none"> - Harmonic Roll - Bounce • Truck component relative motions during perturbed track tests. (Required for validation of the large signal responses.) • All measured car truck responses shall be correlatable with the track geometry within ± 6 inches. 	Not Measured	N/A
		Not Measured	N/A
		Measured	Satisfactory
		Measured	Satisfactory
		Track geometry measured	Track geometry and response data cannot be correlated.

Table 3-13. Test Data Required for Engineering Analysis

Performance Regime	Performance Index	Necessary Test Data	Availability of Test Data from Phase I
<u>Lateral Stability</u>	● Critical Speed	Lateral Acceleration of one or more representative points on the truck measured as a function of speed and such variables as: wheel/rail contour, rail surface conditions, car bodies (truck spacing, stiffness), and lading (empty, full, ...)	Lateral acceleration available on axle and car body. Data are taken at constant speeds of 40, 50, 60, 70, and 79 mph. Varying speeds exist between these constant speeds. Variables such as wheel profile, rail surface conditions, car body parameters, and lading is noted in the test header. No rail contour data are available. Tests were not run for a full matrix of variables.
	● Magnitude of Lateral Acceleration	Magnitude of lateral acceleration at or near the hunting speed, for the same set of variables mentioned above.	Lateral acceleration data on axles.
<u>Curve Negotiation</u>	● Lateral force on leading outer wheel per 1000 pounds axle load per degree of curve under, at and over balance speed.	Lateral force on leading outer wheel as a function of lading, degree of curvature at, under, and above balance speed.	No measurements made of lateral force.
	● Wear Index	Angle of attack as a function of lading, and degree of curvature under, at, and above balance speed.	No measurements made of angle of attack.
	● Derailment Potential	L/V ratio as a function of speed, lading, wheel/rail contour.	No measurements made from which to calculate L/V.
<u>Trackability</u>	● Wheel Unloading Index	Simultaneous loads under the wheels as a function of track twist in degrees as a function of lading.	No measurements made of vertical load at wheel. Vertical loads measured at bearing adapters, but cannot be correlated to track geometry.
	● Max. Roll Amplitude	Max. roll amplitude as a function of excitation (amp. and frequency) for different lading conditions.	Roll angle of car body/truck bolster and roll acceleration of car body were measured, however, they cannot be correlated to track geometry.
	● Rate of Energy Dissipation	Level of friction force, displacement (i.e., spring travel), rate of increase of friction level with spring compression, as a function of lading.	No friction snubber force measurements were made.
	● Derailment Potential	L/V ratio as a function of speed, lading, wheel/rail contour.	No measurements made from which to calculate L/V.
<u>Ride Quality</u>	● Transmissibility	Acceleration response, referred to one or more specific locations on the car body, as a function of speed, track quality and lading within the normal operating range of speeds.	Vertical acceleration made on car body. Speed, trackability, and lading were varied, however, a complete matrix of these variables was not tested.

SECTION 4 - PILOT PROGRAM

Phase I data will be used as part of the analytical and engineering tasks in model validation and quantification of performance indices. To gain familiarity and confidence in the data and to demonstrate a technique for data analysis, a small pilot program was conducted during the Phase I data evaluation and analysis. This pilot study, which was intended to show how the data are to be used, was limited to one specific performance regime, ride quality.

A number of test runs were analyzed to investigate ride quality. The pilot program used rms acceleration versus speed plots to provide a visual display of the data, and regression analysis to quantify the relative magnitude of the various parameters considered during Phase I testing. A large number of variables were tested during Phase I (e.g., loading condition, carbody, rail type, wheel profile, and truck type). This analysis attempted to address which variables had significant impact on the ride quality level.

4.1 DATA ANALYSIS METHODOLOGY

For the purposes of this analysis, the rms versus speed analysis capability from the Phase I Post Processing Program was used. The Post Processing Program was given a specific series of track sections and asked to calculate the rms acceleration value. In each case, two test zones were chosen at the speed rating indicated in Table 4-1. The purpose of dividing the speed zone in half was to give some indication of the amount of spread which could occur in the results from one track section to the next.

The test runs to be considered were selected using the TDOP data sorting routine. The intent is to analyze as wide a variation of parameters as available from the Phase I testing. After much discussion, Wyle decided to concentrate on carbodies other than the 70-ton refrigerator car because there was considerable criticism of Phase I for using this test car so extensively. This limited our analysis to a workable number of runs without severely compromising the number of variables to be considered. Later in the program, the refrigerator car was included in some of the regression runs to help separate the effects of wheel profile from those of carbody types. This had little effect on the results, however.

4.2 METHODOLOGY FOR SUMMARY REGRESSIONS

Wyle used a descriptive regression to summarize the results of the investigation of the TDOP Phase I data with respect to ride quality. A descriptive regression quantifies the relative effects of a number of variables. For the purposes of this study, ride quality was quantified by rms acceleration (i.e., acceleration was taken as the ride quality performance index). This was then considered a function of a number of variables such as train speed, load conditions, carbody, etc. The slope of the acceleration with respect to the train speed was estimated for speeds in the range of 30 to 79 mph. Other influences such as car loaded, car empty, jointed rail, CWR rail, etc., were represented by dummy variables, e.g., a variable whose value is either 0 or 1 depending upon which category the measurement fell into. The average change in rms level for each category was estimated. The results of this analysis are indicated in Table 4-2.

It was discovered early in the analysis that the response data measured on the axles was different from the response data measured on the carbody. This results from the fact that the axle measurements are made on the unsprung portion of the truck while the remainder of the measurements were made on the truck component and carbody which are separated from the rail input by the truck suspension system. As the rail inputs feed directly into the axle-mounted accelerometers, it is reasonable that they would have much higher accelerations than the accelerometers mounted elsewhere. It was decided to separate the axle measurements into one regression analysis and the remainder of the measurements into their own regression analysis. An analysis of variance was run with the early regressions. This analysis showed that the cross effects between the axle and the other parameters were larger than most of the primary effects which strongly suggested that this was an appropriate division to make.

It is important to note that a descriptive regression does not attempt a curve fit of the data. Individual curves could be fitted using the least-square techniques, each curve having a separate equation. Individually fitted curves would provide a more accurate representation of the data. However, information regarding the relative size of the effects would be obscured. Since the purpose of this analysis is to determine the relative importance of the various parameters, we have chosen to describe the data with the regression, obtaining an indication of the average size of each effect. Similarly, the equations used do not force the acceleration to go through zero when the train speed is zero. The equations should be regarded as linear approximations to the "real" function in the range of the train speed variable considered (e.g., 30 to 79 mph).

The following example clarifies the use of the data in Table 4-2. Suppose it were desired to estimate the rms acceleration level in the lateral direction at the A-end roof of a fully loaded 70-ton box car traveling at a train speed of 40 mph on CWR with new wheels and a Barber truck. The total rms acceleration is calculated by adding the rms acceleration contribution of each of the variables, as shown in the following equation:

$$\begin{aligned} \text{g rms acceleration} = & \text{g rms/mpH} \times \text{speed} + \text{g rms} \\ & (\text{acceleration location}) + \text{g rms} (\text{loading}) + \text{g rms} \\ & (\text{carbody}) + \text{g rms} (\text{rail type}) + \text{g rms} (\text{acceleration} \\ & \text{direction}) + \text{g rms} (\text{wheel profile}) + \text{g rms} (\text{truck} \\ & \text{type}) + \text{constant rms acceleration} = .00172 \text{ g} \\ & \text{rms/mpH} \times 40 \text{ mph} + .0609 \text{ g rms} + 0.0 + .0189 \text{ g rms} \\ & - .0186 \text{ g rms} - .0185 \text{ g rms} + 0.0 + 0.0 - .0188 \text{ g rms} \\ & = .0927 \text{ g rms.} \end{aligned}$$

This predicted value of .0927 g rms based upon the regression analysis may be compared to measured test values taken on the above configuration of .0728 g rms and .0702 g rms. The error here is typical, 68% of the data may be expected to have an error within $\pm .0329$ g rms.

However, the importance of the analysis is not so much a quantitative prediction of the g rms levels, but a qualitative prediction of how the variables affect the measured g rms level. While it is an accepted practice in the railroad industry to report ride quality as an rms level, this is not necessarily appropriate for all modes of deterioration. Rms is an average level. It may be that certain types of lading can accept an rms level of around 1 g rms, but suffer damage if some peak accelerations are exceeded (e.g., 10 g peak).

Table 4-1. Track Sections

Speed-mph	Mile Post Numbers	
	Jointed	CWR
30	48.5 - 48.25	42.5 - 42.75
30	48.25 - 48.0	42.75 - 43.0
40	47.75 - 47.5	43.25 - 43.625
40	47.5 - 47.25	43.625 - 44.0
50	46.75 - 46.38	44.5 - 45.0
60	46.38 - 46.0	45.0 - 45.5
60	45.0 - 44.75	46.94 - 47.37
70	43.75 - 48.38	42.5 - 43.38
70	43.38 - 43.0	43.38 - 44.25
79	41.6 - 41.15	45.5 - 46.44
79	41.15 - 40.7	46.44 - 47.37

Note: Samples per zone: 3000 to 9300

Table 4-2. Summary Regressions

	Measurement Not On Axle	Measurement On Axle
Slope of Speed	.00172 ± .00008 g rms/mph	.00376 ± .00025 g rms/mph
Accelerometer Location		
Axle	N/A	Nominal
Truck Side Frame	.0884 ± .0062 g rms	N/A
Roof of Car	.0609 ± .0064 g rms	N/A
Car Center	.0029 ± .0061 *g rms	N/A
Center Sill	Nominal	N/A
Empty as Opposed to Loaded:	.0287 ± .0085 g rms	No Data
Car Body		
70-ton box	.0189 ± .0045 g rms	.0242 ± .0107 g rms
89-ft flat	-.0004 ± .0079 *g rms	No Data
100-ton Hopper	-.0115 ± .0105 *g rms	No Data
100-ton box	Nominal	Nominal
CWR as Opposed to Jointed:	-.0186 ± .0041 g rms	-.0496 ± .0107 g rms
Lateral as Opp. to Vertical Accel:	-.0185 ± .0062 g rms	-.1561 ± .0107 g rms
Worn as Opp. to New Wheels:	.0131 ± .0105 *g rms	No Data
ASF as Opp. to Barber Truck:	-.0013 ± .0105 *g rms	No Data
Constant:	-0.188 ± .0066 g rms	.1109 ± .0164 g rms
R ² **	72.6%	82.8%
Std. Error	.0329 g rms	.0545 g rms
Number of Samples	338	104
*Cannot be distinguished from nominal at 5% significance level.		
**Ratio of explained variance to total variance.		

To demonstrate what information may be extracted from Table 4-2, consider the following: as the speed goes up, so does the g rms acceleration level; the empty car has a rougher ride than the loaded; the 100-ton hopper gives the best ride; wheel profile and truck type have too small an effect to be distinguished from zero, based upon these data.

4.3 PRIMARY INFLUENCES ON RIDE QUALITY

The primary influence on ride quality, as measured by rms acceleration readings from the TDOP Phase I data, was train speed. As the train speed increased, the g rms level increased. Another major difference in the measurements was the significantly higher g rms levels measured on the truck axle, as opposed to measurements made elsewhere on the truck and carbody. Train speed was expected to play a major role in determining ride quality. As the train moves faster, there is more kinetic energy available to excite the car. Thus, one expects the accelerometer readings to increase roughly as the square of the train speed. Similarly, the distinction between measurements on the car and measurements on the truck were expected because the truck is designed to cushion the car from the rail. The unsprung mass at the wheelsets should respond more violently than the much heavier carbody. The difference in level between axle and car measurements merely indicates that the truck is operating as expected.

The effect of train speed on ride quality is clearly visible throughout the data. This is shown in Figures 4-1 through 4-4 where rms acceleration is plotted against train speed. An rms value is plotted for the first and second half of each speed zone. The true data are represented by the symbols. The lines connecting the symbols are for visual clarity only, and are not intended to represent any information at other speeds. Figures 4-1 and 4-2 show vertical and lateral acceleration as a function of train speed for travel over jointed rail and Figures 4-3 and 4-4 are for travel over continuous welded rail (CWR). The expected trend may be seen in each of the figures where the rms level tends to increase with speed. In particular the effect becomes more pronounced as the measurement is taken at locations closer to the rail. However, it tends to be obscured by a resonance phenomenon (e.g., buildup due to rocking at 50 mph in the 100-ton box car). This caused problems in estimating a squared relationship in the summary regression. A least square curve fit of the data in Figure 4-1 will tend to bow down because of the resonance. Without the resonance points in the data, the curve would tend to bow up, which is the desired effect. Thus, it was decided to use a straight line approximation instead of a least squared curve fit.

The cushioning effect of the truck is also illustrated in Figures 4-1 to 4-4. In Figure 4-1, the highest rms accelerations are shown to occur on the axle, with lower levels occurring on the side frame, and the lowest levels on the carbody itself. This is also indicated in the summary regression from Table 4-2. As mentioned earlier, data measured on the axle were separated from data measured elsewhere to obtain a more accurate representation. The marked differences between coefficients in these regressions indicate the size of distinction in the data, e.g., a slope of .00376 g rms/mph for data measured on the axle compared with .00172 g rms/mph for the rest of the data.

The ratio between vertical and lateral accelerations on the axle is quite different from that on the carbody. The lateral acceleration is a smaller proportion of the vertical acceleration on the axle than on the carbody. The distinction between CWR and jointed rail is larger numerically but is roughly the same proportion in the axle data as in the other data, and the distinction in carbodies is even smaller in the axle data than in the other data. Finally, the accelerometer on the side frame of the truck reads higher (.0884 g rms as shown in Table 4-2) than any of the other locations considered in the carbody regression. This suggests the extent to which the truck succeeds in cushioning the car.

Considering the car alone, the major influences on rms ride quality seem to be speed, and the distinction between empty and loaded cars. The level of the measured rms is dependent on the location of the transducer. The highest rms levels were measured on the axle with significantly lower levels being measured elsewhere on the truck and the carbody.

Empty and loaded rms acceleration plots in Figures 4-5 and 4-6 for both jointed and CWR show the empty car to have consistently higher levels. There seems to be little difference between the average level on jointed versus CWR. Accelerations on the roof averaged 0.0609 g rms higher than accelerations at the A-end center sill or at the center of the carbody on the floor of the car. No significant distinction was found between the center sill and the center of the car indicating that the carbody was fairly rigid, e.g., flexible modes of the car do not play a major role in these data. On the average, empty cars rode rougher than fully loaded cars by 0.0287 g rms. This was expected when the mass of the system decreases (i.e., the car is empty) the acceleration must increase if the force causing the motion does not decrease in proportion. Another interpretation is that the friction snubbers are sized for fully loaded cars; hence, they over-damp the empty cars.

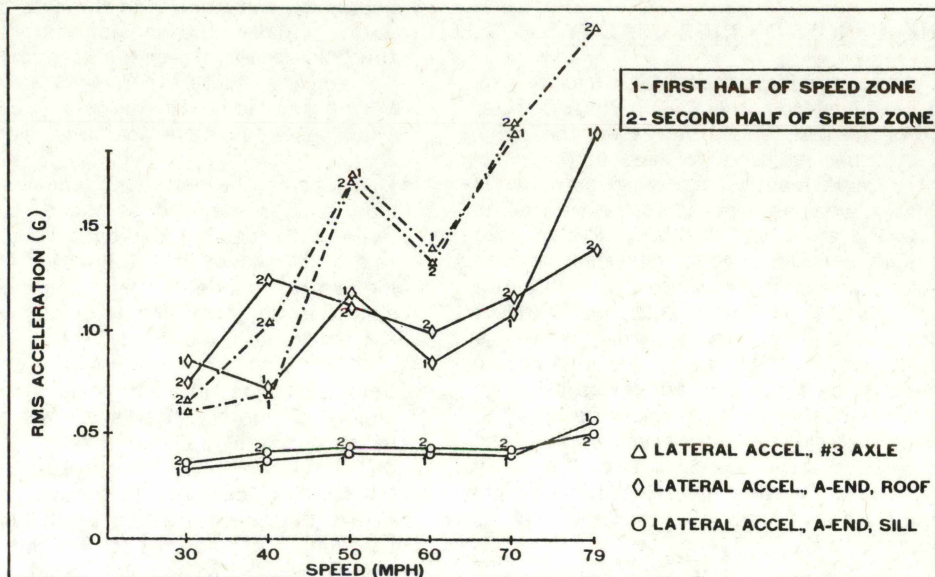


Figure 4-1. Fully Loaded, 100-Ton Box Car with Barber Trucks Over Jointed Tracks (Vertical Acceleration, R-1 Axle)

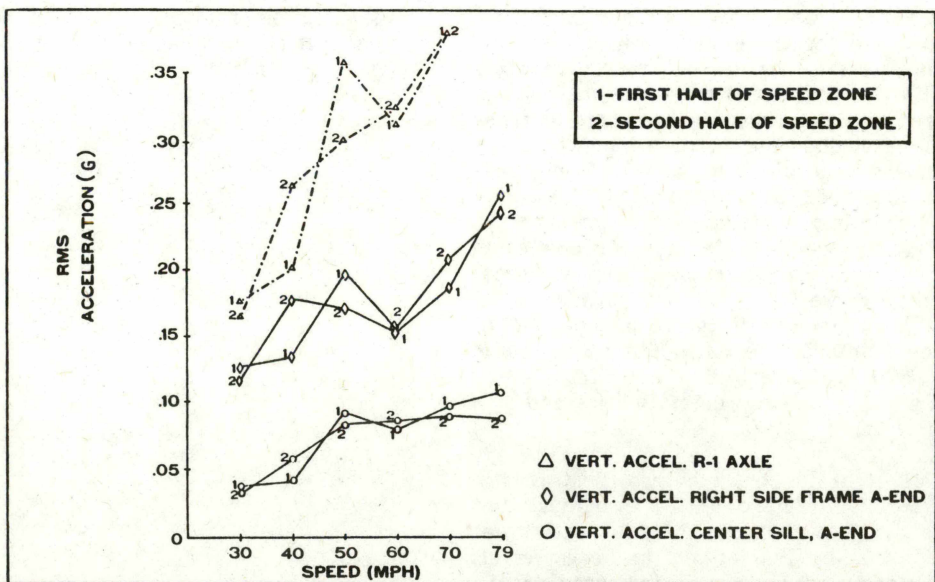


Figure 4-2. Fully Loaded, 100-Ton Box Car with Barber Trucks Over Jointed Track (Lateral Acceleration, #3 Axle)

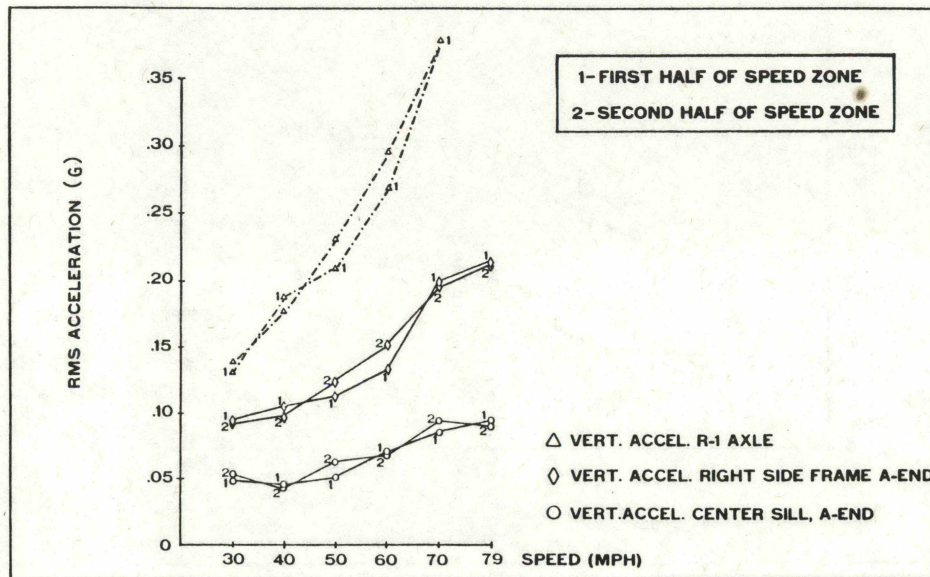


Figure 4-3. Fully Loaded, 100-Ton Box Car with Barber Trucks Over CWR Track (Vertical Acceleration, R-1 Axle)

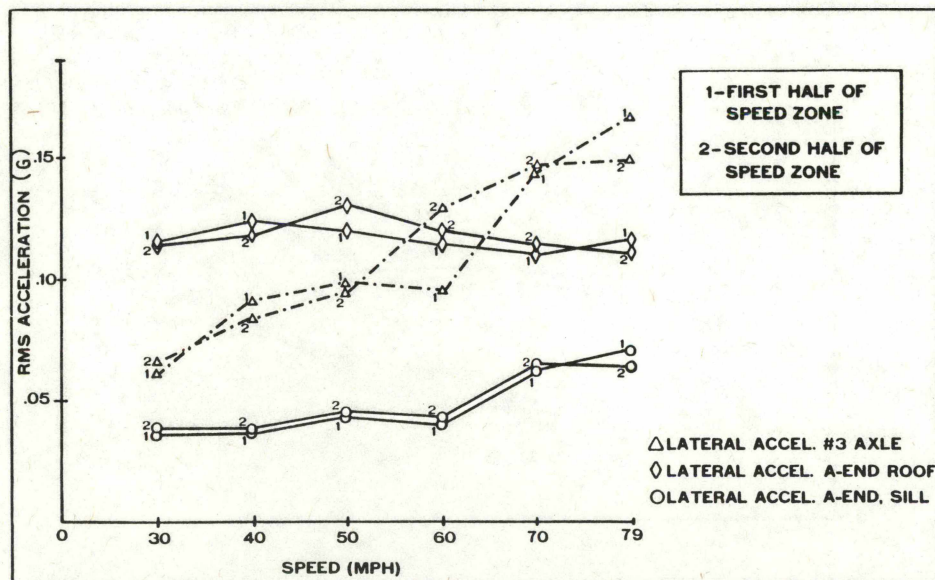


Figure 4-4. Fully Loaded, 100-Ton Box Car with Barber Trucks Over CWR Track (Lateral Acceleration, #3 Axle)

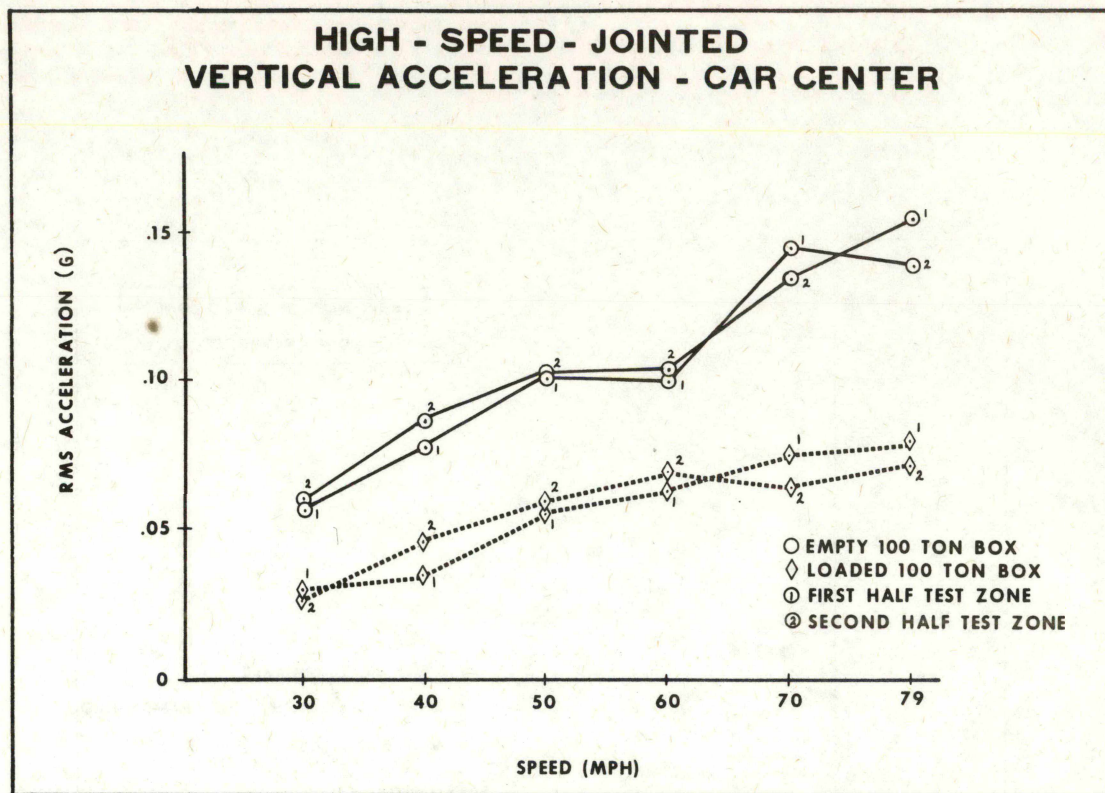


Figure 4-5. Empty vs Loaded rms Acceleration Plots (Jointed Track)

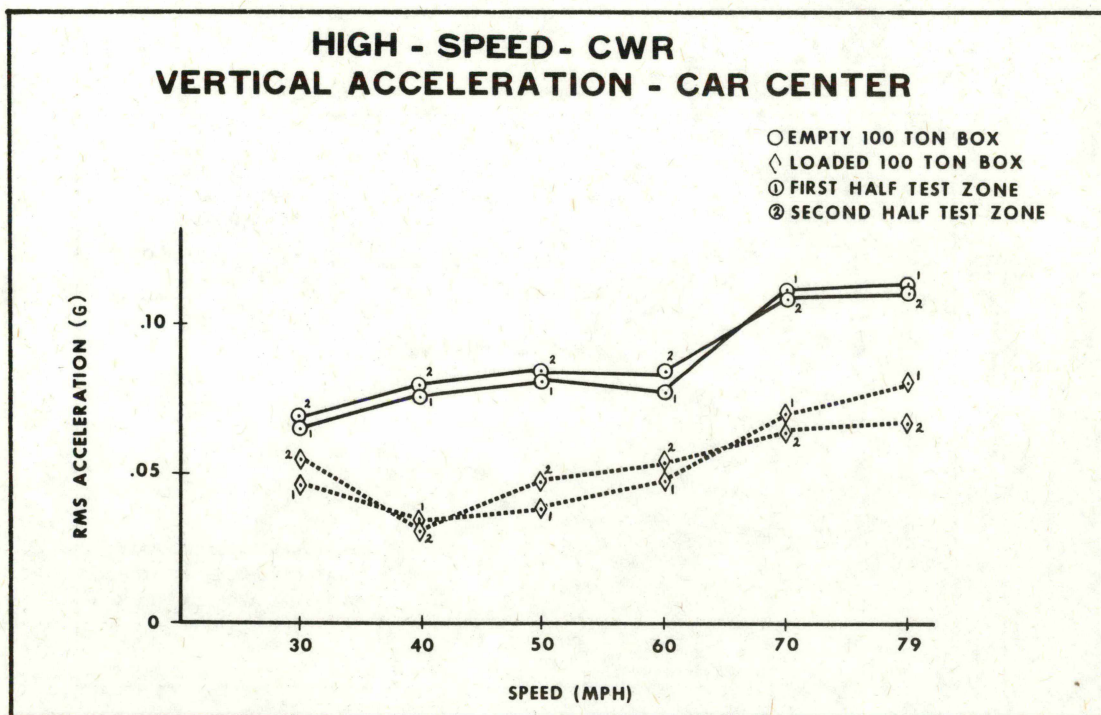


Figure 4-6. Empty vs Loaded rms Acceleration Plots (CWR Track)

4.4 SECONDARY INFLUENCES ON RIDE QUALITY

Most of the distinctions (carbody type, wheel profile, rail type) investigated had only a secondary influence on the ride quality. These are shown in Figures 4-7 through 4-9 which compare plots of rms acceleration for new vs. worn wheels, Barber vs. ASF truck, and 100-ton vs. 70-ton box cars. In each of these cases, the difference in rms acceleration is less than in previous plots. In particular, the type of carbody, the type of rail, and the accelerometer orientation all exhibited about .019 g rms effects. Regardless of the truck manufacturer, new wheels did not exhibit any influence that could be distinguished from zero at 5% confidence level.

Four carbodies were investigated: the 100-ton box car, the 70-ton box car, the 89-ft. flat car and the 100-ton hopper car. Only the 70-ton box car was significantly different from the other cars (averaging .0189 g rms more than the others). Interpreting the results for the 89-ft. flat car is compounded by the lack of data taken on trucks similar to the ones used in the other tests.

Only the ASF low-level truck was run under the 89-ft. flat car, and this truck was not run under any other carbody. Data for the 89-ft. flat car in a loaded condition have not been considered to date (where flexible behavior might be expected). The similarity of the results for the different carbodies tends to suggest the cars were behaving rigidly.

The rail type (i.e., CWR or jointed rail) showed the expected effect: the CWR averaged .0186 g rms less than the jointed rail. This supports the hypothesis that the joints are one of the causes of the excitation.

Similarly, laterally oriented accelerometers averaged .0185 g rms less than vertically oriented accelerometers. This suggests that most of the motion excited from the rail is vertical (at least in the ride quality regime).

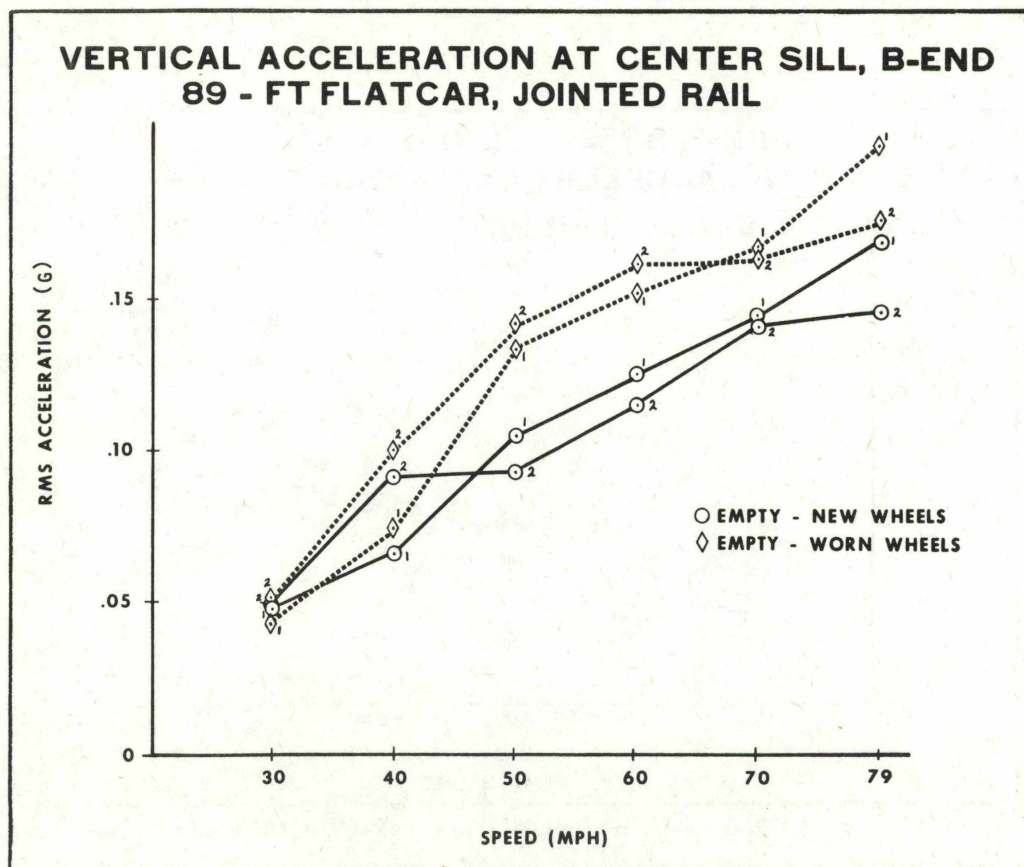


Figure 4-7. Comparative Plots of rms Acceleration (New vs Worn Wheels)

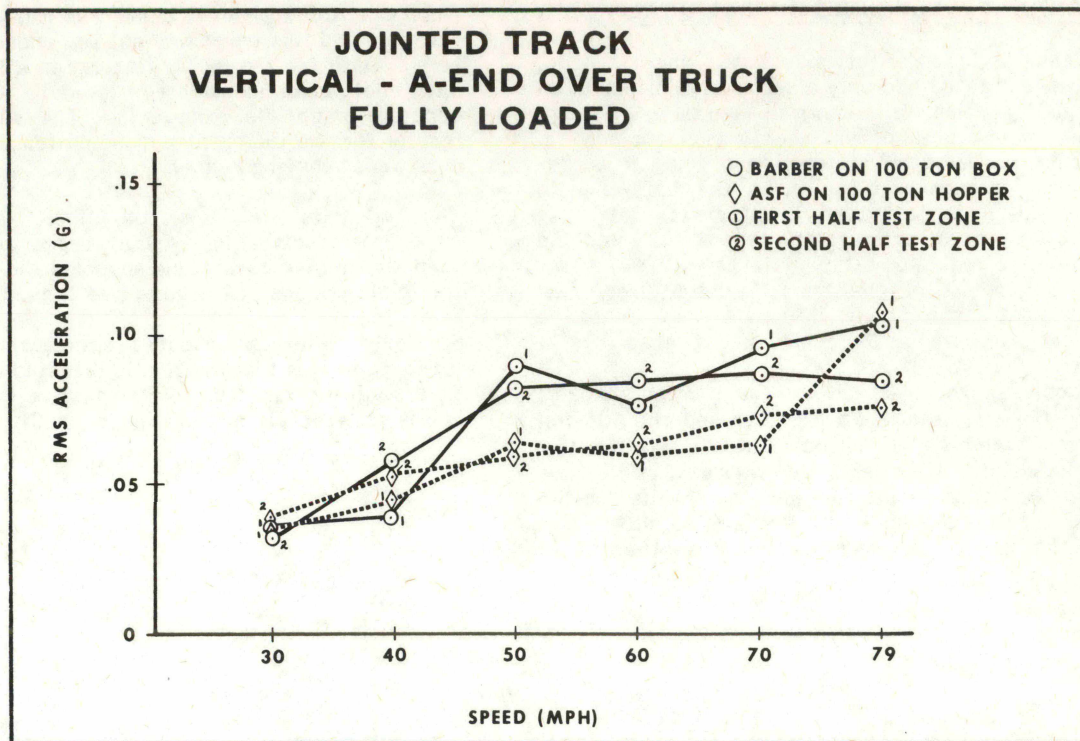


Figure 4-8. Comparative Plots of rms Acceleration (Barber vs ASF Truck)

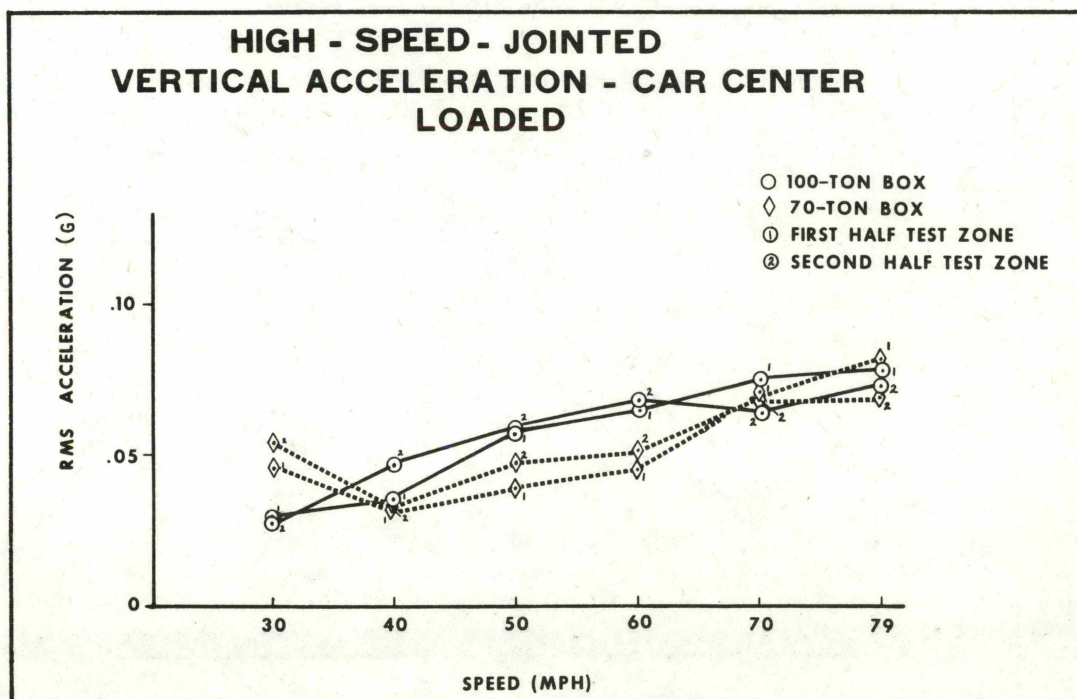


Figure 4-9. Comparative Plots of rms Acceleration (100 ton vs 70 ton)

4.5 RELATION TO ECONOMICS

Based upon the size of the measured effects (level of g rms), it seems doubtful that the levels and distinctions being reported here are large enough to have a major effect on lading damage (i.e., differences in the .02 g rms range probably are too small to play a major role in lading damage). The one exception is speed. From this analysis, it appears that operational considerations (train handling, humping, etc.) and sensitivity of the lading probably play a larger role in lading damage than the items discussed in Table 4-2. This is not to say that resonant phenomenon like harmonic roll or instabilities like hunting are not important in determining lading damage; however, for this pilot program, the concentration was on ride quality, rather than on these other performance regimes.

Another interpretation of these results is that the performance index selected (acceleration) does not measure the source of the problem. For example, the peak acceleration levels might be quite different from the rms. To assess this, the preceding analysis was rerun with peak acceleration and average absolute amplitude as performance indices rather than rms. Naturally, this changed the numbers obtained. However, the ratio between the numbers did not change significantly. Essentially, rms acceleration was as good a prediction of the size of differences between carbodies (for example) as peak acceleration. One exception was observed during the course of this analysis, but it was traced back to an accelerometer with insufficient sensitivity.

SECTION 5 - CONCLUSIONS

After acquiring the Phase I data, necessary changes were made to the SPTCo software (the Post Processing Program) to make it operational on the Interdata computer. The task of modifying the Post Processing Program to run on the Interdata computer proved to be quite difficult. The sorting routine that Wyle developed made it possible to obtain an accurate idea of what was available in terms of test configurations from the Phase I data. While there were many gaps in the available data as noted in this report, it was still possible to derive useful information from the data. This was demonstrated by the pilot program for the ride quality regime.

5.1 APPLICABILITY OF DATA TO PHASE II

The applicability of the Phase I test data to Phase II was evaluated from three points of view. The first was completeness of the test matrix. To determine this Wyle developed the TDOP data sorting routine; the results of this routine are shown in the series of matrix tables in Section 3.2. This analysis showed that the preponderance of the testing was conducted on the 70-ton refrigerator car with the ASF truck and new wheels. Wyle believes that the refrigerator car was not typical of most cars in service and that using these data from these measurements in any extensive manner might tend to bias the results of the analytical work. Thus, these data were not used in the pilot program and may be used only sparingly in the analytical and engineering tasks. Furthermore, some significant configuration combinations are missing from the test matrix.

The second manner in which the Phase I data were evaluated was from the point of measurement accuracy; how well did a given combination or set of channels reproduce the desired measurement parameter? In all areas except two, the quality of measurements was acceptable. These two unacceptable areas were in measurement of lateral wheel force at the wheel/rail interface and in the detection of ALD targets. In particular, the lack of lateral forces at the wheel/rail interface is of critical importance to TDOP Phase II. Without it, there is little that may be done in validating curving models or assessing the curve negotiation performance indices on the Type I truck. Also, these missing data will have a secondary influence on the analysis of lateral stability because the time-domain models cannot be validated. The lack of ALD target detection (not being able to correlate ALD targets with response) limits the usefulness of the data for analysis of the trackability regime. The lack of ALD correlation hampers the ride quality evaluation to a lesser degree.

The third point of view was in the Phase I data's adequacy to perform the Type I truck model validation and specification of performance indices. In paragraph 3.4, the required data vs. available data from Phase I is shown. The data in the regimes of ride quality and lateral stability appear to be adequate for the Phase II effort. In the regimes of curve negotiation and trackability, the lack of adequate measurements of wheel/rail forces makes it more difficult to extract meaningful information from the data.

In summary, the one critical flaw with the Phase I data was the lack of lateral force measurements at the wheel/rail interface. This deficiency will require correction via additional testing of the Type I truck during TDOP Phase II.

5.2 USAGE OF DATA DURING PHASE II

This evaluation and analysis study was conducted to determine the applicability of data acquired during Phase I to the analytical and engineering effort being conducted on TDOP Phase II. The results of this applicability were discussed in detail in Section 3. The usage of data will be addressed in the analytical and engineering task efforts.

The data analysis routines in the Post Processing Program were reviewed and corrected so that correctly analyzed data will be obtained from the data analysis.

5.3 RECOMMENDATIONS FOR FUTURE TESTING

The critical lack in Phase I data of lateral and vertical force measurements at the wheel/rail interface must be corrected in TDOP Phase II. The primary goal of testing on the Type I truck during Phase II will be to measure these forces. The first step will be to conduct an extensive study of available techniques for measuring these forces and to prepare recommendations for a technique to be used during Phase II. The second step will be to develop the transducers necessary to provide the required measurements. To measure the angle of attack of the wheels relative to the rail, we plan to use displacement transducers which will measure the relative position of the wheel and rail.

To provide a positive correlation between track geometry and carbody response data, an ALD system will be developed which will explicitly determine the test car location relative to the track. The ALD systems will consist of a buried magnet or tuned coil and a detector system on the instrumentation car which will sense the field of the buried target as the car passes over it. By placing the ALD system prior to starting Phase II testing and by using the same ALD system on all testing (track geometry, friction snubber, Type I truck, and Type II truck), it will be possible to correlate all measured data taken during Phase II of TDOP.

To complete the test matrices, test carbodies will be tested, using new and worn wheel profiles. The primary concentration of Phase II testing on Type I trucks will be in the curve negotiation performance regime. However, tests will be conducted as well for the trackability and lateral stability performance regimes. Data for the ride quality may then be extracted from the other three regimes. These tests should be run on vehicle configurations already prepared and instrumented for the curve negotiation tests and should not require any additional preparation time. The purposes for running the additional trackability and lateral stability tests are:

- a. To complete information not previously obtained during Phase I (i.e., 100-ton hopper car on the ASF truck and a hopper car with worn wheels).
- b. To provide some degree of continuity between Phase I and Phase II data (by repeating one or two Phase I runs, a comparison may then be made between data from the two programs).
- c. To provide a final validation check of models (i.e., models validated using the Phase I data may be used to predict Phase II test results).
- d. To provide test data over yard track to assess the ability of the truck to traverse severe changes in track configuration.

REFERENCES

1. FRA Report No. FRA/ORD-78/34, "TDOP Phase I Data Evaluation and Analysis Plan," September 1978.
2. FRA Report No. FRA/ORD-78/12.XIII, Volume 12, "TDOP Post Processing Program Manual," February 1978.
3. Wyle document TDOP TR-OX, "User Operator's Manual for Post Processing Program Operation on Interdata 8/32," May 1978.
4. FRA Report No. FRA/ORD-78/12, "Freight Car Truck Design Optimization, Volume I, Executive Summary; Volume VI, Critique of Phase I - Test Series Results Reports; Volumes VII through X, Results Report for Test Series 1 through 5," February 1978.
5. FRA Report No. FRA/ORD-79/24, "Friction Snubber Force Measurement System Field Test Report," August 1979.

APPENDIX A INVENTORY OF PHASE I DATA

01/26/78

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TAPE 440 INPUT 3 040301TSM001 040301TSR001	BOX 12
TAPE 441 INPUT 9 040301TEM001 040301TEH001 040301TWA001	BOX 12
TAPE 442 INPUT 7 040301TWA001 040301TSM001 040301TSR001	BOX 12
TAPE 443 INPUT 7 040301TEM001 040301TEH001	BOX 12
TAPE 444 INPUT 8 040301TWA001 040301TEM001 040301TEH001	BOX 12
TAPE 445 040301TSM001 040301TSR001	BOX 12
TAPE 446 INPUT 3 040301TSM001 040301TSR001	BOX 12
TAPE 447 INPUT 22 040301TFH001 040301TFH001 040301TWA001	BOX 12
TAPE 448 INPUT 37 040401CFH001 040401CFH001 040401CFH001	BOX 12
TAPE 449 INPUT 38 040401CFH001 040401CFH001 040401CFH001	BOX 12
BOX 13 TAPES 450-460	
TAPE 450 INPUT 1 040401TSM001 040401TSR001 040401TSM001	BOX 13

DESCRIPTION	LOCATION
TAPE 451 040401TSR001 040401TSM001 040401TSR001	BOX 13
TAPE 452 INPUT 1 040401TWA001 040401TWA001 040401TWA001	BOX 13
TAPE 453 INPUT 2 040401TEM001 040401TEH001 040401TEM001 040401TEH001	BOX 13
TAPE 454 INPUT 3 040401TEM001 040401TEH001	BOX 13
TAPE 455 INPUT 10 040401CNE001 040401CNO001	BOX 13
TAPE 456 INPUT 15 040401CNR001 040401CFH001 040401CNO001	BOX 13
TAPE 457 INPUT 11 040401CNE001 040401CNO001 040401CNR001	BOX 13
TAPE 458 INPUT 7 040401TEM001 040401TEH001 040401TEH001 040401TEH001	BOX 13
TAPE 459 INPUT 7 040401TEM001 040401TEH001	BOX 13
TAPE 460 INPUT 4 040401TWA001 040401TWA001 040401TWA001	BOX 13
BOX 14 TAPES 461-470	
TAPE 461 INPUT 16 040401TEM001 040401TSM001 040401TSR001	BOX 14
TAPE 462 INPUT 1 040401TSM001 040401TSR001	BOX 14
TAPE 463 INPUT 2 040401TSM001 040401TSR001	BOX 14
TAPE 464 INPUT 3 040501TEM001 040501TEH001 040501TSM001	BOX 14
TAPE 465 INPUT 4 040501TSR001 040501TWA001	BOX 14
TAPE 466 040501CNE001 040501CNR001 040501CNO001	BOX 14
TAPE 467 INPUT 13 040501CNE001 040501CNO001	BOX 14
TAPE 468 INPUT 16 040501TEM001 040501TEH001	BOX 14
TAPE 469 INPUT 10 040501TSM001	BOX 14

DESCRIPTION	LOCATION
040501TSR001 040501TWA001	
TAPE 470 050104CNE001 050104CNO001	BOX 14
472 INPUT 7 050104TWA001 050104TEA001	BOX 15
471 INPUT 5 050104MOD001 050104TSM001 050104TWA001	BOX 15
473 INPUT 10 050401MOD001 050401TSM001	BOX 15
474 INPUT 12 050104TEA002	BOX 15
475 INPUT 11 050104TWA002	BOX 15
476 INPUT 10 050401TWA001	BOX 15
477 INPUT 14 050104CNE001 050104CNO001	BOX 15
478 INPUT 15 050104MOD002 050104TSM003	BOX 15
479 INPUT 13 050104MOD001 050104TSM001 050104TWA001	BOX 15
480 INPUT 14 050104TEA001	BOX 15
481 INPUT 1 050104TSM002	BOX 15
BOX 16 TAPES 482-492	
TAPE 482 INPUT 15 050104CNE001 050104CNO001	BOX 16
TAPE 483 INPUT 21 050201MOD001 050201TSM001 050201TWA001	BOX 16
TAPE 484 INPUT 22 050201TEA001	BOX 16
TAPE 485 INPUT 16 050201CNE001 050201CNR001 050201CNO001	BOX 16
TAPE 486 INPUT 15 050401CFH001 050401CFH001 050401CNO001	BOX 16
TAPE 487 INPUT 16 050401CNE001 050401CNR001 050401CNO001	BOX 16
TAPE 488 INPUT 11 050401MOD001 050401TSM001	BOX 16
TAPE 489 050401TWA001 050401TEA001	BOX 16
TAPE 490 050401CNR001 050401CNE001 050401CNO001	BOX 16
TAPE 491 INPUT 12 050401MOD001 050401TSM001	BOX 16
TAPE 492 INPUT 23 050401TEA001 050401TWA001	BOX 16

DESCRIPTION	LOCATION	DESCRIPTION	LOCATION	DESCRIPTION	LOCATION
040201TEM/H		040501CNE		050101TSM001	
040202TEM/H		040401CNE		050101TSM001	
040203TEM/H		040501CNO		050101TSM001	
040204TEM/H		040401CNO		050201TSM001	
040201TWA002		040501TSM/R		050201TSM001	
040202TWA001		040401TSM/R		050201TSM001	
040203TWA001		040401TEM/H		050201TSM001	
040204TWA001		040501TEM/H		050201TSM001	
040301TWA001		040501CNR		050201TSM001	
040302TWA001		040401CNR		050201TSM001	
040303TWA001		040401CNR		050201TSM001	
040201TSM/R		OUTPUT PAPER	BOX 35	050201TSM001	
040202TSM/R		040301TWA		050201TSM001	
040203TSM/R		040201TWA		050201TSM001	
040204TSM/R		040301TWA		050201TSM001	
040301TSM/R		040201TWA		050201TSM001	
040302TSM/R		040301TWA		050201TSM001	
040303TSM/R		040101TWA		050201TSM001	
040401TWA001		010201TEM		050201TSM001	
040402TWA001		040301TWA		050201TSM001	
040403TWA001		040301TWA001		050201TSM001	
040404TWA001		040302TWA001		050201TSM001	
040405CNE		040303TWA001		050201TSM001	
040406CNE		040304TWA001		050201TSM001	
040407CNE		050101MOD		050201TSM001	
040408CNE		050102MOD		050201TSM001	
040501CNE		050101TEA002		050201TSM001	
040401CNE		050102TEA001		050201TSM001	
040402TSM/R		050103TEA001		050201TSM001	
040403TSM/R		050104TEA001		050201TSM001	
040404TSM/R		050105TEA001		050201TSM001	
040405TSM/R		050106TEA001		050201TSM001	
040406TSM/R		050107TEA001		050201TSM001	
040407TSM/R		050108TEA001		050201TSM001	
040408TSM/R		050109TEA001		050201TSM001	
040409TSM/R		050110TEA001		050201TSM001	
040410TSM/R		050111TEA001		050201TSM001	
040411TSM/R		050112TEA001		050201TSM001	
040412TSM/R		050113TEA001		050201TSM001	
040413TSM/R		050114TEA001		050201TSM001	
040414TSM/R		050115TEA001		050201TSM001	
040415TSM/R		050116TEA001		050201TSM001	
040416TSM/R		050117TEA001		050201TSM001	
040417TSM/R		050118TEA001		050201TSM001	
040418TSM/R		050119TEA001		050201TSM001	
040419TSM/R		050120TEA001		050201TSM001	
040420TSM/R		050121TEA001		050201TSM001	
040421TSM/R		050122TEA001		050201TSM001	
040422TSM/R		050123TEA001		050201TSM001	
040423TSM/R		050124TEA001		050201TSM001	
040424TSM/R		050125TEA001		050201TSM001	
040425TSM/R		050126TEA001		050201TSM001	
040426TSM/R		050127TEA001		050201TSM001	
040427TSM/R		050128TEA001		050201TSM001	
040428TSM/R		050129TEA001		050201TSM001	
040429TSM/R		050130TEA001		050201TSM001	
040430TSM/R		050131TEA001		050201TSM001	
040431TSM/R		050132TEA001		050201TSM001	
040432TSM/R		050133TEA001		050201TSM001	
040433TSM/R		050134TEA001		050201TSM001	
040434TSM/R		050135TEA001		050201TSM001	
040435TSM/R		050136TEA001		050201TSM001	
040436TSM/R		050137TEA001		050201TSM001	
040437TSM/R		050138TEA001		050201TSM001	
040438TSM/R		050139TEA001		050201TSM001	
040439TSM/R		050140TEA001		050201TSM001	
040440TSM/R		050141TEA001		050201TSM001	
040441TSM/R		050142TEA001		050201TSM001	
040442TSM/R		050143TEA001		050201TSM001	
040443TSM/R		050144TEA001		050201TSM001	
040444TSM/R		050145TEA001		050201TSM001	
040445TSM/R		050146TEA001		050201TSM001	
040446TSM/R		050147TEA001		050201TSM001	
040447TSM/R		050148TEA001		050201TSM001	
040448TSM/R		050149TEA001		050201TSM001	
040449TSM/R		050150TEA001		050201TSM001	
040450TSM/R		050151TEA001		050201TSM001	
040451TSM/R		050152TEA001		050201TSM001	
040452TSM/R		050153TEA001		050201TSM001	
040453TSM/R		050154TEA001		050201TSM001	
040454TSM/R		050155TEA001		050201TSM001	
040455TSM/R		050156TEA001		050201TSM001	
040456TSM/R		050157TEA001		050201TSM001	
040457TSM/R		050158TEA001		050201TSM001	
040458TSM/R		050159TEA001		050201TSM001	
040459TSM/R		050160TEA001		050201TSM001	
040460TSM/R		050161TEA001		050201TSM001	
040461TSM/R		050162TEA001		050201TSM001	
040462TSM/R		050163TEA001		050201TSM001	
040463TSM/R		050164TEA001		050201TSM001	
040464TSM/R		050165TEA001		050201TSM001	
040465TSM/R		050166TEA001		050201TSM001	
040466TSM/R		050167TEA001		050201TSM001	
040467TSM/R		050168TEA001		050201TSM001	
040468TSM/R		050169TEA001		050201TSM001	
040469TSM/R		050170TEA001		050201TSM001	
040470TSM/R		050171TEA001		050201TSM001	
040471TSM/R		050172TEA001		050201TSM001	
040472TSM/R		050173TEA001		050201TSM001	
040473TSM/R		050174TEA001		050201TSM001	
040474TSM/R		050175TEA001		050201TSM001	
040475TSM/R		050176TEA001		050201TSM001	
040476TSM/R		050177TEA001		050201TSM001	
040477TSM/R		050178TEA001		050201TSM001	
040478TSM/R		050179TEA001		050201TSM001	
040479TSM/R		050180TEA001		050201TSM001	
040480TSM/R		050181TEA001		050201TSM001	
040481TSM/R		050182TEA001		050201TSM001	
040482TSM/R		050183TEA001		050201TSM001	
040483TSM/R		050184TEA001		050201TSM001	
040484TSM/R		050185TEA001		050201TSM001	
040485TSM/R		050186TEA001		050201TSM001	
040486TSM/R		050187TEA001		050201TSM001	
040487TSM/R		050188TEA001		050201TSM001	
040488TSM/R		050189TEA001		050201TSM001	
040489TSM/R		050190TEA001		050201TSM001	
040490TSM/R		050191TEA001		050201TSM001	
040491TSM/R		050192TEA001		050201TSM001	
040492TSM/R		050193TEA001		050201TSM001	
040493TSM/R		050194TEA001		050201TSM001	
040494TSM/R		050195TEA001		050201TSM001	
040495TSM/R		050196TEA001		050201TSM001	
040496TSM/R		050197TEA001		050201TSM001	
040497TSM/R		050198TEA001		050201TSM001	
040498TSM/R		050199TEA001		050201TSM001	
040499TSM/R		050200TEA001		050201TSM001	
040500TSM/R		050201TEA001		050201TSM001	
040501TSM/R		050202TEA001		050201TSM001	
040502TSM/R		050203TEA001		050201TSM001	
040503TSM/R		050204TEA001		050201TSM001	
040504TSM/R		050205TEA001		050201TSM001	
040505TSM/R		050206TEA001		050201TSM001	
040506TSM/R		050207TEA001		050201TSM001	
040507TSM/R		050208TEA001		050201TSM001	
040508TSM/R		050209TEA001		050201TSM001	
040509TSM/R		050210TEA001		050201TSM001	
040510TSM/R		050211TEA001		050201TSM001	
040511TSM/R		050212TEA001		050201TSM001	
040512TSM/R		050213TEA001		050201TSM001	
040513TSM/R		050214TEA001		050201TSM001	
040514TSM/R		050215TEA001		050201TSM001	
040515TSM/R		050216TEA001		050201TSM001	
040516TSM/R		050217TEA001		050201TSM001	
040517TSM/R		050218TEA001		050201TSM001	
040518TSM/R		050219TEA001		050201TSM001	
040519TSM/R		050220TEA001		050201TSM001	
040520TSM/R		050221TEA001		050201TSM001	
040521TSM/R		050222TEA001		050201TSM001	
040522TSM/R		050223TEA001		050201TSM001	
040523TSM/R		050224TEA001		050201TSM001	
040524TSM/R		050225TEA001		050201TSM001	
040525TSM/R		050226TEA001		050201TSM001	
040526TSM/R		050227TEA001		050201TSM001	
040527TSM/R		050228TEA001		050201TSM001	
040528TSM/R		050229TEA001		050201TSM001	
040529TSM/R		050230TEA001		050201TSM001	
040530TSM/R		050231TEA001		050201TSM001	
040531TSM/R		050232TEA001		050201TSM001	
040532TSM/R		050233TEA001		050201TSM001	
040533TSM/R		050234TEA001		050201TSM001	
040534TSM/R		050235TEA001		050201TSM001	
040535TSM/R		050236TEA001		050201TSM001	
040536TSM/R		050237TEA001		050201TSM001	
040537TSM/R		050238TEA001		050201TSM001	
040538TSM/R		050239TEA001		050201TSM001	
040539TSM/R		050240TEA001		050201TSM001	
040540TSM/R		050241TEA001		050201TSM001	
040541TSM/R		050242TEA001		050201TSM001	
040542TSM/R		050243TEA001		050201TSM001	
040543TSM/R		050244TEA001		050201TSM001	
040544TSM/R		050245TEA001		050201TSM001	
040545TSM/R		050246TEA001		050201TSM001	
040546TSM/R		050247TEA001		050201TSM001	
040547TSM/R		050248TEA001		050201TSM001	
040548TSM/R		050249TEA001		050201TSM001	
040549TSM/R		050250TEA001		050201TSM001	
040550TSM/R		050251TEA001		050201TSM001	
040551TSM/R		050252TEA001		050201TSM001	
040552TSM/R		050253TEA001		050201TSM001	
040553TSM/R		050254TEA001		050201TSM001	
040554TSM/R		050255TEA001		050201TSM001	
040555TSM/R		050256TEA001		050201TSM001	
040556TSM/R		050257TEA001		050201TSM001	
040557TSM/R		050258TEA001		050201TSM001	
040558TSM/R		050259TEA001		050201TSM001	
040559TSM/R		050260TEA001		050201TSM001	
040560TSM/R		050261TEA001		050201TSM001	
040561TSM/R		050262TEA001		050201TSM001	
040562TSM/R		050263TEA001		050201TSM001	
040563TSM/R		050264TEA001		050201TSM001	
040564TSM/R		050265TEA001		050201TSM001	
040565TSM/R		050266TEA001		050201TSM001	
040566TSM/R		050267TEA001		050201TSM001	
040567TSM/R		050268TEA001		050201TSM001	
040568TSM/R		050269TEA001		050201TSM001	
040569TSM/R		050270TEA001		050201TSM001	
040570TSM/R		050271TEA001		050201TSM001	
040571TSM/R		050272TEA001		050201TSM001	
040572TSM/R		050273TEA001		050201TSM001	
040573TSM/R		050274TEA001		050201TSM001	

DESCRIPTION	LOCATION	DESCRIPTION	LOCATION
050602TEA001		EMPTY TEST 030202	
050401TEA001		SERIES 3 BUNK 5 50-FT 70-TON BOX CAR	BOX 50
050401TWA001		LOADFN TEST 030301	
050402CNO001		SERIES 3 BUNK 6 50-FT 70-TON BOX CAR	BOX 50
050401TWA		EMPTY TEST 030302	
050402TWA		SERIES 3 BUNK 7 100-TON COVERED HOPPER	BOX 50
050403TWA		LOADFN TEST 030401	
050301TWA		SERIES 3 BUNK 8 100-TON COVERED HOPPER	BOX 50
050302TWA		EMPTY TEST 030402	
050303TWA		SERIES 3 BUNK 9 89-FT FLAT CAR LOADED	BOX 51
		TEST 030502	
BOX 40 OUTPUT PAPER SERIES 4-4-X		SERIES 3 BUNK 10 89-FT FLAT CAR EMPTY	BOX 51
OUTPUT PAPER	BOX 40	TEST 030501	
050402TSM001		SERIES 2 BUNK 1 RMS, HISTOGRAMS AND	BOX 52
050402TSM001		PSD's	
050402TSM001		SERIES 2 BUNK 2 TIME DOMAIN GRAPHS	BOX 52
050402MOD001		AVERAGE WORN WHEELS (EMPTY)	
050401MOD001		SERIES 4 BUNK 5 CURVED RAIL EMPTY	BOX 52
050402TWA001		TESTS 040X0YCNE, 040X0YCNO	
050402TEA001		SERIES 4 BUNK 4 CURVED RAIL LOADED	BOX 52
050402CNO001		TESTS 040X0YCNR, 040X0YCNE	
050402CNE001		040X0YCNO	
050402CNR001		SERIES 4 BUNK 3 MEDIUM SPEED JOINTED	BOX 52
050402CNR001		RAIL TESTS 040X0YTSN, 040X0YTSR	
050402CNR001		SERIES 2 BUNK 3 TIME DOMAIN GRAPHS	BOX 53
050402CNR001		WORN WHEELS D-5 SPRINGS (LOADED)	
050402CNE001		SERIES 2 BUNK 4 TIME DOMAIN GRAPHS	BOX 53
050402MOD001		WORN WHEELS D-3 + D-7 SPRINGS	
050402TWA001		(LOADED)	
050402CNO001		SERIES 2 BUNK 5 TIME DOMAIN GRAPHS	BOX 53
050402MOD001		WORN WHEELS D-5 SPRINGS (EMPTY)	
050402TWA001		SERIES 2 BUNK 6 TIME DOMAIN GRAPHS	BOX 53
050402TWA001		WORN WHEELS D-3 + D-7 SPRINGS	
050402TWA001		(EMPTY)	
050402TSM		SERIES 2 BUNK 7 TIME DOMAIN GRAPHS	BOX 53
050402TSM		NEW WHEELS, D-5 SPRINGS, 2/3 SNIRRING	
050402TSM		(LOADED AND EMPTY)	
050402TSM		SERIES 3 BUNK 11 89-FT FLAT CAR EMPTY	BOX 53
050402TSM		WORN WHEELS TEST 030503	
050402TSM		SERIES 4 BUNK 1 HIGH SPEED JOINTED	BOX 53
050402TSM		RAIL TEST 040X0YTWA	
050402TSM		SERIES 4 BUNK 2 HIGH SPEED WELDED	
050402TSM		RAIL TEST 040X0YTEM, 040X0YTEM	
050402TSM		SERIES 2 GRAPHS BOOK A	BOX 54
050402TSM		SERIES 5 GRAPHS BOOK 1-4	BOX 54
050402TSM		SERIES 5 BUNKS 70-TON CAR CURVED TRACK	BOX 55
050402TSM		OVER-FQ. SPEED	
050402TSM		05010XCNO	
050402TSM		05020XCNO	
050402TSM		BOOK 4 70-TON CAR CURVED	
050402TSM		TRACK EQUILIRRIUM SPEED	
050402TSM		05010XCNE	
050402TSM		02020XCNE	
050402TSM		BOOK 7 70-TON CAR CURVED	
050402TSM		TRACK RESONANT SPEED	
050402TSM		05020XCNR	
050402TSM		BOOK 8 100-TON CAR HIGH-SPEED	
050402TSM		JOINTED RAIL	
050402TSM		05030XTWA	
050402TSM		05040XTWA	
050402TSM		BOOK 9 100-TON CAR HIGH-SPEED	
050402TSM		WELDED RAIL	
050402TSM		05030XTWA	
050402TSM		05040XTWA	
050402TSM		BOOK 10 100-TON CAR MODIFIED	
050402TSM		TRACK	
050402TSM		05030XTWA	
050402TSM			
SERIES 1 ORIGINALS	BOX 41		
SERIES 2 PLOTS TIME DOMAIN 8.5X11	BOX 42		
SERIES 3 PLOTS, REEFER, BOX RCL	BOX 43		
SERIES 3 PLOTS FLAT RCL	BOX 44		
SERIES 4 PLOTS	BOX 45		
SERIES 3 PLOTS REEFER, BOX RCL			
SERIES 5 PLOTS 8.5 X 11	BOX 46		
SERIES 2 ORIGINALS	BOX 47		
SERIES 5 TIME PLOTS, ORIGINALS	BOX 48		
SERIES 1 BUNK 1 LOADED REEFER	BOX 49		
SERIES 1 BUNK 2 HALF LOADED	BOX 49		
SERIES 3 BUNK 1 MECHANICAL REFRIGER-	BOX 49		
ATOR LOADED TEST 030102			
BUNK 2 MECHANICAL REFRIGER-			
ATOR LOADED TEST 030101			
BUNK 3 60-FT 100-TON BOX			
CAR LOADED TEST 030201			
SERIES 3 BUNK 4 60-FT 100-TON BOX CAR	BOX 50		

DESCRIPTION	LOCATION
0504nXMOD	
SERIES 5 ROKK 11 100-TON CAR MEDIUM SPEED JOINTED RAIL 05030XTSM 0504nXTSM	BOX 56
SERIES 5 ROKK 12 100-TON CURVED TRACK OVER-FQ. SPEED 05030XCNO 05040XCNO	BOX 56
SERIES 5 ROKK 13 100-TON CAR CURVED TRACK EQ. SPEED 05030XCNE 05040XCNE	BOX 56
SERIES 5 ROKK 14 100-TON CAR CURVED TRACK RESONANT SPEED 05040XCNR	BOX 56
CLEAN TAPER	BOX 100
16240-77-2-40	
16240-77-2-38	
16240-77-2-30	
16240-77-2-44	
16240-77-2-42	
16240-80-3-41	
16240-80-2-39	
16240-80-3-31	
16240-80-3-29	
16240-80-3-49	
TDRP DRAWINGS	BOX 57
ROLL A	
ROLL R SCL BOX CLASS X-5-B	
ROLL R R-70-24	
ROLL R B-100-33	
ROLL F L + N HOPPER	
ROLL F F-70-65	
ROLL R 70 TON ASF RIDE CONTROL	
ROLL H 70 TON BARBER S-2 STARLIZED	
ROLL Y 100 TON BARBER S-2 STARLIZED LOW PROFILE	
ROLL J 70 TON ASF LOW LEVEL	
ROLL K 100 TON ASF AT RIDE CONTROL	
ROLL I TIMKEN MOD + STD ADAPTERS 70 + 100 TON	
BOX 101 BLANK TAPES	BOX 102
16240-67-3-15	
16240-67-3-05	
16240-69-3-38	
16240-67-3-13	
16240-67-3-11	
16240-69-3-42	
16240-69-3-40	
16240-69-3-46	
16240-69-3-44	
16240-67-3-17	
BLANK TAPER	BOX 103
16240-77-1-36	
16240-77-1-50	
16240-77-1-30	
16240-77-1-28	
16240-77-1-32	
16240-77-1-48	
16240-77-1-34	
16240-77-1-20	
16240-77-1-1	
16240-77-1-16	
BLANK TAPER	BOX 104
16240-80-4-31	
16240-80-4-35	
16240-77-1-14	
16240-80-4-25	
16240-80-4-27	
16240-77-2-14	
16240-80-4-17	
16240-80-4-21	
16240-80-4-47	
16240-80-4-09	
BLANK TAPER	
16240-80-4-37	

DESCRIPTION	LOCATION
16240-77-3-18	
16240-77-3-49	
16240-77-4-48	
16240-77-4-41	
16240-77-3-43	
16240-77-4-34	
16240-77-4-46	
16240-77-3-10	
16240-77-3-09	
BLANK TAPER	BOX 105
16240-69-2-46	
16240-69-2-44	
16240-69-2-42	
16240-69-2-40	
16240-69-1-10	
16240-69-1-16	
16240-69-1-46	
16240-69-1-20	
16240-67-3-26	
16240-69-2-30	
16111:30 /ENDJNR	
16111:35 /FINI	

APPENDIX B - COMPUTER PROGRAMS

PHASE I DATA SORTING ROUTINE

Wyle Laboratories developed a data sorting routine to provide a ready access to, and analysis of, Phase I data. A subsequent upgrading consisted of bringing the sort program parameter into agreement with the information contained in the Phase I Final Report, (1) and in the magnetic data tape headers. The tape header for those runs that the data were reduced are contained in Appendix C.

The sorting routine permits the user to specify a given set of test conditions; the program then lists all test runs which meet that set of requirements. The program sorts on the nineteen parameters or test conditions listed in Table B-1. Shown below each parameter in this table are the possible variations of the parameter and the user code which is specified for a search on that parameter. Any combination of the user codes may be used when making a search. However, many combinations will produce a null set. As an example, a sort for 100-ton box car and 70-ton ASF ride control truck will produce a sort with no entries.

An example of a typical sort printout is contained in Table B-2. This was a sort for all tests which were run over high-speed, jointed track on a fully loaded 100-ton box car, with a Barber 100-ton truck equipped with new wheels. The result was one test run consisting of six entries, one each at 30, 40, 50, 60, 70 and 79 mph. The Test ID, inventory box location, and the tape number are contained in the first two lines. With this information, it is possible to determine the required tape and retrieve it for data reduction. This sorting program was used extensively in the evaluation of the Phase I data.

POST PROCESSING PROGRAM

The Post Processing Program developed by SPTCo. was used to analyze the Phase I data. It was received from the FRA on magnetic tape and converted to run on Wyle's Interdata computer. Documentation for the program was provided by the Post Processing Program manual.(2) The effort required to convert this program to the Interdata computer proved more difficult than originally anticipated. The original program was supplied in EBCDIC and required conversion to seven-bit ASCII for the Interdata computer. The assembly language subroutines were completely rewritten using the Interdata assembly language and one subroutine was modified to work with the ASCII decimal equivalents rather than the EBCDIC decimal equivalents. To get the Post Processing Program to fit on the Interdata computer, it had to be overlayed and the number and size of plots that could be requested was reduced to 10 plots with 20 lines.

(1) FRA Report No. FRA/ORD-78/12.II, "Freight Car Truck Design Optimization Volume II, Phase I Final Report," February 1978.

(2) FRA Report No. FRA/ORD-78/12.XIII, Volume 12, "TDOP Post Processing Manual," February 1978.

The original manual for the program was revised (3) to include the changes that were made for operation on the Interdata computer. It contains samples of all files used in building the load module, a sample request deck, and the output from the request deck.

Some problems were also experienced in the original version of the program and have been noted. These problems were discovered during the course of reducing data and show that the program is not yet fully checked out. For example, at least one plot must be requested as there are many FORTRAN do-loops which use the total number of plots as the upper limit, and zero is not an acceptable value. Also, at least one set of equation cards must be included, even if the data are not required, because a zero equation is unacceptable to the program logic. Also, the original manual was not clear as to where blank cards must be positioned nor where they may be detected. These problems were eliminated in the revised version of the program.

SPTCo. recommended that at least 450k bytes of memory core be available for operation of the Post Processing Program. This total did not include space for such things as the operating system or any enhancements to the system, which would rule out use of the program without extensive modification on any machine with less than 512k bytes of core. Problems were also encountered by going to a non-IBM system, e.g., file structures were different, routines required rewriting, the EBCDIC-to-ASCII conversion was necessary, etc.

Difficulty may be encountered even when trying to install the Post Processing Program on an IBM computer. The manner in which the program handles peripheral devices, such as I/O and disk storage, would make its use on any IBM machine difficult, unless the system had a nearly identical set of peripherals to that for which the program was written. For example, the program was written to output to three separate line printers, and would require modification if a lesser number of line printers were available.

As previously discussed, a program manual is available from NTIS for running the Post Processing Program. While this manual was an invaluable guide in making the program operational on the Interdata computer, anyone attempting to use it should be aware of certain errors and unclear passages in the manual, as discussed in this report. Some of these have been noted in this report, however, not all analysis combinations were tried and some errors may still exist.

Program Validation

To assure the accuracy of data analyzed by using the Post Processing Program, Wyle executed a sequence of procedures to validate the operation of the program on the Interdata computer.

(3) Wyle Document TDOP TR-0X, User Operator's Manual for Post Processing Program Operation on Interdata 8/32," May 1978.

Table B-1. Sort Program Parameters

TAPE NO.			D6	USER CODE = 13, 6,
WHERE N = TAPE NO.	USER CODE = 1, N,		D7	USER CODE = 13, 7,
			D8	USER CODE = 13, 8,
CARBODY TYPE			SNUBBING (OUTER)	
70 TON M RFR	USER CODE = 2, 1,		ASF 3091	USER CODE = 15, 1,
100 TON 60' BOX	USER CODE = 2, 2,		2/3 NORMAL	USER CODE = 15, 2,
70 TON 50' BOX	USER CODE = 2, 3,		B 432	USER CODE = 15, 3,
89' FLT CAR	USER CODE = 2, 4,		ASF 3221	USER CODE = 15, 4,
100 TON C HPR	USER CODE = 2, 6,		B 421	USER CODE = 15, 6,
			B 422	USER CODE = 15, 8,
TRUCK TYPE			SNUBBING (INNER)	
ASF 70 TON R.C.	USER CODE = 3, 1,		B 433	USER CODE = 16, 1,
ASF 70 TON L.L.	USER CODE = 3, 2,		ASF 3222	USER CODE = 16, 2,
ASF 100 TON R.C.	USER CODE = 3, 3,		ASF 3092	USER CODE = 16, 3,
BARBER 70 TON	USER CODE = 3, 4,		8442	USER CODE = 16, 4,
BARBER 100 TON	USER CODE = 3, 5,		B 422	USER CODE = 16, 5,
			B 433	USER CODE = 16, 7,
			3091	USER CODE = 16, 8,
TRUCK CENTER			SNUB. AUG.	
45' 9"	USER CODE = 4, 1,		NO AUGMENTATION	USER CODE = 17, 1,
46' 3"	USER CODE = 4, 2,		VOLUTE	USER CODE = 17, 2,
41' 3"	USER CODE = 4, 3,		HYDRAULIC	USER CODE = 17, 3,
40' 10"	USER CODE = 4, 4,		TRUCK CEER AUG.	USER CODE = 17, 4,
64' 0"	USER CODE = 4, 5,			
PER CENT LOAD			C PLT. FRICTION	
EMPTY	USER CODE = 5, 1,		STEEL-MOLY	USER CODE = 18, 1,
HALF FULL	USER CODE = 5, 2,		COMP. STEEL	USER CODE = 18, 2,
FULLY LOADED	USER CODE = 5, 3,		STEEL-STEEL	USER CODE = 18, 3,
WHEEL PROFILE			FILE NO.	
1-20 (NEW)	USER CODE = 6, 1,		N	USER CODE = 19, N,
1-40 (NEW)	USER CODE = 6, 3,		WHERE N = FILE NO.	
CYLINDRICAL	USER CODE = 6, 3,			
HALF WORN	USER CODE = 6, 4,			
WORN	USER CODE = 6, 5,			
NO. OF OUTER SPG			TRACK	
N	USER CODE = 7, N,		CURVED	USER CODE = 20, 1,
WHERE N = NO. OF SPGS.			SHIMMED	USER CODE = 20, 2,
			HI SPD JTD	USER CODE = 20, 3,
SPG TYPE (OUTER)			HI SPD CWR	USER CODE = 20, 4,
D1	USER CODE = 9, 1,		MED SPD JTD	USER CODE = 20, 5,
D2	USER CODE = 9, 2,			
D3	USER CODE = 9, 3,		SPEED	
D4	USER CODE = 9, 4,		N	USER CODE = 21, N,
D5	USER CODE = 9, 5,		WHERE N = SPEED	
D6	USER CODE = 9, 6,			
D7	USER CODE = 9, 7,			
D8	USER CODE = 9, 8,			
NO. OF INNER SPG			OUTER GIB CLEARANCE	
N	USER CODE = 11, N,		1/4 OGC	USER CODE = 22, 1,
WHERE N = NO. OF SPGS.			5/8 OGC	USER CODE = 22, 2,
			SIDE BEARING	
SPG TYPE (INNER)			3/8 SB CLR	USER CODE = 23, 1,
D1	USER CODE = 13, 1,		1/4 SB CLR	USER CODE = 23, 2,
D2	USER CODE = 13, 2,		5/8 SB CLR	USER CODE = 23, 3,
D3	USER CODE = 13, 3,		1/8 SB CLR	USER CODE = 23, 4,
D4	USER CODE = 13, 4,		2.5 K PRELOAD	USER CODE = 23, 5,
D5	USER CODE = 13, 5,		5. K PRELOAD	USER CODE = 23, 6,
			7.5 K PRELOAD	USER CODE = 23, 7,

Table B-2. Sort Example

TEST CONDITIONS, CAR TYPE		100 TON 60' BOX	USER CODE = 2, 2,		
TEST CONDITIONS, TRUCK TYPE		BARBER 100 TON	USER CODE = 3, 5,		
TEST CONDITIONS, WHEEL PROFILE		1-20 (NEW)	USER CODE = 6, 1,		
TEST CONDITIONS, PER CENT LOAD		FULLY LOADED	USER CODE = 5, 3,		
TEST CONDITIONS, TRACK		HI SPD JTD	USER CODE = 20, 3,		NUMBER OF ENTRIES = 6
ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001	ID. 030201TWA001
BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389	BOX 7 TAPE 389
FILE NO. 1	FILE NO. 1	FILE NO. 1	FILE NO. 1	FILE NO. 1	FILE NO. 1
TAPE NO. 89	TAPE NO. 89	TAPE NO. 89	TAPE NO. 89	TAPE NO. 89	TAPE NO. 89
100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX	100 TON 60' BOX
BARBER 100 TON	BARBER 100 TON	BARBER 100 TON	BARBER 100 TON	BARBER 100 TON	BARBER 100 TON
46' 3''	46' 3''	46' 3''	46' 3''	46' 3''	46' 3''
FULLY LOADED	FULLY LOADED	FULLY LOADED	FULLY LOADED	FULLY LOADED	FULLY LOADED
1-20 (NEW)	1-20 (NEW)	1-20 (NEW)	1-20 (NEW)	1-20 (NEW)	1-20 (NEW)
7-D50/7-D5I	7-D50/7-D5I	7-D50/7-D5I	7-D50/7-D5I	7-D50/7-D5I	7-D50/7-D5I
8-B432	8-B432	8-B432	8-B432	8-B432	8-B432
8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS	8-B433 SNUBBERS
NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION	NO AUGMENTATION
STEEL-MOLY	STEEL-MOLY	STEEL-MOLY	STEEL-MOLY	STEEL-MOLY	STEEL-MOLY
HI SPD JTD	HI SPD JTD	HI SPD JTD	HI SPD JTD	HI SPD JTD	HI SPD JTD
30 MPH	40 MPH	50 MPH	60 MPH	70 MPH	79 MPH
5/8 OGC	5/8 OGC	5/8 OGC	5/8 OGC	5/8 OGC	5/8 OGC
1/4 SB CLR	1/4 SB CLR	1/4 SB CLR	1/4 SB CLR	1/4 SB CLR	1/4 SB CLR

This first step in these procedures was to run the test case described in the SPTCo. documentation for the program. The test run used was 050101TWA002 and the results are documented. The results agreed exactly with those the SPTCo obtained except the Interdata computer plotted only two time-history plots on a page versus six for the SPTCo case.

In the second step of the validation process, Wyle compared independently developed time histories to verify conversion of raw, multiplexed data (in volts) to a correct time history in engineering units. During the initial TDOP Phase II proposal effort, Wyle independently developed software for a limited analyses of the TDOP Phase I tapes from NTIS. A five-second, time-history plot was made, using an in-house program, which is shown in Figure B-1 for each speed zone run during the test. The measurements used consisted of channels 5 and 9. These same channels and corresponding mileposts were run using the Post Processing Program and are shown in Figures B-2 to B-6. The starting time for each of the plots in Figure B-1 is noted on the corresponding plot in Figures B-2 through B-6.

The two curves agree exactly in amplitude indicating that the conversion to engineering units was made correctly. However, when comparing the response at a specific milepost between the two time histories, a difference is noted in that the two time histories show a shift in the milepost at which a particular event occurs. As shown in Table B-3, the milepost comparison grew progressively worse the longer the test was run. This problem is discussed in more detail in the problems with the automatic location detector (ALD).

PSD Calculations

PSD calculations in the Post Processing Program do not use the same technique described by MITRE for the frequency domain model.⁽⁴⁾ The PSDs are calculated by summing the squares of the Fourier coefficients (sine and cosine) at each of 200 evenly spaced frequency points from 0.1 to 20 Hz. According to the TDOP Post Processing Program Manual, all of the TDOP data were

filtered by 20 Hz low-pass filters, and therefore, the higher frequency points of the PSD were not calculated. The data were acquired at a rate of 200 samples per second requiring 2000 data points for a 0.1 Hz resolution. Each PSD plot uses 4000 data points in the following manner: 10 PSD calculations are made; each using 2000 points. (The first PSD calculation is made using points 1-2000, the second from 200-2200, etc.) As may be seen, 1800 points of each PSD calculated overlap those of the previous calculation. These 10 PSDs are then averaged to form one PSD for the plot. Smoothing in the frequency domain followed the summing of the PSD values. This smoothing in the frequency domain is the equivalent of applying a Hanning window in the time domain.

The Fourier coefficients are calculated by FORIT, an IBM scientific subroutine. This subroutine uses a variation of the direct method for calculating Fourier coefficients with the sine and cosine terms calculated recursively to reduce computational times. The subroutine allows calculation of less frequency points than what is possible, therefore, allowing only 200 frequency points to be calculated rather than the possible 1000 points. Although not nearly as efficient as an FFT routine, this method for calculating the Fourier coefficients is valid.

Statistics

The statistical quantities calculated by the Post Processing Program are the mean value, the mean value of the rectified signal, the rms of the signal, and the standard deviation. The standard deviation calculated is not that of the rectified signal; however, there is evidence in the program listing that it was once done this way. There was controversy in the past when the SPTCo. calculated the standard deviation using the rectified signal. There was no agreement as to what the significance of the calculation was in relation to interpreting the data. This practice was apparently dropped later in Phase I and the standard deviation is now calculated using an unrectified signal. No errors were found in the calculation of these statistics. It should be noted that the rms and standard deviation of a signal with a zero mean are mathematically identical and provide no additional information. For those signals with a mean, the only significant information is derived from the standard deviation and is the value used in the analytical work.

(4) FRA Report No. FRA/ORD-78/12.III, Freight Car Truck Design Optimization, Phase I Frequency Domain Model, February 1978.

Table B-3. Time-History Comparison

<u>Speed</u>	<u>Time</u> <u>(Figure B-1)</u>	<u>Milepost</u> <u>(Post Processing)</u>	<u>Milepost (Ft)</u>	<u>Error</u>	<u>Figure</u>
30	25	48.2902	48.2879	12'	B-2
40	129	47.2587	47.2424	86'	B-3
50	174	46.7110	46.6761	184'	B-4
60	292	44.9764	44.9266	263'	B-5
79	463	41.7914	41.7006	479'	B-6

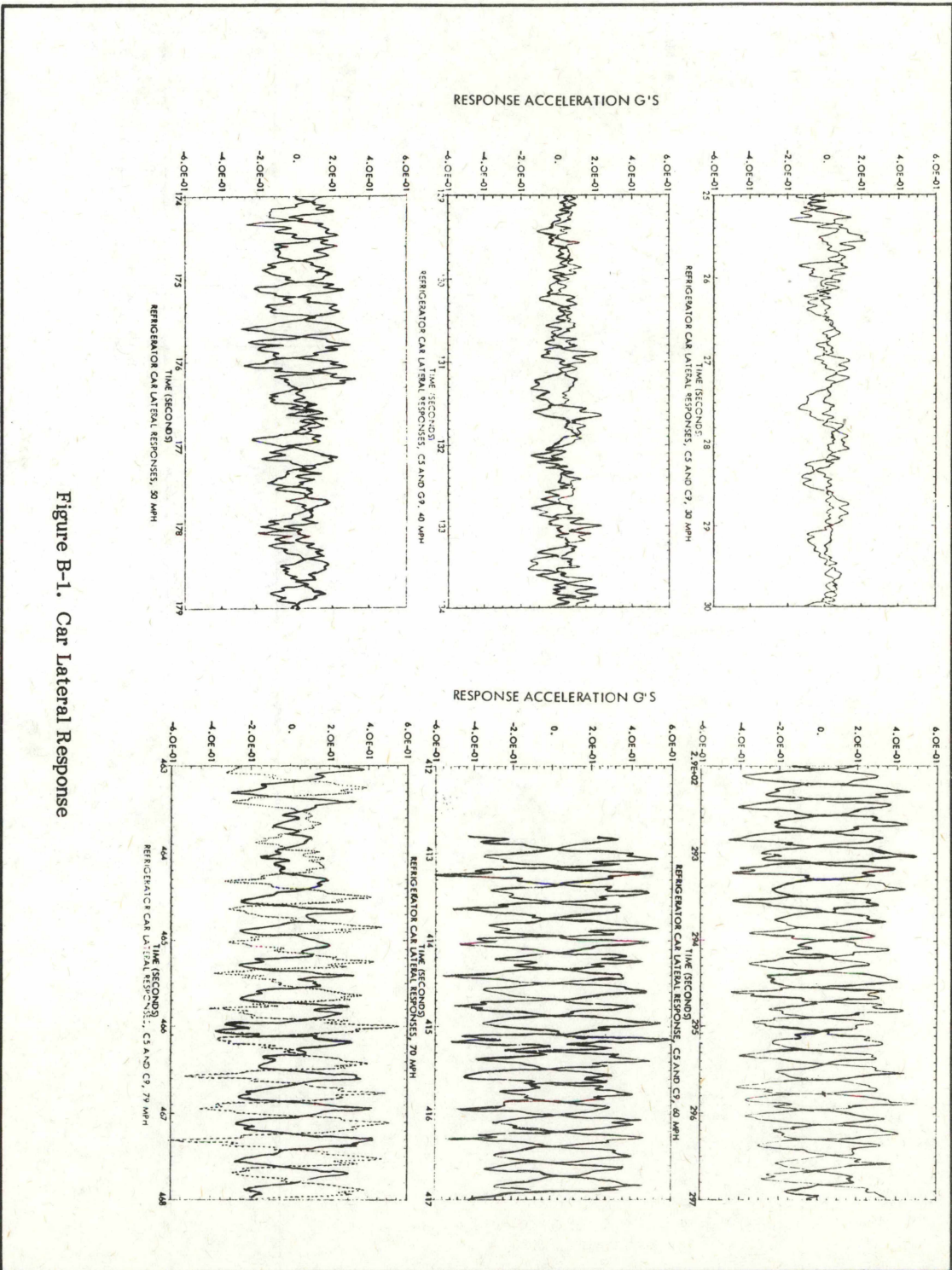
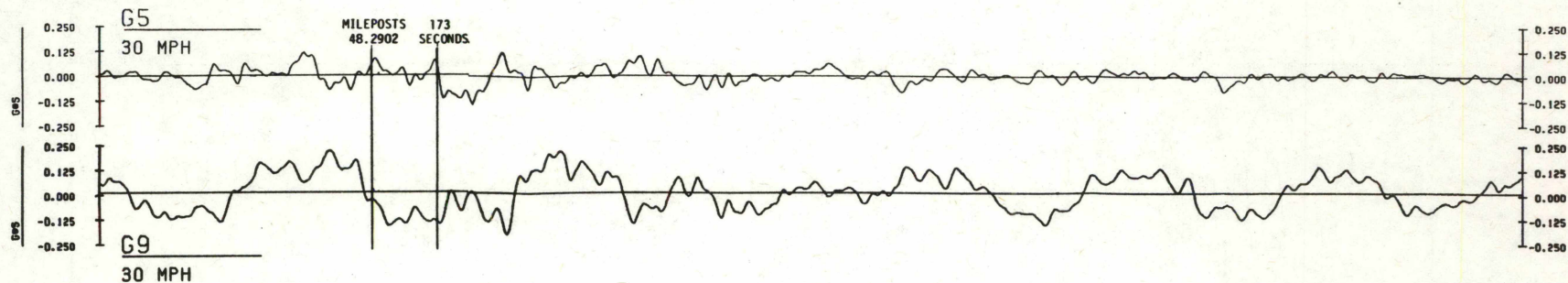
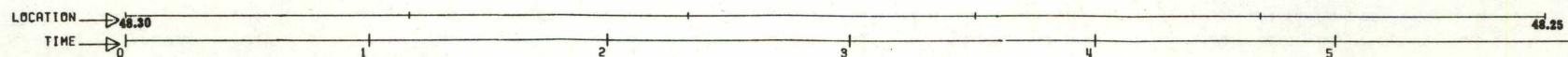


Figure B-1. Car Lateral Response



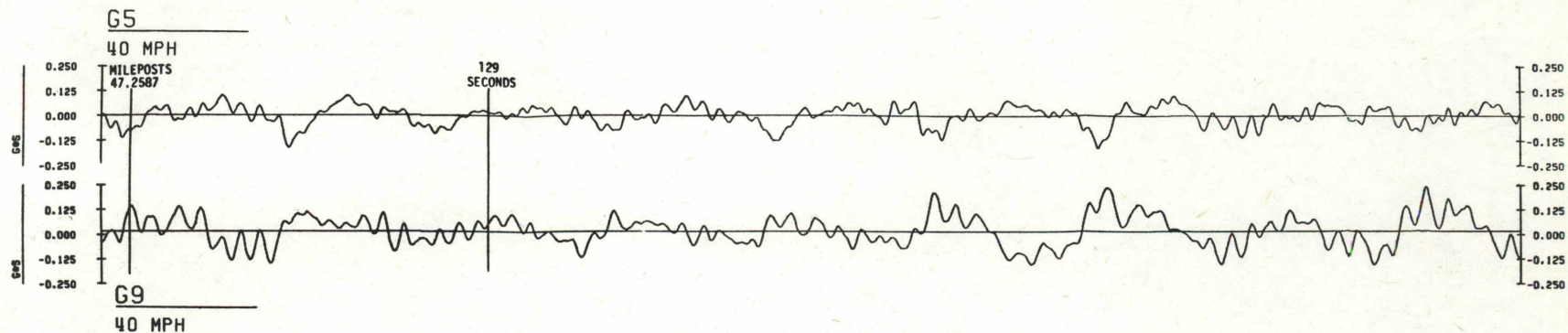
TEST NAME: 010203THA001
 SPEED RANGE: 28 -- 32 MPH
 M.P. LIMITS: 48.30 TO 48.24

PLOT NAME: 12JUN78 0001



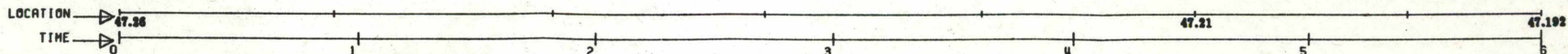
REFRIGRATOR CAR LATERAL RESPONSES
 COMPARISON OF WYLE : POST PROCESSING PROGRAM OUTPUT

Figure B-2



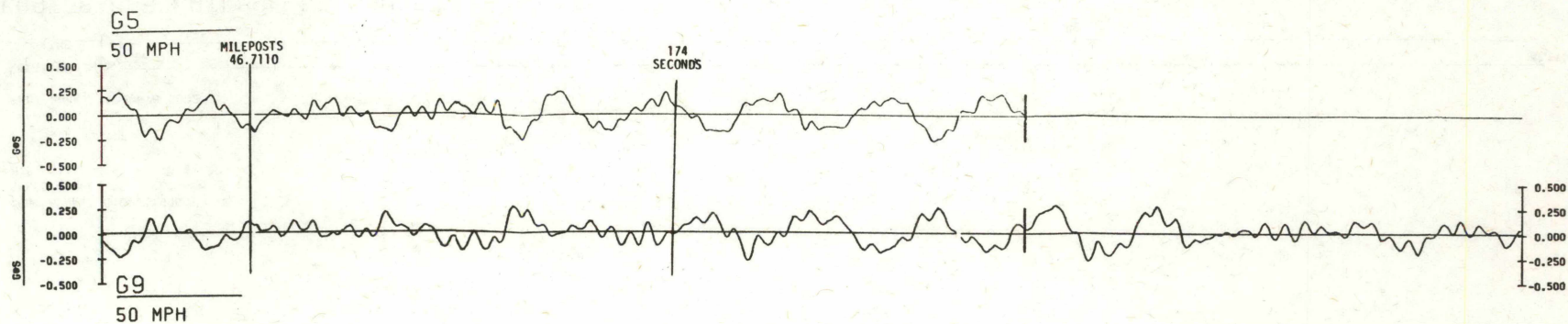
TEST NAME: 010203TMA001
SPEED RANGE: 38 -- 42 MPH
M.P. LIMITS: 47.26 TO 47.20

PLOT NAME: 12JUN78 0002

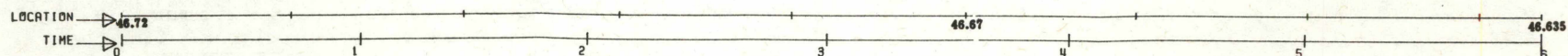


REFRIGRATOR CAR LATERAL RESPONSES
COMPARISON OF WYLE : POST PROCESSING PROGRAM OUTPUT

Figure B-3



TEST NAME: 010203TWA001
 SPEED RANGE: 48 -- 52 MPH
 M.P. LIMITS: 46.72 TO 46.64



REFRIGRATOR CAR LATERAL RESPONSES
 COMPARISON OF WYLE : POST PROCESSING PROGRAM OUTPUT

Figure B-4

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Truck Design Optimization Project: Phase II:
Phase I Data Evaluation and Analysis
Program, 1978
US DOT, FRA