Implementation Package FHWA-IP-80-13

BICYCLE-SAFE GRATE INLETS DESIGN MANUAL

U.S. Department of Transportation **Federal Highway Administration**

December 1980

NOTICE

This document is disseminated under the sponsorship of the Department of Transportation in the interest of information exchange. The United States Government assumes no liability for its contents or use thereof.

The contents of this report reflect the views of the author who is responsible for the facts and the accuracy of the data presented herein. The contents do not necessarily reflect the official views or policy of the Department of Transportation.

This report does not constitute a standard, specification, or regulation.

The United States Government does not endorse products or manufacturers. Trade or manufacturers' names appear herein only because they are considered essential to the object of this document.

BICYCLE-SAFE GRATE INLETS DESIGN MANUAL

Feder୍ତା Highway Administration Research Library Turner-Fairbank Highway Research Ctr. 6300 Georgetown **Pike McLean. VA 22101**

 \mathcal{E}

Prepared for

U.S. DEPARTMENT OF TRANSPORTATION

Federal Highway Administration Offices of Research and Development Implementation Division

> For sale by the Superintendent of Documents, U.S. Government Printing Office Washington, D.C. 20402

> > \bar{z}

ABSTRACT

This report presents equations for computing the hydraulic efficiency and discharge for three bicycle-safe grate inlets on a continuous grade and under sump conditions. These three grates were selected based on previous testing of various grates conducted by the Engineering and Research Center of the Water and Power Resources Service in Denver, Colorado. The parallel bar with transverse rods (P-1-7/8-4), the parallel bar with transverse spacers (P-1-1/8), and the curved vane (CV) grates showed the best overall characteristics in safety, hydraulic efficiency, and debris handling. The equations were derived empirically to fit the data within \pm 10%. Computer and calculator programs are also included for the user's convenience.

ACKNOWLEDGEMENT

The author wishes to express hearty appreciation to Dr. D.C. Woo and Mr. Robert D. Thomas, Offices of Research and Development, Federal Highway Administration, for their guidance and technical assistance. Without their cooperative effort, this study would not have been completed.

Sincere Thanks are also extended to Mr. J. Sterling Jones, Offices of Research and Development, for assistance in developing the calculator programs and to Mr. Robert H. Baumgardner, Hydraulics Branch, Federal Highway Administration, for his review of draft materials.

Fred F.M. Chang

NOTATION

 A_g = gross cross-sectional area of grate, W x L
 A_0 = cross-sectional area of curb opening, h x l A_0^9 = cross-sectional area of curb opening, h x L
a = exponent for the curvature of side flow a^* = exponent for the curvature of side flow
b = experimental coefficient for water surfa b = experimental coefficient for water surface drawdown, y_d
C_a = discharge coefficient of grate in a sump condition $=$ discharge coefficient of grate in a sump condition
 $=$ discharge coefficient of curb inlet C_{ω}^{σ} = discharge coefficient of curb inlet
 C_{ω} = weir discharge coefficient = weir discharge coefficient
= experimental coefficient f c = experimental coefficient for critical grate length, L_c
E = hydraulic efficiency of grate ΔE = adjustment for efficiency for unintercepted frontal flow g = gravitational acceleration
h = height of curb opening k = experimental coefficient for L_O
L = length of grate
L_C = critical grate length to interce L_C = critical grate length to intercept entire frontal flow
 L_C = distance from unstream edge of grate to intersection L_0 = distance from upstream edge of grate to intersection of side flow and gutter line (see Figure 4) ΔL = adjustment of weir length to account for flow contraction $m =$ coefficient related to Manning formula
 $n =$ Manning's roughness coefficient n = Manning's roughness coefficient $p =$ coefficient for orifice discharge
 Q_{I} = intercepted gutter flow
 Q_{T} = total gutter discharge Q_T = total gutter discharge
 Q_W = discharge through grate under weir flow condition
 Q_O = discharge through grate under orifice flow condit Q_0^{\dagger} = discharge through grate under orifice flow condition
q = exponent for orifice discharge coefficient q = exponent for orifice discharge coefficient
 R_f = ratio of unintercepted frontal flow to tote R_f = ratio of unintercepted frontal flow to total gutter flow
 R_S = ratio of unintercepted side flow to total gutter flow R_S = ratio of unintercepted side flow to total gutter flow
 S_O = longitudinal roadway slope
T = width of spread $=$ longitudinal roadway slope
 $=$ width of spread $T =$ width of spread
 $W =$ width of grate
 $y =$ flow depth y_d = flow depth
 y_d = depth of water surface drawdown
Z = reciprocal of roadway cross slo = reciprocal of roadway cross slope

TABLE OF CONTENTS

 $\mathcal{L}(\mathcal{A})$ and $\mathcal{L}(\mathcal{A})$.

 $\mathcal{L}(\mathcal{A})$ and $\mathcal{L}(\mathcal{A})$

 $\label{eq:2.1} \frac{1}{2} \sum_{i=1}^n \frac{1}{2} \sum_{j=1}^n \frac{$

LIST OF TABLES

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2.$

 $\label{eq:2} \mathcal{L}_{\mathbf{q}_i} = \mathcal{L}_{\mathbf{q}_i}$

LIST OF FIGURES

 $\langle \rangle_{\rm{c}}$

 $\tilde{\mathcal{L}}$

V

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^{2} \left(\frac{1}{\sqrt{2}}\right)^{2} \left(\$

 $\label{eq:2.1} \frac{1}{\sqrt{2}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\left(\frac{1}{\sqrt{2}}\right)^2\frac{1}{\sqrt{2}}\frac{1}{\sqrt{2}}\frac{1}{\sqrt{2}}\frac{1}{\sqrt{2}}\frac{1}{\sqrt{2}}\frac{1}{\sqrt{2}}$

INTRODUCTION

The Engineering and Research Center of the Water and Power Resources Service (formerly the Bureau of Reclamation) in Denver, Colorado, has completed testing of 11 inlet grates for the Federal Highway Administration. Seven of these grates were found to be bicycle-safe. These seven grates were then tested thoroughly for their hydraulic efficiency and debris handling capability on continuous grades.

Two grate sizes were tested: 2 ft by 2 ft (0.61 m by 0.61 m) and 2 ft by 4 ft (0.61 m by 1.22 m). From the test results, three grates were selected as possessing the best overall characteristics. These findings are presented in FHWA-RD-77-24 $(1)^*$. These three grates were further tested for various sizes, and the findings are presented in FHWA-RD-78-4 (2). The three grates were then tested in a sump condition, and the results are presented in FHWA-RD-78-70 (3).

The three grates selected as possessing the best overall characteristics are the parallel bar with transverse rods (P-1-7/8-4), the parallel bar with transverse spacers $(P-1-1/8)$, and the curved vane (CV) . The dimensions of these three grates are shown in Figures 1, 2, and 3, respectively. Performance testing of these three grates covered the following ranges:

Based on the data from this testing, an analysis was made to develop empirical equations relating flow characteristics, road geometry and the type and size of grate.

This manual presents the resulting equations for determining the hydraulic efficiencies of the three grates on a continuous grade and the dischargedepth relationship in a sump condition. The detailed analyses of the hydraulic characteristics, the derivation of the equations, and a comparison of the results with experimental data are included in the Appendix.

^{*}Number in parentheses refers to References on page 34.

2 ft by 4 ft (0.61 m by 1.22 m) fabricated steel Figure $\mathbf{1}$. $P - 1 - 7/8 - 4$ grate.

 ϵ

Figure 2 2 ft by 4 ft $(0.61 \text{ m by } 1.22 \text{ m})$ fabricated steel P - 1-1/8 grate (Note: $1 \text{ in } = 25.4 \text{ mm}$).

 $\mathbf{\omega}$

Figure 3 2 ft by 4 ft $(0.61 \text{ m} \text{ by } 1.22 \text{ m})$ curved vane grate (Note: 1 in = 25.4 mm).

 \rightarrow

DESIGN CONSIDERATIONS

In grate inlet design, the determination of type, size, and spacing of the grate to intercept a specified portion of the design gutter discharge is of general concern. In addition, for optimum design of the entire drainage system and for the analysis of flow conditions in the system, the discharge capacity curve for each grate must be developed. Using this curve and the hydrograph of the design gutter flow, the inlet hydrograph of the grate can be derived to simulate flow in the drainage system. In this report, an emphasis was placed on the development of hydraulic performance curves for the three grates.

Selection of.Grate Type

Grate type selection should consider such factors as safety, structural strength, debris handling characteristics, hydraulic efficiency, and cost. Among various grates tested, Burgi and Gober (1) recommended the parallel bar with transverse rods (P-1-7/8-4), the parallel bar with transverse spacers (P-1-1/8), and the curved vane (CV) for their superior overall performance. The dimensions of these grates are shown in Figures 1, 2, and 3.

For grates on a continuous grade, the curved vane grates performed better in debris handling but not as well in bicycle safety than the other two grates. For grates in a sump condition, the P-1-1/8 grate performed slightly better in debris handling. For the three grates on a continuous grade, no significant differences in hydraulic efficiencies were recognized
at lower discharges. The CV and the P-1-7/8-4 grates, however, performed The CV and the $P-I-7/8-4$ grates, however, performed slightly better at higher discharges. For grate inlets in a sump condition, the performance of the grates under weir control was identical for larger grates with low discharges. For smaller grates with higher discharge and orifice flow, the P-1-7/8-4 grates performed slightly better.

Since no single grate type outranked the others in all aspects, judgment must be exercised in selecting a grate. Bicycle safety and debris handling capabilities must be weighed carefully in the final grate selection.

Grate Size and Spacing

•

For economical reasons, grate sizes are usually standardized by the State for use on its highways. The selection of those standard sizes is normally dictated by such factors as cost (construction and maintenance cost of the grate, the catch basin, and the connection to the main sewer), safety for passing traffic, structural strength, hydraulic efficiency, and debris

accumulation. The determination of size and spacing of a grate in the field is based on the criteria of allowable maximum gutter flow width and the grate's efficiency characteristics. In a sump condition, the allowable maximum water depth at the low point of a sag vertical curve is the dominant factor in selecting the size and spacing of a grate.

It is common practice to specify the grate one size larger than actually needed as a precautionary measure against complete plugging by debris. The needed as a precautionary measure against complete plugging by debris. analytical method presented here can be used in the preliminary investigation for the selection of the standardized size.

Summary of Equations for Hydraulic Efficiency and Discharge of Grate Inlets

A. Grate Inlets on a Continuous Grade:

Flow at a grate inlet on a continuous grade is often divided into two parts:
(1) frontal flow that directly passes over the upstream edge of the grate,
and (2) the remaining part generally called side flow. The amount of ception of the flow depends on discharge, road geometry, and the type and size of the grate.

A sketch of a typical flow pattern at a grate inlet on a continuous grade is shown in Figure 4. This flow pattern actually represents that of higher flow when both the frontal flow and side flow are partially intercepted. For lower flow, however, the flow pattern is different: the frontal flow and side flow may be totally intercepted. With decreases in the gutter discharge, the side flow diminishes and the gutter flow consists of only the frontal flow.

Because of these differences in the flow pattern, a set of equations was developed to determine the hydraulic efficiency of an inlet grate for all ranges
of gutter discharges. These equations are summarized in Table 1. Equation (1) is a different form of the integrated Manning's equation (with n = 0.016) derived by Izzard (4) . Flow depths y_1 at the curb and y_2 at a distance W (grate width) from the curb can be computed from Eqs. (1) and (2). The distance L_0 where the water surface intersects with the gutter line can be obtained from Eq. (3). Length L_c in Eq. (4) is the critical length of the grate over which total frontal flow interception occurs. Equation (5) gives hydraulic efficiency when the entire frontal flow is intercepted ($L_c < L$). The second term on the right side of this equation is the ratio of unintercepted side flow to the total gutter flow. If y_2 is negative or L_0 is less than L, indicating no side flow or total interception of the side flow, this term must equal zero and the hydraulic efficiency becomes 100 percent. In case of partial frontal flow interception (L_C > L), the hydraulic efficiency
determined above must be reduced by the amount <code>AE. The value of AE</code> is the ratio of the unintercepted frontal flow to the total gutter flow and can be determined from either Eq. (6) or Eq. (7), whichever is appropriate.

In surmnary, to determine the hydraulic efficiency of a grate, the flow pattern must be identified first, and the correct equations must be selected. For the user's convenience, a nomograph for Eq. (5) is shown in Figure 5, and Eqs. (6) and (7) are jointly plotted in Figure 6.

FIGURE 4 FLOW AT A GRATE INLET ON A CONTINUOUS GRADE

 $\bar{\mathbf{r}}$

分ける

Values of Experimental Coefficients:

 \sim

FIGURE 5 Nomograph for Grate Efficiency for 100 percent Frontal Flow Interception

 $\mathbf \omega$

FIGURE 6 Ratio of Unintercepted Frontal Flow to Total Flow

 \sim

 \star

 \sim

 $\overline{5}$

B. Grate Inlets in a Sump Condition:

Flow at a grate inlet in a sump condition assumes weir flow, orifice flow, or transitional flow conditions depending on flow depth. Weir flow condition prevails at shallow depth and orifice flow at a depth which submerges the grate. Transitional conditions exist between these flows. In this study, the transition zone was found to be at a depth of about $0.43 \text{ } L0.7$ for all three grates.

The general equations for weir flow and orifice flow were modified to account for the roadway cross slope and a curb opening provided along the entire length of the sump grate. These modified weir and orifice equations are presented in Table 2. The discharges in Eqs. (8) and (9) are total discharges assuming the flow converges equally from both sides of the grate at a dip in the road. The first term on the right side of Eq. (9) is the discharge through the curb opening. If no curb opening is present for a sump grate, this term must be eliminated.

For very shallow flow when the flow spread is narrower than the grate width $(T < W, or y2 < 0)$, $y2 \in (8)$ equals zero. In the transitional zone, either Eq. (8) or Eq. t9) may be used; this leads to only a minor discrepancy as demonstrated later in Figure 9. A better result can be obtained by taking the average of both values.

TABLE 2 Equations for Grate Inlets in a Sump Condition

I. Weir Flow Condition:
$$
y_1 \le 0.43 \text{ L}^{0.7}(\text{English}); y_1 \le 0.3 \text{ L}^{0.7}(\text{SI})
$$

\n $Q_W = C_W[(L - y_2)y_2^{1.5} + 0.707W(y_1 + y_2)^{1.5}]$ (8)
\n $C_W = 3.5 \text{ (English)}, 1.93 \text{ (SI)}$

II. Orifice Flow Condition:

$$
Q_0 = 0.7 A_0 [2g(y_1 - h/2)]^{0.5} + C_g A_g [g(y_1 + y_2)]^{0.5}
$$

\n
$$
C_g = p(y_1 + y_2)^{q} / L^{0.5}
$$
\n(9)

Note: If no curb opening is present, delete the first term from Eq. (9)

EXAMPLE PROBLEMS

Applications of the equations in Tables l and 2 to develop hydraulic performance curves for grate inlets are presented below. Two example ,
problems are included: one is for grate inlets on a continuous grade
and the other for grates in a sump condition. In both cases an 18 in. and the other for grates in a sump condition. In both cases an 18 in.
by 18 in. P-1-1/8 grate is used as a State standard.

A. Grate Inlets on a Continuous Grade

Three different sizes are used to demonstrate the effect of grate width and length on hydraulic efficiency: 18 in. X 18 in. (W X L), 36 in. X 18 in. (two grates aligned side by side), and 18 in. X 36 in. (two grates aligned end to end). Step-by-step example computations are shown below, and the hydraulic performance curves for a gutter discharge of up to 5 cfs are presented in Figures 7 and 8 for longitudinal roadway slopes of l and 6 percent, respectively.

Example Computation:

Given: $Q_T = 3.0 \text{ cfs}; S_0 = 6\%; Z = 24; W = 18 \text{ in.}; L = 18 \text{ in.};$ Grate: P-1-1/8

Required: Compute hydraulic efficiency

Solution:

1. Compute flow depths y_1 and y_2 :

$$
y_1 = \left(\frac{0_1}{34.8 S_0^{0.5} Z}\right)^{0.375} = \left(\frac{3.0}{34.8(0.06)^{0.5}(24)}\right)^{0.375} = 0.205 \text{ ft}
$$

$$
y_2 = y_1 - W/Z = 0.205 - 1.5/24 = 0.143 \text{ ft}
$$

2. Compute lengths L₀ and L_c:
\nL₀ = kS₀^{0.65} y₂^{1.17} z = 65(0.06)^{0.65}(0.143)^{1.17}(24) = 25.74 ft
\nL_c = c (
$$
\frac{Q_T S_0}{Z}
$$
)^{0.67} = 50 ($\frac{3 \times 0.06}{24}$)^{0.67} = 1.88 ft

3. Compute efficiency E :

$$
E = 1 - (y_2/y_1)^{2.67} (1 - (L/L_0)^a)^{2.67}
$$

= 1 - (0.143/0.205)^{2.67} (1 - (1.5/25.74)^{1.5})^{2.67} = 0.632

Alternately, the nomograph in Figure 5 can be used:

For $y_2/y_1 = 0.143/0.205 = 0.698$ and $L/L_0 = 1.5/25.74 = 0.0582$, $E = 0.632$

4. Compute efficiency adjustment $\triangle E$, if needed.

Since $L/L_{c} = 1.5/1.88 = 0.798 < 1$, an adjustment is needed. Equation (6) should be used for the adjustment since $y_2/y_1 = 0.698$ is less than (L/L_C)^{0.56} = (1.5/1.88)^{0.56} = 0.881.

$$
\Delta E = (1 - (y_2/y_1)^{2.67}) \left(\frac{(1 - (L/L_c)^{0.56})^2}{1 - (y_2/y_1)^2}\right)
$$

= (1 - (0.698)^{2.67}) \left(\frac{(1 - 0.881)^2}{1 - (0.698)^2}\right) = 0.017

Alternately, from the chart in Figure 6, for $y_2/y_1 = 0.698$ and $L/L_c = 0.798,$

 $\Delta E = 0.017$

 \mathcal{L}

5. The hydraulic efficiency of the grate is

 $E - \Delta E = 0.632 - 0.017 = 0.615$

FIGURE 7 HYDRAULIC EFFICIENCY VS. GUTTER FLOW FOR THE P-1-1/8 GRATE, $S_0 = 0.01$

 ϵ

 $\hat{\textbf{v}}$

FIGURE $\,$ 8 $\,$ HYDRAULIC EFFICIENCY VS. GUTTER FLOW FOR THE P-1-1/8 GRATE, S $_{\rm 0}$ = 0.06 $\,$

B. Grate Inlets in a Sump Condition

Grate inlets in a sump condition are required to intercept all gutter flow at a sump depth not exceeding the maximum allowable depth. Therefore, for a sump grate, the major concern is to develop the depth-discharge relationship. Two different grate sizes are used to show the effect of size: 18 in. X 18 in. (W X L) and 18 in. X 36 in. (two grates aligned end to end). Example computations are presented below, and the depth-discharge relationships for the two grates up to discharges of 18 cfs are shown in Figure 9. In all cases, the sump grate has a curb opening of 4 in. height and with a length equal to the length of the grate.

Example Computation:

Given: $y_1 = 0.5$ ft; $Z = 24$; $W = 18$ in.; $L = 18$ in.; Grate: P-1-1/8

Required: Determine discharge

Solution:

l. Determine the type of flow (weir or orifice):

Since $y_1 = 0.5$ ft < 0.43 $L^{0.7} = 0.43$ (18/12)^{0.7} = 0.571 ft,

the flow is under weir flow condition, and Eq. (8) must be used to determine the discharge.

2. Compute the discharge:

$$
y_2 = y_1 - W/Z = 0.5 - 1.5/24 = 0.438 \text{ ft}
$$

\n
$$
Q_W = 3.5[(L - y_2)y_2]^{1.5} + 0.707W(y_1 + y_2)^{1.5}]
$$

\n= 3.5[(1.5 - 0.438)(0.438)]^{1.5} + 0.707(1.5)(0.5 + 0.438)]^{1.5}]
\n= 4.45 cfs

FIGURE 9 INLET CAPACITY CURVE, P-1-1/8 SUMP GRATES, $Z = 24$

 $\overline{\mathcal{L}}$

COMPUTER AND CALCULATOR PROGRAMS

Computer and calculator programs for computing hydraulic efficiencies for three bicycle-safe grates on a continuous grade and for computing discharges in a sump condition were developed. The computer programs are written in FORTRAN IV and the calculator programs are developed for the TI58/TI59 and for the HP67/97 calculators. All equations are listed in Tables l and 2.

..

A. Hydraulic Efficiencies of Grates on a Continuous Grade

Input and output parameters for all programs for computing hydraulic efficiencies for the three grates on a continuous grade are as follows:

Input Data:

-5

-
-
- Q_T gutter discharge
S_O longitudinal roadway slope
Z reciprocal of roadway cross Z - reciprocal of roadway cross slope
W - width of grate
L - length of grate
-
- length of grate
- m coefficient related to Manning's equation
k experimental coefficient for L_0
- experimental coefficient for L_0
- a exponent for the curvature of side flow
- c experimental coefficient for critical grate length

Output:

- Q_T gutter discharge
E hydraulic efficie
- hydraulic efficiency of grate

The following examples may be used to test the programs for hydraulic efficiency of grate inlets on a continuous grade:

 \sim

 $\overline{}$

 $\sim 10^{11}$

© 19TT Texas Instruments Incorporated

r $\begin{matrix} \downarrow \\ \downarrow \end{matrix}$

TITLE Efficiency of Grate Inlets

01 1977 Texas Instruments Incorporated

I, e
R

@ 1977 Texas Instruments Incorporated

 $\ddot{}$

USER INSTRUCTION

HP67/97

TITLE Efficiency of Grate Inlets PAGE 1 OF 3

Efficiency of Grate Inlets **TITLE**

PAGE 2 OF 3

 $HP67/97$
Coding

 \mathbf{r}

 $\hat{\mathcal{L}}$

 $PAGE$ 3 OF 3

HP 67/97 Coding

B. Discharge of Grate in a Sump Condition

Input and output parameters of all programs for computing discharges for the three grates in a sump condition are as follows:

Input Data:

- Z reciprocal of roadway cross slope
W width of grate
- W width of grate
L length of grate
-
- L length of grate h height of curb opening
- p coefficient for orifice discharge
- q exponent for orifice discharge coefficient
 y_1 sump depth at the curb
- y_1 sump depth at the curb

Output:

- Y1 sump depth at the curb
- Q_w or Q_0 sump discharge

The following examples may be used to test these programs for discharge of grate inlets in a sump condition:

 $\label{eq:2.1} \frac{1}{\sqrt{2\pi}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2\pi}}\left(\frac{1}{\sqrt{2\pi}}\right)^2\frac{1}{\sqrt{2\pi}}\int_{\mathbb{R}^3}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{1}{\sqrt{2\pi}}\frac{$

 $\mathcal{L}(\mathcal{L}(\mathcal{L}))$ and $\mathcal{L}(\mathcal{L}(\mathcal{L}))$. The contribution of $\mathcal{L}(\mathcal{L})$

The Contract of the Contract

 $\bar{1}$ Ĭ.

TITLE Sump Grate Discharge

01977 Texas Instruments Incorporated

÷

PROGRAMMER

to 1977 Texas Instruments Incorporated

USER INSTRUCTIONS

HP67/97

TITLE Sump Grate Discharge PAGE 1 OF 3

REGISTERS

PAGE 2 OF 3

HP67/97 Coding

COMMENTS

KEY CODE

 35 11 $36\sqrt{03}$ 36 $\overline{\phi}$ 8 -45 $36\overline{\phi}8$ \emptyset 1 -62 $\overline{\phi}$ 5 $\overline{31}$ -35 -62 $Ø7$ øø \emptyset 7 36 \varnothing 2 -35 $36\overline{00}$ \emptyset 1 -62 $\overline{\phi}$ 5 $\overline{31}$ -35 -55 36 11 -35 $35 15$ 22 \emptyset 3 21 Ø2 $\overline{\phi}$ 3 $\mathbf{0}2$ -62 $\mathbf{\underline{\emptyset2}}$ <u>35 12</u> 3605 36 Ø9 36 Ø6 31 -35 36003 <u>54</u> -24 35 13 36 03 36 Ø4 -35 -62 $0/2$ -35 3514 \varnothing 2

 $\frac{1}{\sqrt{2}}$

 $\frac{1}{\epsilon}$

 $\hat{\mathcal{A}}$

 $\frac{1}{2}$

 $\sim 10^6$

 \bar{z} $\frac{1}{2}$

 \bar{z}

REFERENCES

- 1 Burgi, P.H., and Gober, D.E., 11Bicycle-Safe Grate Inlets Study. Volume 1 Hydraulic and Safety Characteristics of Selected Grate Inlets on Continuous Grades," Federal Highway Administration, Report No. FHWA-RD-77-24, June 1977.
- 2 Burgi, P.H., "Bicycle-Safe Grate Inlets Study. Volume 2 Hydraulic Characteristics of Three Selected Grate Inlets on Continuous Grades, ¹¹ Federal Highway Administration, Report No. FHWA-RD-78-4, May 1978.
- 3 Burgi, P.H., 11 Bicycle-Safe Grate Inlets Study. Volume 3 Hydraulic Characteristics of Three Selected Grate Inlets in a Sump Condition," Federal Highway Administration, Report No. FHWA-RD-78-70, September 1978.
- 4 Izzard, Carl F., "Hydraulics of Runoff from Developed Surface," Highway Research Board, Vol. 26, 1946.

Cited in the APPENDIX

- 5 Li, W.H., 11The Design of Stonn-Water Inlets," Department of Sanitary Engineering and Water Resources, Johns Hopkins University, June 1956.
- 6 Davis, C.V., Handbook of Applied Hydraulics, McGraw-Hill Co., 1952.
- 7 U.S. Department of Transportation, Federal Highway Administration, Design of Urban Highway Drainage - The State-of-the-Art,FHWA-TS-79-225, August 1979.

APPENDIX

ANALYSIS OF FLOW AT A GRATE INLET

I. Grates on a Continuous Grade

Gutter flow at a grate inlet is generally divided into two parts: (1) frontal flow that directly passes over the upstream edge of the grate and (2) the remaining part generally called side flow. Varying portions of these flows will be intercepted by the grate depending on the flow condition, the geometry of the road and the type of grate inlet.

The hydraulic efficiency, E, of a grate is defined as the ratio of the flow intercepted, Q_i , to the total gutter flow, Q_T ,

$$
E = Q_{\mathbf{i}} / Q_{\mathbf{T}} \tag{A-1}
$$

This can also be expressed in terms of the portion of the flow which is not intercepted:

$$
E = 1 - R_c - R_f \tag{A-2}
$$

where

 R_S = ratio of unintercepted side flow to the total gutter flow R_f = ratio of unintercepted frontal flow to the total gutter flow.

A. Side Flow

A typical flow pattern of gutter flow at a grate inlet is shown in Figure 4. The side flow, designated with subscript 2, is subject to a lateral motion as the portion of the frontal flow is drained and the water surface drawn down. Because of this lateral motion, the side flow curves toward the grate, and the flow near the grate will be intercepted by the grate. Assuming a constant lateral acceleration, Li (5) found that the flow line at a curb inlet formed a parabolic curve. A similar flow pattern prevails for the side flow at a grate inlet. However, the curve is expected to be milder because the resistance of the grate tends to impede the lateral motion of the flow, thus pushing the flow downstream. In this study, a curve of "a"th power was assumed, i.e. the width, y₂Z, of the side flow is proportional to ${\sf L}_{\bf 0}$ to the "a"th power, where ${\sf L}_{\bf 0}$ is the distance from the upstream edge of the grate to the intersection of the water edge and the qutter line (see Figure 4). Then the ratio of the width of the portion of the side flow intercepted to the total width of the side flow becomes

$$
\frac{y_2 z - y_3 z}{y_2 z} = (L/L_0)^a \qquad \text{or} \qquad 1 - y_3/y_2 = (L/L_0)^a \qquad (A-3)
$$

where

With the integrated Manning equation for gutter flow,

$$
Q_T = \frac{0.557}{n} Z S_0^{0.5} y_1^{2.67}
$$
 (A-4)

the ratio R_S of the unintercepted side flow to the total gutter flow can
be expressed as

$$
R_{S} = \left\{\frac{y_{3}}{y_{1}}\right\}^{2.67} \quad \text{or} \quad R_{S} = \left\{\frac{y_{2}}{y_{1}}\right\}^{2.67} \left\{\frac{y_{3}}{y_{2}}\right\}^{2.67} \quad (A-5)
$$

When Eq. (A-3) is substituted into the above equation, the result is

$$
R_{s} = {\frac{y_{2}}{y_{1}}}^{2.67} \left(1 - {\frac{L}{L_{0}}}^{a} \right)^{2.67}
$$
 (A-6)

According to Li's study (5) for curb inlets, for $Z \ge 8$,

$$
L_0 = 23.15 S_0^{0.5} y_1^{1.17} Z
$$
 (A-7)

In order to determine the value of the exponent a and of L_0 for the three inlet grates, the data with low discharge (for which 100 percent of the frontal flow was intercepted) were selected. For these data, $R_f = 0$ and the hydraulic efficiency obtained in the testing can be expressed as

$$
E = 1 - R_{s} = 1 - {\frac{y_{2}}{y_{1}}}^{2.67} \left(1 - {\left(\frac{L}{L_{0}} \right)}^{a} \right)^{2.67}
$$
 (A-8)

Regression analysis revealed that the value of the exponent a is 1.5 for the P-1-7/8-4 and the P-1-1/8 grates and 1.8 for the curved vane grate. The following equation for L_0 was found most adequate for all three grates:

$$
L_0 = 65 S_0^{0.65} y_1^{1.17} Z
$$
 (A-9)

B. Frontal Flow:

For high velocity flows, some frontal flow may skip over the grate and be unintercepted. The drawdown, y_d, of the water surface in this case is less than the flow depth y_1 at the curb. If no change in the mean velocity of the frontal flow over the grate is assumed, then the ratio of the unintercepted frontal flow to the total frontal flow would be the ratio of the cross-sectional area (A_1) ' of the flow at the downstream edge of the grate to the cross-sectional area of the frontal flow at the upstream edge of the grate. The shape of the area $(A_1)'$ may be triangular or trapezoidal depending on the drawdown, y_d. For y_d> y₂, the shape
is a triangle. For y_d < y₂, the shape is trapezoidal. The ratio of the unintercepted frontal flow to the total frontal flow for these cases is,

for
$$
y_d/y_1 \ge y_2/y_1
$$

\nRatio =
$$
\frac{(1 - y_d/y_1)^2}{1 - (y_2/y_1)^2}
$$

\nor, for $y_d/y_1 < y_2/y_1$
\nRatio =
$$
1 - \frac{2(y_d/y_1)}{1 + (y_2/y_1)}
$$
 (A-10b)

To investigate drawdown, y_d , photographs of all runs were carefully studied to find the cases in which the water surface just intersected the downstream edge of the grate. In these cases, Yd equals y1 and the relationship between y_d and the other parameters can be determined. The analysis yielded the following equation for y_d :

$$
y_{d} = \frac{1}{b} (L/S_0)^{0.56}
$$
 (A-11)

where $b =$ experimental coefficient.

The coefficient b was found to be 46 for the P-1-7/8-4 grate and 34 for both the P-1-1/8 grate and the curved vane grate. If 100 percent frontal flow interception is desired, then the length of the grate must be chosen so that the value of $y_{\bf d}$ equals flow depth $y_{\bf l}$, which means that curve a-c of the frontal flow in Figure 4 diminishes just at the downstream edge of the grate. This length shall be referred to as the critical length, $L_{\mathbf{C}}$, in this report. By using the Manning equation for gutter flow with $n = 0.016$, the depth of flow, y_1 , in feet, can be expressed as

$$
y_1 = \left(\frac{Q_T}{34.8 S_0^{0.5} Z}\right)^{0.375}
$$
 (A-12)

where Q_T = discharge in cfs.

When Eq. (A-11) is set equal to Eq. (A-12), the critical length L_c can be found:

$$
L_{\rm C} = c (Q_{\rm T} S_0 / Z)^{0.67}
$$
 (A-13)

The value c is 85 for the P-1-7/8-4 grate and 50 for the P-1-1/8 and the curved vane grates.

By definition, $y_d = y_1$ for $L = L_c$; therefore, from Eq. (A-11), the following relation can be established:

$$
y_{d}/y_{1} = (L/L_{c})^{0.56}
$$
 (A-14)

Since the ratio of the frontal flow to the total gutter flow is $l - (y_2/y_1)$ the ratio Rf of the unintercepted frontal flow to the total gutter flow can be derived from Eqs. $(A-9)$, $(A-10)$, and $(A-14)$ as

for
$$
y_2/y_1 \le (L/L_c)^{0.56} < 1
$$
,
\n
$$
R_f = \begin{bmatrix} 1 - (y_2/y_1)^{2.67} \end{bmatrix} \begin{bmatrix} \frac{(1 - (L/L_c)^{0.56})^2}{1 - (y_2/y_1)^2} \end{bmatrix}
$$
\n(A-15)

or, for
$$
(L/L_c)^{0.56} < y_2/y_1 < 1
$$
,
\n
$$
R_f = \left[1 - (y_2/y_1)^{2.67}\right] \left[1 - \frac{2(L/L_c)^{0.56}}{1 + (y_2/y_1)}\right]
$$
\n(A-16)

The term in the first bracket in the above equations is the ratio of the frontal flow to the total gutter flow, and the term in the second bracket represents the ratio of the unintercepted frontal flow to the total frontal flow. The ratio R_f is same as efficiency adjustment, ΔE , in Table 1.

II. Grates in a Sump Condition

Flow at a grate inlet in a sump condition can be analyzed as either weir flow or orifice flow depending on the depth of flow. The weir flow condition controls until the flow depth is sufficiently deep to submerge the grate. The general equations for weir flow and orifice flow can be found elsewhere (6). For gutter flow where a cross slope exists, however, these general equations must be modified. In addition, a curb opening is generally added along the entire length of a sump grate to provide extra protection

against debris accumulation. Although this type of curb opening does not affect weir flow, the curb opening discharges a substantial portion of gutter flow when the water is deep and orifice flow prevails. Therefore, the discharge through the curb opening must be considered for orifice flow. With these considerations, the weir equation and the orifice equation can be modified and expressed as

$$
Q_{w} = C_{w} \{ (L - \Delta L) (y_{2})^{1.5} + 0.707W(y_{1} + y_{2})^{1.5} \}
$$
 (A-17)

and

$$
Q_0 = C_0 A_0 \{ 2g(y_1 - h/2) \}^{0.5} + C_0 A_0 \{ g(y_1 + y_2) \}^{0.5}
$$
 (A-18)

where

The first term on the right side of Eq. (A-18) is the discharge through the curb opening, and the second term is the discharge through the grate inlet. If no curb opening is provided, the first term should be eliminated.

Grate Inlet in Weir Flow Condition:

For moderate or low flow, a weir flow condition prevails at a grate inlet, and Eq. (A-17) can be used to determine the discharge. Generally, the weir flow pattern is similar for all types of grates regardless of their configurations. Therefore, all grates of larger sizes, i.e. 2 ft by 4 ft (0.61 m by 1.22 m) and 3 ft by 4 ft $(0.61 \text{ m}$ by 1.22 m) were analyzed to determine the weir coefficient and to investigate simultaneously the weir length adjustment, Δ L, for Eq. (A-17). The analysis resulted in a weir coefficient of 3.5 and ΔL = y₂ which adequately fit all the data. Later, these values were tested and found to be adequate for other grates of smaller size in the weir flow condition. Equation (A-17) may be rewritten as

$$
Q_W = C_W[(L - y_2)y_2]^{1.5} + 0.707W(y_1 + y_2)^{1.5}]
$$
 (A-19)

B. Grate Inlet in Orifice Flow Condition:

While the effect of a curb opening located along the grate inlet can be ignored when the flow is in a weir flow condition, the discharge through the curb opening becomes a significant portion of the total intercepted flow when flow depth increases and orifice flow conditions develop. Analysis of data from curb opening tests revealed that the orifice coefficient for curb

opening is 0.7 as was found in report FHWA-RD-78-70 (3). Next, in order to determine the orifice coefficient, C_0 , for the three grates, the data of. smaller grates with larger flow depths were analyzed, assuming that orifice flow conditions prevailed. The orifice coefficients were found to be a function of average flow depth $(y_1 + y_2)/2$ and the length of the grate:

$$
C_g = p(y_1 + y_2)^{q} / L^{0.5}
$$
 (A-20)

Values of Experimental Coefficients

	English		
$P - 1 - 7/8 - 4$	0.34	0.49	$0.8\,$
$P-1-1/8$	0.30	0.38	
Curved Vane	.32	0.24	0.25

Next, in order to find the depth y_1 for which the orifice equation applies, discharges were computed by using Eq. (A-18) for the whole range of flow depths; a comparison was then made with actual data. A distinct deviation of the data from the equation was recognized for flows with shallow depths where weir flow existed. The minimum flow depth for which the orifice equation can be applied was determined from this analysis to be

$$
y_1 = 0.43 \, \text{L}^{0.7} \tag{A-21}
$$

In other words, for $y_1 \geq 0.43 \; {\mathsf{L}}^{0.7},$ orifice flow exists and Eq. (A-18) applies. For $y_{\vphantom{\{1}}\vphantom{\{1}}}< 0.43$ L $^{0.7},$ Eq. (A-19) must be used.

III. Comparison of Results

The efficiencies of the three grates on a continuous grade were ·computed by using the equations developed from the analysis on the preceding pages. They are plotted against the measured data in Figures 10 to 12. Approximately, 90 percent of the points lie within an error range of 5 percent and all within a range of 10 percent. As expected, the errors are larger for grates of shorter length for which some frontal flow splashes over and is unintercepted.

The same data **were** used to evaluate the method of determining grate efficiency presented in FHWA Report TS-79-225 (7). The results are
presented in Figures 13 to 15. Although most points (except a few points for lower efficiencies) lie within the error range of 10 percent, this method tends to overestimate the efficiencies. As for the validity of the discharges for the three grates in a sump condition, the computed discharges were compared with the measured discharges in Figure 16. Only 3 out of 230 points slightly exceed the 10 percent error limit.

COMPARISON OF MEASURED AND COMPUTED EFFICIENCIES FIGURE 10

FOR $P = 1 - 7/8 - 4$ GRATE

COMPUTED EFFICIENCY

FIGURE 11 COMPARISON OF MEASURED AND COMPUTED EFFICIENCIES

FOR $P-1-1/8$ GRATE

COMPUTED

MEASURED EFFICIENCY

COMPARISON OF MEASURED EFFICIENCY AND COMPUTED FIGURE 13 EFFICIENCY BY USING FHWA-TS-79-225 METHOD, FOR $P-1-7/8-4$ GRATE

MEASURED EFFICIENCY

COMPARISON OF MEASURED EFFICIENCY AND COMPUTED FIGURE $|4$ EFFICIENCY BY USING FHWA-TS-79-225 METHOD FOR $P-1-1/8$ GRATE

MEASURED EFFICIENCY

COMPARISON OF MEASURED EFFICIENCY AND COMPUTED
EFFICIENCY BY USING FHWA-TS-79-225 METHOD
FOR CURVED VANE GRATE FIGURE 15

FIGURE 16 COMPARISON OF MEASURED AND COMPUTED DISCHARGES OF THE THREE GRATES IN A SUMP CONDITION