July 1982 Final Report

DOT HS-806-323



### Improved Performance of Production Belt System Assembly for the Plymouth Horizon and Plymouth Reliant

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Contract No. DTNH-22-81-C-07111 Contract Amount \$200,000



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#### TECHNICAL REPORT STANDARD TITLE PAGE

	2. Government A	ccession No.	3. Recipient's Catalog	No.
DOT HS 806 323				
4. Title and Subtitle			5. Report Date	
Improved Performance of Pro			July 1982	
Assembly for the Plymouth Plymouth Reliant	llorizon and	DEPARTMENT OF TRANSPORTATION	6. Performing Organiza	ation Code
7. Author(s)			8. Performing Organiza	ation Report
Robert A. Galganski		JUN - 1983	6829-V-8	
9. Performing Organization Name and Address	S	LIBRARY	10. Work Unit No.	
Calspan Corporation Advance	ed Technolog	8	E37 Series	
P.O. Box 400		Chattaneonin (Cloud Col)	11. Contract or Grant DTNH22-81-C-0	-
Buffalo, NY 14225			13. Type of Report and	d Period Cov
12. Sponsoring Agency Name and Address			Final Report:	Janua
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#### PREFACE

This final report is submitted in partial fulfillment of the documentation requirements of Contract No. DTNH22-81-C-07111. It presents an overview of the research effort previously documented in technical reports 6829-V-1 through V-7 and in eighteen progress reports.

The author gratefully acknowledges the contributions of Messrs. Sheridan Smith and Saverio Pugliese, both of Calspan ATC. Mr. Smith supervised the construction of the Reliant sled body buck and was responsible for the detail design and preparation of engineering drawings of mounting hardware for the Takata Kojyo webbing clamp assembly and the Calspan-developed breakaway steering shaft used in the compartment-modified Reliant and Horizon, respectively. He also assisted in high-speed film data analysis of the two New Car Assessment crash tests and the Phase I and Phase II Reliant sled test series. In addition to providing valuable advice and constructive review of all technical reports, Mr. Pugliese suggested that the author investigate the concept of seat cushion stiffening to help control occupant head rotation/acceleration. This concept proved to be instrumental in the successful completion of this program.

The author would also like to thank the following individuals for their help during the program: Mr. T. Albert Yamada of Mike Masaoka Associates, Mr. David J. Romeo of Romeo Kojyo Co.. Inc., and Mr. Robert Simpson of the Chrysler Corporation. Messrs. Yamada and Romeo provided reliable Takata Kojyo restraint system hardware used in developmental and evaluation testing while Mr. Simpson supplied the author with essential information regarding production Chrysler belt restraint system design.

The NHTSA Contract Technical Monitor for this program was Mr. Lee Stucki of the Occupant Packaging Branch of the Vehicle Engineering Research Division. Mr. Stucki's interest in the program and willingness to accommodate several changes in the original work plans were an integral part of this highly successful program.

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The opinions, findings, and conclusions expressed in this publication are those of this author and not necessarily those of the National Highway Traffic Safety Administration.

This report has been reviewed and approved by:

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#### 1.0 INTRODUCTION

Many different small domestic and foreign cars have been evaluated in nominal 35 mph frontal impact crash tests as part of the NHTSA's ongoing New Car Assessment (NCA) program. Test results have indicated that while many of these cars provide generally satisfactory crash energy management and occupant compartment integrity, they still fail to offer adequate protection from injury to one or both of their front-seated occupants. This unsatisfactory performance can be attributed, at least in part, to inherent deficiencies in the vehicle standard-equipment belt restraint system, seat, steering column or dashboard design. In such cases, the application of straightforward, relatively minor modifications to one or more of these systems may offer the potential for full compliance with all FMVSS 208 occupant injury criteria.

Consistent with the above observation and hypothesis, the NHTSA selected two current-production Chrysler Corporation automobiles as the basis for the research program described in this report. The vehicles selected, a Plymouth Horizon TC-3 2-door hatchback (L-body car) and a Plymouth Reliant 2-door sedan (K-car), each were subjected to nominal 35 mph flat frontal barrier crash tests in the NCA program.\* In addition, 1980 model year Horizons were also tested in nominal 70 mph closing speed,\*\* colinear, car-to-car impacts against a 1980 Chevrolet Citation 2-door sedan and a 1980 Ford Mustang 2-door sedan in the same program. Occupant performance data were obtained for Part 572 50th percentile male anthropomorphic test devices (ATDs) placed in the driver and right-front seating positions of each vehicle.

Although both occupants of the barrier-tested Horizon (Reference 1) satisfied all FMVSS 208 occupant injury criteria, their counterparts in the carto-car exposures each failed to comply with one or more of these requirements. Of these two tests, the Mustang impact (Reference 2) generated a more severe

<sup>\*</sup>Vehicle model years 1979 and 1981, respectively.

<sup>\*\*</sup>Impact speed 35 mph, each car.

collision environment for the Horizon occupants than the Citation exposure (Reference 3). In the Reliant barrier test (Reference 4), the driver occupant alone complied fully with all injury criteria.

The original objectives of the subject research program entitled "Improved Performance of Production Belt System Assembly for the Plymouth Horizon and Plymouth Reliant," hereafter referred to as the BSA program, are delineated below:

- Horizon: (1) identify areas of the seat belt assembly, steering column and occupant compartment design which could be improved to enhance driver and right-front passenger performance in nominal 35 mph frontal impact exposures; and (2) make minor, readily producible modifications to one or more of the above systems which will insure compliance with FMVSS 208 occupant injury criteria in colinear, car-to-car frontal crash tests at a nominal 70 mph closing speed.
- Reliant: (1) identify those belt restraint system design characteristics which permit excessive front seat passenger excursion during a nominal 35 mph flat frontal barrier collision exposure; and (2) improve the production belt restraint system to insure that the occupant complies fully with all FMVSS 208 injury criteria in such a crash test.

On the basis of information which became available during the course of the program, the Ford Mustang was selected as the opposition vehicle for the demonstration of improved Horizon occupant performance in the car-to-car impact test mode. Also, satisfaction of program objectives for the Reliant required modification of more than just its production belt restraint system.

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The BSA program, which was initially formulated for the Plymouth Horizon only, was subsequently expanded early in the original effort to include the Plymouth Reliant. The latter work was assigned higher priority by the NHTSA and thus was completed before the Horizon part of the program.

Implementation of the above program objectives commenced with a detailed data review/evaluation of baseline Horizon and Reliant occupant protection performance in each of the four, previously mentioned NCA crash tests. Knowledge obtained from these studies, discussed in Section 2.0, was instrumental in the formulation of our initial planned technical approach to vehicle occupant compartment interior systems redesign.

Subsequent validation and developmental sled testing provided essential additional insight into this process. Systems were developed which provided improved (relative to their production counterparts) occupant performance in appropriate simulated collision environments. These efforts are summarized in Section 3.0.

The same (or similar) substitute/modified compartment interior systems were employed in a production 1981 Reliant and evaluated under the corresponding NCA frontal barrier crash test condition. Also, a production 1982 Horizon was equipped with alternate occupant compartment systems and similarly evaluated in a car-to-car head-on impact against a 1982 Ford Mustang. Both modified vehicles afforded their occupants significantly improved crash protection relative to their baseline counterparts. The comparison of critical Head Injury Criterion (HIC) and maximum resultant chest acceleration ( $C_R$ ) responses listed in Table 1 vividly typifies this improved overall performance. Complete summaries of these highly successful tests are presented in Section 4.0.

The last section, 5.0, outlines the significant conclusions drawn from the BSA research effort. Recommendations for additional, future work of a similar nature are also presented therein.

	œ					]
	NJURY RESPONSES AFTER COMPT. SYSTEMS MOD. <sup>1</sup>	$c_R \sim g's$	33 (34)	<b>3</b> 8 (37)	44	38
	INJURY RESP COMPT. SYS	HIC	367 (391)	NA <sup>2</sup> (296)	727	510
	INJURY RESPONSES BEFORE COMPT. SYSTEMS MOD.	$c_{ m R}\sim { m g's}$	52	29	61	52
	INJURY RESPONSES BEFOR COMPT.SYSTEMS MOD.	HIC	605	1731	1817	2096
	OCCUPANT	LOCATION	DRIVER	<b>PASSENGER</b>	DRIVER	PASSENGER
the second se		VEHICLE	PLYMOUTH RELIANT		PLYMOUTH HORIZON	TC-3
		TEST CONFIGURATION	FLAT FRONTAL BARRIER		HEAD-ON IMPACT WITH FORD MUSTANG	

<sup>1</sup>NUMBERS IN PARENTHESES REFLECT RESULTS OBTAINED BY AN INDEPENDENT CONTRACTOR.

<sup>2</sup>PRODUCTION BUCKLE ASSEMBLY WEBBING FAILED. HEAD Z ACCELERATION DATA CONTAMINATED BY ANOMALOUS NOISE SPIKES.

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## Table 1

# PLYMOUTH RELIANT AND PLYMOUTH HORIZON OCCUPANT PERFORMANCE IN NOMINAL 35 MPH FRONTAL CRASH TESTS

#### 2.0 PRODUCTION VEHICLE CRASHWORTHINESS DATA REVIEW AND EVALUATION

All available data generated during the conduct of the five pertinent NCA crash tests noted in Section 1.0 were first analyzed in an attempt to assess the crashworthiness performance of the production Horizon and Reliant in these 35 mph frontal crash test exposures. Consistent with the scope of the BSA program, attention was focused primarily on the occupant compartment interior systems and components directly affecting occupant performance.

This section summarizes the major conclusions reached during the Horizon and Reliant evaluation studies. They are presented in the proper chronological order, i.e., Horizon first, followed by Reliant-related observations. The reader is referred to References 5 and 6, respectively, for a detailed discussion of these findings.

#### 2.1 Plymouth Horizon Responses

- The car-to-car collision mode is more severe than the flat barrier impact configuration with respect to both Horizon structural and occupant responses.
- (2) The head-on impact configuration places greater kinetic energy dissipation demands on the Horizon (and opposing vehicle) front structure.
- (3) Horizon underride of the opposing vehicle front end in the car-to-car exposures resulted in the generation of greater occupant compartment pitch and velocity change than that produced in the barrier impact. The car-to-car tests also caused larger steering column/ wheel and fire wall intrusions than the barrier test configuration.

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- (4) Vehicle whole-body pitch is an inherent characteristic of the Horizon collision response. This action is more pronounced in the car-to-car impact configuration than in the flat barrier exposure. It also contributes to pitching of the seats themselves relative to the occupant compartment.
- (5) In the head-on impact mode the front seats are prone to relative forward motion along their respective seat tracks.
- (6) Within the car-to-car test mode itself, the Mustang impact exposure generated a more severe collision environment for the Horizon structure and occupants than the corresponding Citation exposure.
- (7) Excessive steering column/wheel intrusion and subsequent head-wheel contacts in the head-on crash tests is the overriding cause for Horizon driver noncompliance with the FMVSS 208 HIC requirement.
- (8) Rigid coupling of the upper steering column to the lower portion of the dashboard contributes to the vertical displacement component of steering system motion upon driver dummy knee contact with the dashboard and/or steering column shroud.
- (9) The passenger dummy in both the flat barrier and car-to-car impact configuration undergoes excessive forward motion relative to the passenger compartment. Passenger noncompliance with the HIC requirement in the latter test mode occurs as a result of head contact with a hard surface (dashboard or dummy knee).

- (10) The standard 3-point Horizon belt restraint system permits excessive webbing spool-off from the retractor.
- (11) The occupants in all three tests have a tendency to submarine to some degree. This motion is attributable to occupant compartment/seat pitching, inadequate dummy lower torso/extremity restraint from the dashboard, seat relative forward motion and perhaps seat design.\*
- (12) Substantial passenger-side retractor housing deformation occurred in all three tests.

#### 2.2 Plymouth Reliant Responses

- Right-front passenger noncompliance with the HIC requirement of FMVSS 208 stemmed directly from head impact with the unpadded metal dashboard top cover.
- (2) Excessive fore-aft occupant stroke (relative to the passenger compartment), combined with substantial upward/rearward dashboard intrusion, permitted the above contact to occur.
- (3) The Plymouth Reliant has a relatively limited amount of available passenger compartment stroking distance (e.g., as provided by a dummy chest-to-dash measurement) compared to other cars in its size/weight class.

<sup>\*</sup>Developmental sled testing later showed that seat cushion design constitutes a critical factor in occupant submarining response.

- (4) The aforementioned dashboard intrusion seriously compromised the (somewhat limited) space available for the occupant to ride down the crash.
- (5) Belt spool-off from the retractor is the principal mechanism responsible for excessive occupant stroking. Significant belt playout and head motion, at relatively low torso belt loadings, occur early in the event.

#### 3.0 VALIDATION AND DEVELOPMENTAL SLED TESTING

Three separate series of sled tests were performed during the BSA program; these efforts are outlined in the ensuing subsections. Consistent with the prudent use of program monetary resources, no attempt is made to describe these activities in complete detail. Rather, the reader is referred to the original test reports which provide excellent, comprehensive documentation of all individual tests. Included therein are test objectives, a description of the occupant compartment interior systems configuration evaluated, electronic data, photographs, analyses and visual observations. High-speed movies of each event are in the possession of the NHTSA.

This final report presents a summary of significant sled test results, including FMVSS 208 occupant injury indicator information, in tabular format. The "best fix" occupant compartment interior systems modification concepts uncovered during each of the three series are also briefly discussed herein.

#### 3.1 Phase I Plymouth Reliant Developmental Sled Tests

Ten (10) single-occupant (right-front passenger) sled tests were performed in this initial (Phase I) series of developmental testing. Complete results are documented in Reference 7.

As indicated in Table <sup>2</sup>, the Phase I test series explored a number of different restraint system-related modifications/concepts in an effort to improve the unsatisfactory right-front passenger performance provided by this vehicle in a New Car Assessment (NCA) 35 mph flat frontal barrier crash test (Reference 4). The most promising of these consisted of a simple, 12-inch length reduction of its standard unibelt (Tests 2749 and 2755). This change substantially reduced the head stroke, belt spool-off from the retractor and HIC number relative to results generated in two replicate baseline (zero belt reduction)

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PHASE I PLYMOUTH RELIANT DEVELOPMENTAL SLED TEST RESULTS Table 2

	REMARKS			TOP OF HEAD GRAZED RIM OF METAL DASH. BDARD TOP COVER.				BELT FAILURE AT WEB- BING CLAMP CAUSED UNRESTRAINED OCCU- PANT MDTIDN	LOCK-UP ACTION NOT TOTALLY EFFECTIVE. BELT SPOULED OFF BELT RECOR AFTER RETRACTOR AFTER AN INITIAL 11 ms. DELAY.	NECK FAILED IN TENSION/HEAD	BROKE LOOSE (CRACK IN NECK OPENED UP AT ABOUT 73 ms.).	NO HEAD-DASH CONTACT OCCURRED EFFECTS OF PADDING ON HEAD RESPONSES NOT ASCERTAINED.	
LTA	MAX. LAP BELT LOAD (Ibs.)			NA6	086	1570	1180		1620	1760	1120	1160	1600
RESTRAINT SYSTEM DATA	MAX. TORSO BELT LOAD (Ibs.)			1910	2100	2130	2110	2200 <sup>7</sup>	2080	2180	2170	2S20	2200
RESTRA	MA XIMUM BELT SPOOL-OFF4 (m.)			4 4	2.8	4.6	2.3		4.1	3.4	2.6	4.7	4.0
	MAX. HEAD DISPLACE. MENT3 (In.)			21.0	19.3	20.7	17.S		8		18.1	20.8	20.2
	R R S S		RIGHT	8	80	110	S40	,	70	8	410	650	8
	MAX. COMPRES- SIVE FEMUR LOADS		LEFT	190	220	160	410		140	130	390	420	160
DATA	ANT	6	TIME (ms.)	65/39 <mark>5</mark>	61/74	6S/78	62/80		63/79	65/80	62/75	62/81	64/79
DUMMY DATA	CR <sup>1</sup> , MAX. RESULTANT CHEST ACCELERATION	100 000	(s.6)	44	47	s	49		20	24	46	54	ß
		, ar	12	111	115	119	112		118		116	113	145
	HIC, HEAD INJURY CRITERION	✓ INIMIT		8	11	72	65	,	72		ß	70	ß
	CRET		2046	974	317	1130	992		1092		964	1341	1492
	ANT	L	(ms.)	97	98	83	. <b>68</b>		92		98	92	06
	HR. MAX RESULTANT HEAD ACCELERATIDN		(g's)	116	82	86	79	,	S6 .	,	76	88	100
	RESTRAINT SYSTEM CONFIGURATION			STANDARD	BELT SHORTENED 12 INCHES	RETRACTOR SPOOL DIAMETER INCREASED S/8 INCHES.	RETRACTOR SPOOL DIAMETER INCREASED S/8 INCHES; BELT SHORTENED 20 INCHES	IRVIN INDUSTRIES WEB- BING CLAMP MOUNTED ON B-PILLAR IN PLACE OF D-RING	RETRACTOR LOCKED UP PRE-TEST TO SIMU- LATE IDEALIZED WEBBING CLAMP.	BELT SHORTENED 8 INCHES.	BELT SHORTENED 12 INCHES. (REPEAT OF RUN 2749)	STANDARD, PADDING FROM PRODUCTION DASH TOP ADDED TO PROJECTING DASH. BOARD RIM.	BELT SHORTENED 8 INCHES. (REPEAT OF RUN 2754)
<u>Av.</u>	LUIAL VELOCITY CHANGE (mph)			39.7	39.8	39.7	39.7	40.0	3. 8.	39.7	39.7	38.7	7.65
	RUN NO.			2748	- 2749	2750	2751	2752	2753	2754	27SS	2756	2757

<sup>1</sup>EXCFEDING A CUMULATIVE DURATION OF 3 MILLISECONDS. <sup>2</sup>THESE RESPONSES ALL EXHIBIT AT LEAST TWO COMPARALE RELATIVE MAXIMUM VALUES. THE TIMESLISTED REREIN RELECT THE TIMING OF THE INITIAL PEAK ACCELERATION AND THE NEXT-HIGHETS VALUE. <sup>3</sup>HORIZONTAL COMPONENT AT CLOSEST APPROACH TO DASHBOARD FRONT SURFACE. <sup>3</sup>HORIZONTAL COMPONENT AT CLOSEST APPROACH TO DASHBOARD FRONT SURFACE. <sup>4</sup>AT CLOSEST HEAD-DASH APPROACH. <sup>6</sup>LEONCE, INTERMEDIATE PEAK AT 81 m. <sup>6</sup>LEONCOLL AMMED AGAINST SEAT FRAME DURING EVENT, PRODUCING AN UNREALISTICELL MAMED AGAINST SEAT FRAME DURING EVENT, PRODUCING AN <sup>7</sup>BELT FAILURE LOAD.

configuration runs (Nos. 2748 and 2756).\* However, in Calspan's opinion, the HIC numbers obtained in the belt-shortened Tests 2749 and 2755 (779 and 964, respectively) did not provide an adequate margin to assure successful occupant performance during the final Reliant crash test. Accordingly, we recommended that an additional (Phase II) series of developmental sled testing be performed (see Section 3.2).

It should be noted that the 12-inch belt-shortening modification also failed to prevent the passenger occupant from submarining during forward excursion in the compartment. This motion resulted in highly undesirable lap belt roping and associated penetration of the dummy's abdomen.

#### 3.2 Phase II Plymouth Reliant Developmental Sled Tests

Although Phase II testing was specifically undertaken in an attempt to decrease the above-noted marginally satisfactory occupant HIC response, this effort also addressed other (interrelated) aspects of occupant performance: (1) further reduction of retractor belt spool-off and head displacement magnitudes and (2) minimization/elimination of occupant submarining and consequent lap belt roping/abdominal penetration. The results of this effort are documented in Reference 8.

As indicated in Table <sup>3</sup>, Phase II testing demonstrated that the Reliant's right-front passenger performance could be considerably enhanced by employing two basic modifications: (1) addition of a Takata Kojyo webbing clamp mounted near the intersection of the B-pillar and roof header and substitution of 8%-elongation Takata Kojyo webbing for the production belt material in the standard Reliant restraint system; (2) stiffening the front portion of the Reliant's relatively "soft" bench seat cushion. In the three tests which utilized these system changes (Nos. 2844, 2846 and 2847), the modified restraint system limited

<sup>\*</sup>A slight improvement in maximum resultant chest acceleration was also noted.

	REMARKS					OCCUPANT SUB- MARTIEDILAP BELTIROPED ANO PENE ANO PENE ABOONINAL CRACK OPENEO ABOONINAL CRACK OPENEO OF DUMMYS OF DUMMYS OF CNASTALL SECUENT RUNS)		REPEAT OF RUN 2846
ATA	MAX. LAP BELT LOAD	1.00		17.30	1750	0204F4400008ma	1610	1730 R
SYSTEM O	MAX. TORSO BELT LOAO			2130	2120	2340	2360	2550
RESTRAINT SYSTEM OATA	MAXIMUM BELT SPOOL- OFF			2.1	S. 0	ν. O	0.2	0.1
	MAX. PELVIC OISPLACE. MENT4			9.6	ю. 8	14.7	6. 8	7.6
	MAX, HEAD OISPLACE. MENT <sup>3</sup>			21.3	16.7	16.0 0	15.8	16.7
	ATION		VERT. COMP.	55	23	20	<del>3</del> 4	35
	MAX. PELVIC ACCELERATION (g's)		HORIZ. COMP.	ß	2	ē	R	33
		3~	RIGHT	00	3	670	8	3
TA	MAX. COMPRES SIVE FEMUR		Ľ	2605	2505	320	40	9
DUMMY DATA	TANT		TIME <sup>2</sup> (ms.)	66/83	62/78	61/85 <sup>8</sup>	81/80	63/78
00	CR <sup>1</sup> , MAX RESULTANT CHEST		MAGN. (g's)	45 45	42	£	41	43
				127	126	119	127	126
	HIC, HEAD INJURY CRITERION	TIMING ~ ms	-1-	89	5	226	60	64
	LINI CRIT		MAGN.	795	531	14897	562	634
	C. TANT		TIME (ms.)	S O	න හ	8 9 0 0	68	68
	HR. MAX. RESULTANT HEAO		MAGN. (g's)	77	20	78 846	5	28
IRATION	SEAT			STIFFENEO (MOO.1)	STIFFENED (MOO. 1)	STANDARO	STIFFENEO (MOO. 2)	STIFFENEO (MOD. 2)
CONFIGURATION	RESTRAINT SYSTEM			BELT SHORT. ENEO 12 INCHES	TAKATA KOJ. YO WEBBING/ WEBBING CLAMP CLAMP CLAMP MOUNTED ON B.PILLARAFT ANO ABOVE D.RING POSITION	AS PER 2844.	AS PER 2844.	AS PER 2844.
Av.	VELOCITY CHANGE (mph)			40.3	0.04	40.5	40.6	40.5
	RUN NO.			2843	2844	2845	2846	2847

 Table 3
 PHASE II PLYMOUTH RELIANT DEVELOPMENTAL SLED TEST RESULTS

EXCEEDING A CUMULATIVE OURATION OF 3 MILLISECONOS.

<sup>2</sup>THESE RESPONSES ALL EXHIBIT AT LEAST TWO COMPARABLE RELATIVE MAXIMUM VALUES. THE TIMES LISTEO HEREIN RFFLECT THE TIMING OF THE INITIA! PFAK ACCELERATION ANO THE NEXT-HIGHEST VALUE.

<sup>3</sup>HORIZONTAL COMPONENT AT CLOSEST APPROACH TO OASHBOARO FRONT SURFACE.

<sup>4</sup>HORIZONTAL COMPONENT.

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5ESTIMATED VALUE: OATA TRACE CONTAMINATEO BY NOISE SPIKES.

<sup>6</sup>CAUSEO BY HEAO CONTACT WITH TOP OF SLAT BACK/ENO OF HEAOREST ON REBOUNO.

<sup>2</sup>INCLUOES THE EFFECTS OF HEAD SEAT BACK/HEAOREST CONTACT.

<sup>8</sup>LOWER, INTERMEDIATE PEAK AT 71 ms.

belt spool-off to minimal levels, provided early occupant upper torso restraint and significantly reduced occupant head excursion, insuring avoidance of head contact with an intruding dashboard surface in an actual 35 mph flat frontal barrier crash test. The stiffened seat cushion provided badly needed pelvic restraint to prevent occupant submarining motion and associated lap belt roping/abdominal penetration. All occupant injury indicator parameters obtained in these three tests were all well below FMVSS 208-stipulated limits.

#### 3.3 Plymouth Horizon Validation and Developmental Sled Tests

A total of seven (7) double-occupant validation and developmental sled tests of the Plymouth Horizon were performed to simulate the 70 mph closing speed, NCA car-to-car head-on impact with a Ford Mustang (Reference 2). This effort was completed in two stages: (1) a combination validation/developmental series consisting of four tests performed in August 1981 (Nos. 2807-2810) and (2) a strictly developmental series of three tests performed in March 1982 (Nos. 2953-2955). Results from all seven tests are documented in Reference 9.

The initial test series basically provided occupant and restraint/ occupant compartment interior systems performance data for the baseline Horizon configuration. All four tests utilized production bucket seats on both the driver-side and right-front passenger side. Both production and modified unibelt restraint systems were employed in this series. In addition, the steering wheel position was adjusted to simulate various degrees of column/wheel intrusion in the passenger compartment.

Table 4 presents a summary of the results obtained from the driverside exposures of Tests 2807-2810. The first three validation runs led to the selection of Test 2808 as the baseline Horizon driver configuration.\* Developmental Test 2810, which used a separate Takata Kojyo webbing clamp assembly behind a reverse-mounted production Horizon retractor, showed that this clamp

<sup>\*</sup>None of these runs provided an entirely satisfactory overall simulation of the driver HIC response, resultant chest acceleration or kinematics/body contacts exhibited in the NCA Horizon-Mustang crash test. Test 2808 was, however, so selected because its zero-intruded steering wheel position allowed for the generation of the highest occupant velocity change during crash ride down.

_	2	2	CONFIGURATION	N						5	OUMMY DATA	DATA						RESTRAINT SYSTEM DATA	SYSTEM L	DATA	
RUN NO.	UIAL VELOCITY CHANGE (mph)	RESTRAINT SYSTEM	SEAT	STEERING WHEEL POSITION	HR. MAX. RESULTANT HEAD	R- X. VD	J	HIC, HEAD INJURY CRITERION		C <sub>R</sub> <sup>1</sup> , MAX. RESULTANT CHEST	TANT	MAX. COMPRES- SIVE FEMUR	La RES	MAX. PELVIC ACCELERA TION	R R R	VISIBLE BOOY CONTACTS		MAXIMUM BELT SPOOL- OFF	MAX TORSO BELT LOAD	MAX LAP BELT LOAD	REMARKS
					ACCELE	ACCELERATION		TIMING	_	ACCELERATION	ATION	LOADS	SO -	(s,6)	()		120110	(in.)	ŝ	. sqi)	
					MAGN. (g's)	TIME (ms)	MAGN.	L1	t2	MAGN. (g s)	TIME (ms.)	LEFT	GHT	HORIZ. COMP.	VERT. COMP.	НЕ ИО	ABOOMEN				
2807	42.3	PRDDUCTION	PRDDUCTION PRODUCTION	MAXIMUM INTRUSION	140 <sup>2</sup> (126)	77	1\$25	65	85	54	62	300	4,0	20	σ	FOREHEAD WITH UPPER STEERING WHEEL RIM,	UPPER CHEST WITH LOWER WHEEL RIM.	Ω. Ω	1410	1850	
2808	42.6	PRODUCTION	PRODUCTION PRODUCTION	ZERO INTRUSION	270 <sup>2</sup> (160)	8	2350	8	е 6	9	793	10	420	51	თ	FOREHEAD WITH UPPER WHEEL RIM. FACE WITH WHEEL HUB.	LOWER CHEST WITH LOWER WHEEL RIM.	4 0	1850	1980	
2809	42.5	PRODUCTION	PRODUCTION PRODUCTION	ONE-THIRD MAXIMUM INTRUSION	240 <sup>2</sup> (183)	Ω Ø	2400	83	837	ŝ	ũ	720	280	19	01	GRAZING CONTACT WITH UPPER WHEEL RIM, WHEEL HUB	LOWER CHEST WITH LOWER WHEEL RIM.	4 N	1610	1920	
2810	42 2	PRODUCTION RETRACTOR ANOTAKATA KOJYO WEBBING CLAMP	PRODUCTION RETRACTOR RETRACTOR RETACTOR KOJYO WEBBING CLAMP CLAMP	MAXIMUM INTRUSION			1			I			1	- 1	1		1	1	l	i	DATA NDT VALID ROOF RAIL MOUNT FOR RESTRAINT SYSTEM BENT DURING TEST,

Table 4 PLYMOUTH HORIZON SLED TEST RESULTS: DRIVER OCCUPANT SLED TESTS 2807-2810

<sup>3</sup>SLIGHTLY LOWÉR PEAK AT 64 ms.

<sup>2</sup>SHORT DURATION SPIKE CAUSED BY HEAD CONTACT WITH THE STEERING WHEEL THREE MILLISECONO CUTOFF VALUE SHDWN IN PARENTHESIS.

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was capable of substantially reducing belt spool-off and limiting occupant head displacement. Early upper-body restraint, however, failed to prevent the dummy from submarining during its forward excursion. This clearly indicated the need for increased occupant pelvic restraint.

Passenger-side results from the initial Horizon fied test series are shown in Table 5. Unfortunately, valid, incontrovertible data was obtained in only one of the four tests. The results of the other three tests were invalid because of a belt system integrity failure. In the latter tests, the production seat belt buckle assembly unlatched during occupant forward excursion, resulting in various degrees of partial (or nonexistent) occupant restraint at different event times. This problem, together with buck structural failures in Test 2810 and a higher priority placed in the Reliant part of the program by the NHTSA, caused the Horizon sled test effort to be postponed until March 1982.

The only valid passenger-side exposure (Test 2808) employed a production restraint system with its overall belt length shortened 6 inches. Because of program economy considerations and since results obtained with a full-length belt would not be expected to differ significantly from that generated in this run,\* no further validation-type tests were performed when testing resumed. Thus Test 2808 was selected as the baseline Horizon passenger configuration. Clearly, if occupant performance were improved relative to Test 2808, it follows that the true baseline configuration (i.e., a full-length belt) would be similarly improved.

Test 2808 results clearly indicated the need for occupant compartment interior systems which could simultaneously (1) limit head and pelvic excursions, (2) prevent excessive head acceleration and HIC responses and (3) reduce or at least maintain the present, acceptable chest acceleration response level.

<sup>\*</sup>Occupant stroke and head acceleration responses would probably be higher in such a test.

	REMARKS		DATA NOT VALID DATA NOT VALID UBLI BUCKLE UBLATCHED DURING OCCUPANT FORWARD MOTION.	OCCUPANT SUB MARINED AP BELT ROPEO AND FENE TRATED ADDOMINAL REGION BACK OF DCCUPANT'S HE AO OCCUPANT'S HE AO OF THE SEAT BACK/ REST OM REBOUNO.	DATA NOT VALID BELT BUCKLE UNLATCHED DURING OCCUPANT FORWARD MOTION.	DATA NOT VALID BELT BUCKLE BLLT BUCKLE ULLATCHED DURING OCCUTANT FORWARD MOTION
DATA	MAX. LAP BELT LOAD (Ibs.)		1	1610	I	I
SYSTEM 0	MAX. TORSO BELT LOAO (Ibs )		1	2540	I	1
RESTRAINT SYSTEM OATA	MAXIMUM BELT SPOOL OFF (in.)		I	8 E	I	1
	MAX. PELVIC DISPLACE MENT <sup>3</sup> (in.)		I	13.9	1	i
	MAX. HEAO OISPLACE MENT <sup>2</sup> (in.)		i	21.6	I	1
	X. IC NN SN	VERT. COMP.	I	13	1	1
	MAX. PELVIC ACCELERA TION (9's)	HORIZ. COMP.	1	28	ł	1
	MAX. DMPRES SIVE EMUR LOADS	RIGHT	I	330	I	1
DATA	MAX. COMPRES SIVE FEMUR LOADS	(Ibs.)	1	150	1	1
OUMMY DATA	K TANT ST ATION	TIME (ms).		99		
	CR <sup>1</sup> , MAX. RESULTANT CHEST ACCELERATION	MAGN. (9's)	1	23	,	I
	NO	tiMING ~ πιs. t1 t2		150		
	HIC, HEAD INJURY CRITERION	TIMINO		68) (68)		
	J	MAGN. (g's)	I	1711 (2198) <sup>5</sup>	1	1
	R. X. TAN1 AO RATION	TIME (ms.)		95 (166)		
	HR. MAX. RESULTAN1 HEAO ACCELERATION	MAGN. (9's)	1	93 (106) <sup>4</sup>	I	ı
IRATION	SEAT		PRODUCTION	PRODUCTION	PRODUCTION	PRODUCI LON
CONFIGURATION	RESTRAINT SYSTEM		PROOUCTION PROOUCTION	PROPUCTION PRODUCTION BELTLENCTH SHORTENED 6 INCHES	PRODUCTION RETRACTOR REPOSTOR REPOSTOR TIONED 3 INCHES REARWARD	I-RDDUCTION RETRACTOR AND TAKATA KOJYO WEBBING CLAMP
Δv, TOTAI	VELOCAL CHANGE (mph)		42.3	4 .	42.5	42.2
	RUN NO.		2807	2808	2809	2810

Table 5 PLYMOUTH HORIZON SLED TEST RESULTS: PASSENGER OCCUPANT SLED TESTS 2807-2810

<sup>1</sup>FXFEEDING A CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>2</sup>HORIZONTAL COMPONENT AT CLOSEST APPROACH TO OASHBOARO FRONT SURFACE

<sup>3</sup>HORIZONTAL COMPONENT.

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<sup>4</sup>CAUSEO BY HEAD CONTACT WITH THE SEAT BACK/END OF HEADREST ON REBOUND.

<sup>5</sup>INCI UDES THE EFFECTS OF HEAO SEAT BACK/HEADREST CONTACT

The final three Horizon sled tests (Nos. 2853-2855) utilized restraint system/occupant compartment modifications similar to those employed in the highly successful Phase II Reliant sled test series. In the Hdrizon effort, a Takata Kojyo combination retractor/webbing clamp device, including 8%-elongation Takata Kojyo webbing, was substituted for the standard Horizon belt restraint system in order to limit occupant upper-body stroke. Also, the front portion of both bucket seat cushions were stiffened in an attempt to minimize occupant pelvic displacement. Two seat cushion modifications were evaluated; the second one (Mod. 2) provided the desired pelvic restraint.

Tables 6 and 7 present the results of these tests. Examination of these tables shows that occupant injury indicator parameters obtained in those tests which featured the Takata Kojyo restraint system and the Mod. 2 seat cushion were all well within FMVSS 208-stipulated limits.\* Moreover, these changes also eliminated the inherent tendency for both occupants to submarine and the lap belt to rope by limiting forward pelvic excursion. Furthermore, occupant head excursion was well controlled, insuring avoidance of head contact with an intruding dashboard or steering wheel surface in an actual 35 mph carto-car head-on collision. The kinematics of both occupants were also excellent during forward excursion and rebound motion. The unsatisfactory HIC results from driver-side Test 2955 (steering wheel in its two-thirds intruded position), however, indicated that successful driver occupant performance in such a collision environment mandates that steering wheel intrusion into the occupant compartment be minimized to the greatest extent feasible.

<sup>\*</sup>Tests 2954 and 2955 for the passenger-side exposure; Test 2954 (steering wheel in unintruded position) for the driver-side exposure.

یے ا		CONFIGURATION						-		DUMMY DATA	DATA	ŀ		ſ			RESTRAINT SYSTEM DATA	SYSTEM (	DATA	
VELOCITY CHANGE (mph)	RESTRAINT SYSTEM	r seat	STEERING WHEEL POSITION	HR, MAX. RESULTANT HEAD	DANT		HIC, HEAD INJURY CRITERION		CR <sup>1</sup> MAX. RESULTANT CHEST	TANT	MAX. COMPRES SIVE FEMUR	LE SS	MAX. PELVIC ACCELERA- TION	N ERC.	VISIBLE BODY CONTACTS	LE V CTS	MAXIMUM BELT SPOOL OFF	MAX. TORSO BELT LOAD	MAX. LAP BELT LOAD	REMARKS
				ALLELE	NOIN		TIMING		ACCELERATION	ALION	LUAUS		(s,8)			CHECT/	(in.)	(.sq)	(ibs.)	
				MAGN. (g's)	TIME (ms.)	MAGN.		t_2	MAGN. (9's)	TIME (ms.)	LEFT 1	RIGHT	HORIZ. COMP.	VERT. COMP.		ABDOMEN				
42.2	TAKATA KOJYORE- TRACTOR/ WEBBING CLAMP ASSEMBLY	STIFFENED (MOD. 1)	ZERO INTRUSION	1		1			1		I	1	1	1	i .	1	1	1	1	RETRACTOR CLAMPHOUSING CLAMPHOUSING SPARATEDFROM MOUNTING PLATE RESULTING IN UNRESTRAINED
32.5	AS PER 2953	STIFFENED (MOD. 2)	ZERO INTRUSION	≈ 62 <sup>2</sup>	06 ≈	619	47	121	46	54	8	60	29	68	NONE	NONE	0.5	1770	1270	OCCUPANT NOTION.
42.5	AS PER 2953	STIFFENED (MOD. 2)	TWO-THIRDS 146 <sup>3</sup> MAX. INTRUSION	146 <sup>3</sup> (128)	8	1230	&	91	4 Q.	60	06	20	30	8	FACE WITH STEERING WHEEL HUB	UPPER CHEST WITH LOWER STEERING WHEEL RIM	0.5	1880	1200	LOWER STEEP NG WHEEL RIM SUS TAINED MINIMAL PERMANENT BEND ING DEFORMATION

PLYMOUTH HORIZON SLED TEST RESULTS: DRIVER OCCUPANT SLED TESTS 2953-2955

Table 6

<sup>1</sup>EXCEEDING A CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>2</sup>VALUE QUOTED EXCLUDES AN ANOMALOUS NOISE SPIKE PRESENT IN THE HEAD Z ACCELERATION DATA.

<sup>3</sup>SHORT.DURATION SPIKE CAUSED BY HEAD CONTACT WITH THE WHEEL HUB. THREE MILLISECOND CUTOFF VA: I'FE SHOWN IN PARENTHISIS.

LYMOUTH HORIZON SLED TEST RESULTS: 1 SLED TESTS 2953-2955
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Table 7

REMARKS			OCCUPANT SUB MARINED LAP BELT MARINED LAP BELT TRAYED ABOOMINAL FREIGN BACK ON OCCUPANT SHEAD OCCUPANT SHEAD OCCUPANT SHEAD OCCUPANT SHEAD OCCUPANT SHEAD FREIGNOR FREIGNAR FR		REPEAT OF RUN 2954
RESTRAINT SYSTEM DATA	MAX LAP BELT LOAD (Ibs )		1680	1650	1050
	MAX. TORSO BELT LOAD (Ibs.)		1990	2150	2020
RESTRAINT	MAXIMUM BELT SPOOL. OFF (in.)		0,1	0.5	0.5
DUMMY DATA	MAX. PELVIC DISPLACE. MENT3 (in.)		12.0	5.5	6.0
	MAX. HEAD DISPLACE MENT <sup>2</sup> (in.)		15.6	16.4	14.9
	MAX. PELVIC ACCELERA. TION (9's)	VERT. COMP.	31	46	23
		HORIZ. COMP.	24	31	22
	MAX. COMPRES- SIVE FEMUR LOADS	(łbs.) TRIGHT	2205	220 <sup>5</sup>	210 <sup>5</sup>
		LEF	220	160	80
	CR <sup>1</sup> , MAX, RESULTANT CHEST ACCELERATION	TIME (ms).	0 10	59	58
		MAGN. (g's)	작	43	43
	-5	tiMiNG ~ ms. t1 t2		126	121
		TIMIN 1		53	57
		MAGN. (g'sl	2 4	616	677
	HR, MAX, RESULTANT HEAD ACCELERATION	TIME (ms.1		06	91
		MAGN. {g's}	NA <sup>4</sup>	26	65
CONFIGURATION	SEAT		STIFFENED (MOD 1)	STIFFENED (MOD. 2)	STIFFENED (MOD. 2)
	RESTRAINT SYSTEM		TAKATA KOJYORE. TRACTOR/ TRACTOR/ CLAMP ASSEMBLY ASSEMBLY	AS PER 2953	AS PER 2953
JV. TOTAL VELOCITY CHANGE (mph)			6 2 2	42.5	42.5
NON.			2953	2954	25955

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<sup>1</sup>EXCEEDING & CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>2</sup>HORIZONTAL COMPONENT AT CLOSEST APPROACH TO DASHBOARD FRONT SURFACE.

<sup>3</sup>HORIZONTAL COMPONENT,

<sup>4</sup>h<sub>R</sub> and hic results are ouestionable. Head x acceleration data contain several discontinuities/spikes Caused by intermittent grounding of an accelerometer wire during the event.

5value ouoted discounts anomatious noise spikes present in the DATA.

#### 4.0 FULL-SCALE CRASH TESTING

A 1982 Plymouth Horizon TC-3 2-door hatchback and a 1981 Plymouth Reliant 2-door sedan were each subjected to nominal 35 mph frontal crash tests in order to evaluate the effectiveness of selected occupant compartment interic system changes developed during previous sled testing in an actual collision environment. Calspan attempted to duplicate, to the extent feasible, the test conditions for the corresponding original NCA crash test. Occupant performance data were obtained for two Part 572 50th percentile male anthropomorphic test devices (ATDs) placed in the driver and right-front seating positions of the test vehicles. All seats were in the mid-position on their respective seat tracks.

The following two subsections briefly summarize, in tabular format, the occupant-related results obtained for the compartment-modified vehicles in the two BSA program crash tests. Identical information generated in the corresponding NCA tests is also presented in these tables, enabling a direct comparison of occupant protection provided by both the baseline (NCA) and compartment-modified (BSA) cars.

Complete tests results, including written and pictorial descriptions of the modified compartment systems, electronic data, photographs, pre- and post-test measurements, analyses, and visual observations are documented in two separate test reports. High-speed movies of both crash tests are in the posses: of the NHTSA.

#### 4.1 Plymouth Reliant Flat Frontal Barrier Crash Test

A production 1981 Plymouth Reliant 2-door sedan was subjected to a 35.3 mph flat frontal barrier crash test on October 19, 1981. The Reliant was equipped with a Calspan-developed, modified front bench seat cushion and 3-point

belt restraint systems.\* A complete description of these systems, as well as all test results, is provided in Reference 10. For the convenience of the reader, however, the above modifications are described briefly in the ensuing three paragraphs. Also, electronic data recorded during this test are presented in Appendices A, B and C of this report.

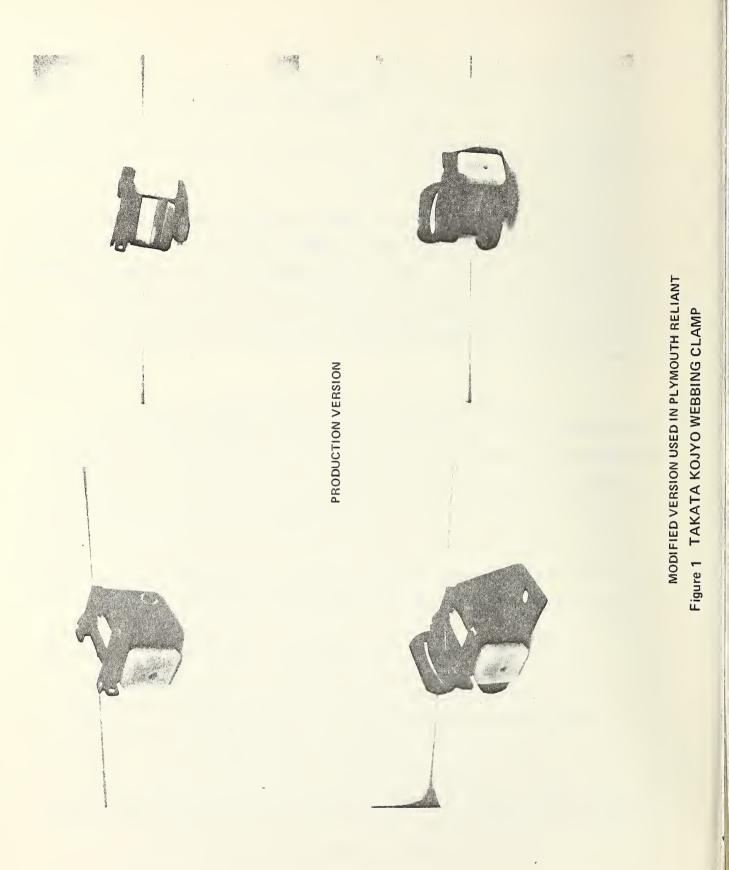
The Reliart's production unibelt restraint system was modified by (1) the installation of a Takata Kojyo webbing clamp and conventional belt guide loop near the B-pillar; (2) removal of the standard-equipment, B-pillar-mounted D-ring; and (3) substitution of 8%-elongation Takata Kojyo webbing for the thinner, 7%-elongation production Reliant webbing. Photographs of the production and Reliant-adapted versions of the Takata Kojyo webbing clamp are presented in Figure 1.

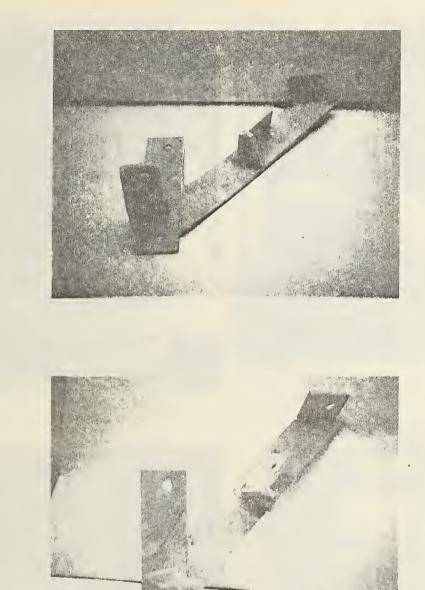
The clamp and a conventional belt guide loop were mounted on a specially-designed bracket, depicted in Figure 2. The bracket was bolted in three places to locally reinforced regions of the B-pillar and side roof rail (see Figure 3). Various views of the passenger-side webbing clamp system and resulting belt configuration are depicted in Figure 4. It can be seen that the belt guide loop serves to remove an inherent belt twist between the production retractor (mounted at the base of the B-pillar) and the webbing clamp. Pre-test photographs of the modified restraint system belt configuration in place around the driver and passenger occupants are presented in Figure 5.

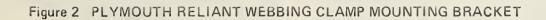
The production Reliant front bench seat cushion was stiffened along its front edge in a rather straightforward manner. Four mild steel angle sections were welded to the underside of the existing cushion frame and the space between it and the cushion springs filled with a single, 5-inch thick shaped piece of closed-cell polystyrene insulation board;\*\* this fill material

<sup>\*</sup>Both the driver and the right-front passenger were restrained with the same modified belt systems.

<sup>\*\*</sup>Dow Chemical Styrofoam SM, with a density of 2.1 lbs./ft.<sup>3</sup>, was used.







1 × 1 ×

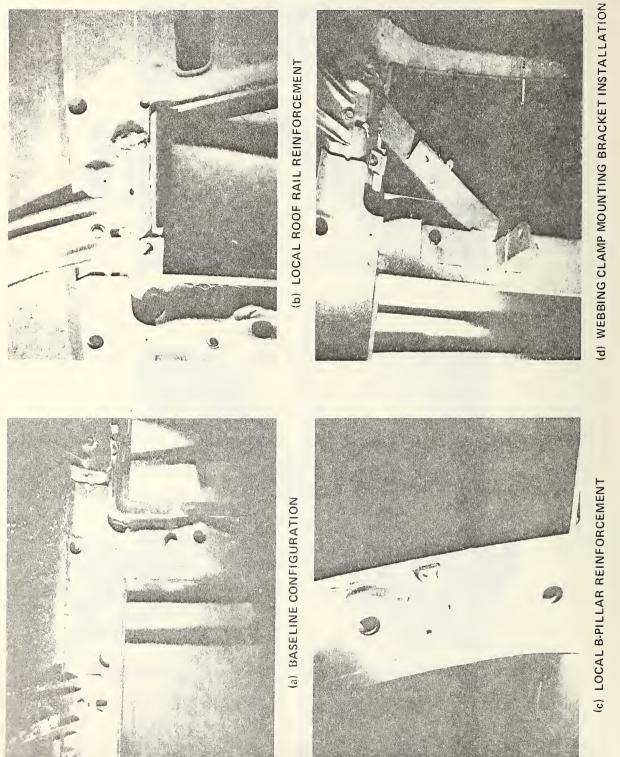
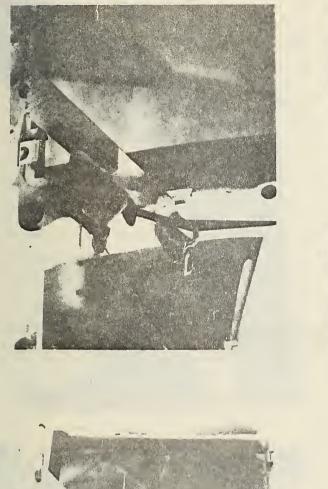


Figure 3 PLYMOUTH RELIANT B-PILLAR/ROOF RAIL MODIFICATIONS

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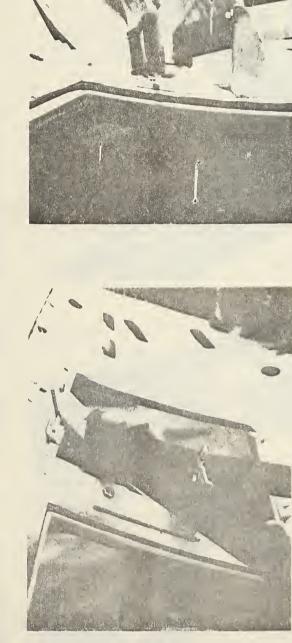
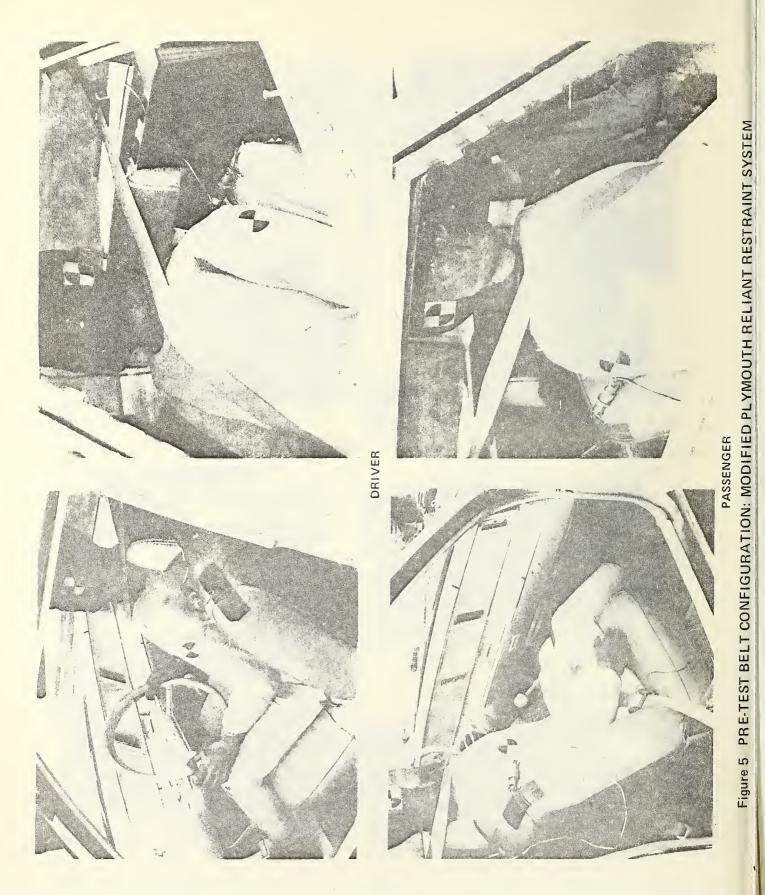


Figure 4 PLYMOUTH RELIANT WEBBING CLAMP SYSTEM/BELT CONFIGURATION

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extended about 8 inches back from the inner cushion front end. Covering the foam over most of its lower surface was a contoured, 0.040-inch thick sheet of 2024 aluminum. Figure 6 depicts the components and various stages of construction of the cushion assembly used in the crash-tested Reliant.

Table 8 presents a summary of Reliant driver responses in both the BSA and NCA barrier impacts. With the possible exception of a low-severity head contact with the upper steering wheel rim,\* the driver in the BSA test exhibited excellent responses in all aspects of occupant performance. Occupant injury indicators were all well below the allowable values stipulated by FMVSS 208. Moreover, the maximum resultant head and chest accelerations (i.e.,  $H_R$  and  $C_R$ , respectively) and HIC number were substantially lower than the already satisfactory magnitudes recorded in the NCA test. On a percentage basis, the BSA HIC and  $C_R$  were 63 percent and 45 percent below their allowable limits while their NCA counterparts were 40 percent and 13 percent below these same respective values. Left and right BSA femur loads were respectively 24 percent and 56 percent below the maximum allowable loading compared to 56 percent and 72 percent below for the corresponding NCA magnitudes.

High-speed films and post-test inspection of the dummy and belt configuration disclosed the absence of lap belt roping/penetration of the driver dummy's abdominal region. The films also showed a maximum retractor belt spooloff of less than 1/4 inch for the BSA test, appreciably less than the approximate 3 inch belt playout recorded in the NCA test at the time of initial head-wheel rim contact.

Unfortunately, an incontrovertible assessment of the Reliant right-front passenger performance in the subject crash test could not be made because of the occurrence of a bizarre restraint system component failure and an unrelated critical head accelerometer malfunction. The production webbing

<sup>\*</sup>As borne out by the head acceleration and HIC number values, driver head contact with the steering wheel was more severe in the NCA test than in its BSA counterpart.

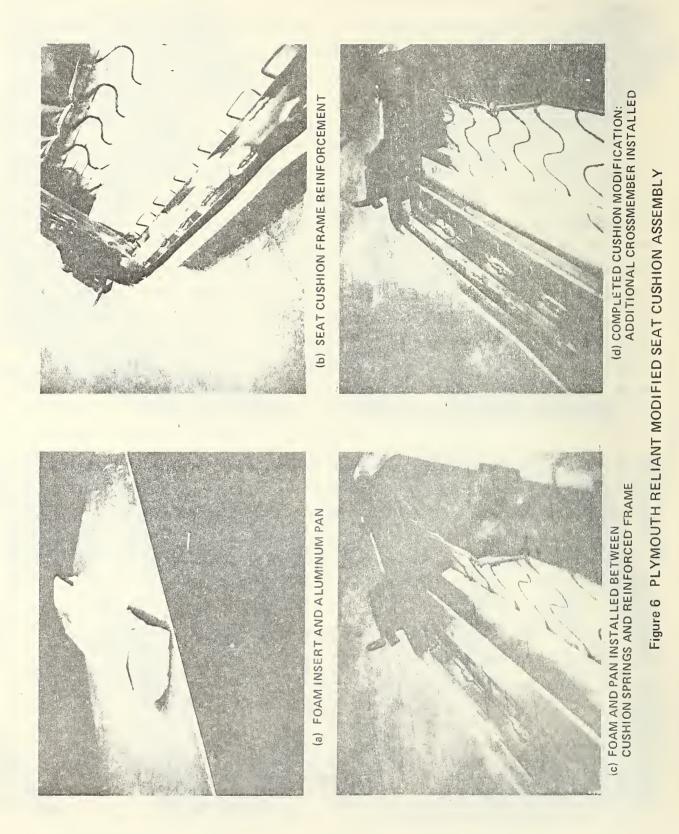


Table 8

# PLYMOUTH RELIANT DRIVER RESPONSES IN 35 MPH FLAT FRONTAL BARRIER CRASH TESTS

REMARKS		EXTENSIVE BENDING OF LOWER STEERING WHEEL RIM INERTIA- SENSITIVE SEAT BACK FAILED TO LOCK UP.	NEGLIGIBLE STEERING WHEEL RIM DEFDRMATION. SEAT WOVED FORWARD RELATIVE TO TRACK
BELT SPOOL-OFF AT INITIAL HEAD WHEEL	CONTACT (in)	e N	< 1/4
VISIBLE BODY CONTACTS	CHEST/ABDOMEN	ABDOMEN INTO LOWER STEERING WHEEL RIM. CHEST INTO WHEEL HUB	ABDOMEN INTO LOWER STEERING WHEEL RIM
VISIBLE BOD'	НЕАО	FOREHEAD WITH UP- PER STEERING WHEL RM, SLIGHT HEAD CONTACT WITH TOP OF UPPER DASH STEERING WHELU COLUMN COLLAPSE.	LOWER FDREHEAD WITH UPPER STEERING WHEEL RIM
	RIGHT	640	086
MAX. FEMUR LOAOS	LEFT	1000	1720
ANT	TIME3 (ms)	60/86	72/96
MAX. RESULTANT CHEST	MAGN (9's)	S2	33
	G~ms t2	104	11S
HIC, HEAO INJURY	MAGN. TIMING~ms	es	S.
-= 8	MAGN	60S	367
MAX. RESULTANT HEAO	S) (ms)	ŵ S	63
RESU	MAGN. (9's)	86 <sup>2</sup> (70)	71 <sup>2</sup> (53)
TEST & VEHICLE	VELOCITY	NEW CAR ASSESSMENT (34.8 mph)	PRODUCTION BELT SYSTEM ASSEMBLY (3S.3 mph)

<sup>1</sup>EXCEEDING A CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>3</sup>THESE RESPONSES EXHIBIT TWO COMPARABLE RELATIVE MAXIMUM VALUES. THE TIMES LISTED HEREIN REFLECT THE TIMING OF THE INITIAL PEAK ACCELERATION AND THE NEXT HIGHEST VALUE <sup>2</sup>SHORT-DURATION SPIKE (LESS THAN 2 MILLISECONDS WIDE) CAUSED BY IMPACT WITH STEERING WHEEL RIM. APPROXIMATE THREE MILLISECONO CUTOFF MAGNITUDES SHOWN IN PARENTHESES.

of the tunnel-mounted unibelt buckle assembly was cut by the edge of the back, transverse production seat cushion frame member,\* causing the belt to break at about 75 milliseconds into the impact. As a result, restraint system integrity was lost and the occupant subsequently experienced head contact with the unpadde metal rim of the dashboard top cover.

Compounding this problem, the passenger head Z acceleration-time history (H<sub>Z</sub>) was contaminated by inexplicable noise spikes over the entire length of the data trace,\*\* precluding the calculation of a HIC number from all three head acceleration components.

As discussed in Reference 10, Calspan believes, based on the arguments presented therein, that the passenger occupant complied with the HIC requirements of FMVSS 208. This contention is based on a combination of evidence comprised of both high-speed film and electronic data, physical mechanics principles and sound engineering judgment.

The belt failure was also unfortunate in that the full stroke-limiting capabilities of the modified restraint system could not be demonstrated complete: for the passenger occupant. Fortunately, this feature was fully evident in the driver's excellent kinematics and injury indicator responses.

Inspection of Table <sup>9</sup> shows that the BSA test passenger dummy experienced a maximum resultant chest acceleration of only 38 g's, considerably lower than the marginally acceptable 59 g magnitude sustained by the NCA test occupant. The former value reflects the effects of two events which occurred

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<sup>\*</sup>The protective plastic sheath enclosing the webbing between its anchorage point and the buckle was removed, as per standard practice, to enable the attachment of a Lebow load cell to obtain the inboard-side lap belt load-time history.

<sup>\*\*</sup>Subsequent investigation traced this problem to a defective accelerometer. This sensor did, however, check out OK both before and immediately after impact.

Table 9

# PLYMOUTH RELIANT PASSENGER RESPONSES IN 35 MPH FLAT FRONTAL BARRIER CRASH TESTS

REMARKS				NO BELT SYSTEM FAILURES DETECTED. INERTIA SENSITIVE SEAT BACK FAILEO TO LOCK UP.	BELT SYSTEM INTE GRITY VOEST AT ABOUT 56 ma AFTER INSACT 5 ALLOWING THE OCCU FANT TU NUBER ACO ABNORMAL (BASED ABNORMAL (BASED
BELT SPOOL-OFF AT INITIAL HEAD-DASH CONTACT (in)			4.1	< 1/4	
VISIBLE BODY CONTACTS				TOP OF HEAD WITH RIM OF METAL OASHBOARD TOP COVER, FOLLOWED BY FOREHEAD CDNTACT WITH PLASTIC OASH BOARD FRDNT SURFACE.	UPPER FOREHEAO WITH RIM OF METAL DASH BOARD TOP COVER.
MAX.	LOADS (Ibs)	LEFT RIGHT		008	420
Σü	-	LEFT		720	00 00
X. TANT	EST	TIME	(sm)	97	က ဆ
MAX. RESULTANT	CHEST	MAGN.	(g's)	23	<sup>∞</sup> Μ
	Z	MAGN. TIMING Vms	t2	66	128
HIC,	CRITERIDN	TIMIT .	5	&	۵ 
	Ū	MAGN		1731	1134
	t <sub>R</sub>	(us)		91	5
	ня	Н <sub>R</sub> (9's)		182 <sup>2</sup> (118)	ب م
MING	t z	(sm)		6	I
MAX. HEAD ACCELERATION AND TIMING	чzн	(s ,6) Z H		122 <sup>2</sup> (80)	e v z
	ځ	t t (∃s)		97	123
	۲	(s,6)		34 <sup>2</sup> (25)	თ
	t,	(ms)		8	120
	х <sup>н</sup>	(5.6)		133 <sup>2</sup> (100)	õ
, TECT 8	VEHICLE	VELOCITY		NEW CAR ASSESSMENT (34.8 mph)	PRODUCTION BELT SYSTEM ASSEMBLY (35.3 mph)

EXCEEDING A CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>2</sup>SHORT DURATION SPIKE (LESS THAN 2 MIL LISECONDS WIDE) CAUSED BY HEAO IMPACT WITH OASHBOARD. APPRDXIMATE THREE MILLISECOND CUTOFF VALUES SHOWN IN PARENTHESES. <sup>3</sup>ENTIRE DATA TRACE CONTAMINATED BY NOISE FROM UNKNOWN SOURCE; NOT POSSIBLE TO EXTRACT TRUE H<sub>2</sub> RESPONSE FROM THIS SIGNAL.

<sup>4</sup>REFLECTS X AND Y ACCELERATIDN COMPONENTS ONLY (SEE NOTE 3)<sub>4</sub>

<sup>5</sup>STANDARO WEBBING OF BUCKLE ASSEMBLY ABOVE INBDARD LAP BELT ANCHOR POINT WAS CUT BY THE EDGE OF THE PRODUCTION DESIGN TRANSVERSE SEAT CUSHIDN FRAME MEMBER.

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during dummy crash ride down: (1) loss of belt restraint system integrity and (2) 1-3/8 inches of forward seat translation (post-test measurement) relative to the seat track. The extent to which these factors contributed to the BSA test  $C_p$  cannot be ascertained.

Table 9 also shows that the BSA test maximum femur loads were essentially the same acceptable order of magnitude as those developed in the NCA test. However, the former values were also influenced to some unknown degree by the aforementioned belt failure and seat translation mechanisms. Had these two events not occurred, it is probable that the BSA test loads would have been even smaller.

Analysis of high-speed films showed that the BSA test passenger demonstrated excellent kinematics prior to the buckle-part lap belt failure. No tendency to submarine was evident in the films of the event.

Finally, it should be noted that another compartment-modified Reliant was recently subjected to a nearly identical, 34.8 mph flat barrier crash test by an independent contractor.\* The Reliant's modified compartment system successfully prevented passenger head-dash contact and produced a HIC of only 296. (The  $C_R$  was 37 g's while left and right femur loads were 780 lbs. and 1290 lbs., respectively.) These results dramatically support our previously mentioned belief that the BSA Reliant passenger would have complied easily with the HIC requirements of FMVSS 208 had the buckle webbing remained intact.

As a matter of interest, corresponding driver occupant parameters in the above test were as follows: HIC = 391,  $C_R = 34$  g's,  $F_L = 1130$  lbs.,  $F_R = 840$  lbs.

<sup>\*</sup>Test SRL 53 performed on 7 July 1982 by the Vehicle Research and Test Center (TRC) in East Liberty, Ohio. See Reference 11.

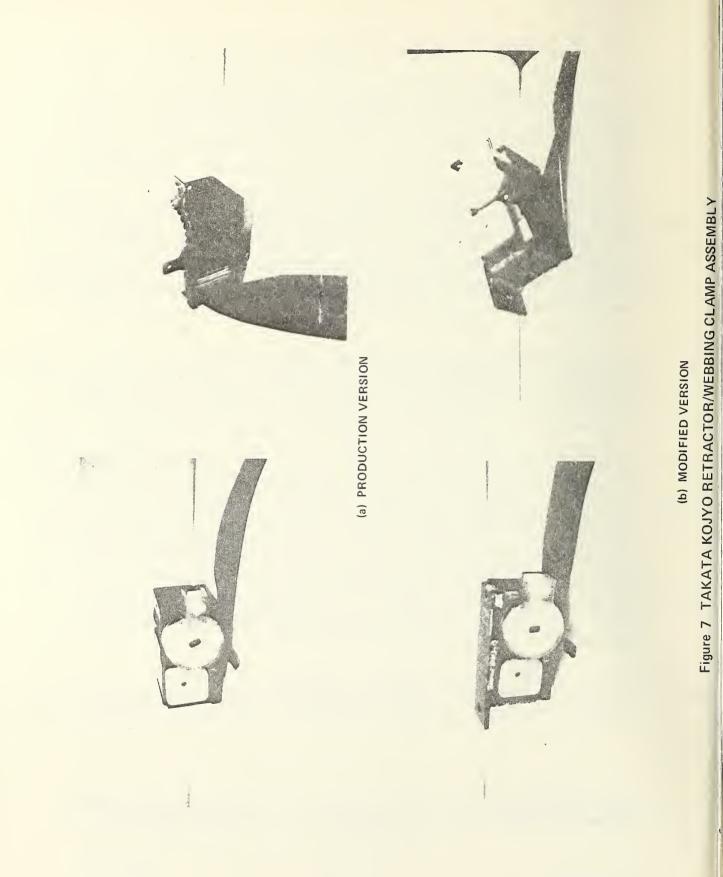
### 4.2 Plymouth Horizon - Ford Mustang Car-to-Car Head-On Crash Test

A production 1982 Plymouth Horizon TC-3 2-door hatchback was subjected to a 70.6 mph closing speed, colinear, car-to-car frontal crash test with a 1982 Ford Mustang 2-door sedan. The Horizon was equipped with Calspan-developed, modified front bucket seat cushions and breakaway-type steering shaft assembly and production Takata Kojyo belt restraint systems. The Mustang was tested in its baseline (i.e., standard-equipment) condition. A complete description of these systems, as well as all test results, is provided in Reference 12. For the convenience of the reader, however, the above modifications are described briefly in the ensuing six paragraphs. Also, electronic data recorded in this test are presented in Appendice's D through G of this report.

Both Horizon production front seat belt restraint systems were replaced by a Takata Kojyo combination retractor/webbing clamp assembly, including 8%elongation Takata Kojyo webbing. This device, shown in Figure 7a, is a variation of a production version used in the 1982 Honda Accord (4-door model). As such, it features the same, proven retractor lock-up/belt gripping mechanism. Only the inclination (26° relative to the horizontal instead of a vertical orientation) is different. Thus with the Takata Kojyo unit mounted near the Horizon's zeroinclination (horizontal) side roof rail, the retractor was effectively locked up prior to the test. While this pre-locked condition certainly provided an advantage for the Horizon occupants during collision ride down, it is highly probable that essentially similar belt spool-off would have occurred if a hypothetical production Takata Kojyo unit designed for horizontal roof rail mounting were employed in this test.\*

The Takata Kojyo unit was adapted for bolt mounting to the roof rail as shown in Figure 7b. Passenger-side views of both the production and substitute

<sup>\*</sup>Supporting arguments for this contention are presented in Section 2.2 of Reference 9.



retractor units at the common side roof rail mounting location are depicted in Figure 8. Figure 9 displays pre-test photographs of the Takata Kojyo restraint system belt configuration in place around the driver and passenger occupants.

In the production Horizon, the retractor is bolted in place using two studs projecting from a cantilevered sheet metal pan (see Figure 10a) which can be pulled downward (elastically) under the action of a small, hand-applied force. In an effort to eliminate the possibility of undesired increased occupant stroke (via extensive permanent pan distortion) during occupant loading, the free edges of the cantilevered pan were pop-riveted to the exterior roof sheet metal as depicted in Figure 10b.

Both production Horizon front bucket seat cushions were stiffened along the front edge in a manner somewhat analogous to the Reliant cushion modification. As shown in Figure 11, mild steel angle sections, polystyrene insulation board and aluminum sheet stock were utilized in the construction of this assembly. As was the case with the stiffened Reliant seat cushion, the added, higher-density foam at the front of the cushion traps the dummy's buttocks in the softer portion of the seat during collision ride down, limiting pelvic excursion.

In an effort to prevent the seats from translating forward relative to their respective seat tracks under occupant loading during the test,\* the upper and lower portions of all four production seat track assemblies were tack-welded together at four locations.

As mentioned in Section 3.3, successful driver occupant performance in an actual 35 mph frontal collision environment required, in addition to the restraint system and seat modifications described above, that steering wheel intrusion be kept to a minimum. In an attempt to accomplish this objective, the solid, production lower steering shaft assembly was modified to form a breakaway-type joint between the upper steering shaft and the short shaft projecting from the steering rack. Photographs of the production and modified

<sup>\*</sup>As noted in Reference 5, the production Horizon bucket seats have a tendency to move forward relative to the seat track assemblies.

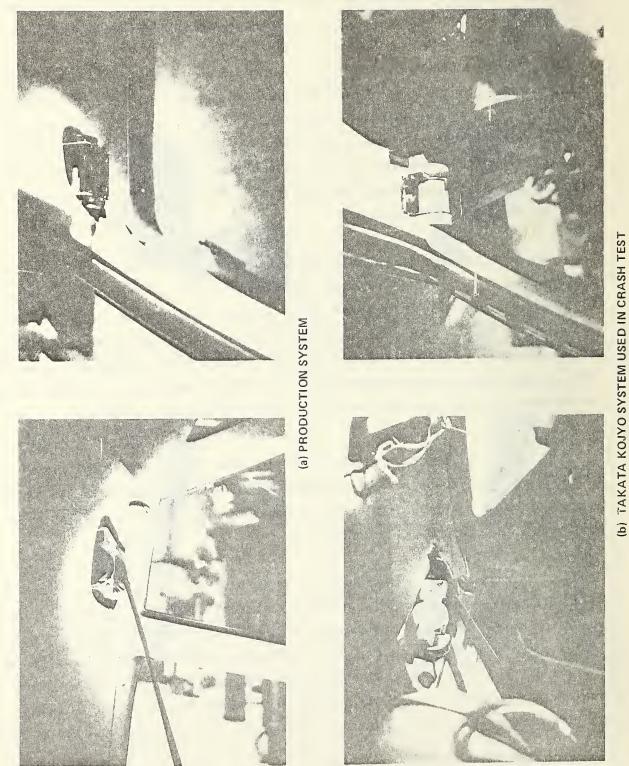
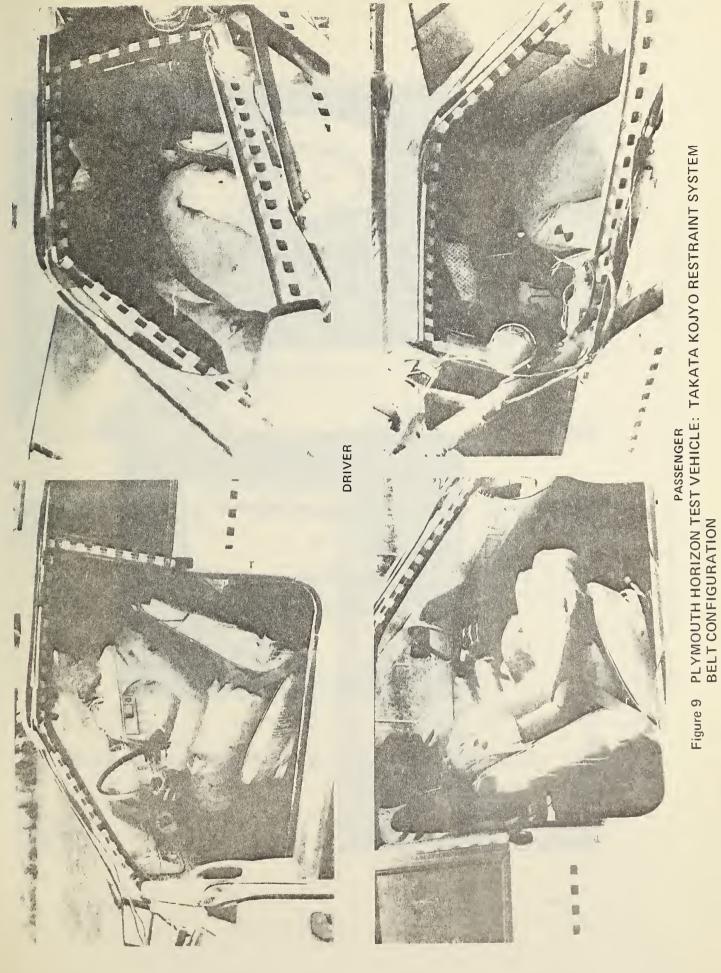
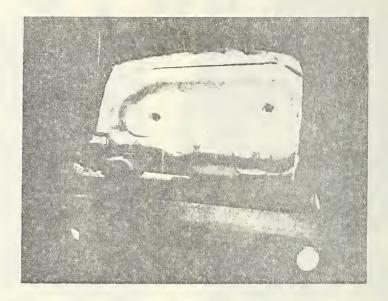
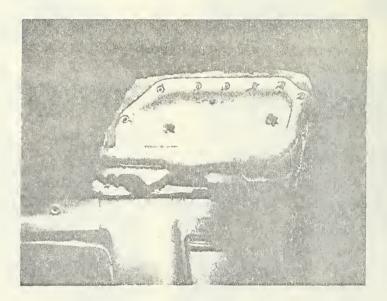


Figure 8 PLYMOUTH HORIZON RETRACTOR AT ROOF RAIL MOUNTING LOCATION





(a) PRODUCTION SYSTEM



(b) REINFORCED SYSTEM

Figure 10 PLYMOUTH HORIZON RETRACTOR MOUNT STRUCTURE

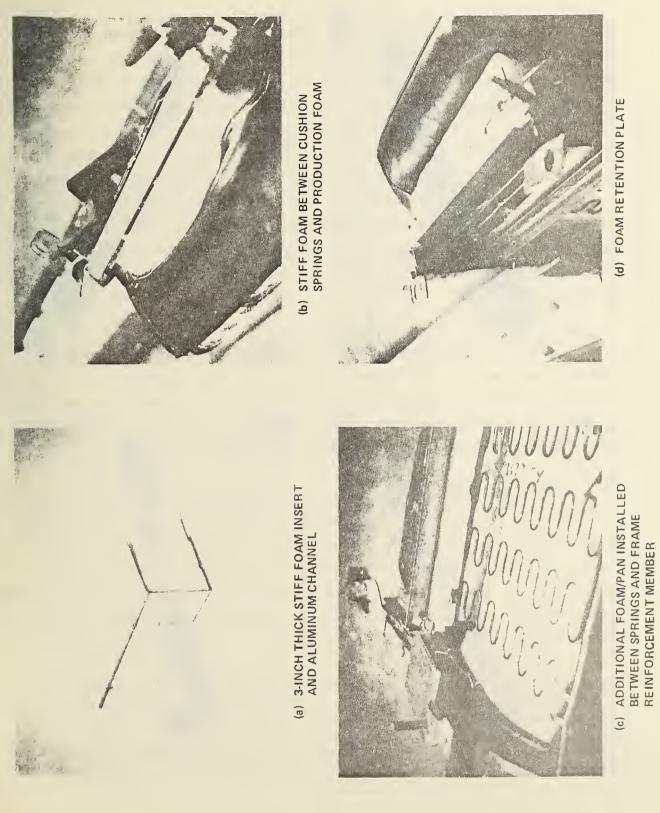


Figure 11 MODIFIED PLYMOUTH HORIZON SEAT CUSHION ASSEMBLY

shafts are presented in Figures 12a and 12b, respectively, while Figures 13a and 13b show these two assemblies in place in the vehicle.

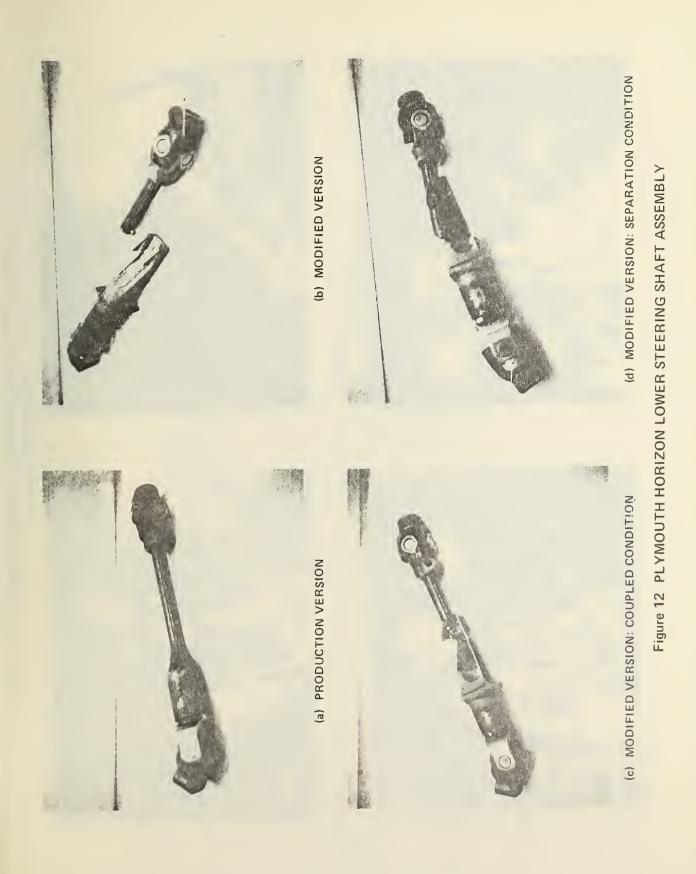
Examination of Table 10 shows that with the possible exception of a low-severity head contact with the upper steering wheel rim,\* the driver in the BSA test exhibited excellent responses in all aspects of occupant performance. Occupant injury indicators were all well below the allowable limits stipulated by FMVSS 208. Moreover, the HIC number and maximum resultant accelerations  $C_R$  and  $H_R$  were all substantially lower than their unsatisfactory counterparts in the NCA collision.\*\* On a percentage basis, the BSA HIC and  $C_R$  were both 27 percent below their allowable limits while their NCA counterparts were 82 percent and 2 percent above these same respective levels. Left and right BSA femur loads were respectively 72 percent and 18 percent below the maximum allowable loading compared to 24 percent and 38 percent below for the corresponding NCA magnitudes.

Despite significant seat pitching motion arising from extensive vehicle floor pan/sill deformation, relative seat translation was prevented and overall driver occupant kinematics were excellent. Submarining motion and lap belt roping were nonexistent and rebound was straight back into the seat back/headrest. Belt spool-off was confined to less than 1/2 inch.

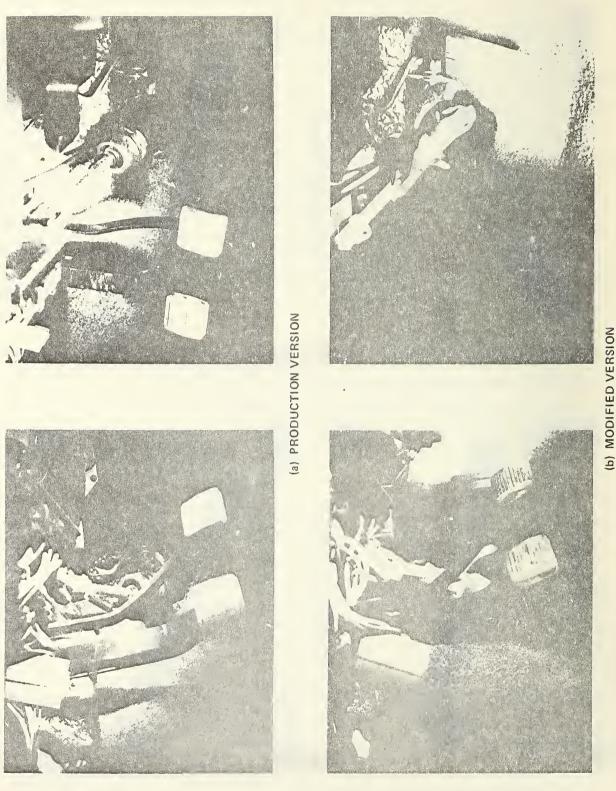
The BSA test passenger displayed outstanding performance relative to both FMVSS 208 occupant injury criteria and kinematic response. With respect to occupant injury indicators, inspection of Table 11 reveals that the HIC number and resultant chest acceleration were both well below the FMVSS 208 tolerance levels and substantially lower than the corresponding values generated in the NCA test. Indeed, the BSA HIC and  $C_R$  were 49 percent and 37 percent lower, respectively, than their allowable limits. Conversely, the same NCA

<sup>\*</sup>As borne out by the head acceleration and HIC number values, driver head contact with the steering wheel was more severe in the NCA test than in its BSA counterpart.

<sup>\*\*</sup>The driver's 1817 HIC and 61 g C<sub>R</sub> exceeded FMVSS 208 tolerance levels in the NCA test.



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## Table 10 PLYMOUTH HORIZON DRIVER RESPONSES IN 70 MPH CLOSING SPEED HEAD-ON FRONTAL IMPACTS WITH A FORD MUSTANG

		REMARKS	SEAT MOVED FDRWARD RELATIVE TO SEAT TRACK, FLDUN FAN RDTATION (PRIMARILY) CAUSED MODERATE SEAT FDRWARD PITCH OCCUPANT SUBMARINED	EXTENSIVE FLOOR PAN DEFORMATION/ROTATION CAUSED THE ENTIAE SEAT TO UNDERGO CDNSIDERABLE PITCHING MOTION
	VISIBLE BODY CONTACTS	CHEST/ABDOMEN	ABDOMEN WITH LOWER STEERING WHEEL RIM.	ABDOMEN WITH LOWER STEERING WHEEL RIM.
	HEAD	EYES/BRIDGE OF NOSE WITH UPPER STEERING RIM, FOLLOWED BY FACIAL CONTACT WITH WHEEL HUB.	BRIDGE OF NOSE WITH UPPER STEERING WHEEL RIM.	
MAX. COMPRES- SIVE FEMUR	(Ibs.)	RIGHT	1400	1850
- <u>0</u> ,	=	LEFT	1700	630
ATION	TIME	(ms.)	76	77
CR <sup>1</sup> , MAX. RESULTANT CHEST ACCELERATION	MAGN	(s.6)	61	44
Z	riming~ms.	t <sub>2</sub>	85	97
HIC, HEAD INJURY CRITERION	INIT	41	69	83
- <del>2</del>	MAGN.	(\$,6)	1817	727
H <sub>R</sub> , MAX. RESULTANT HEAD CCELERATION	TIME	(ms.)	70	83
H <sub>R</sub> , MAX. RESULTANT HEAD ACCELERATION	MAGN.	(s,6)	184 <sup>2</sup> (129)	153 <sup>2</sup> (89)
TEST &	ــــــ ۲	AT IMPACT	NEW CAR ASSESSMENT (70.2 mph)	PRODUCTION 153 <sup>2</sup> BELT SYSTEM (89) ASSEMBLY (70.6 mph)

<sup>1</sup>EXCEEDING A CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>2</sup>SHORT DURATION SPIKE CAUSED BY HEAD CONTACT WITH THE STEERING WHEEL. THREE-MILLISECOND CUTOFF VALUE SHOWN IN PARENTHESIS.

			REMARKS	SEAT MOVEO FORWARO RELATIVE TO SEAT TRACK. FLOOR PAN ROTATION (PRIMARILY) CAUSED MODFRATE SEAT FORWARD PUTCH	OCCUPANT SUBMARINED. EXTENSIVE FLOOR PAN DEFORMATION/ROTATION CAUSED THE ENTIRE SEAT TO UNDERGO	CONSIDERABLE PITCHING MOTION
		CONTACTS	CHEST/ABDOMEN	NONE	NONE	
		VISIBLE BODY CONTACTS	НЕАО	TOP OF HEAO WITH UPPER FRONT SURFACE OF OASHBOARD	NONE	
	MAX. COMPRES. SIVE FEMUR LOAOS	s.)	RIGHT	710	740	
	MAX. COMPRE: SIVE FEMUR LOAOS	<b>q</b> €)	LEFT	1330	NA <sup>3</sup>	
	X. TANT	CELERATION	(g's) (ms.)	100	85	
	C <sub>R</sub> <sup>1</sup> , MAX. RESULTANT CHEST	ACCELERATION	(s,6)	52	38	
		IMING~ms.	t2	101	136	
	HIC, HEAD INJURY CRITERION	TIMIN	1-1-	96	86	
		ATION ATION	.ND MIN	2096	510	
	H <sub>R</sub> , MAX. RESULTANT HEAD	ACCELERATION	(ms.)	86	6	
	H <sub>R</sub> . MAX. RESULTA HEAD	ACCELEF	(s,6)	288 <sup>2</sup> (120)	52	
	TEST &	AT IMPACT	NEW CAR ASSESSMENT (120) (70.2 mph)	PRODUCTION BELT SYSTEM ASSEMBLY	(10.6 mph)	
1		-				

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<sup>1</sup>EXEEOING A CUMULATIVE DURATION OF 3 MILLISECONDS.

<sup>2</sup>SHORT-OURATION SPIKE CAUSED BY HEAD CONTACT WITH THE OASHBOARO. THREE-MILLISECONO CUTOFF VALUE SHOWN IN PARENTHESIS,

<sup>3</sup>NOT AVAILABLE: WIRES PULLEO OUT OF LOAO CELL.

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## Table 11 PLYMOUTH HORIZON PASSENGER RESPONSES IN 70 MPH CLOSING SPEED HEAD-ON FRONTAL IMPACTS WITH A FORD MUSTANG

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parameters were 110 percent above and 15 percent below these limiting values. Peak right femur loadings were comparable in both tests, considerably lower (67 percent and 68 percent less for the BSA and NCA test exposures, respectively) than the maximum allowable loading. A data loss precluded a definitive assessment of corresponding maximum left femur load magnitudes. However, it can be reasonably assumed that this value would not have been significantly different from the acceptable magnitude (1330 lbs.) registered during the NCA collision.

The stroke-limiting features of the BSA vehicle restraint system and seat modifications prevented the life-threatening NCA test head-dashboard contact with margin to spare. Again, occupant submarining and lap belt roping/ abdominal penetration was prevented and rebound was straight back into the seat back/headrest.

As was the case for the driver occupant, the passenger-side restraint system permitted minimal spool-off (<1/2 inch), significantly less than the approximate maximum 5 inches of belt playout noted in the NCA test. Also (as expected), no seat translation relative to the seat tracks occurred during the test.

### 5.0 CONCLUDING REMARKS AND RECOMMENDATIONS

The research program "Improved Performance of Production Belt System Assembly for the Plymouth Horizon and Plymouth Reliant" (i.e., the BSA program), examined the occupant protection potential provided by two current-production, small domestic automobiles in nominal 35 mph frontal impact exposures. It was demonstrated conclusively that this capability can be significantly enhanced by making relatively simple, production-type modifications to the standard-equipment belt restraint system, seats and steering column. As a result of these changes, overall occupant performance as measured by FMVSS 208 injury criteria and dummy kinematic response was upgraded from highly unacceptable in baseline New Car Assessment (NCA) crash tests to very acceptable in corresponding BSA program tests.

Calspan believes that the same basic concepts successfully employed herein could also be effectively utilized in other domestic and foreign automobiles. The NCA frontal impact test series would provide a relatively large sample of candidate vehicles for this purpose. As was the case with the Horizon and Reliant, the vehicle(s) selected for occupant compartment modification would possess good front structure energy management and compartment integrity characteristics but still fail to provide satisfactory frontal impact occupant injury protection.

Because of the knowledge acquired during the BSA effort, the proposed additional effort could be accomplished for perhaps a number of different production vehicles at relatively little expense. Such research could lead to a new generation of safer passenger cars without involving expensive vehicle structure redesign. Also, 40 mph frontal impact occupant protection could be achieved by some of these compartment-modified automobiles.

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### 6.0 REFERENCES

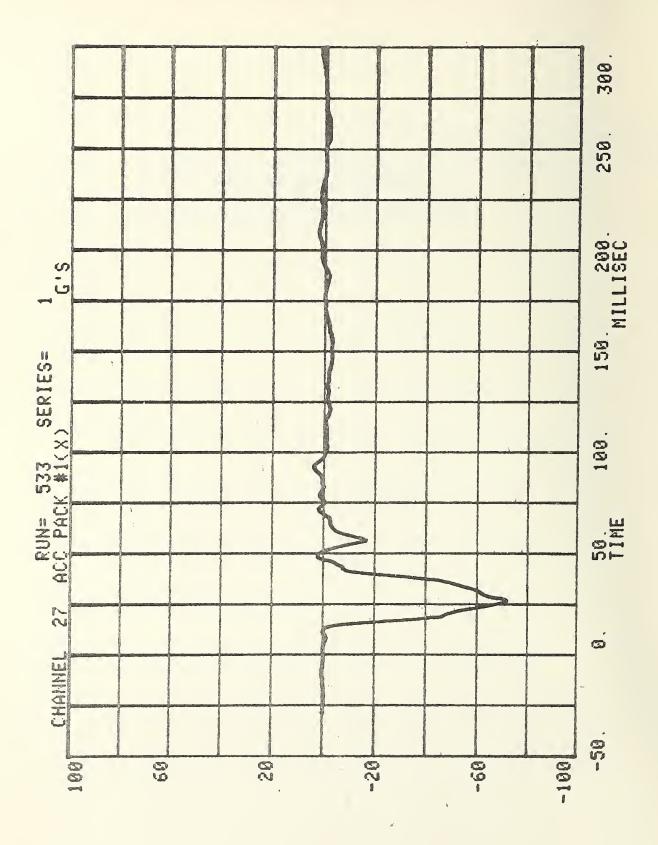
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- 11. Favors, J., "Restraint System Test, Modified Reliant K," Vehicle Research and Test Center Report No. SRL 53, to be published.

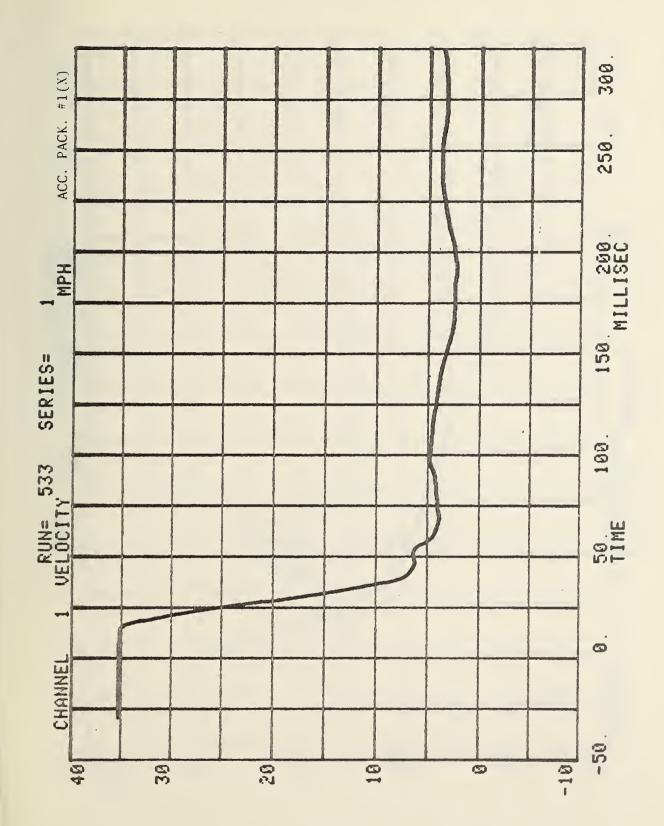
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### APPENDIX A ELECTRONIC CRASH TEST DATA: PLYMOUTH RELIANT VEHICLE-MOUNTED SENSORS

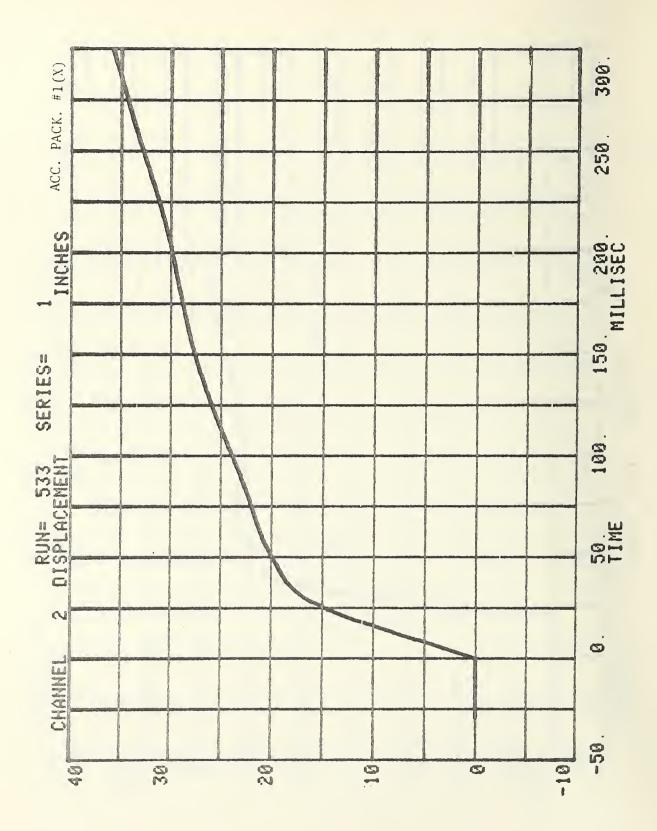
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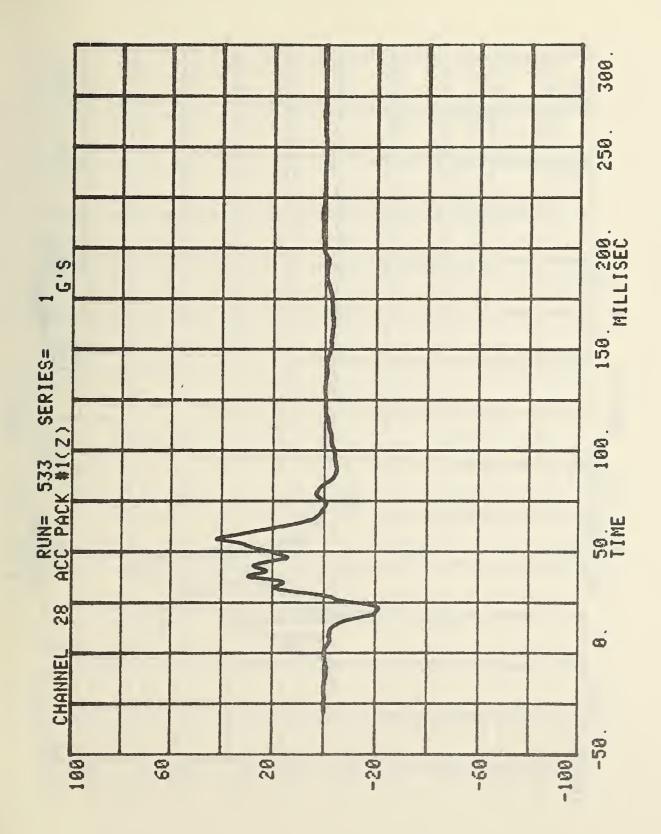


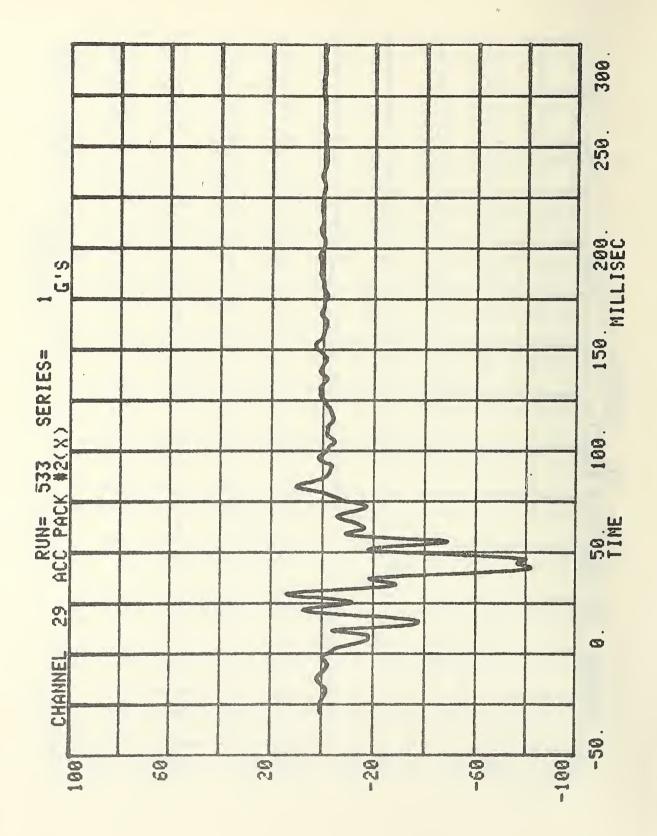


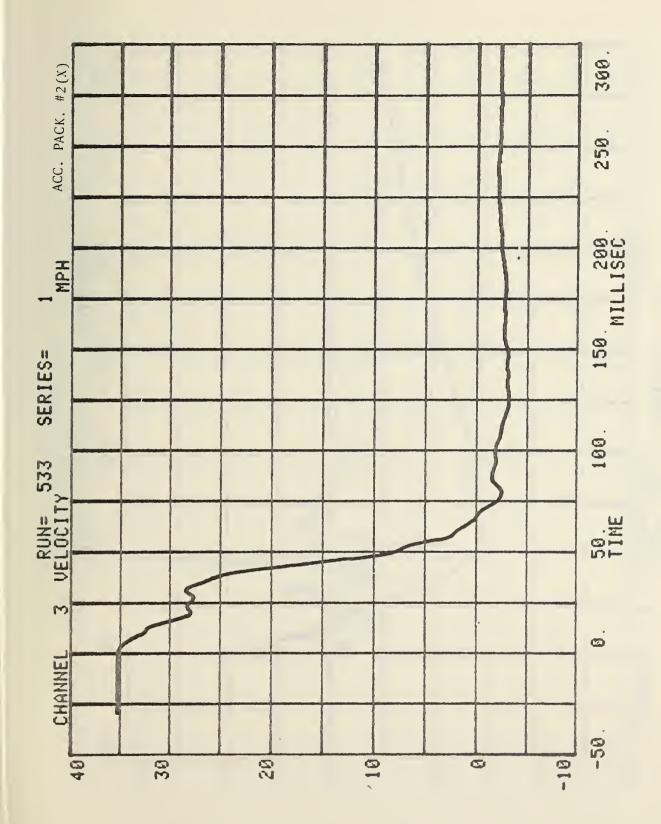
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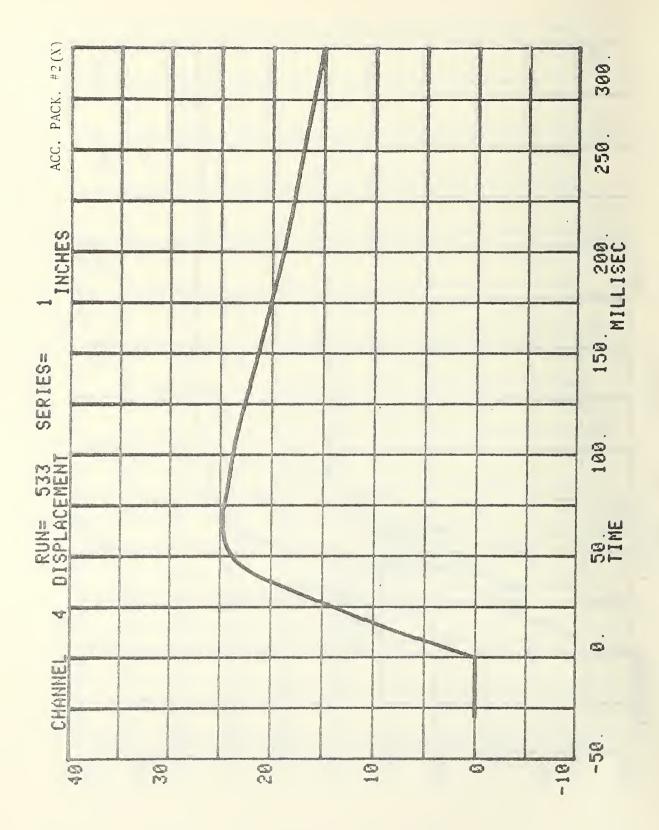
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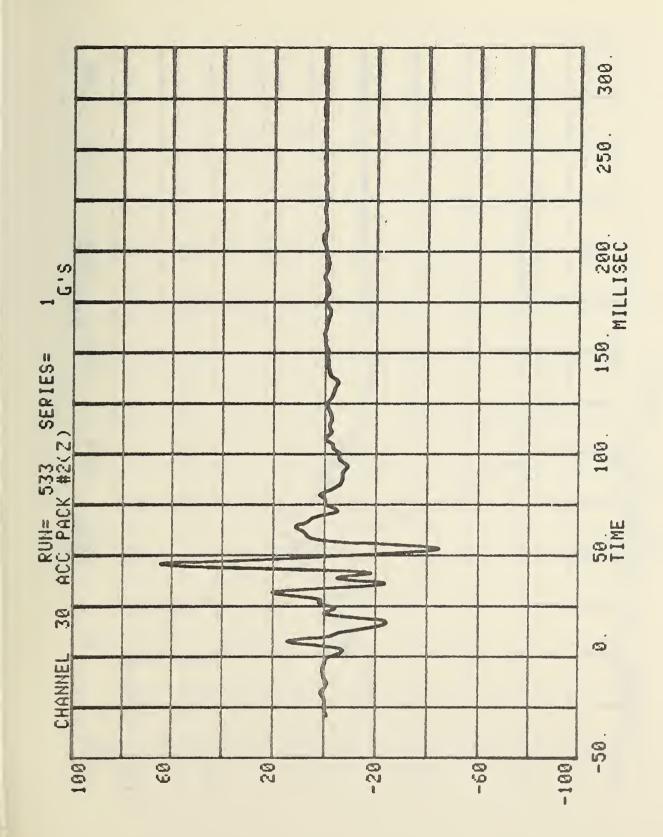


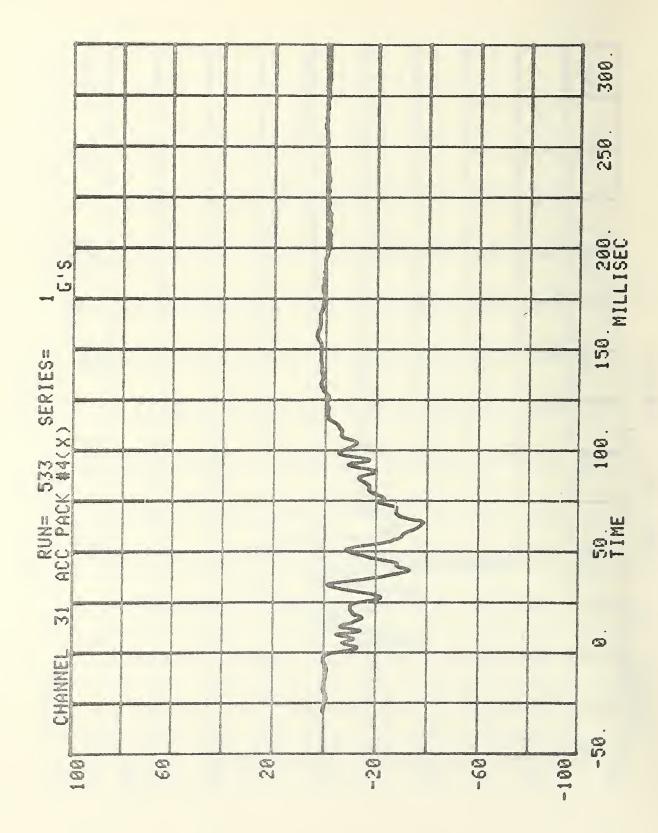


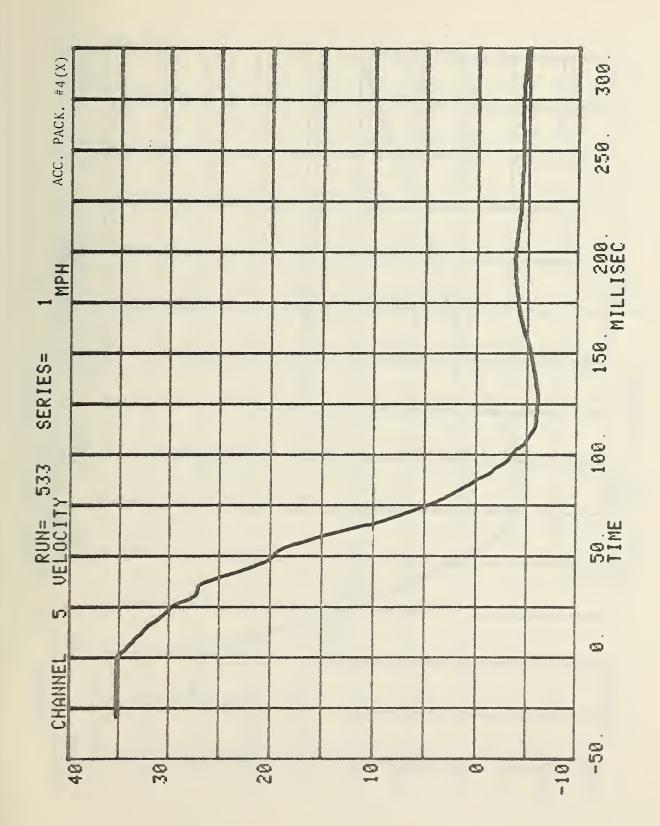


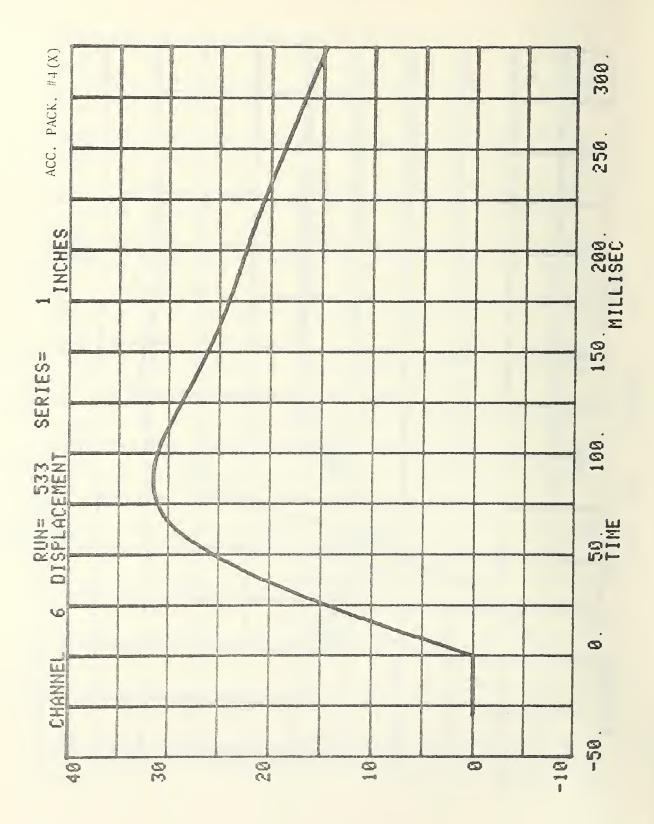


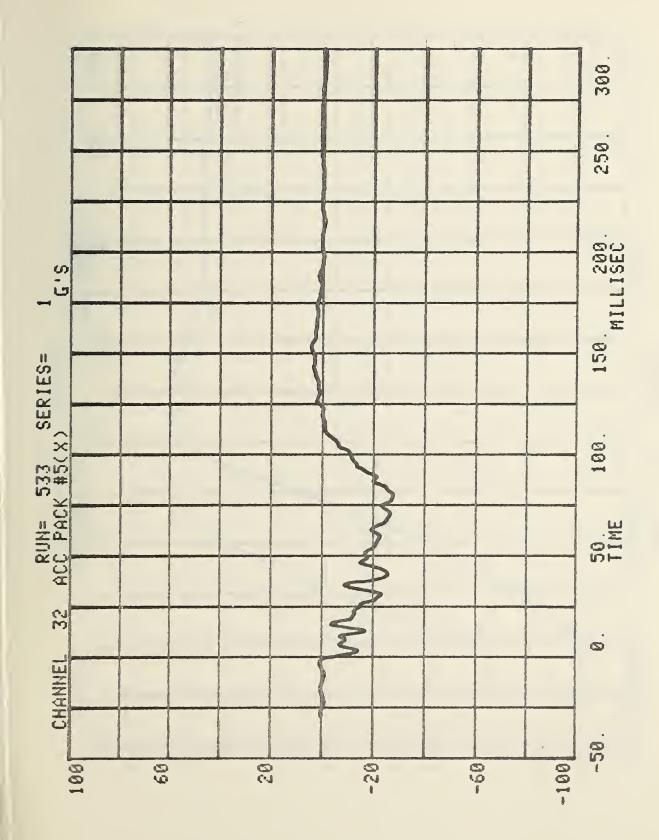


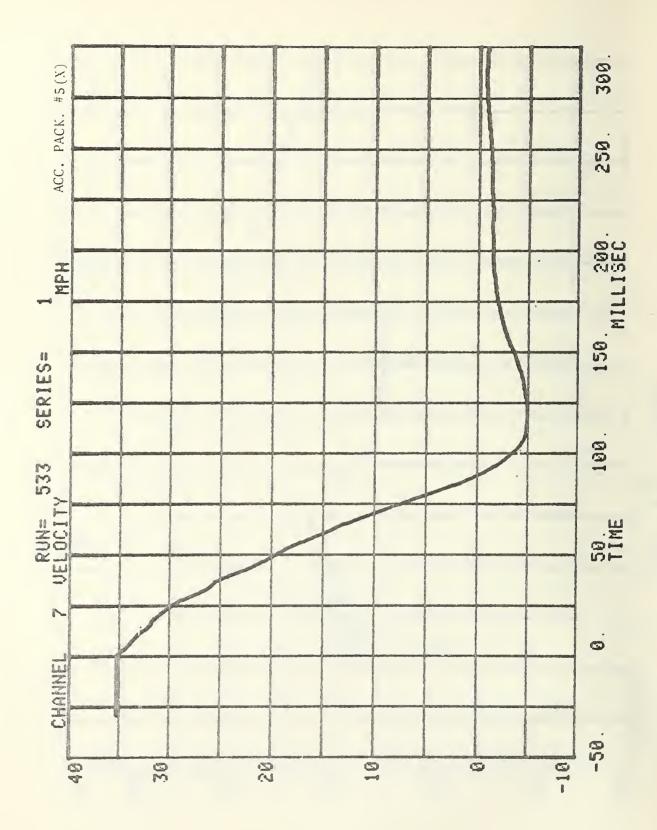


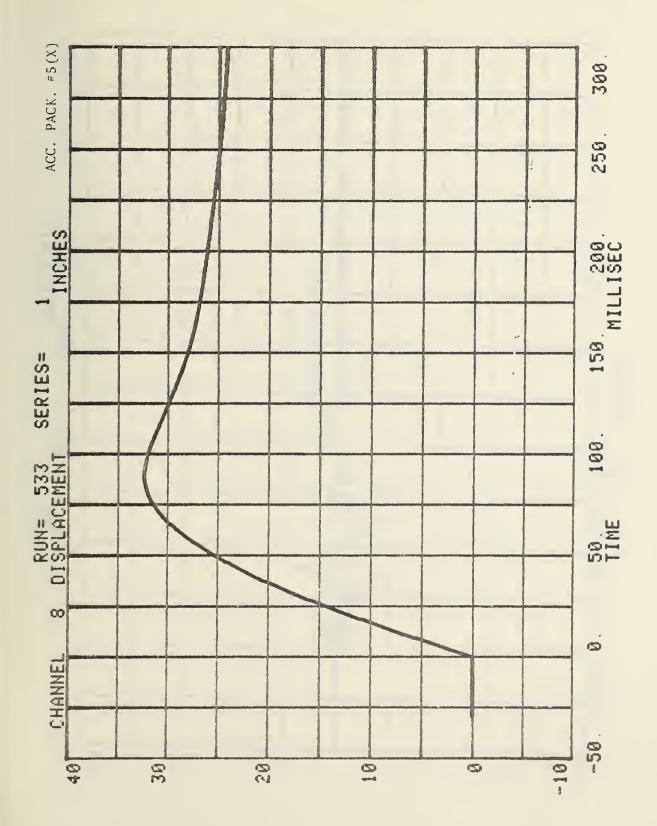


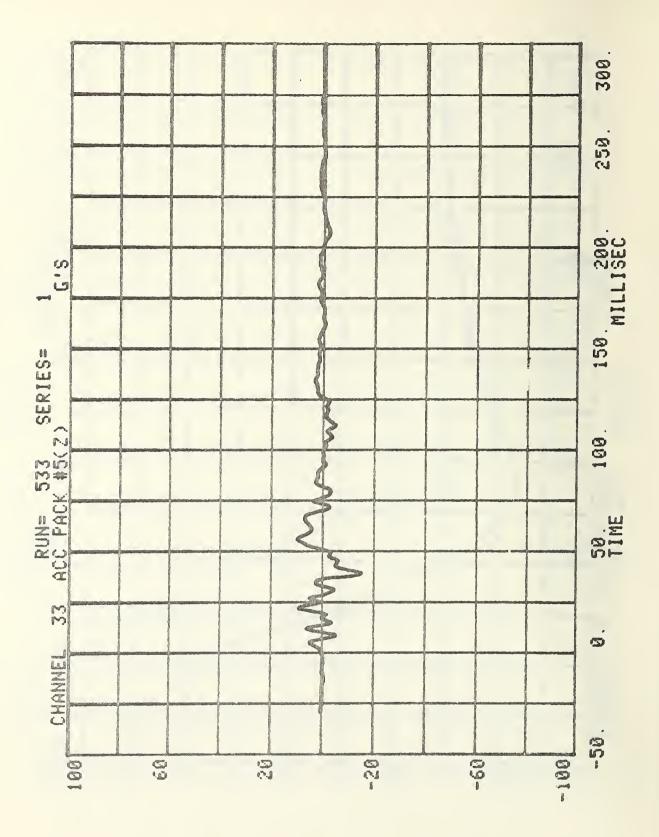


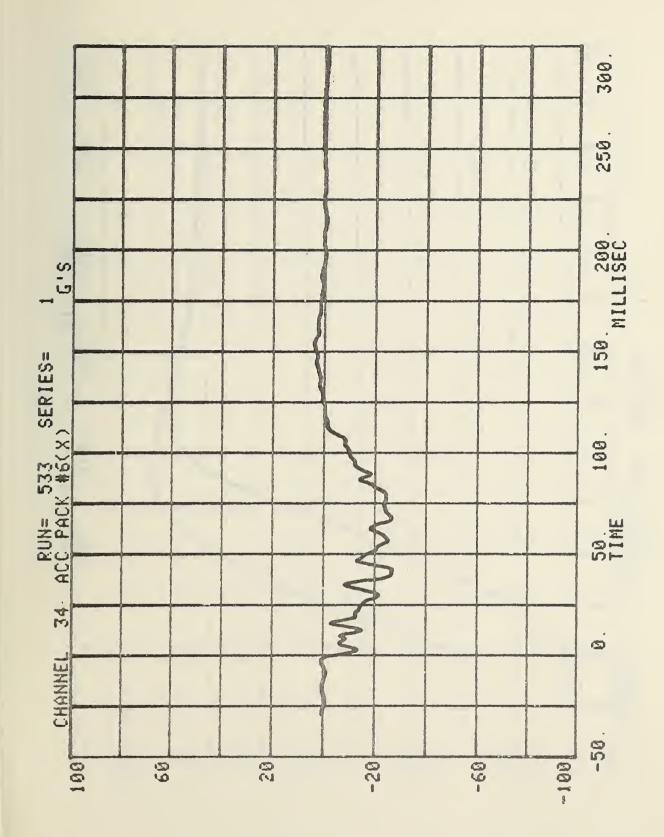


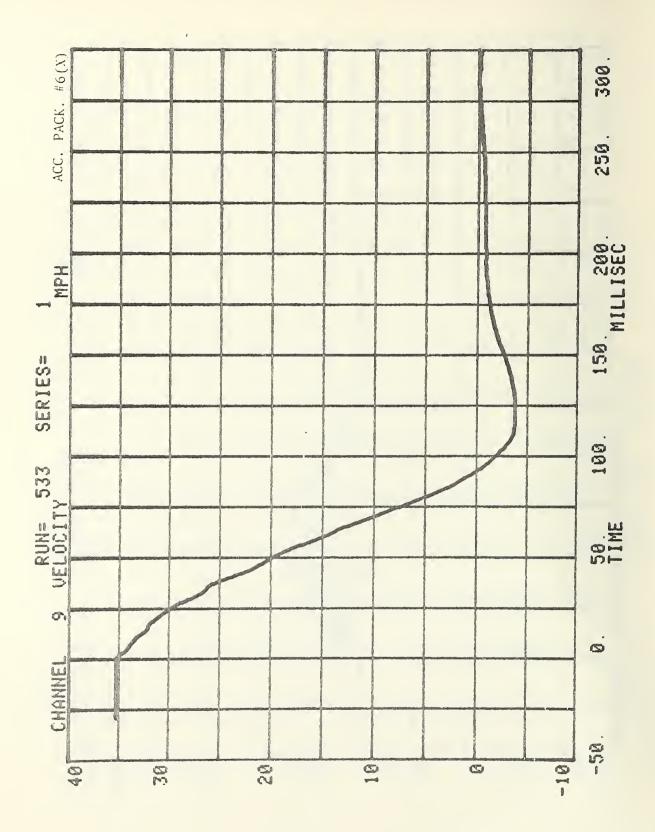




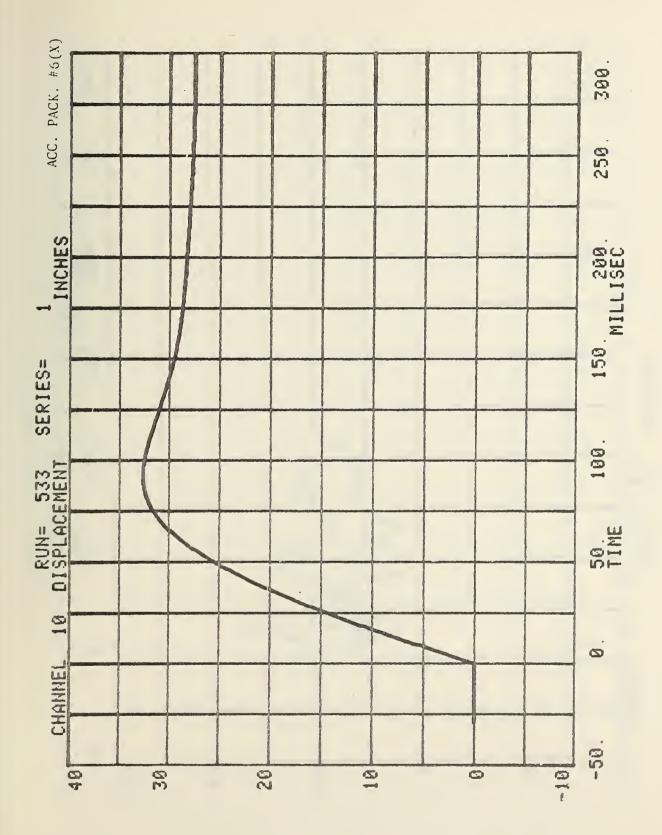


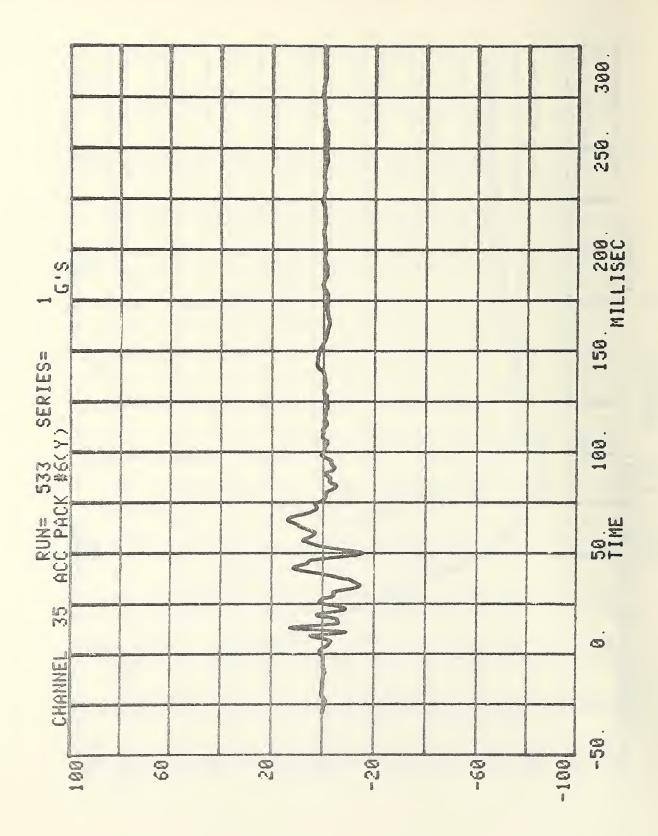


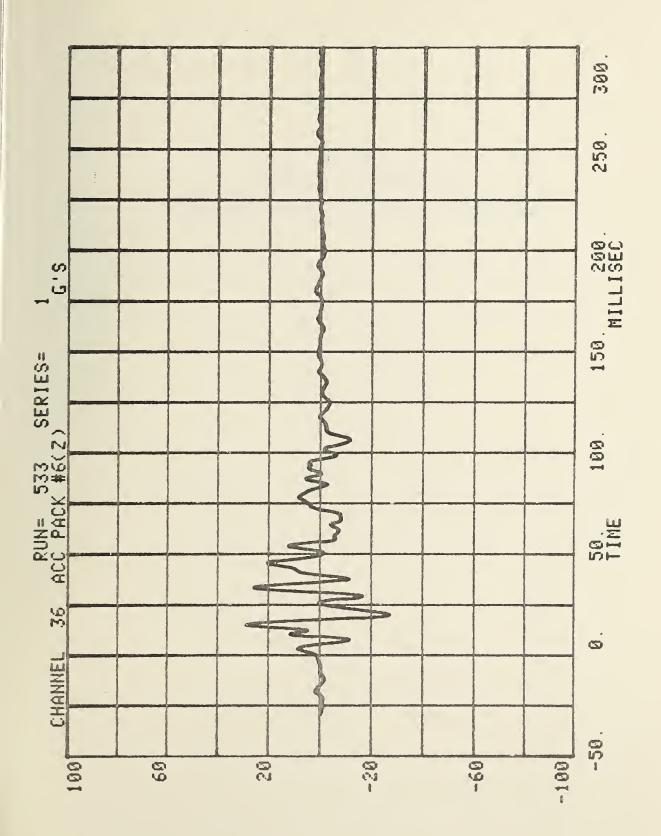




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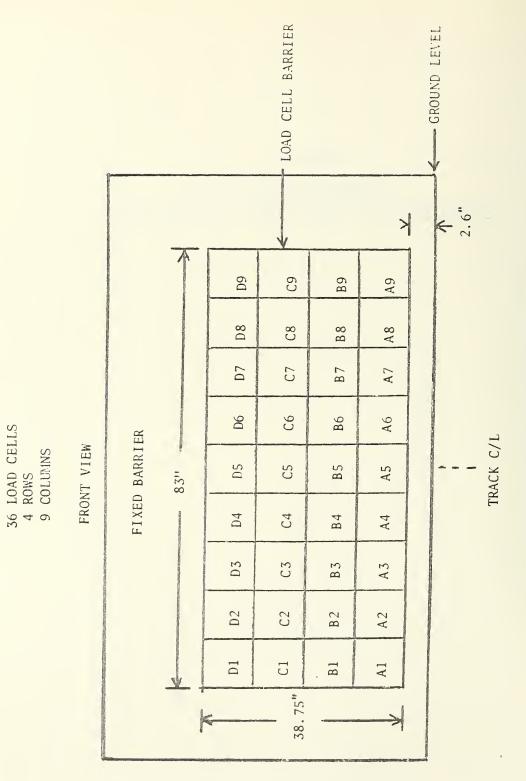


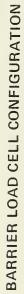


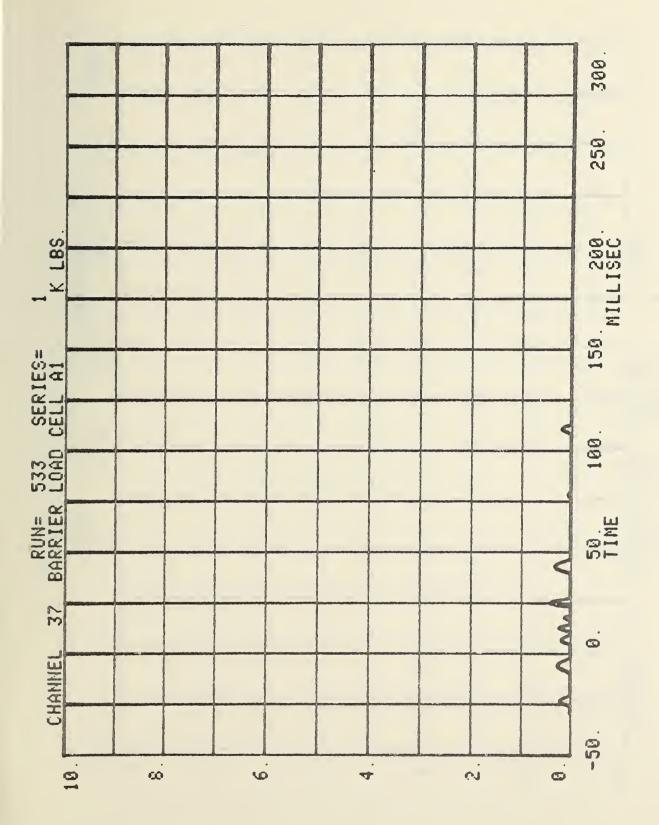


## APPENDIX B

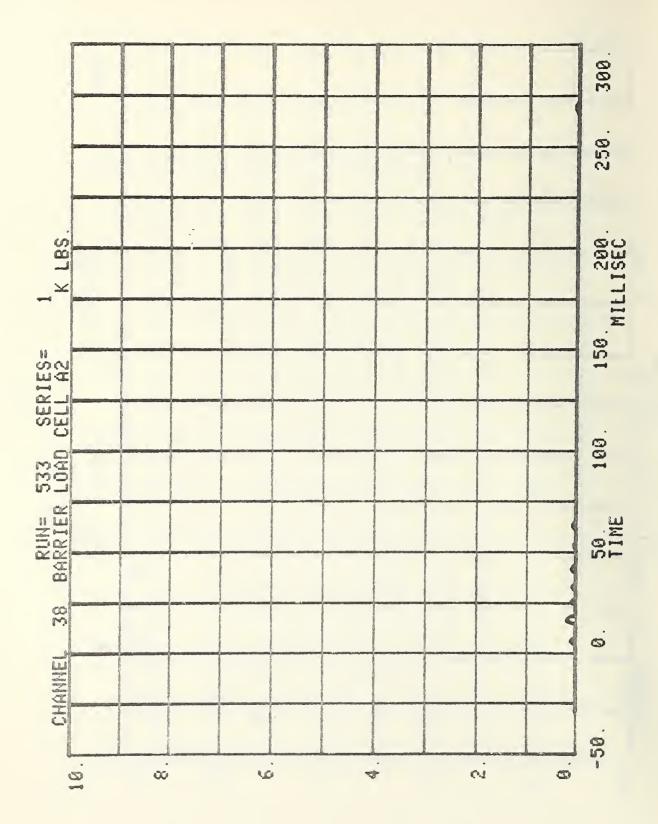
## ELECTRONIC CRASH TEST DATA: BARRIER LOAD CELL FROM PLYMOUTH RELIANT IMPACT

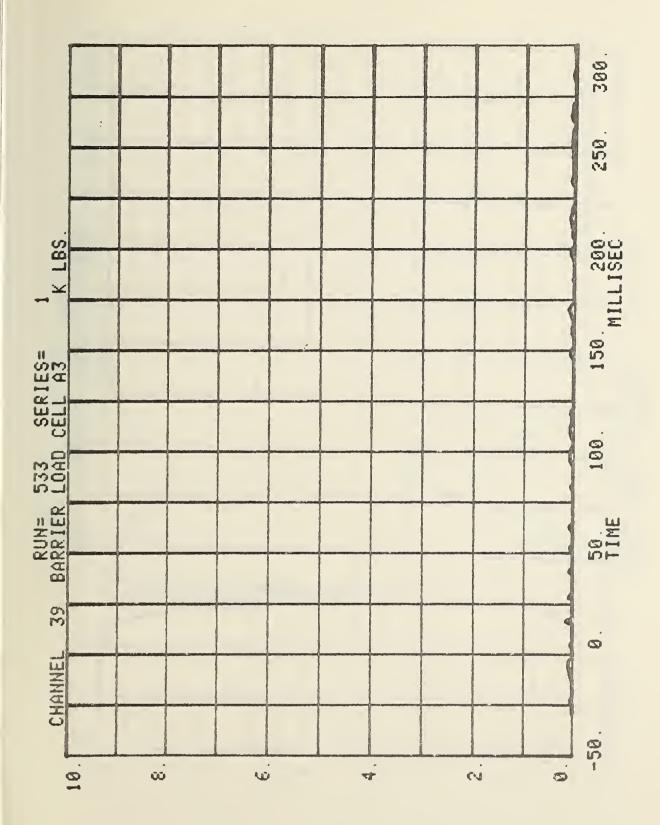


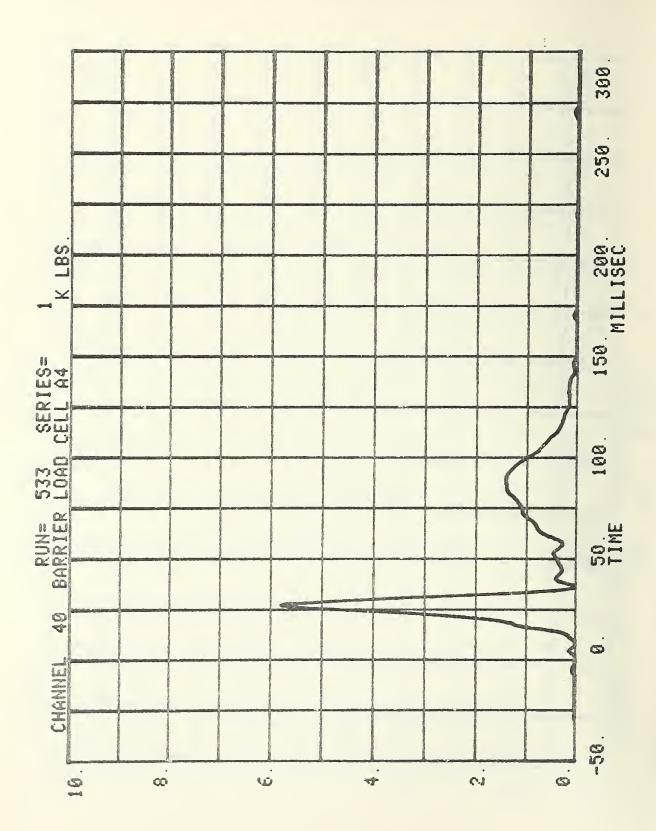




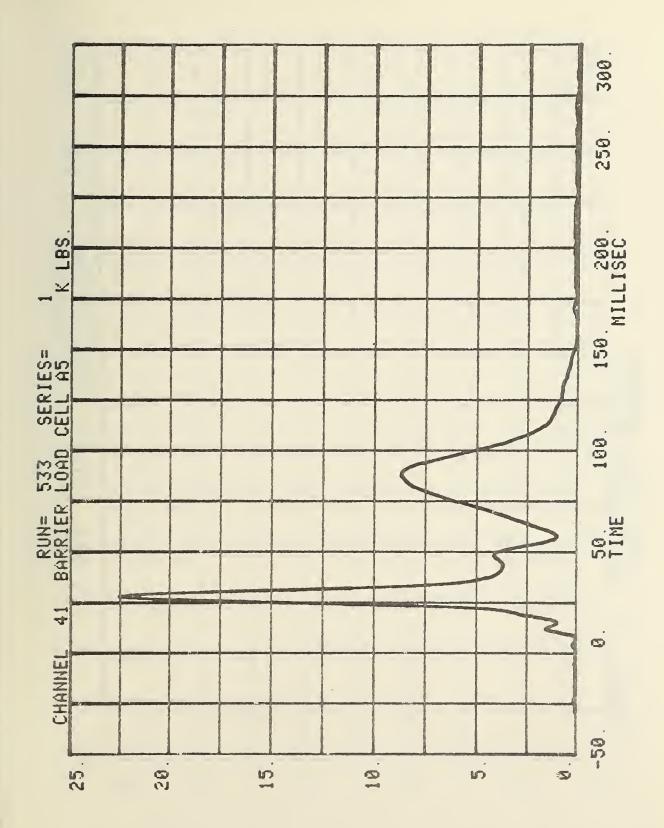
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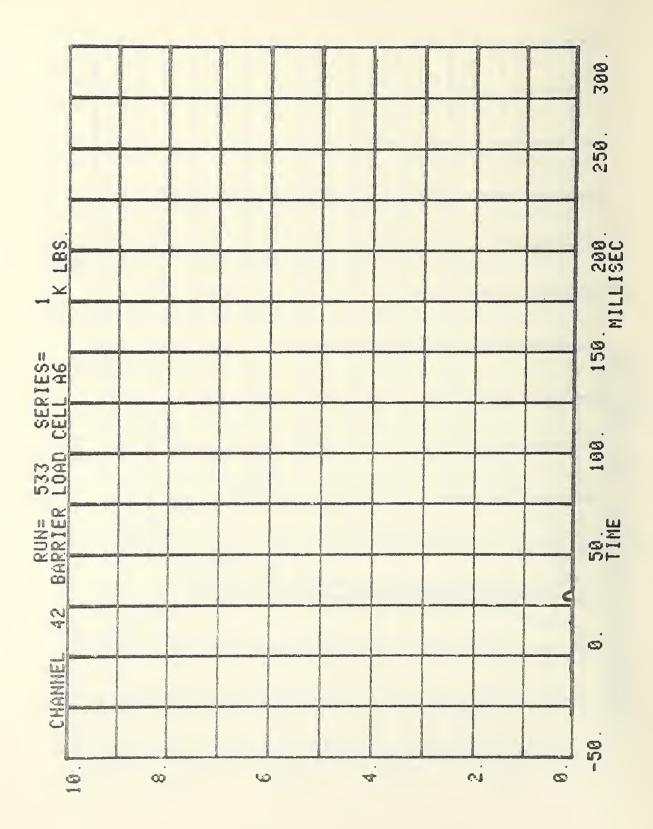


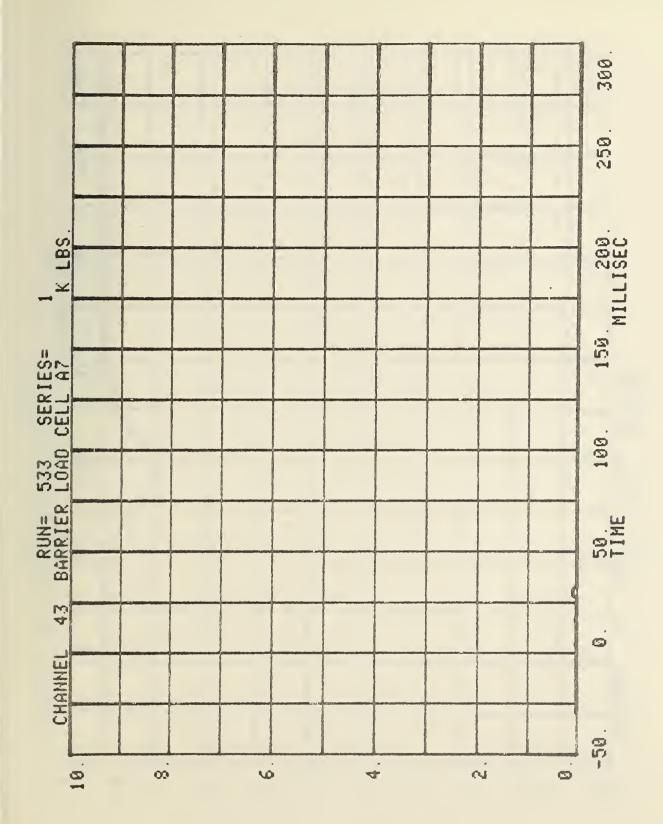


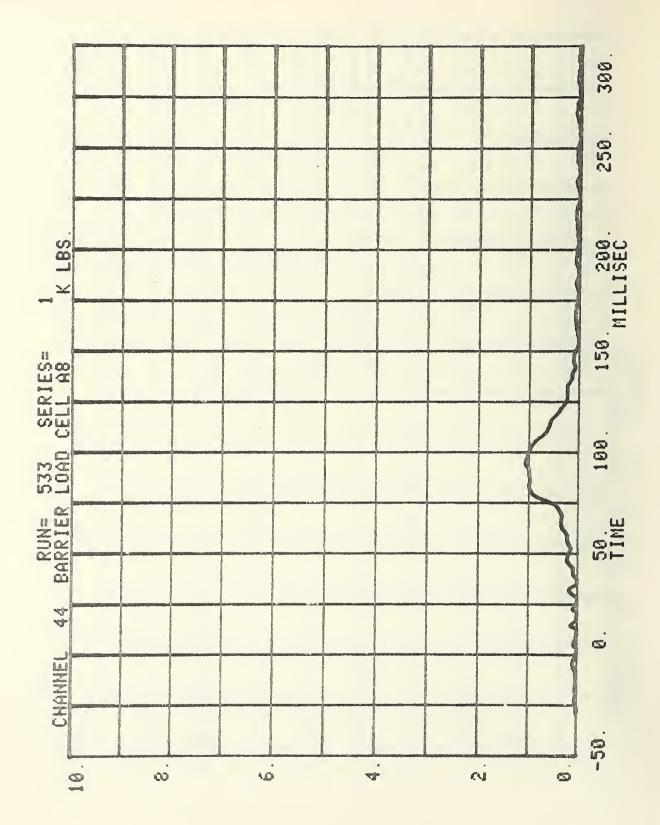


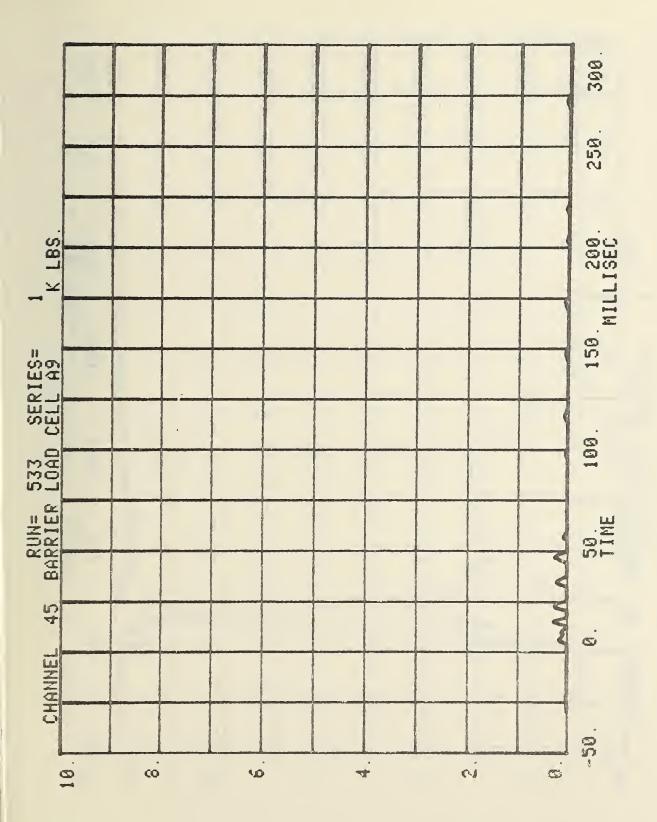
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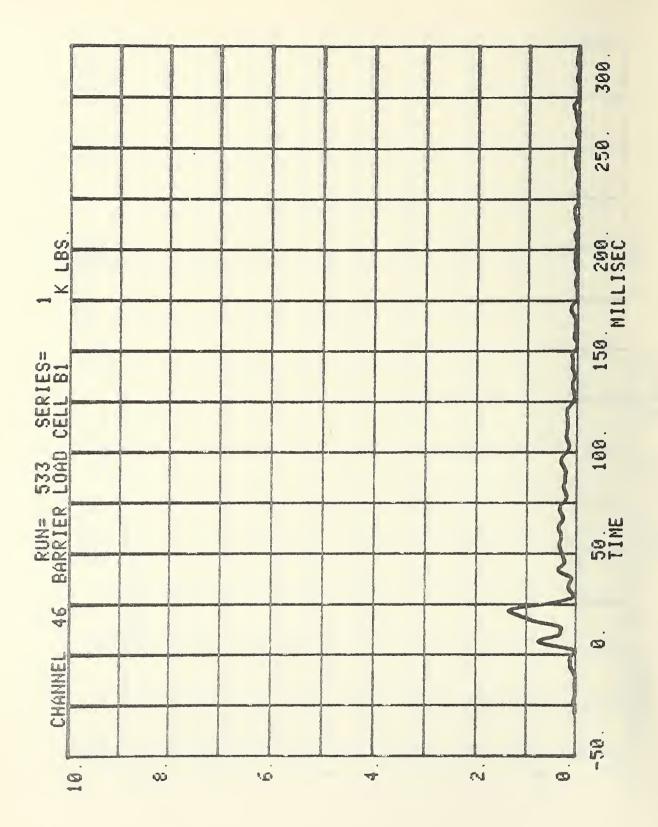


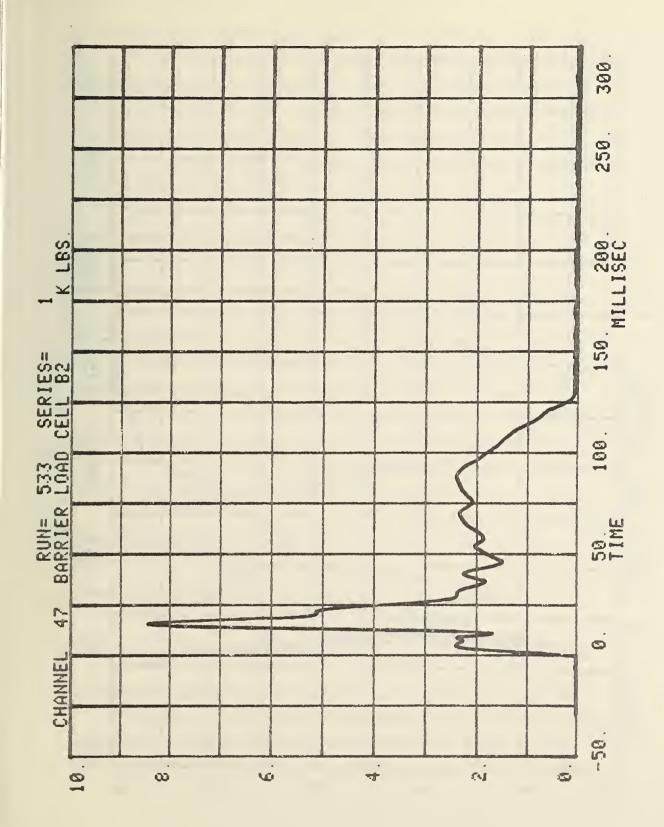


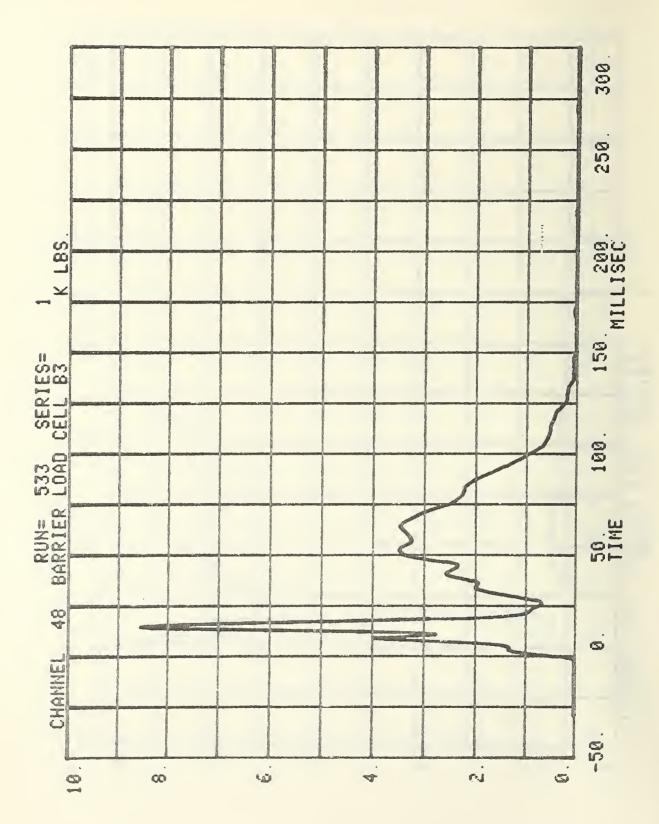


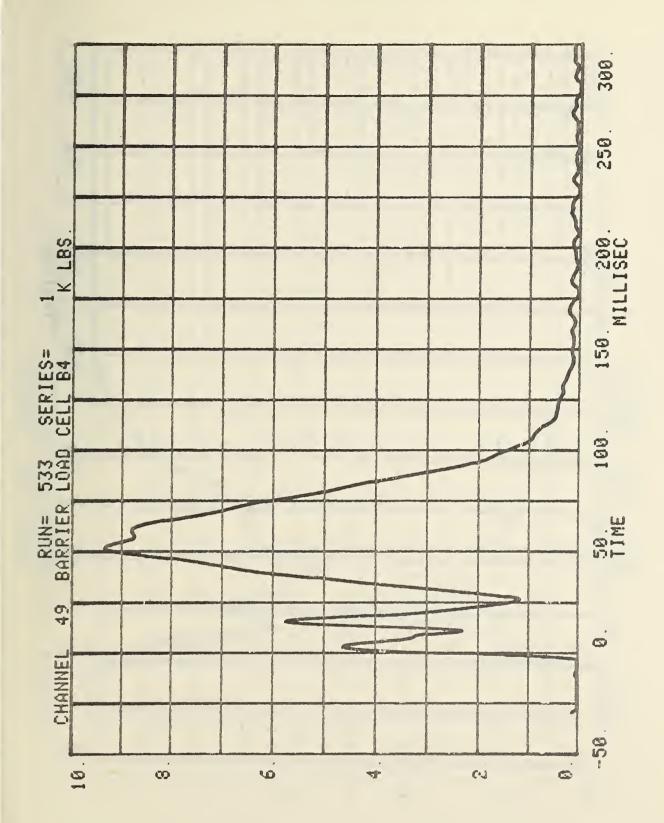


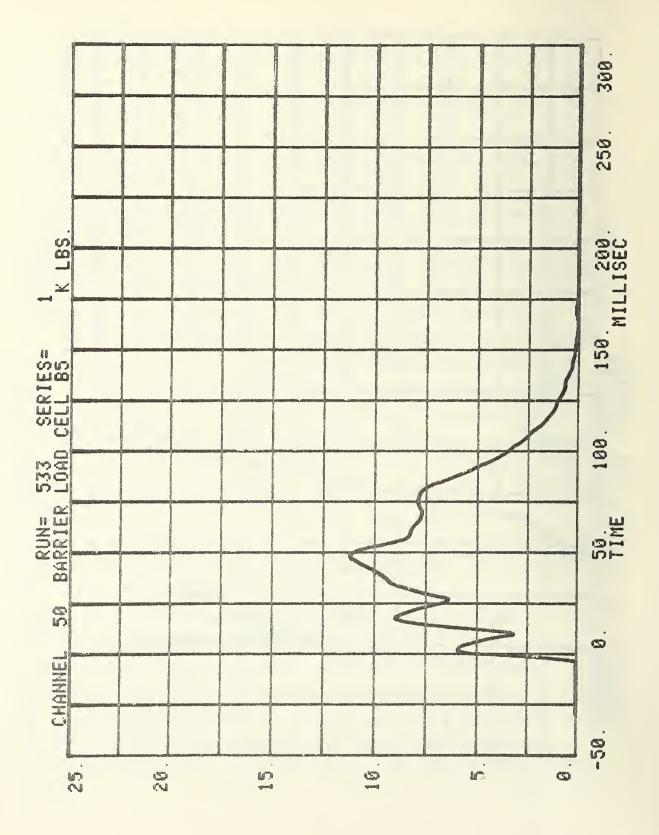






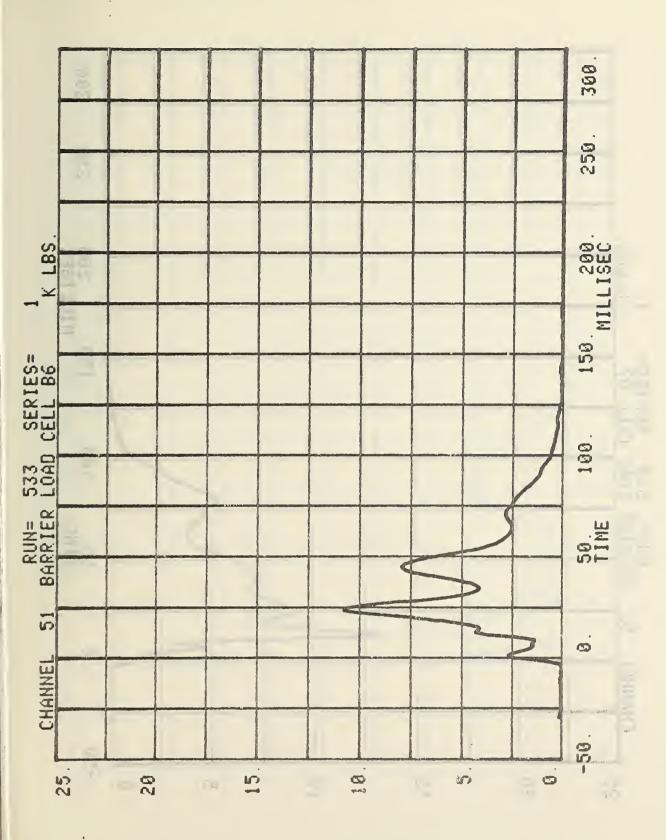


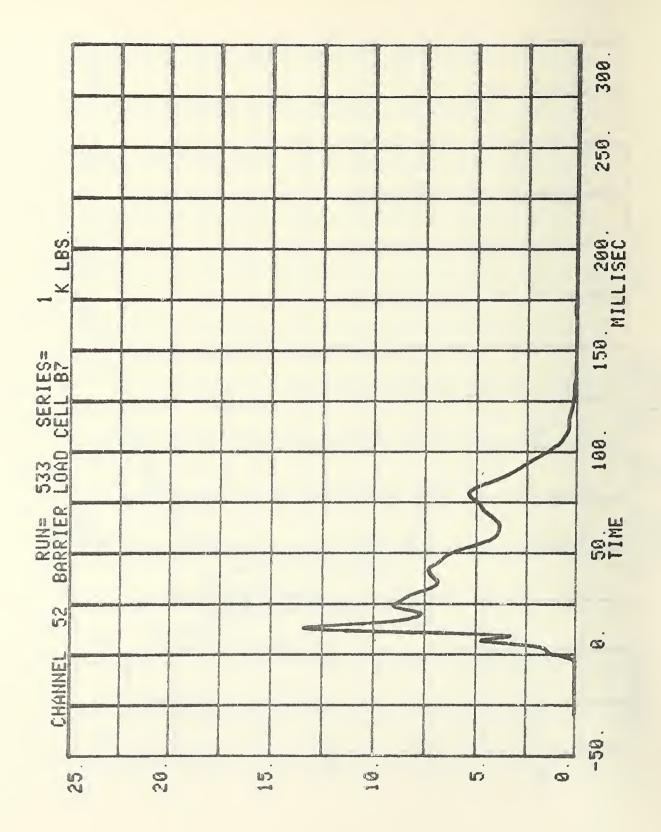




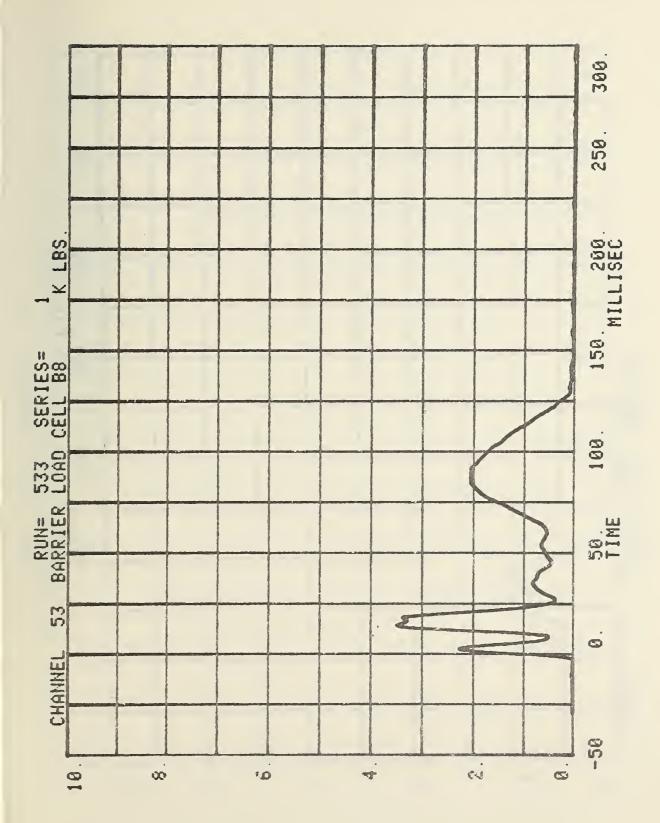
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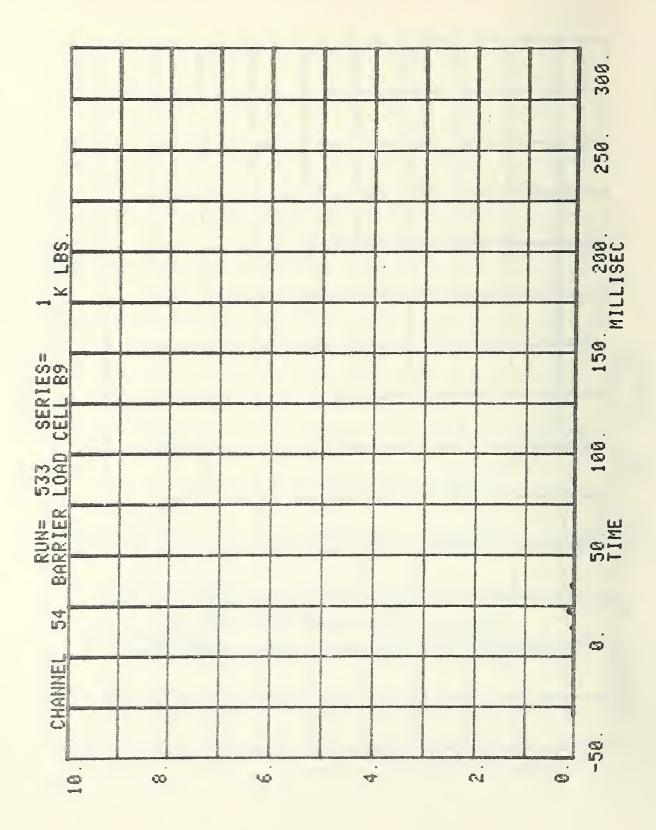
6829-V-8



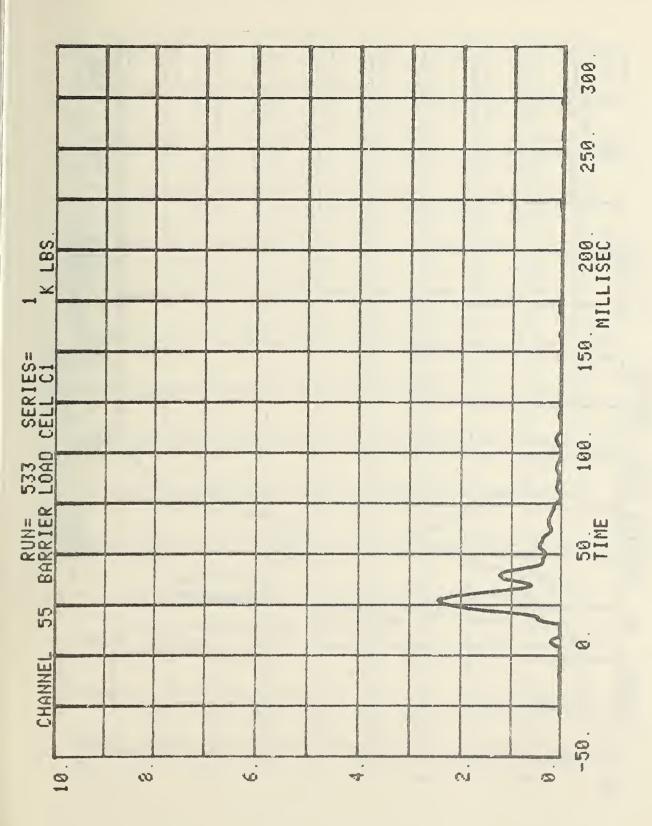


6829-V-8

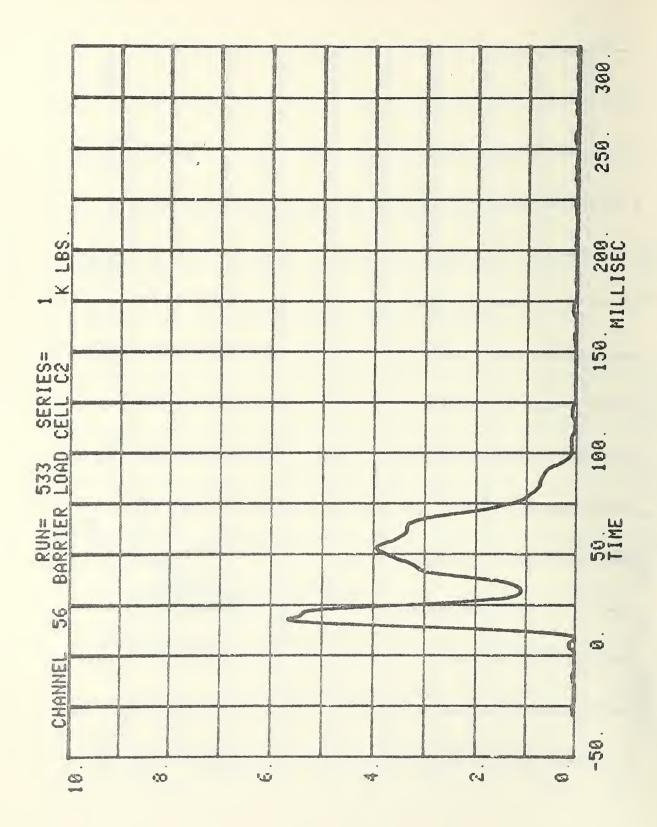


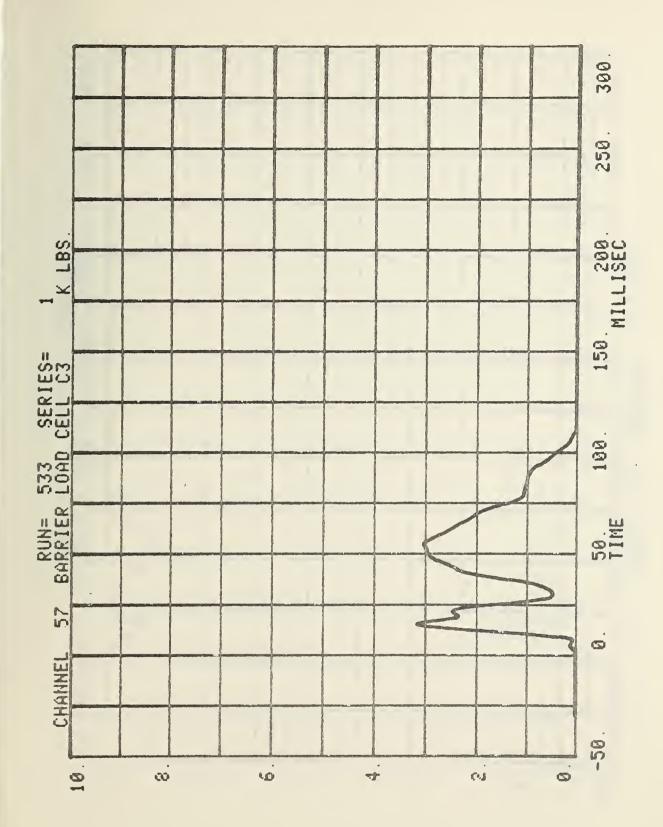


68.29-V-8

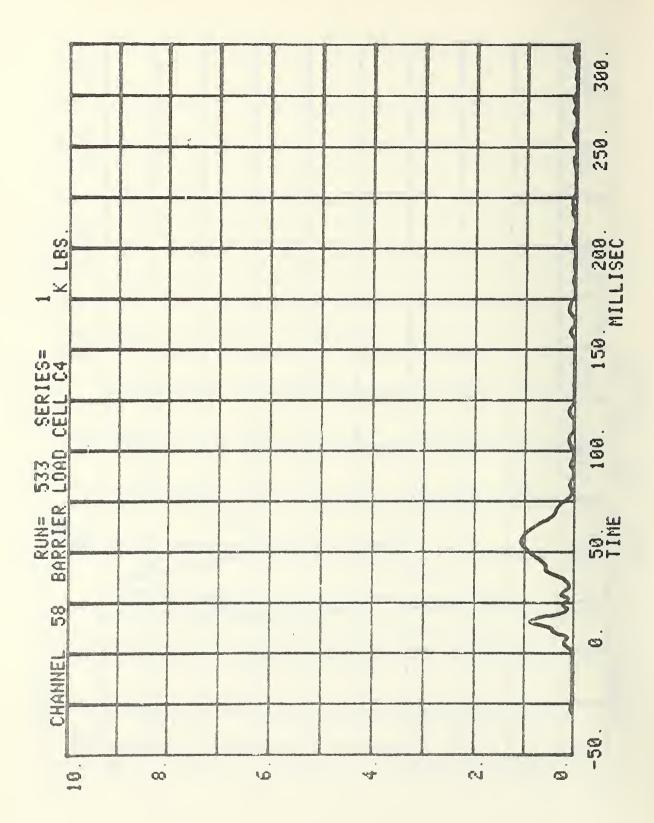


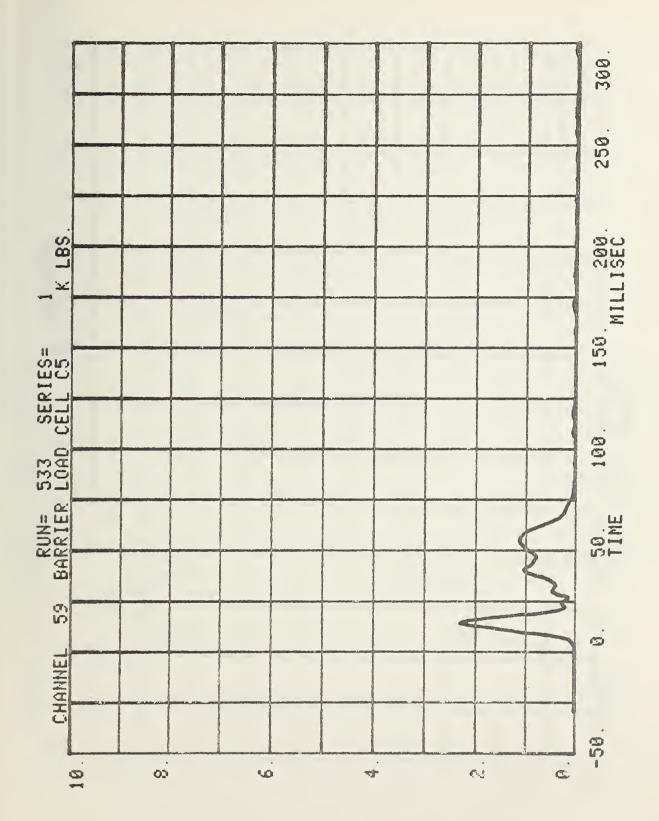
B-21



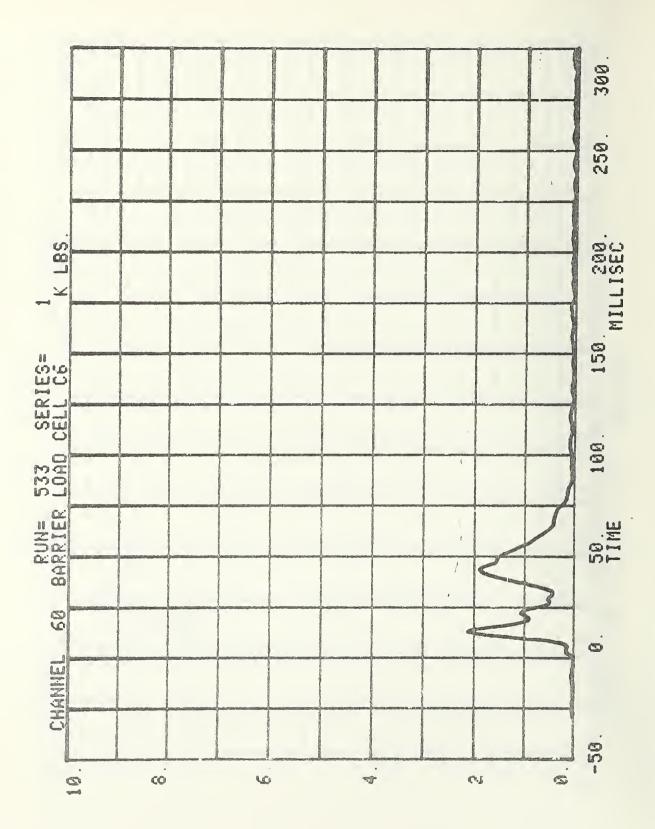


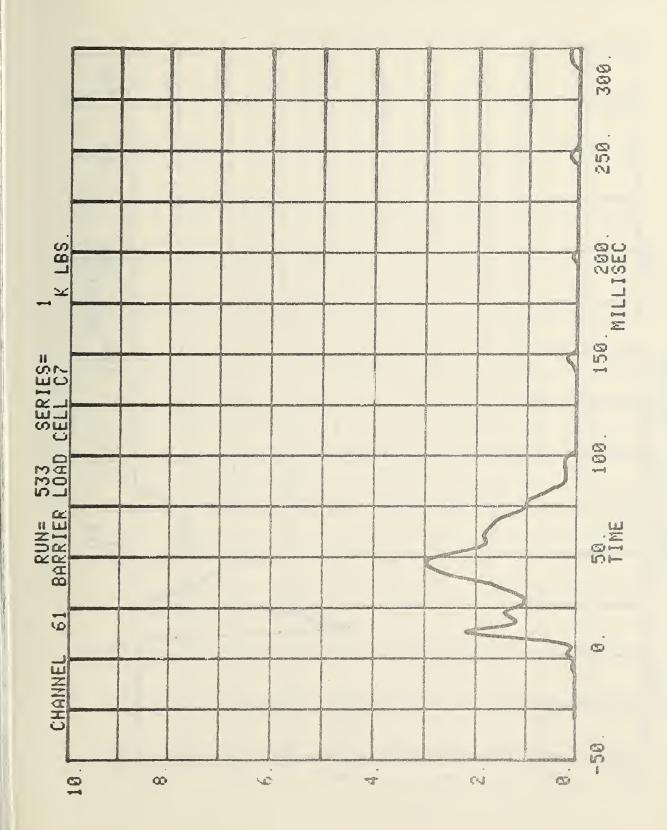
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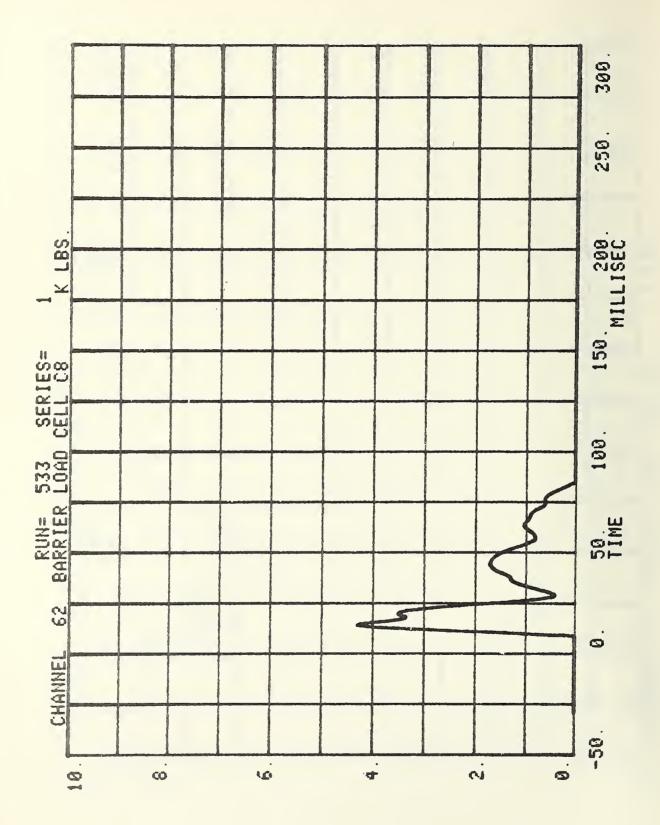


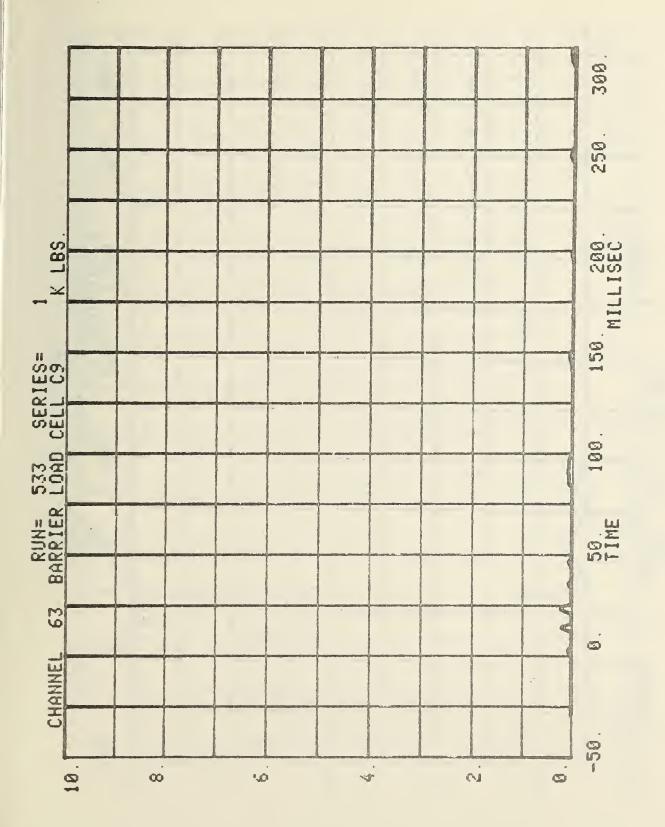


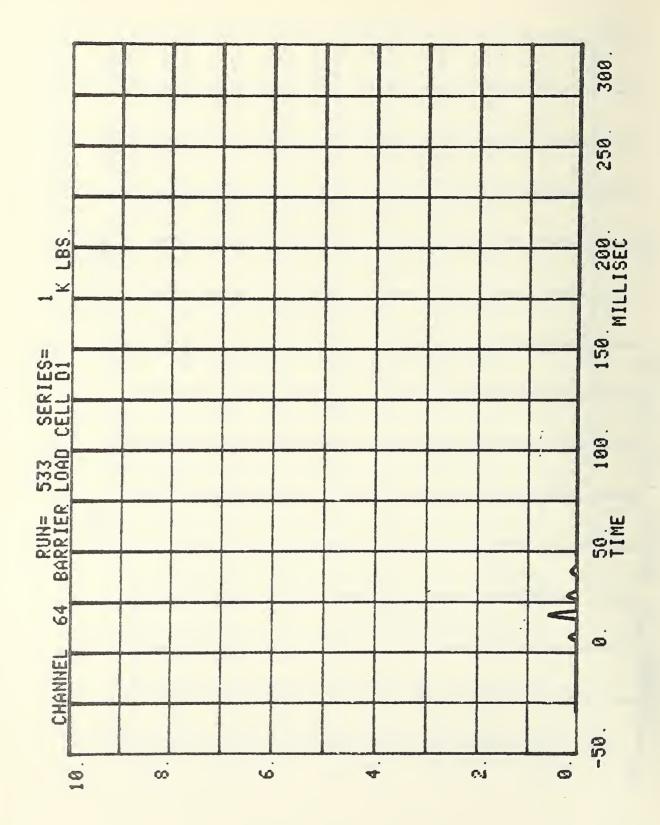
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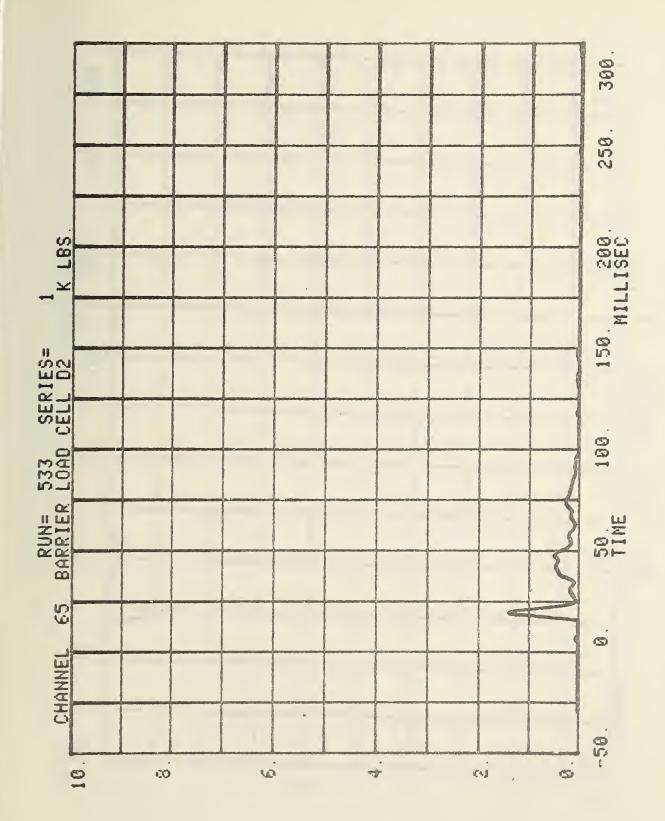


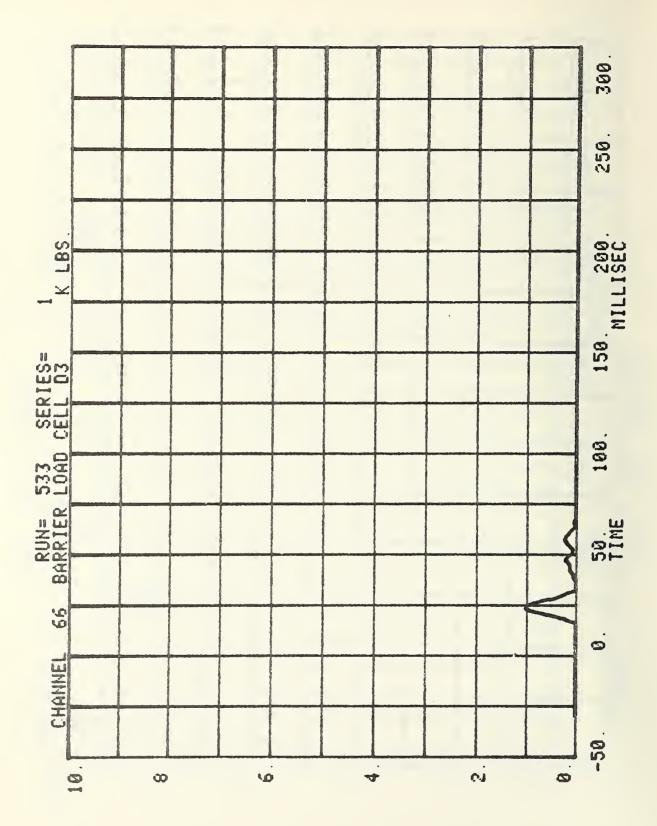


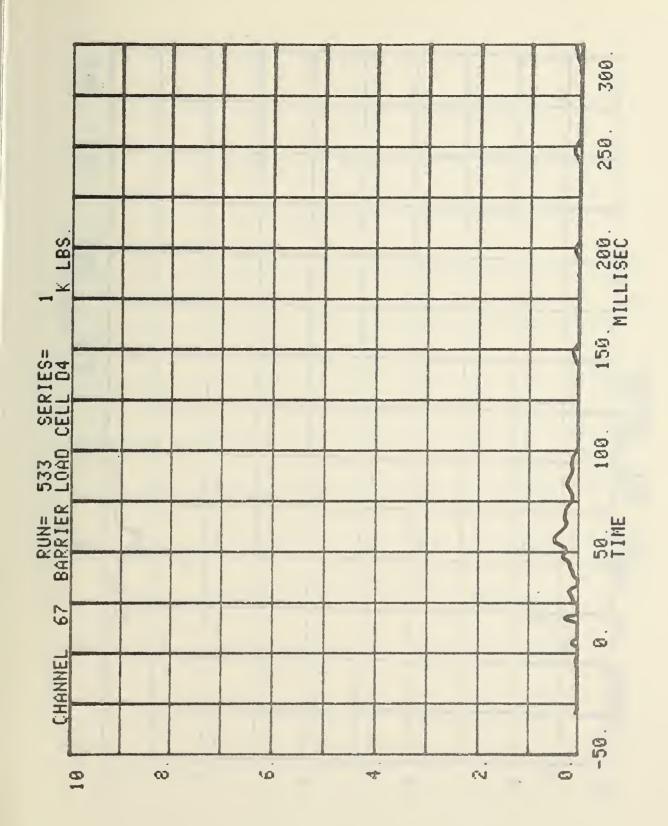


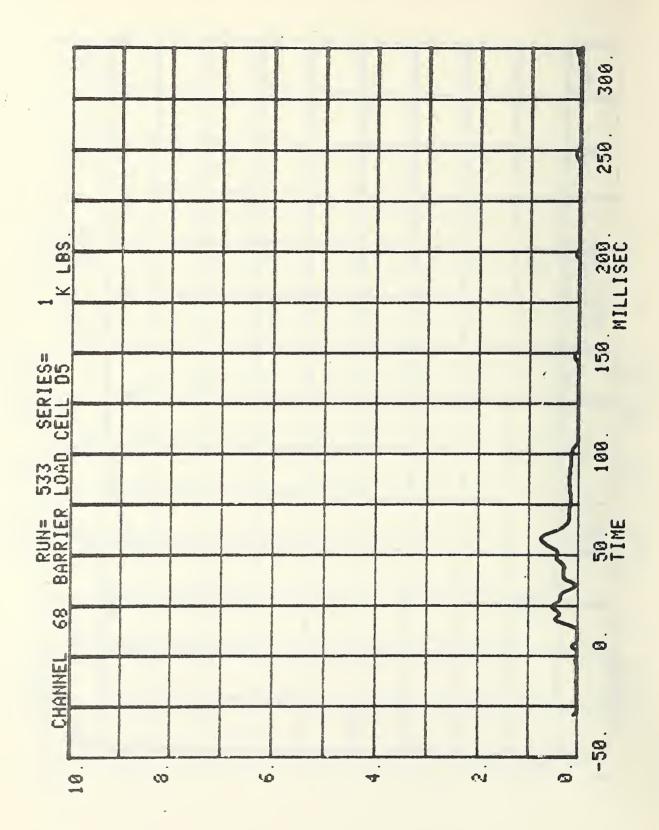


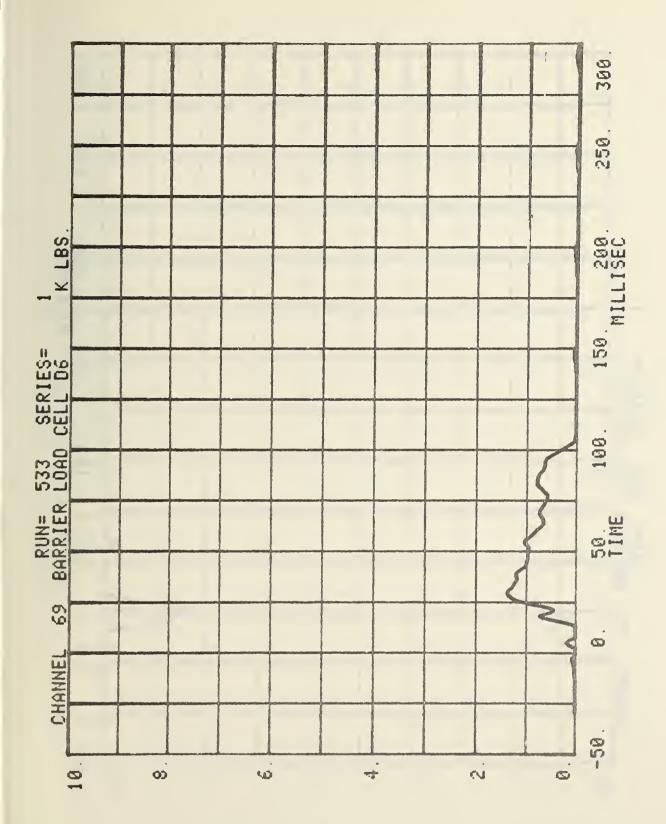


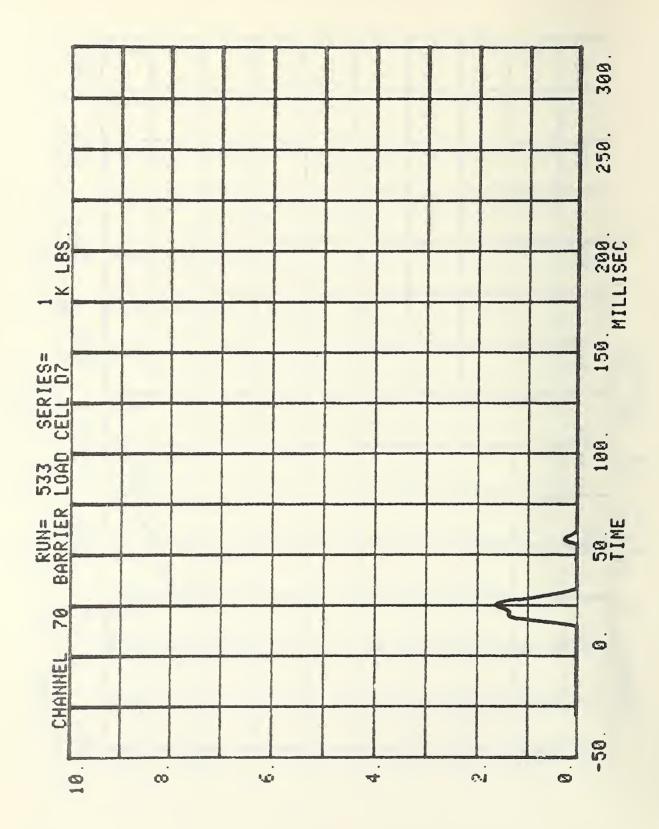


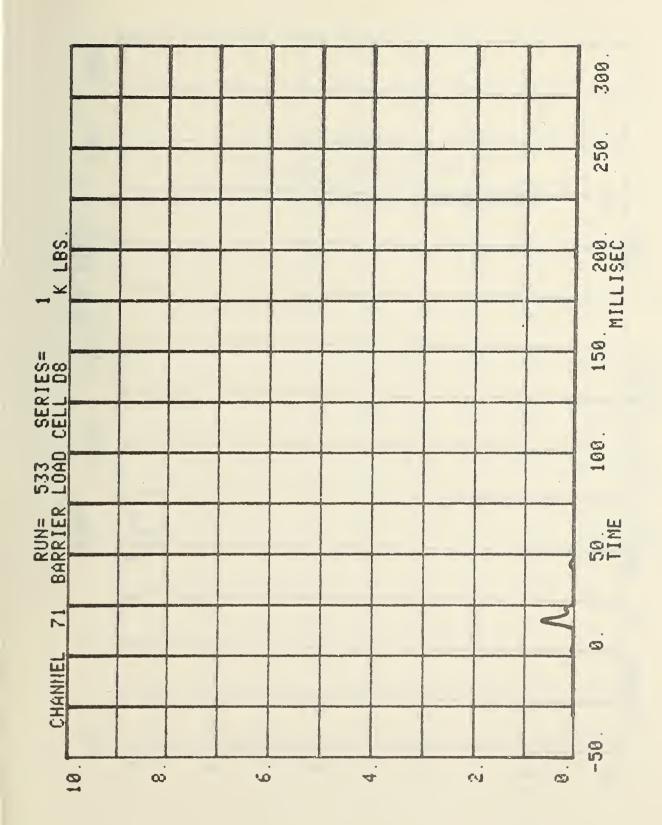


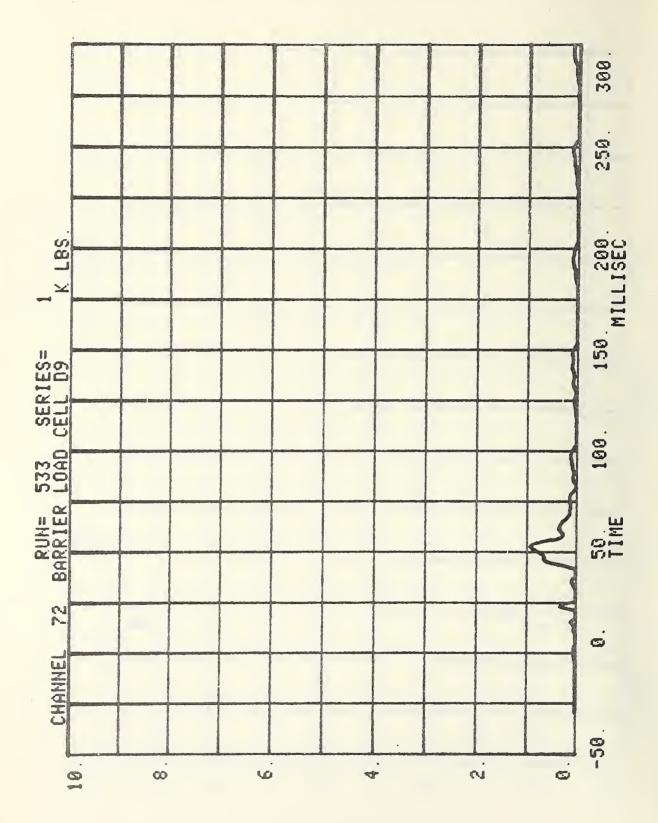












## APPENDIX C ELECTRONIC CRASH TEST DATA: PLYMOUTH RELIANT OCCUPANT AND RESTRAINT SYSTEM

## HEAD INJURY CRITERION HEAD SEVERITY INDEX

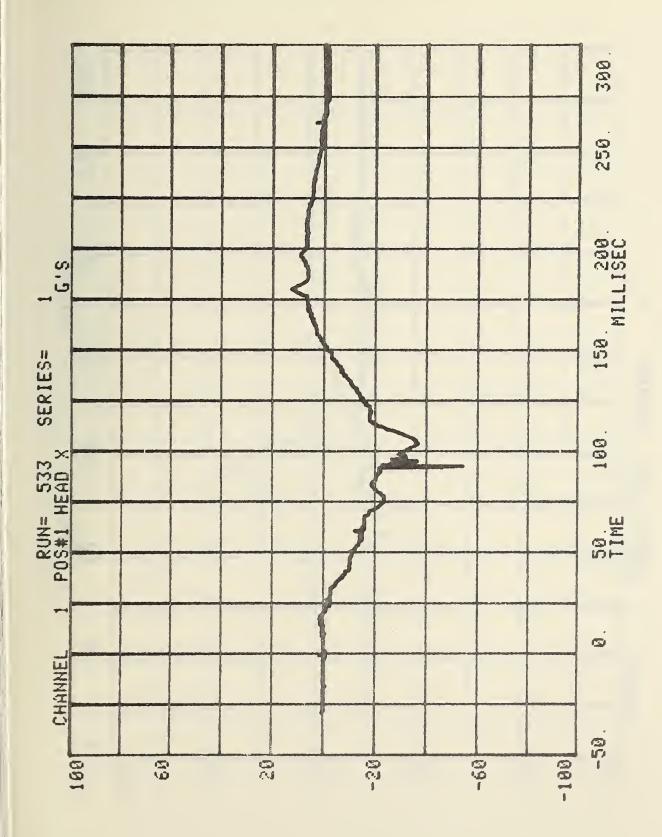
CAR TO LOAD CELL BARRIER

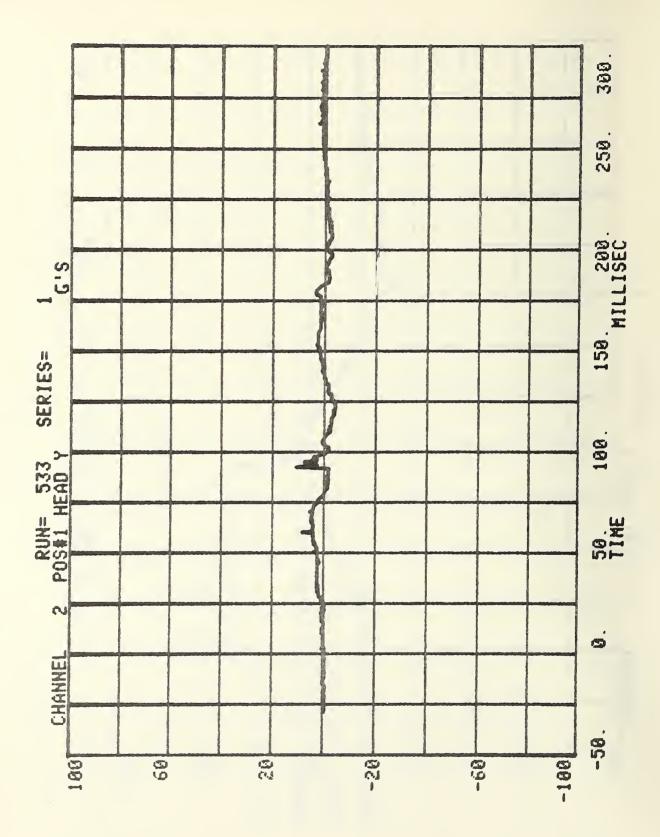
RUN= 533

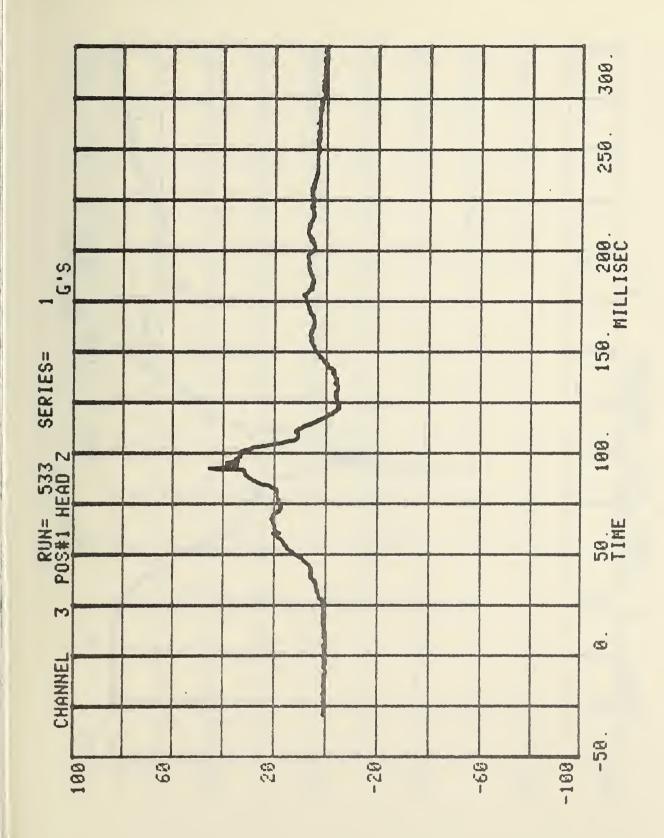
POS#1 HEAD RESULTANT

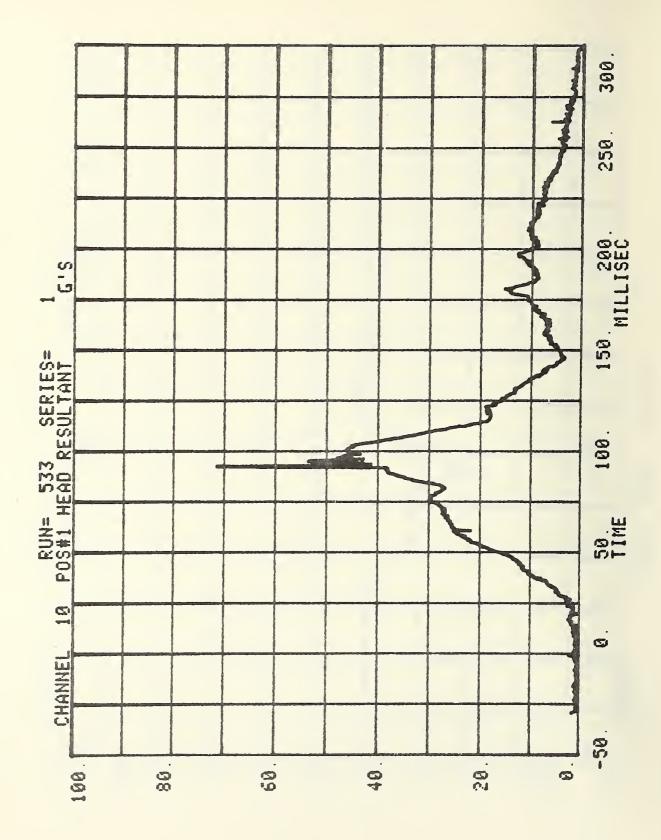
32.26'S TO T2= .11520 AVERAGE ACCELERATION BETWEEN TI AND T2= FROM 11= .05280 MSEC EVENT TIME= 300.0 HIC= 367.3

SEVERITY INDEX= 486.9





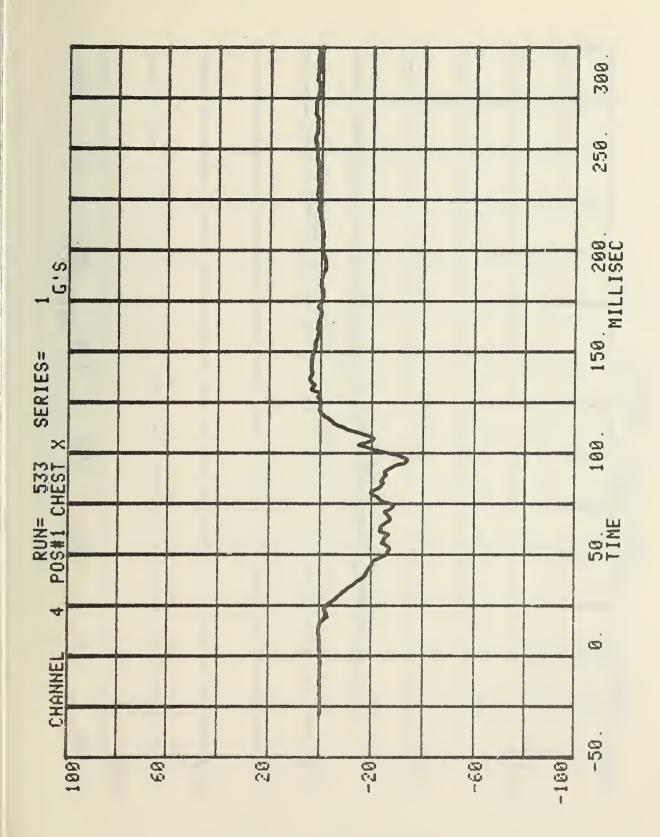


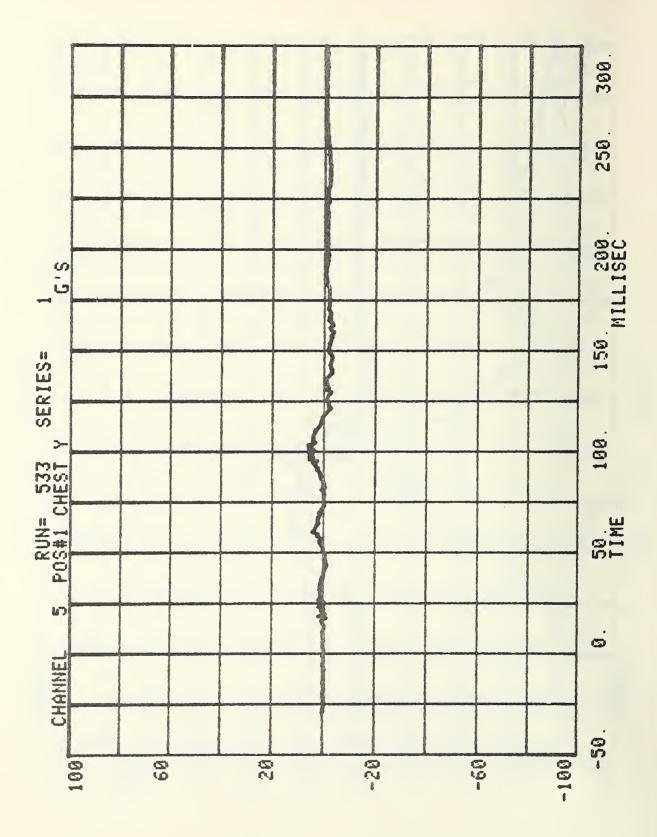


6329-V-3

C-6

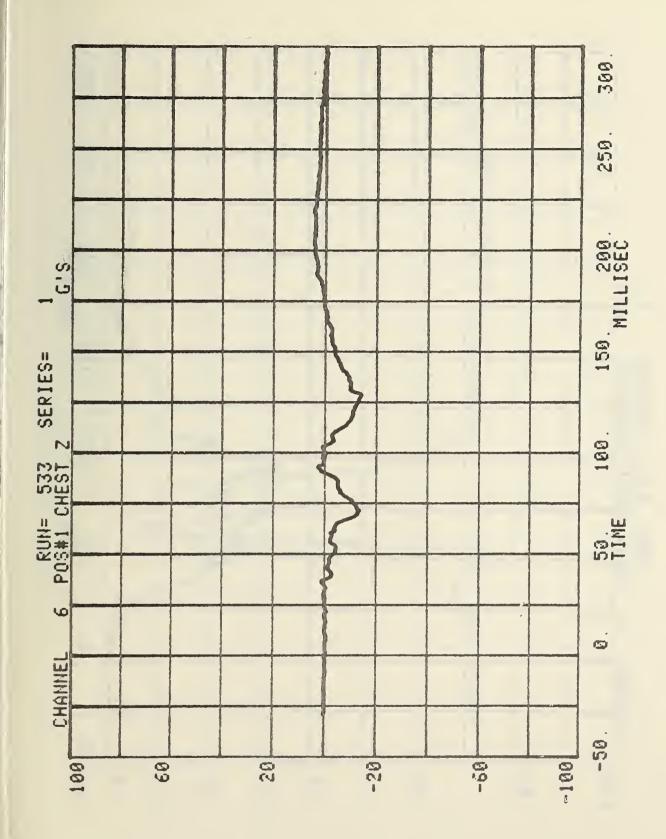
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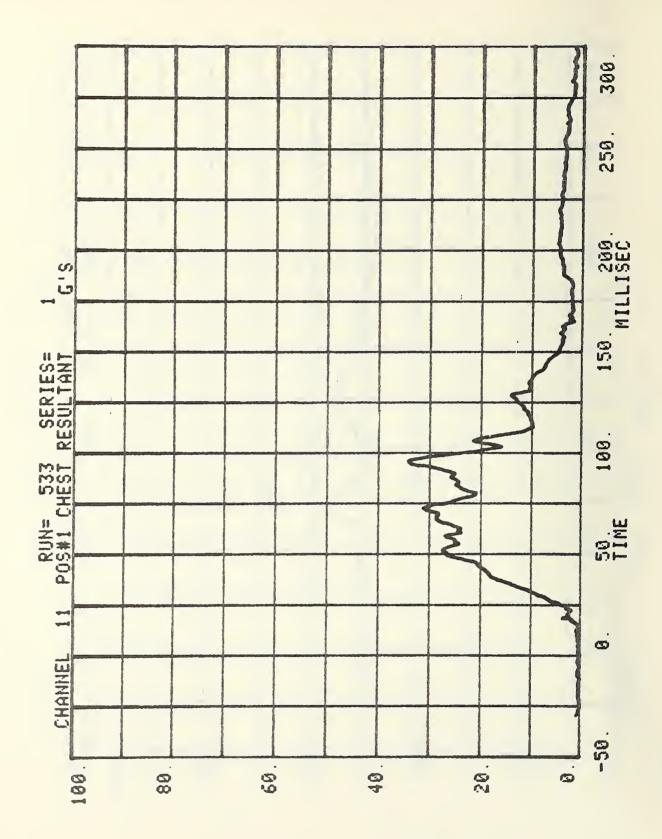


11411

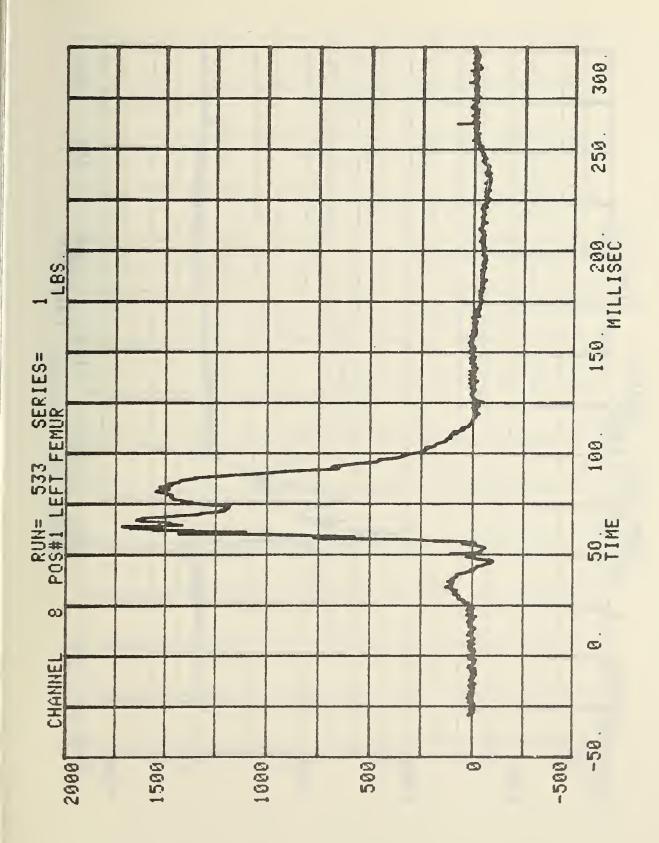
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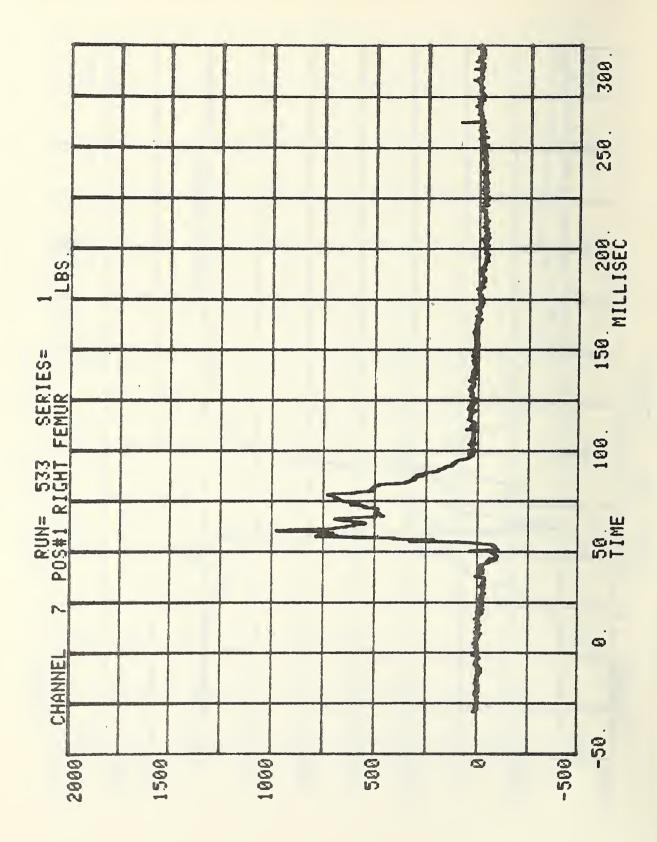


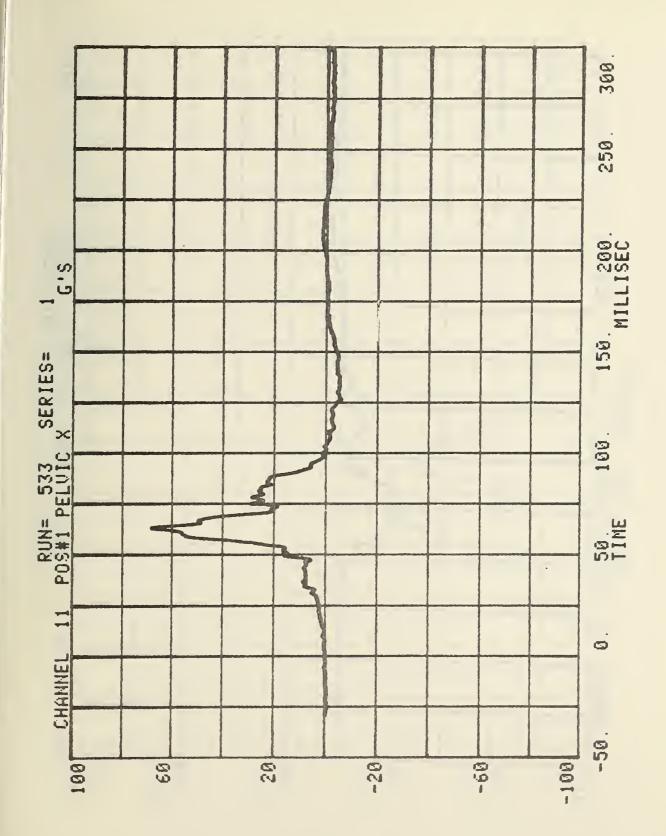
C-9

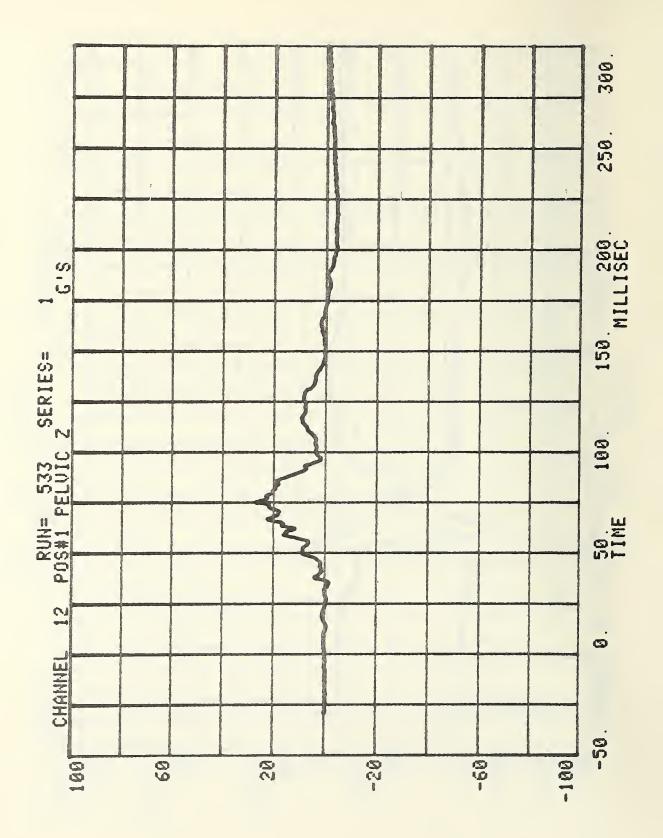


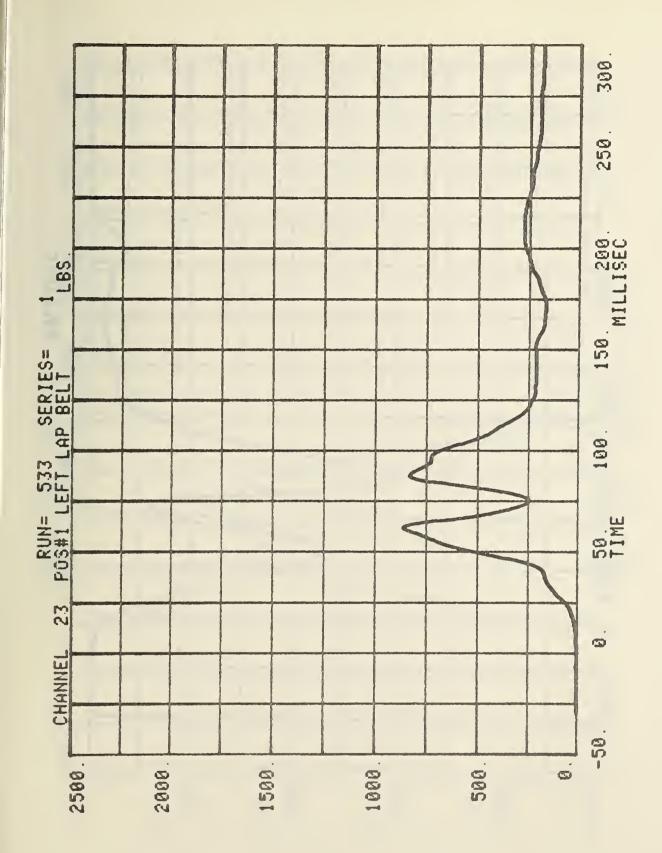
6829-V-8





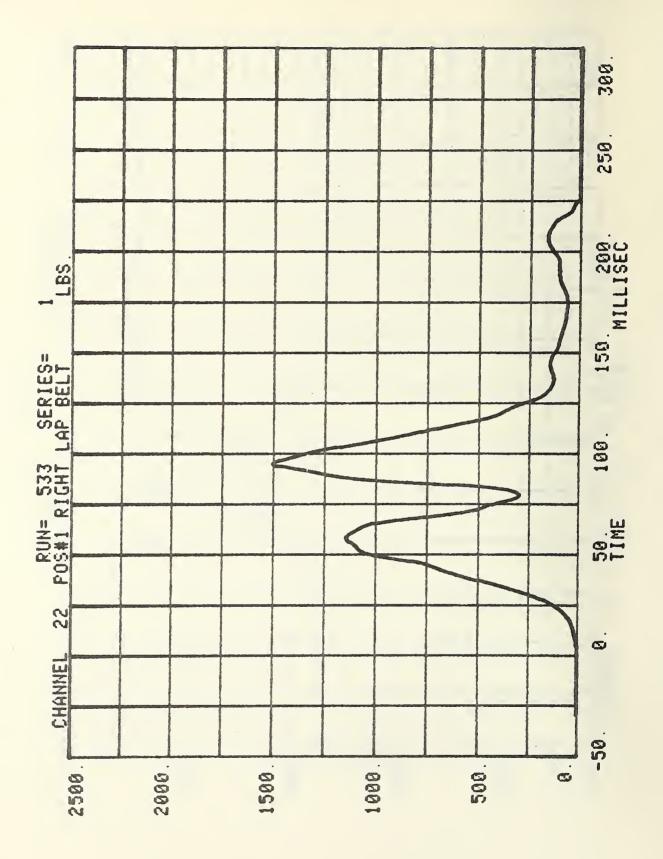




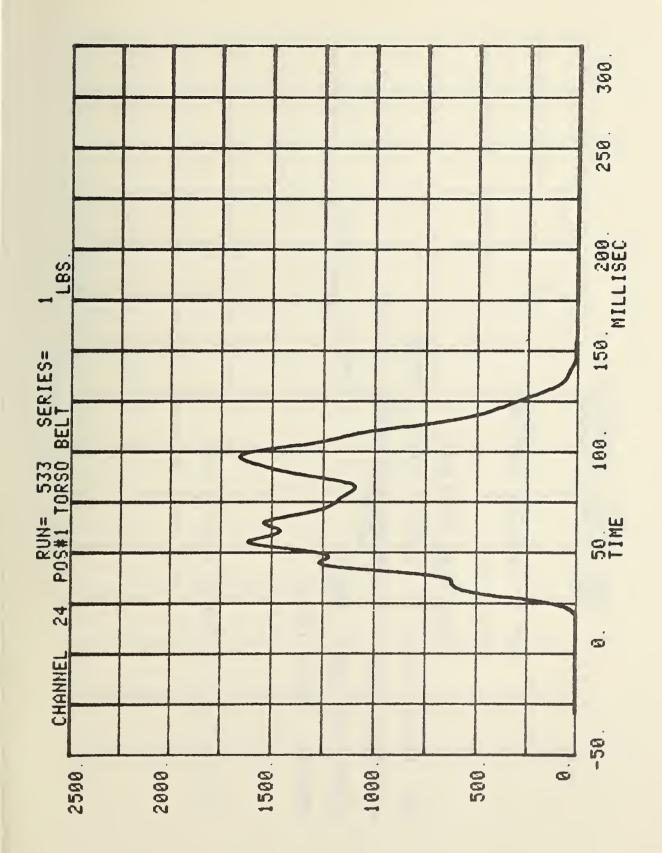


6829-V-3

C-15



C-16



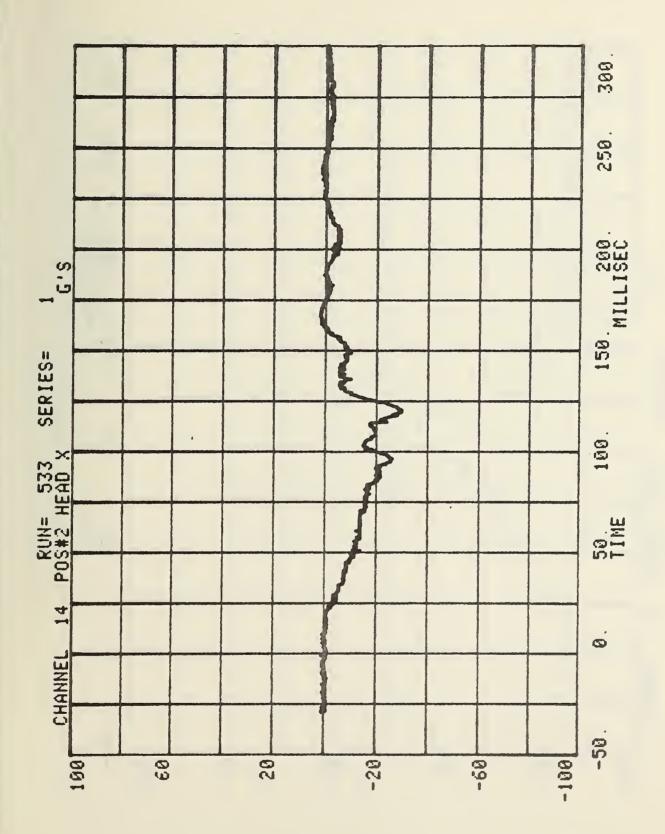
## HEAD INJURY CRITERION HEAD SEVERITY INDEX

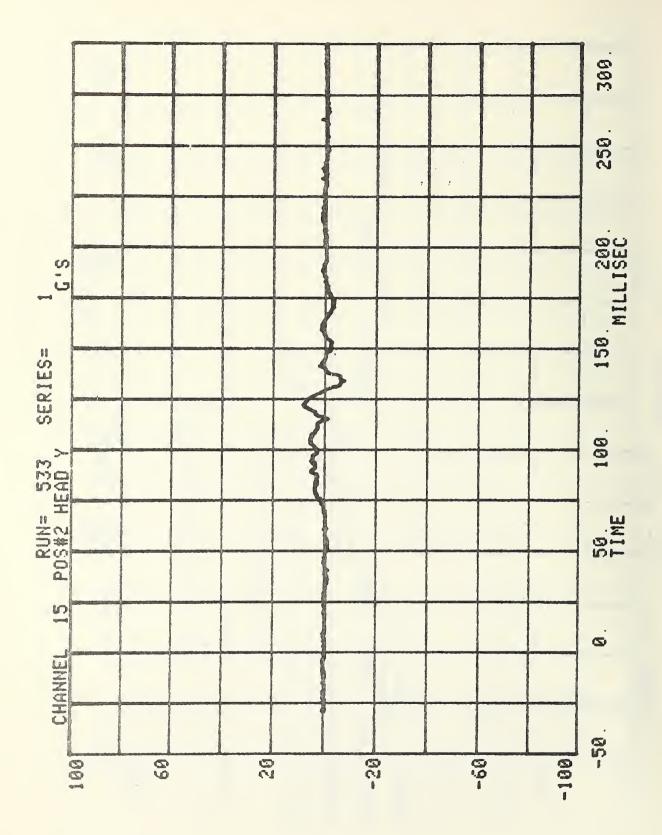
CAR TO LOAD CELL BARRIER RUN= 533

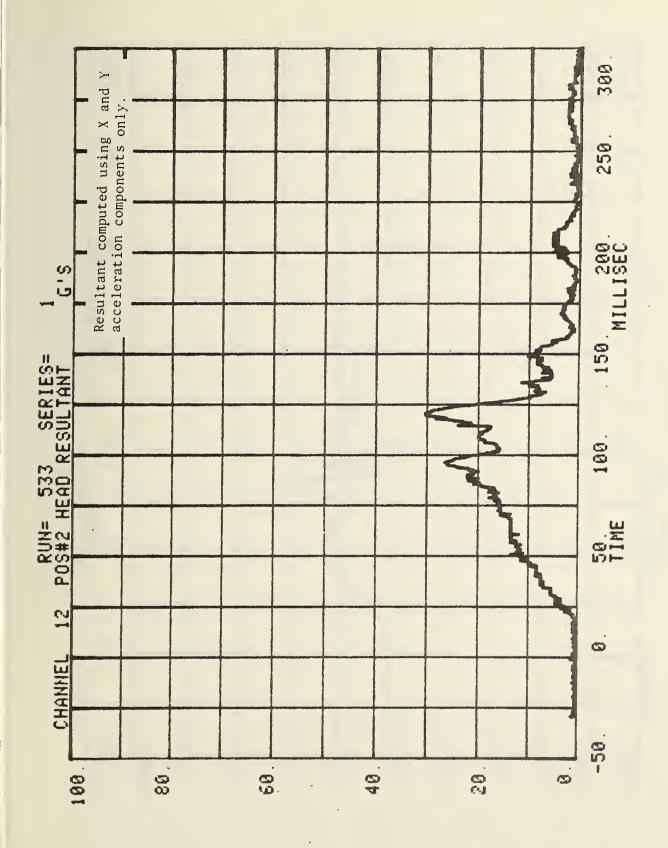
POS#2 HEAD RESULTANT\*

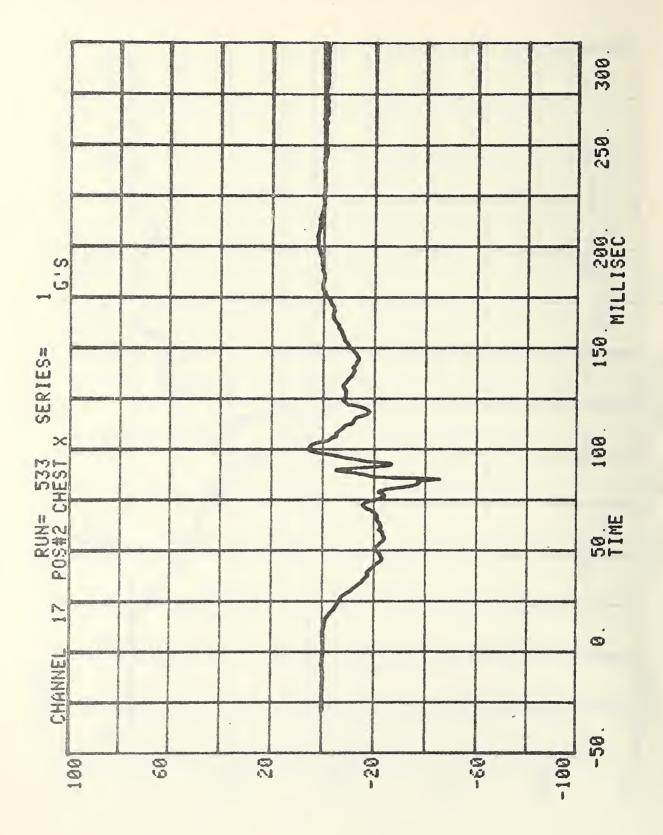
18.36'5 TO T2= .12840 AVERAGE ACCELERATION BETWEEN TI AND T2= FROM T1= .05010 MSEC SEVERITY INDEX= 137.8 EVENT TIME= 300.0 HIC= 112.5

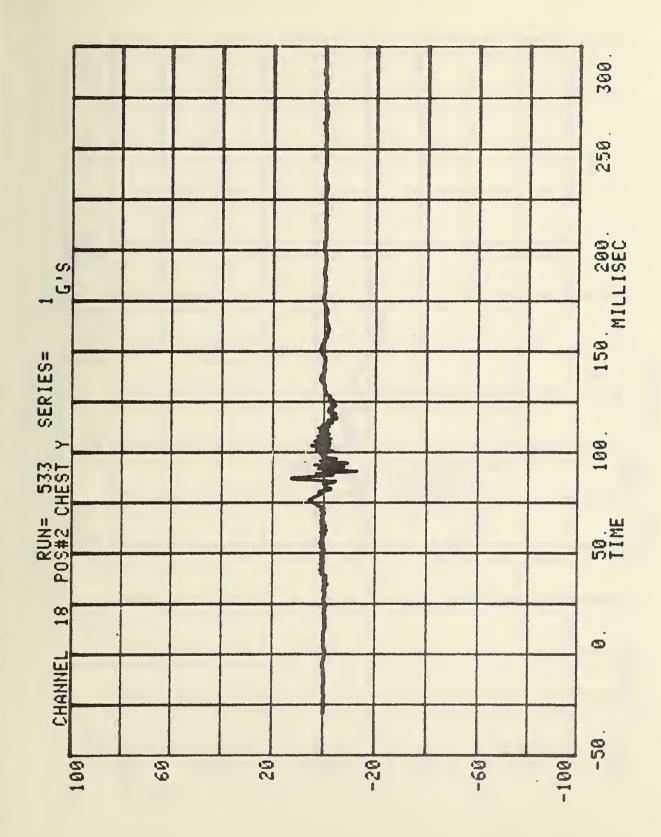
\*Reflects X and Y acceleration components only.

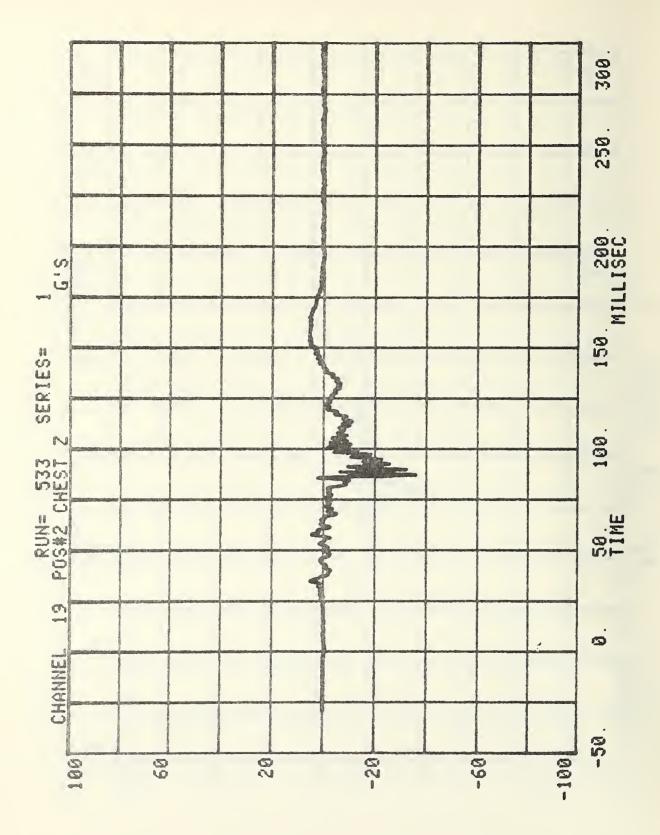


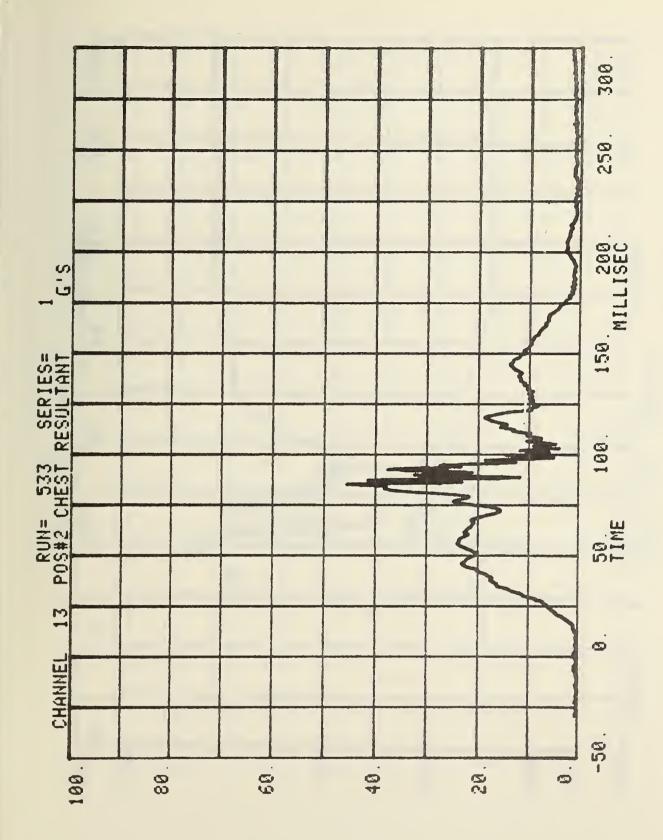


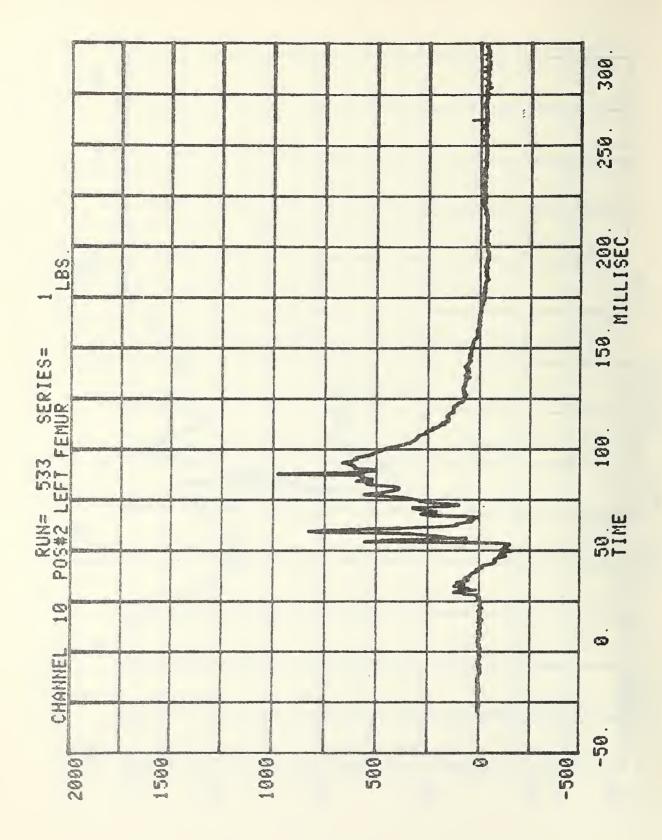


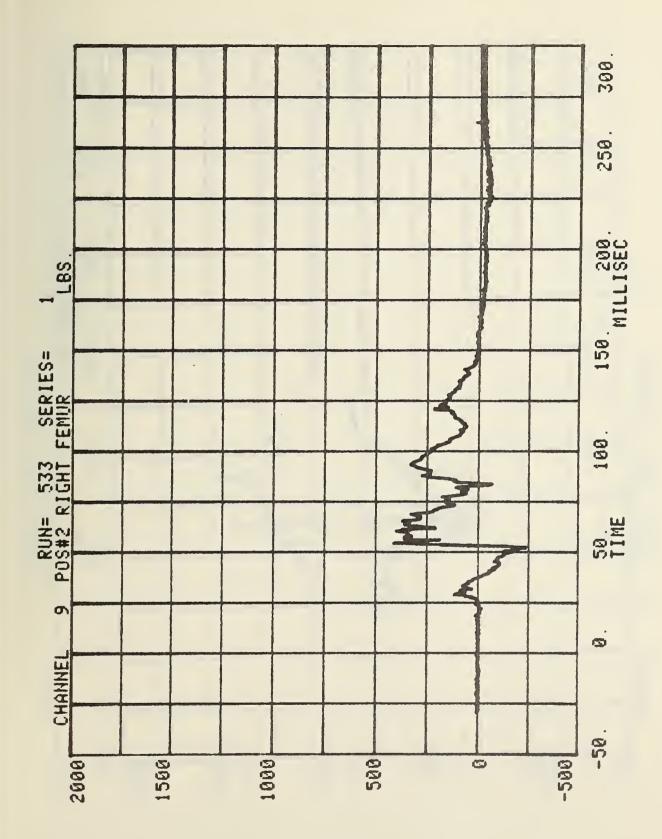


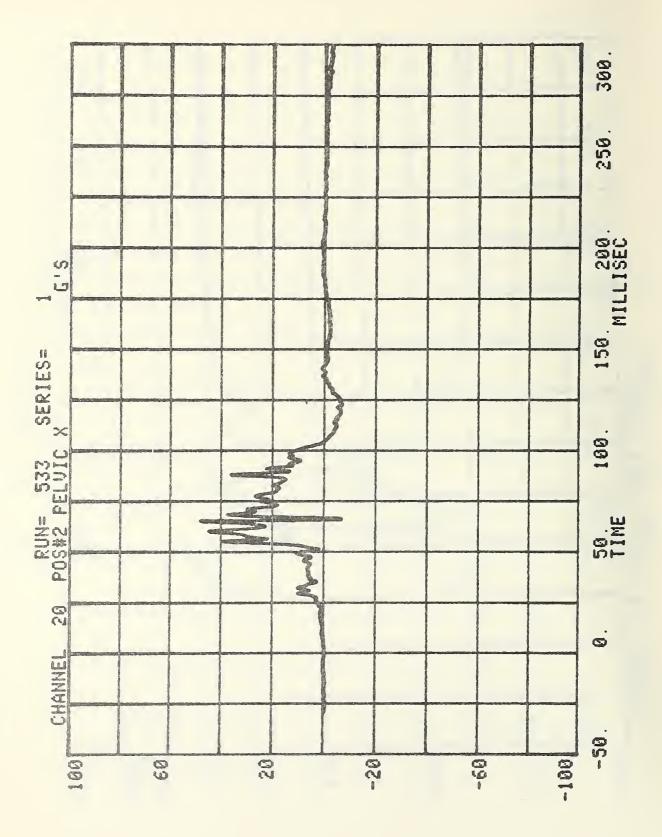


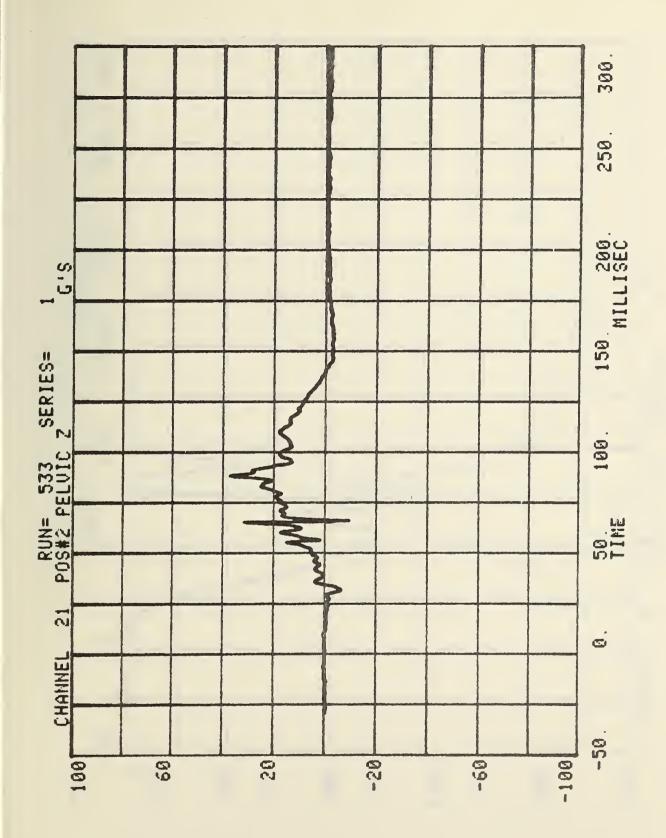


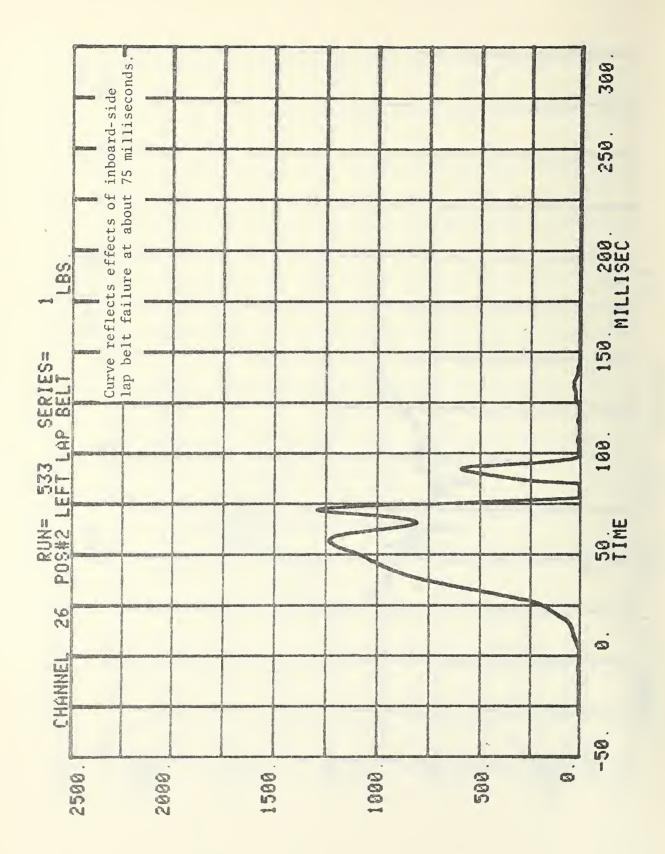


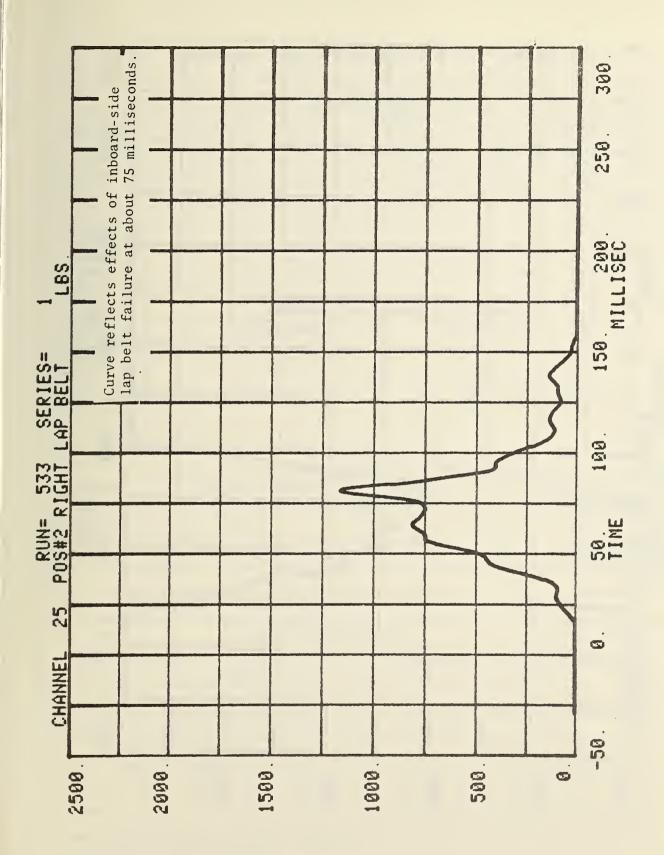


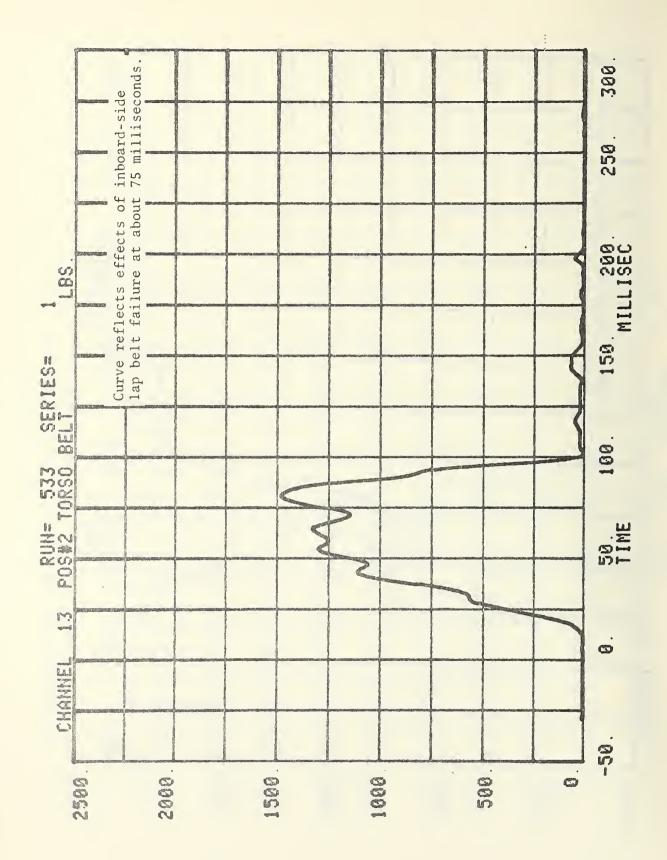












## OCCUPANT AND RESTRAINT SYSTEM DATA SUMMARY

	MAXIMUM ACCELERATION (g)											
DUMMY	HEAD				CHEST			PELVIS				
POSITION	X	Y	Z	R	Х	Y	Z	R	Х	Y	Z	R
Driver (1)	-55	11	45	71	-32	6	-14	33	68		27	
Pass. (2)	- 30	9	NA	30 <sup>3</sup>	- 37	-8	-28	38	48		37	

	MAXIMUM FORCE-FEMUR LOAD (LBS)					
DUMMY POSITION	RIGHT FEMUR	LEFT FEMUR				
Driver (1)	980	1720				
Pass. (2)	420	980				

	MAXIMUM FORCE-SEAT BELT LOADS (LBS)						
DUMMY POSITION	SHOULDER STRAP UPPER BELT LOAD	LAP STRAP RIGHT BELT LOAD	LAP STRAP LEFT BELT LOAD				
Driver (1)	1660	1500	860				
Pass. (2)	1480 <sup>4</sup>	1170 <sup>4</sup>	12804				

		HEAD IN	SEVERITY INDEX			
DUMMY POSITION	ніс	t <sub>1</sub> (SEC)	t <sub>2</sub> (SEC)	AVE. ACC. (g) t <sub>1</sub> TO t <sub>2</sub>	HEAD	CHEST
Driver (1)	367	0.053	0.115	32	487	
Pass. (2)	113 <sup>3</sup>	0.050	0.128	18 <sup>3</sup>	1383	

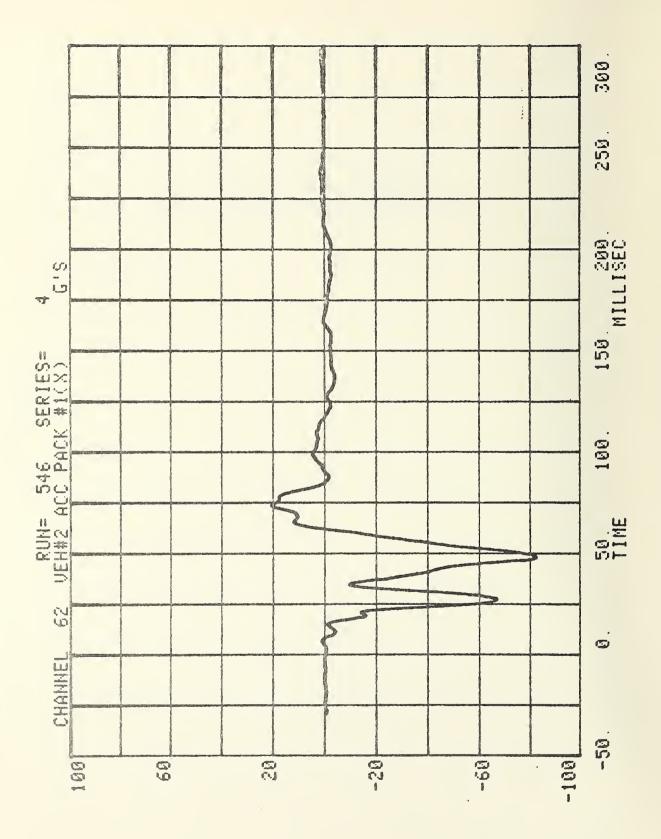
<sup>1</sup>DEFINED AS EXCEEDING 0.003 SEC. DURATION

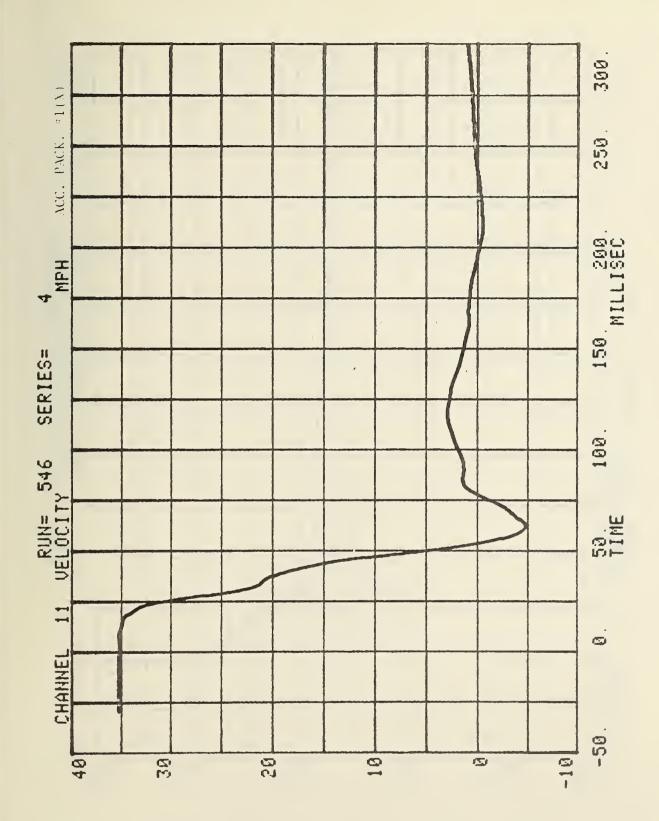
<sup>2</sup>AS DEFINED IN FMVSS NO. 208

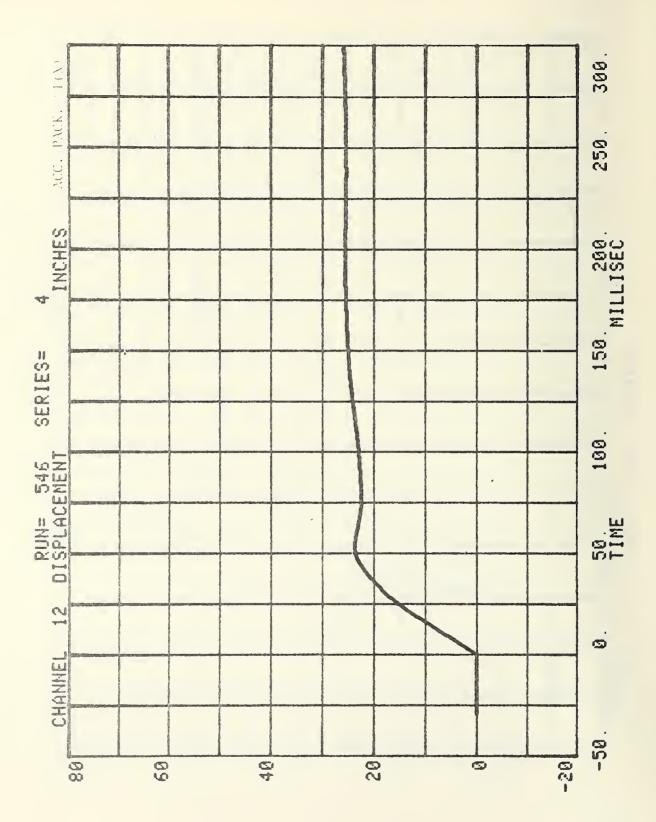
<sup>3</sup>COMPUTED USING X AND Y ACCELERATION COMPONENTS ONLY

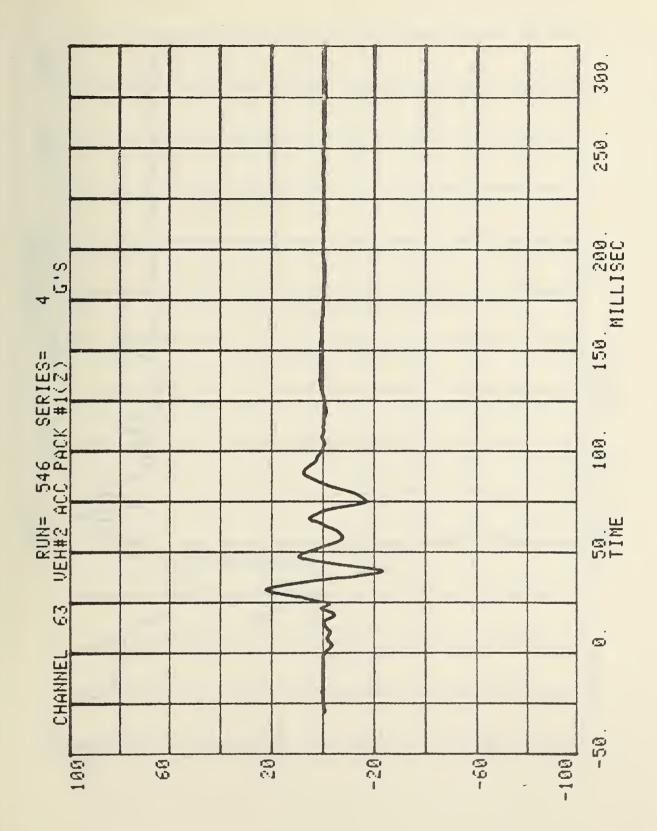
<sup>4</sup>MAXIMUM LOADING PRIOR TO LEFT LAP BELT WEBBING FAILURE

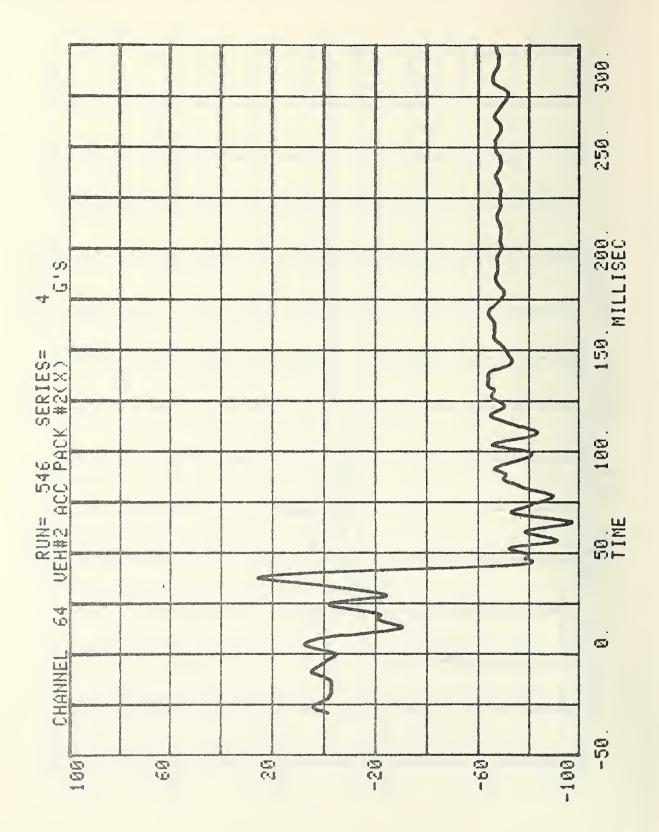
## APPENDIX D <u>ELECTRONIC CRASH TEST DATA:</u> PLYMOUTH HORIZON VEHICLE-MOUNTED SENSORS

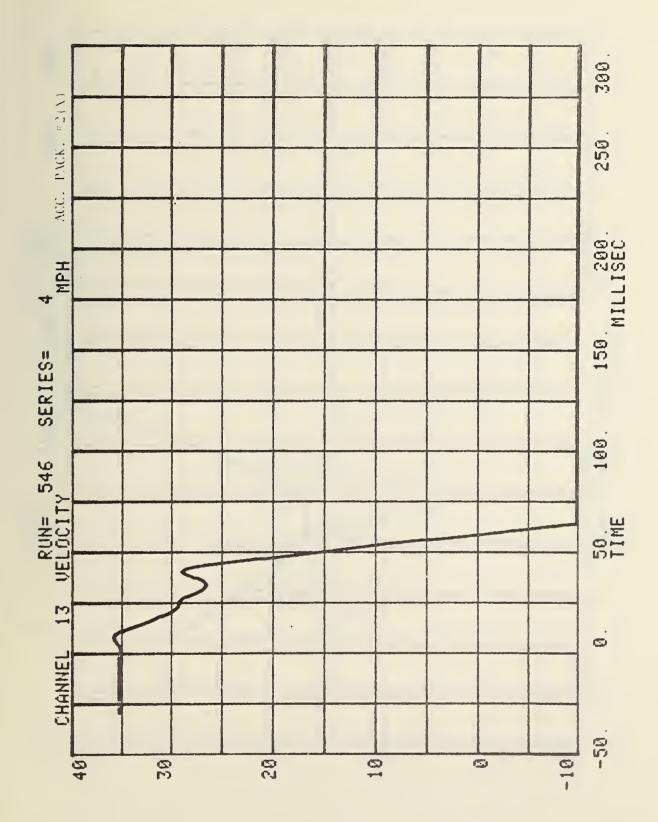




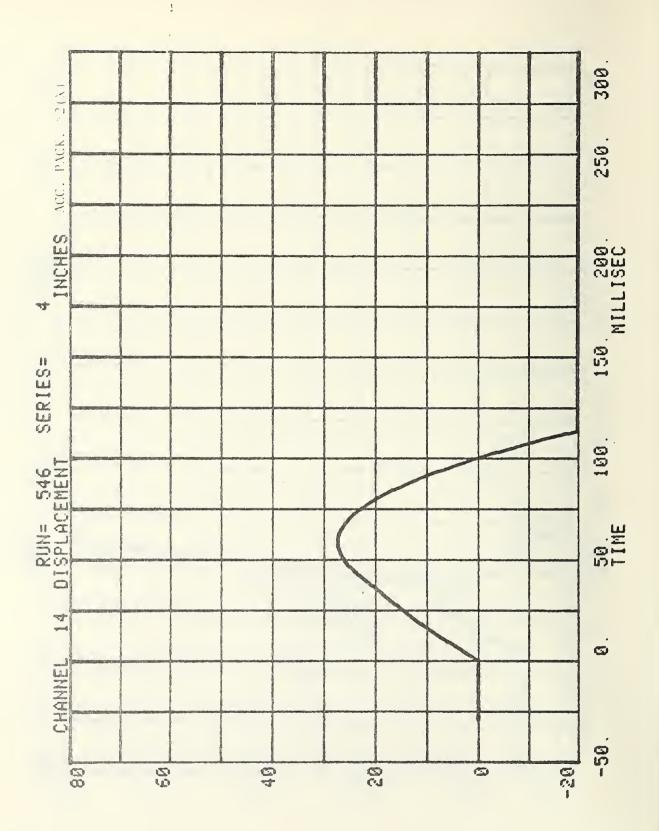


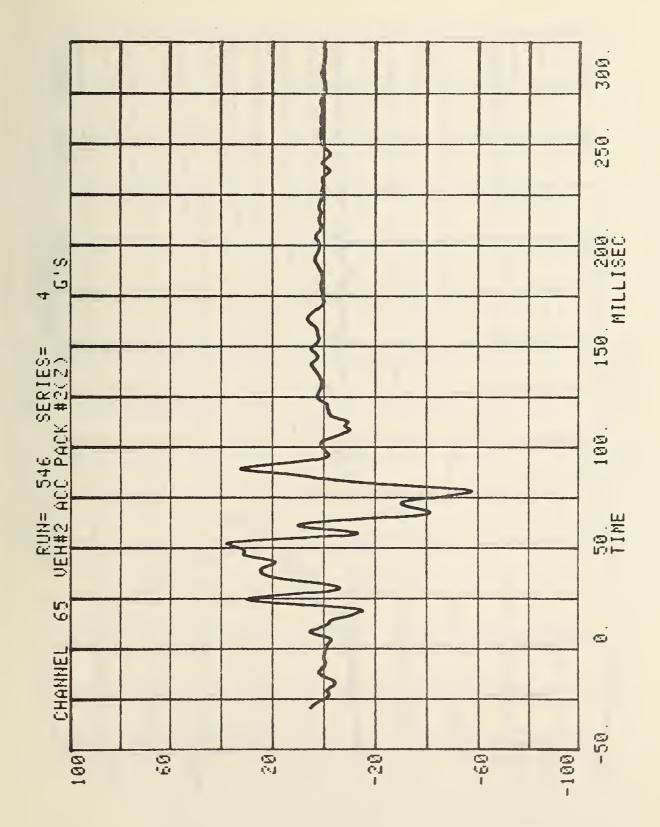


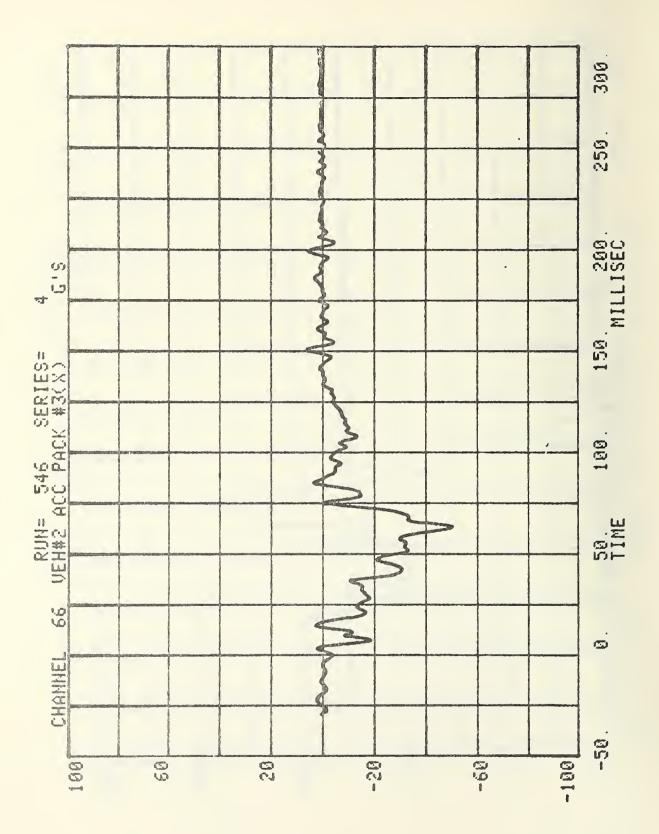


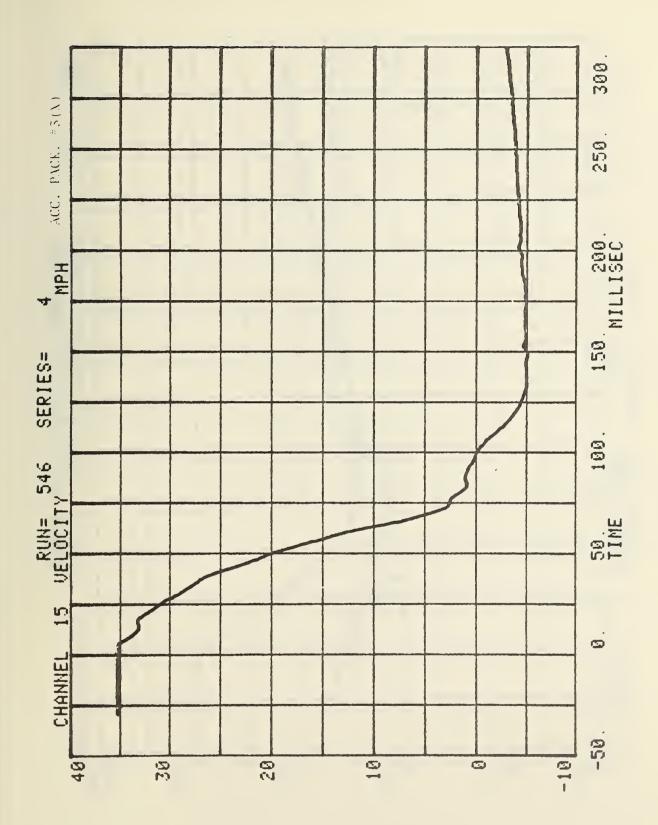


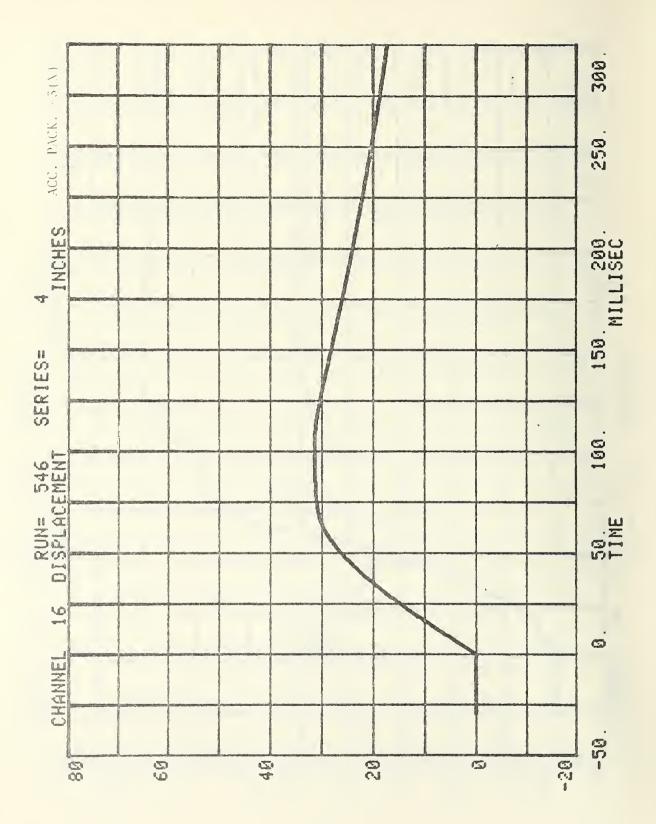
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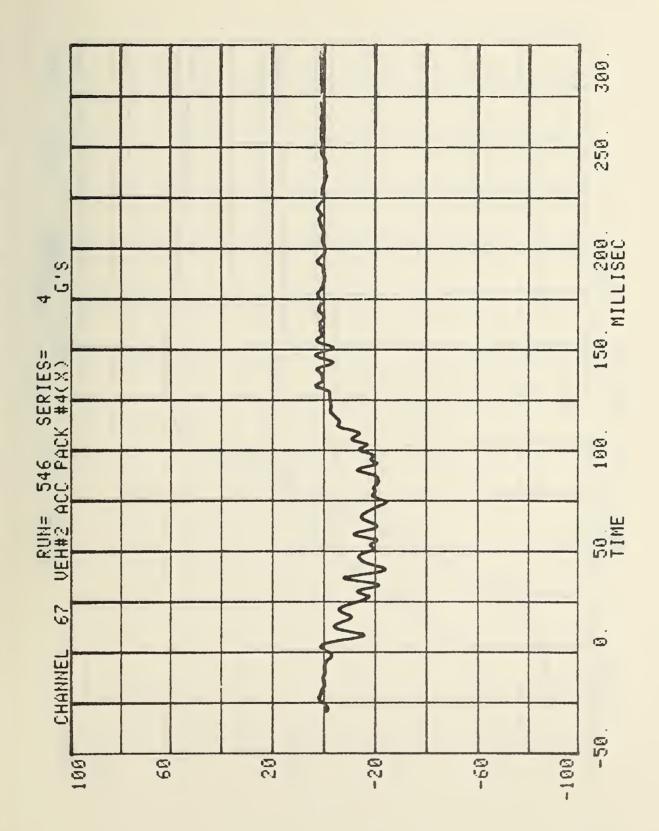




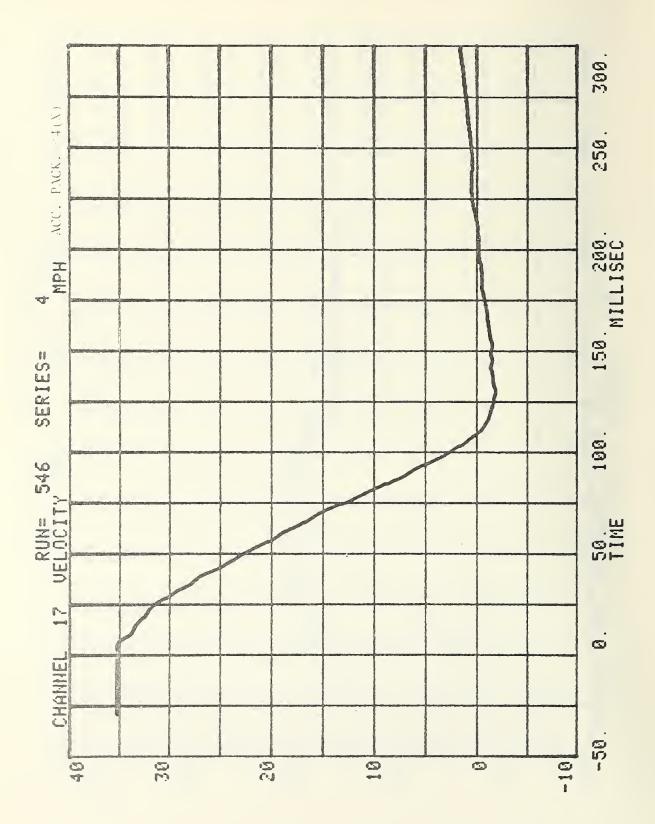


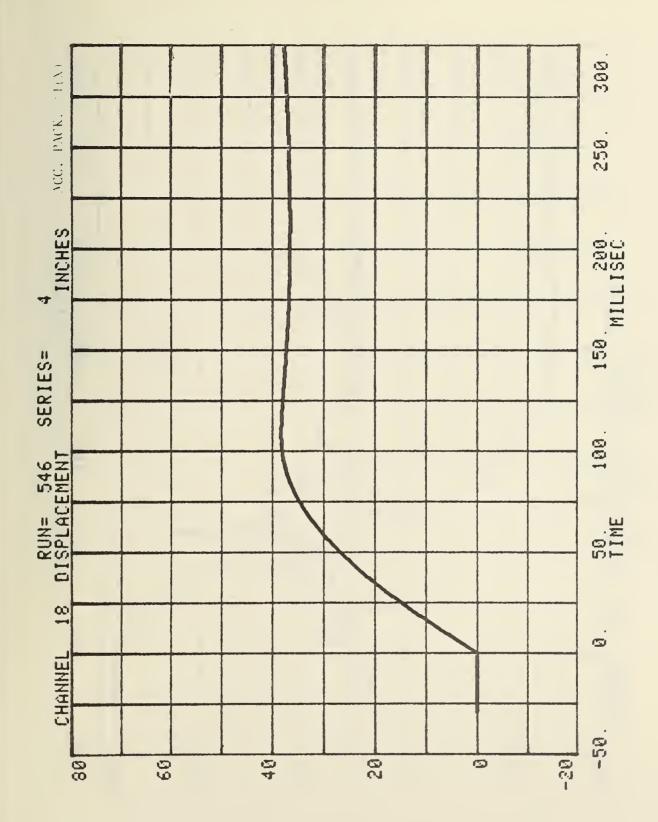


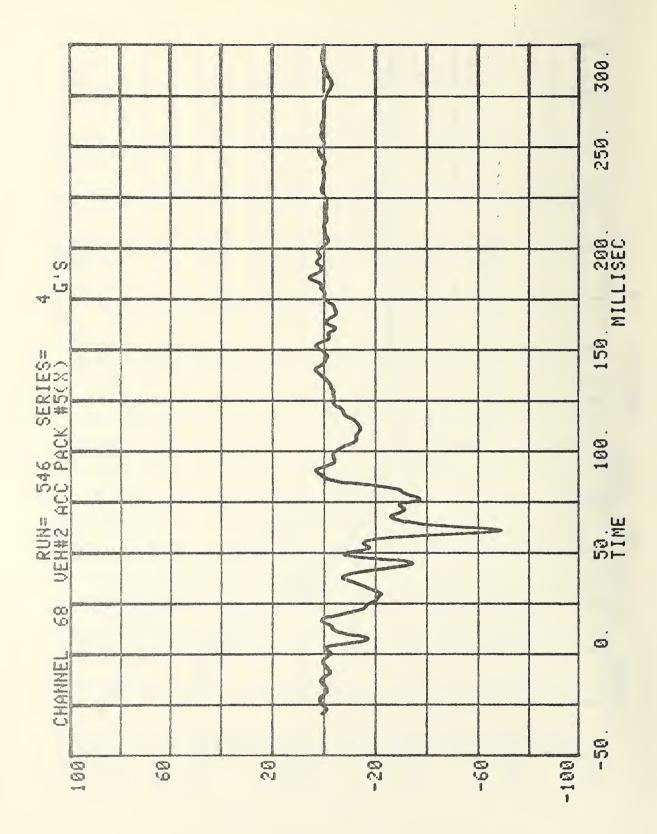


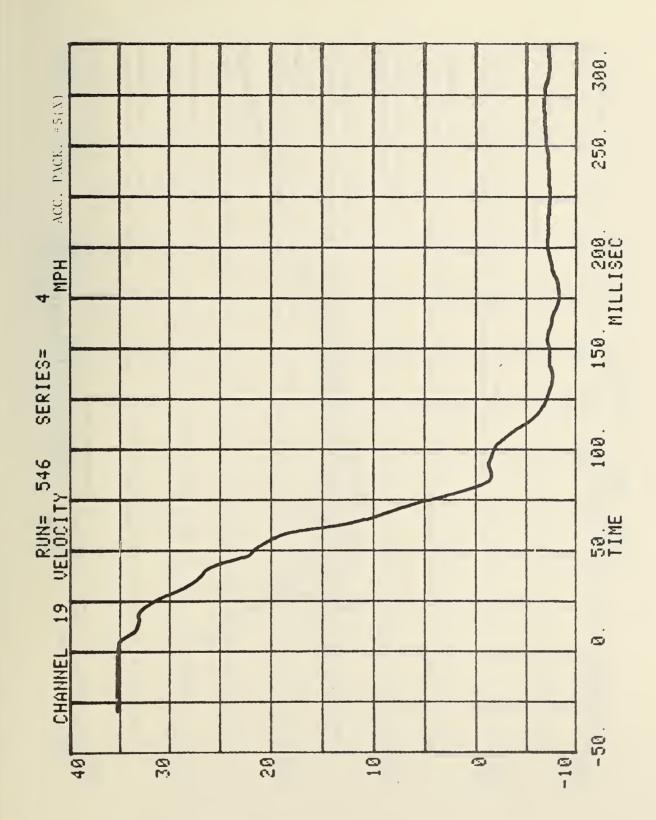


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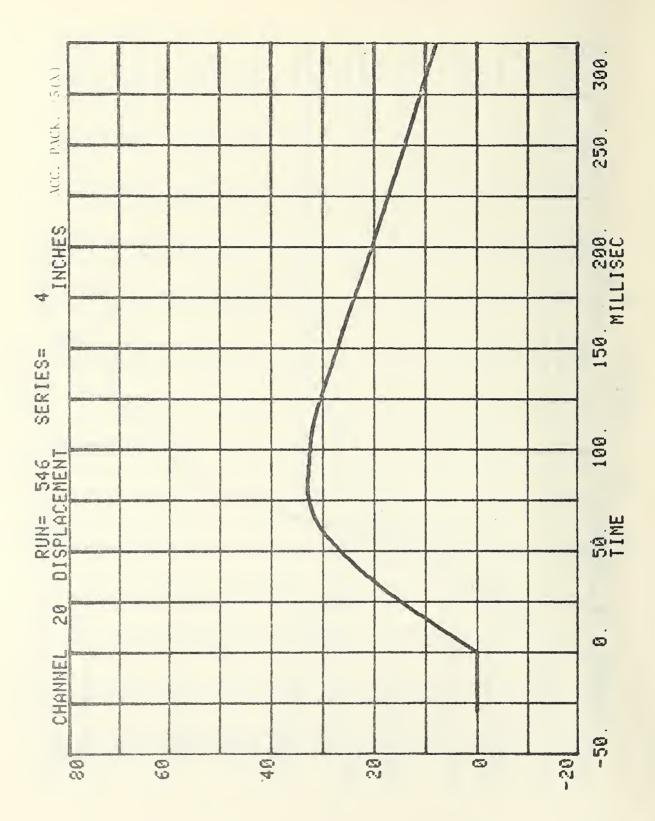


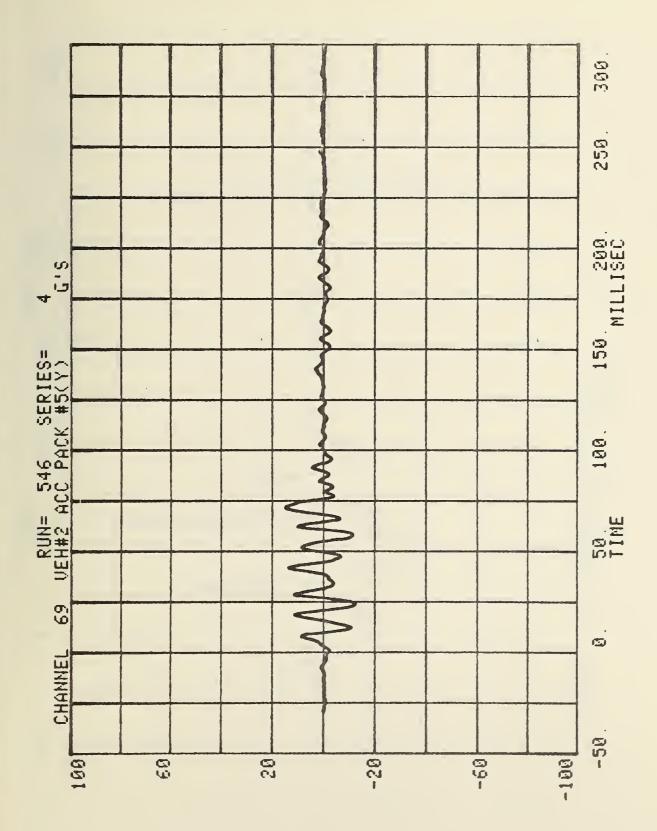


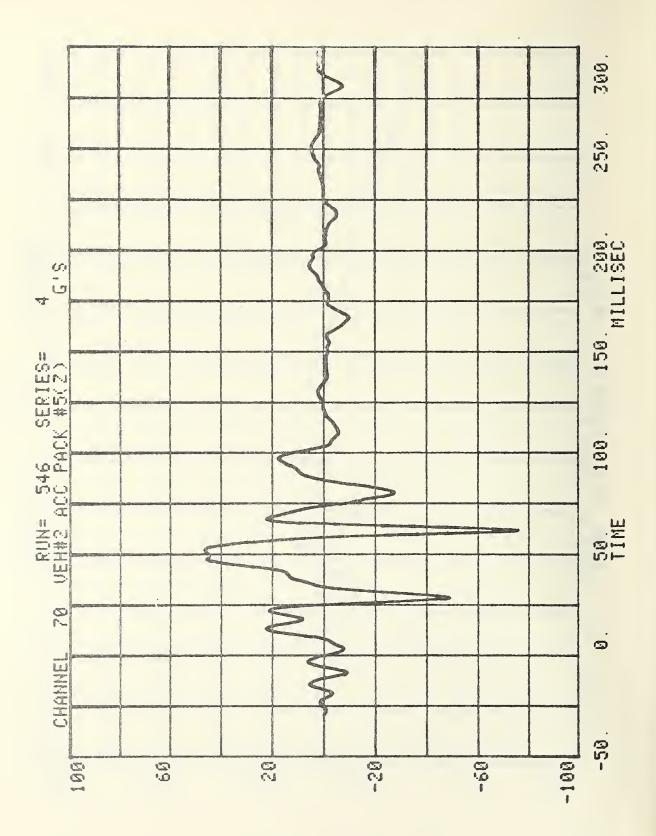


D-17

\*







## APPENDIX E ELECTRONIC CRASH TEST DATA: PLYMOUTH HORIZON OCCUPANT AND RESTRAINT SYSTEM

## HEAD INJURY CRITERION HEAD SEVERITY INDEX

IBSA CAR-TO-CAR TEST #4

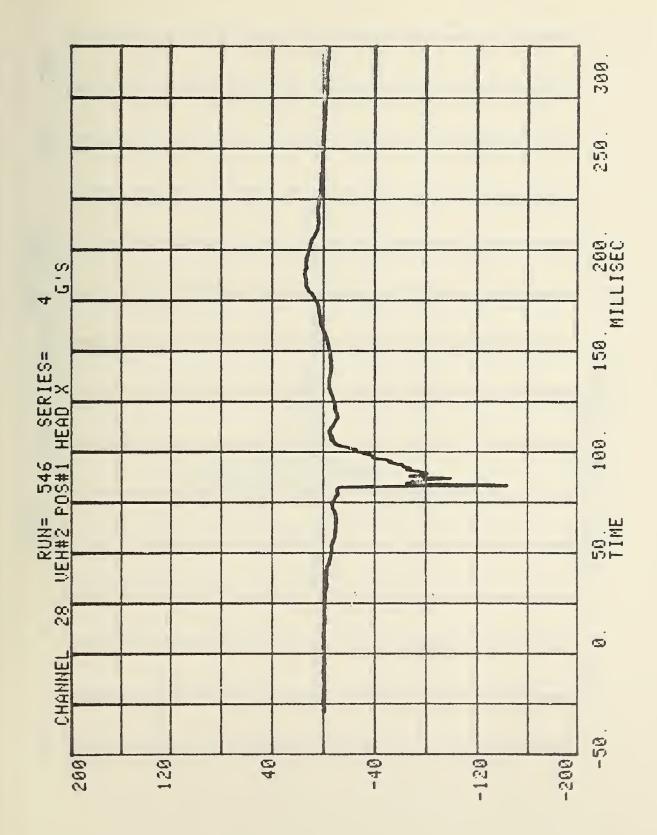
RUN= 546

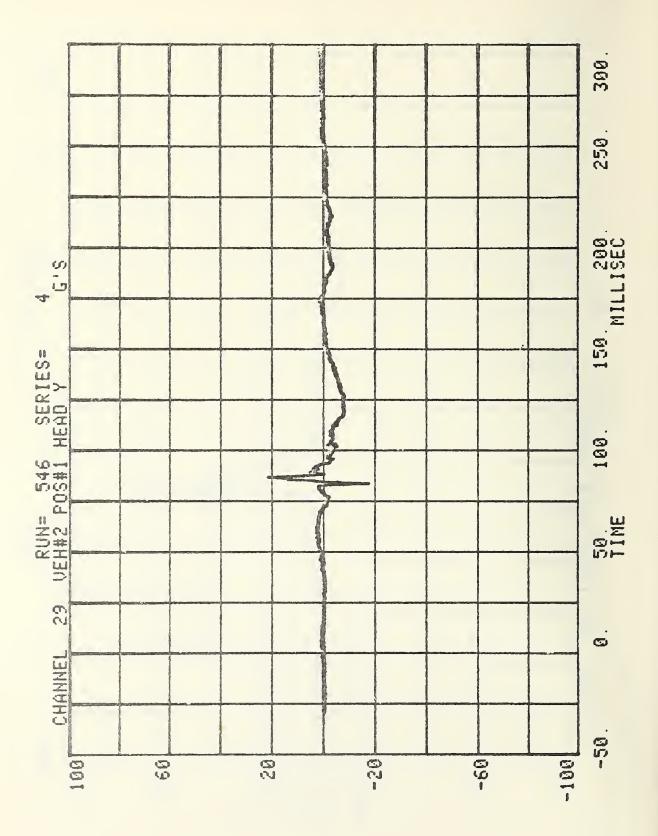
VEH#2 POS#1 HEAD R

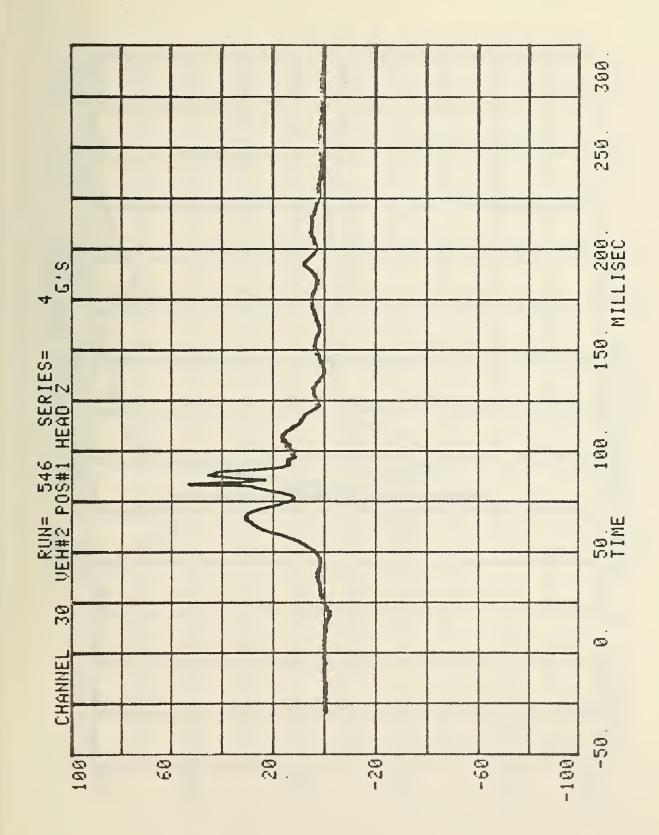
77.46'S 10 12= .09690 AUERAGE ACCELERATION BETWEEN TI AND T2= FRON T1= . 68310 HIC= 727.1

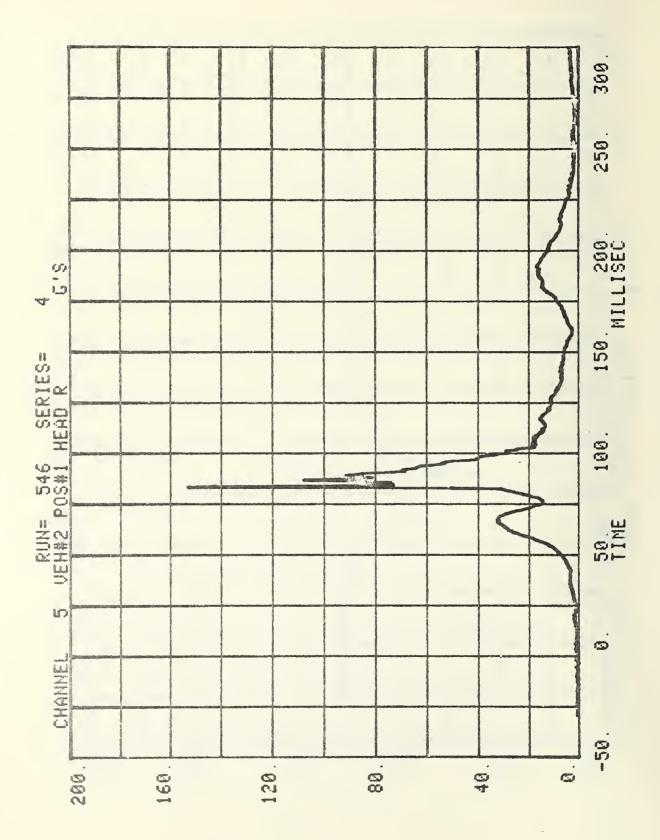
EVENT TIME= 300.0 MSEC

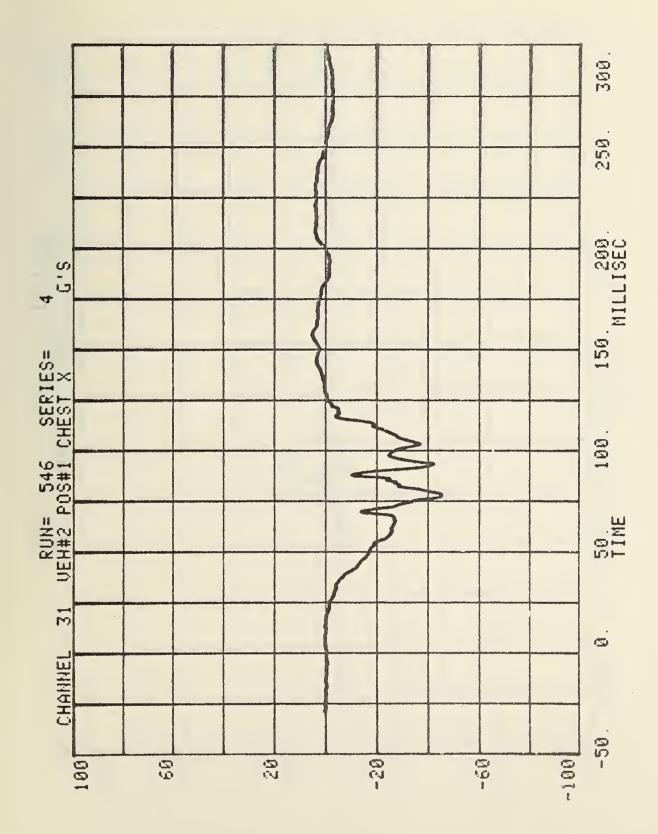
SEVERITY INDEX=1031.1

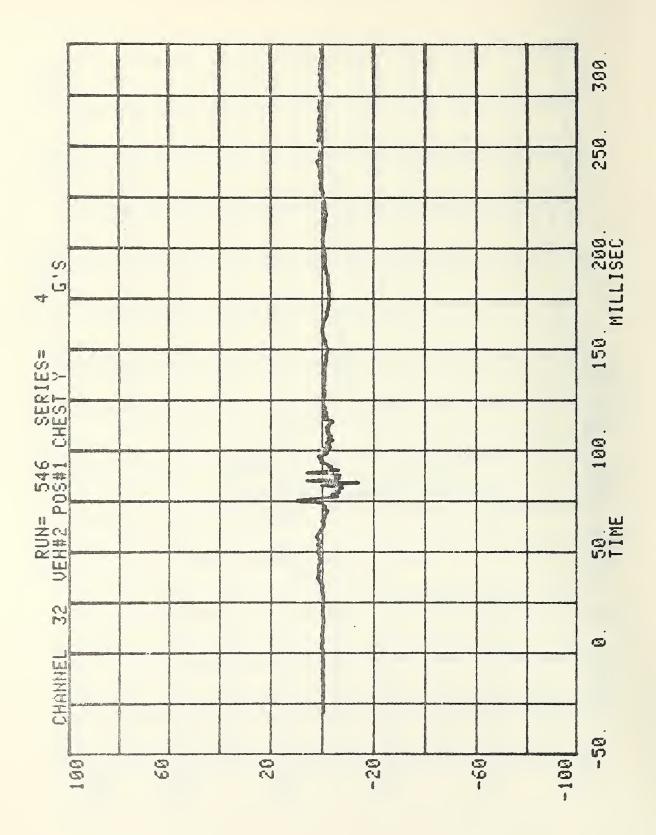


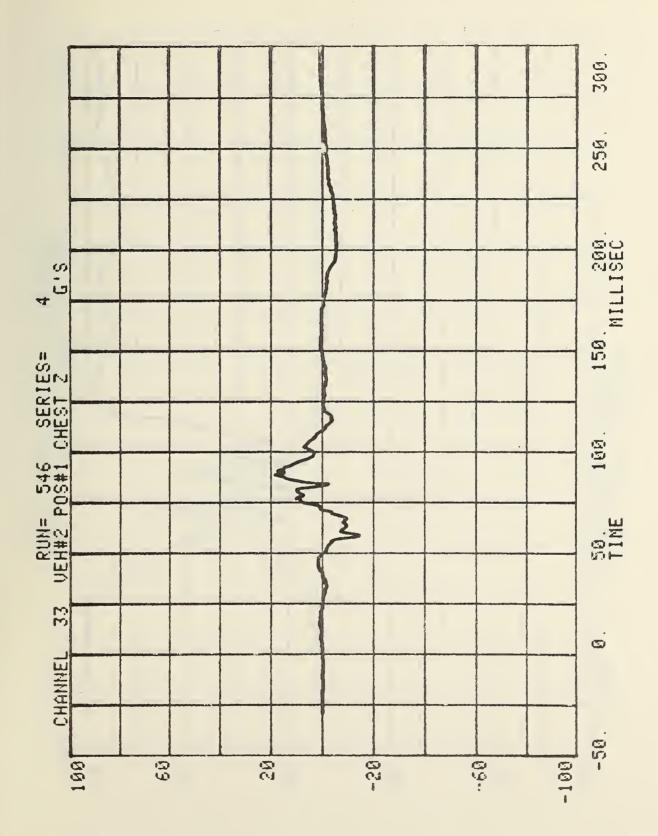


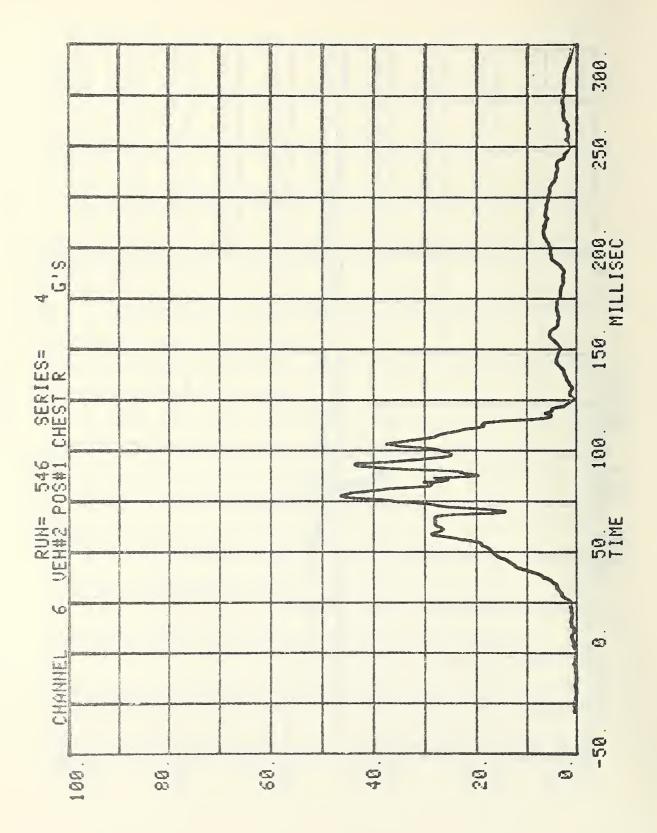


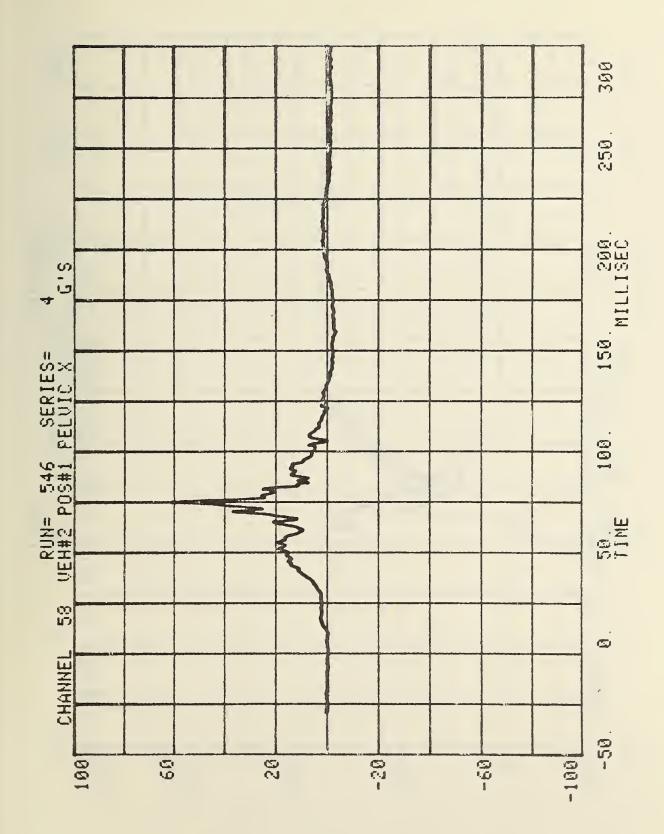


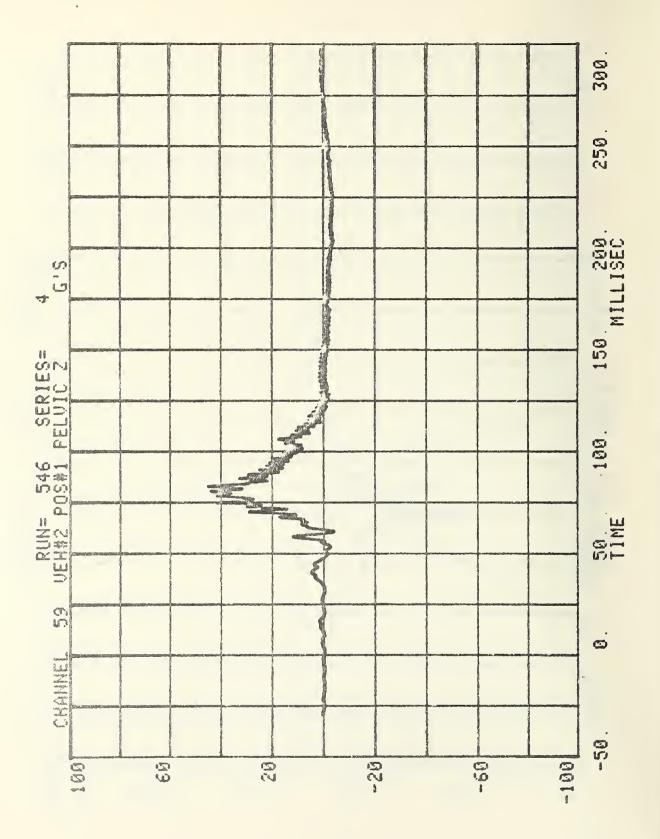


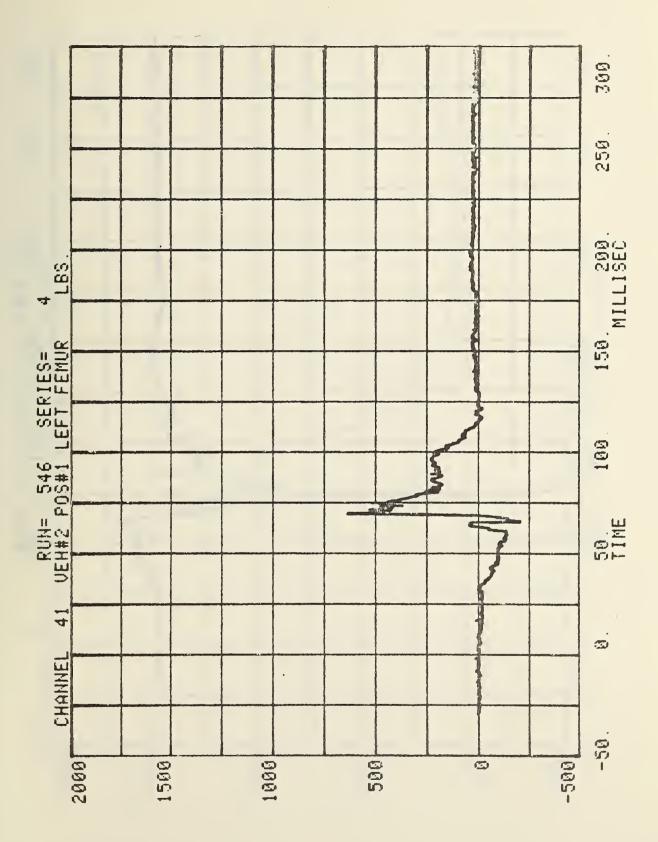


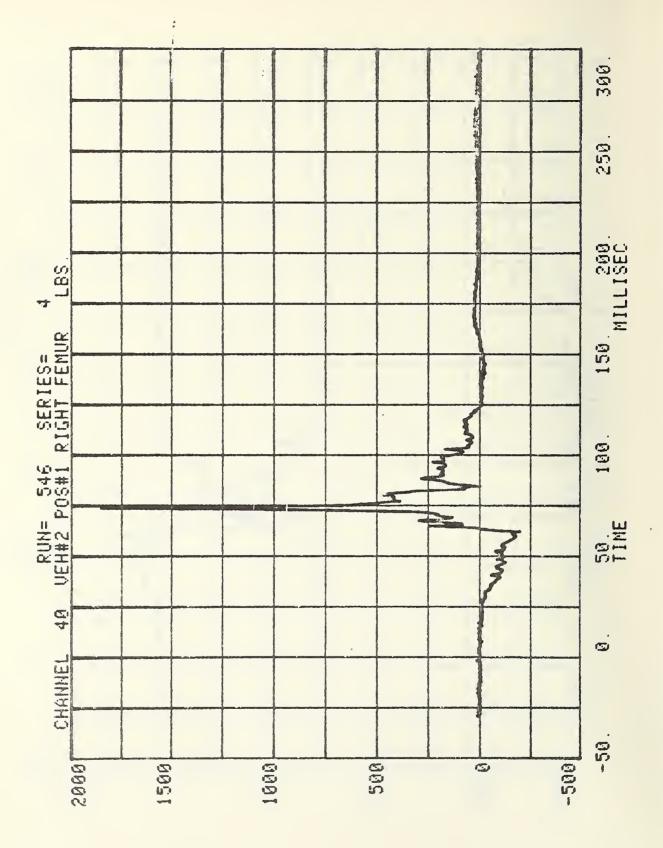


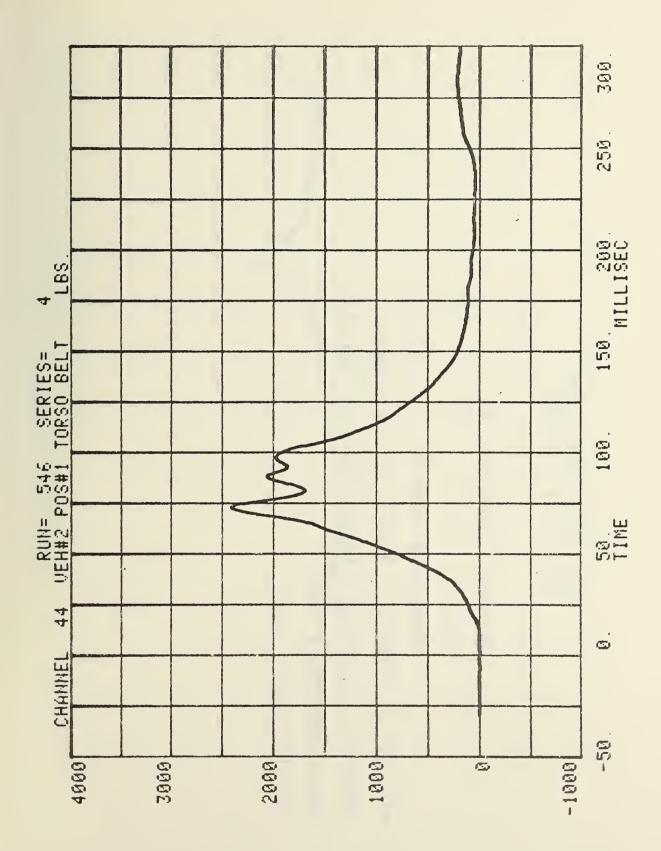










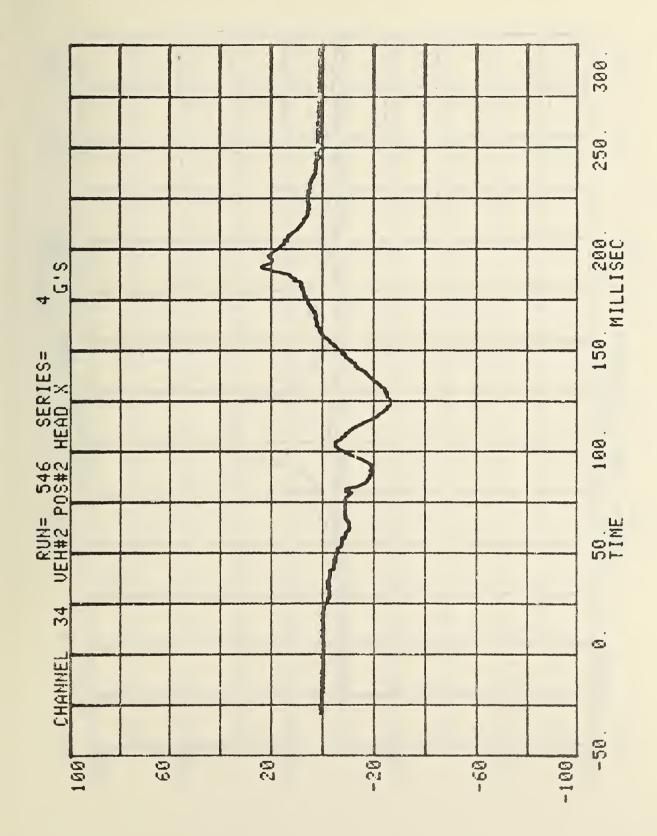


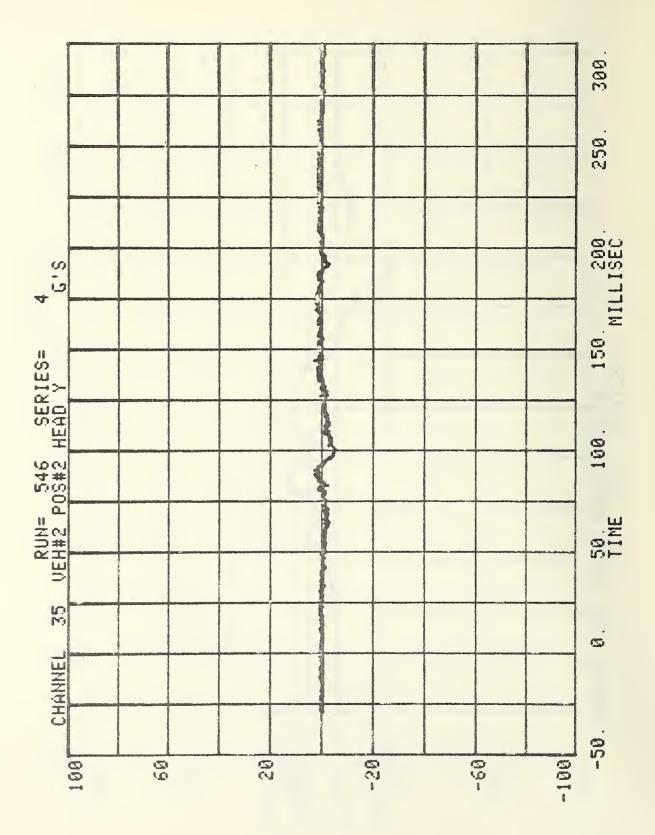
## HEAD INJURY CRITERION HEAD SEVERITY INDEX

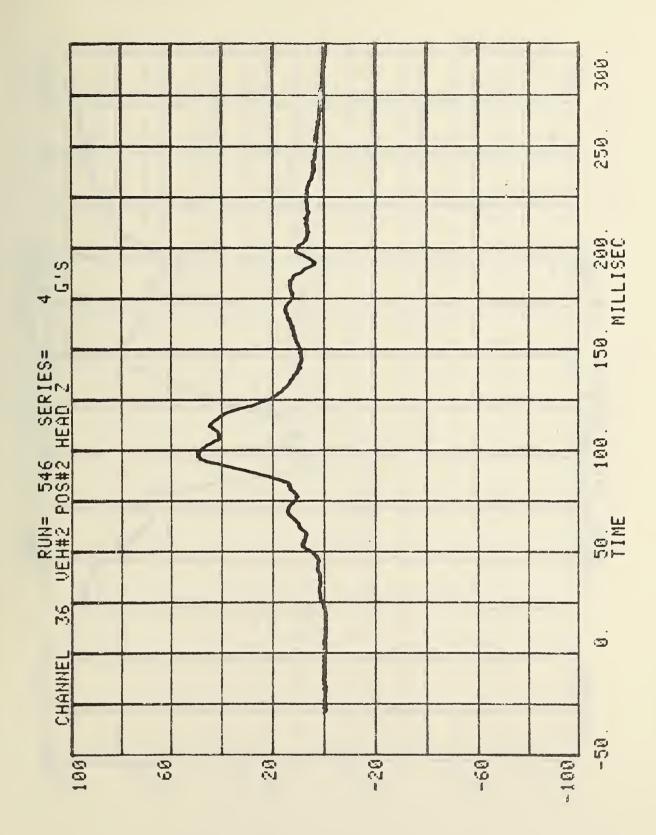
IBSA CAR-TO-CAR TEST #4 RUN= 546

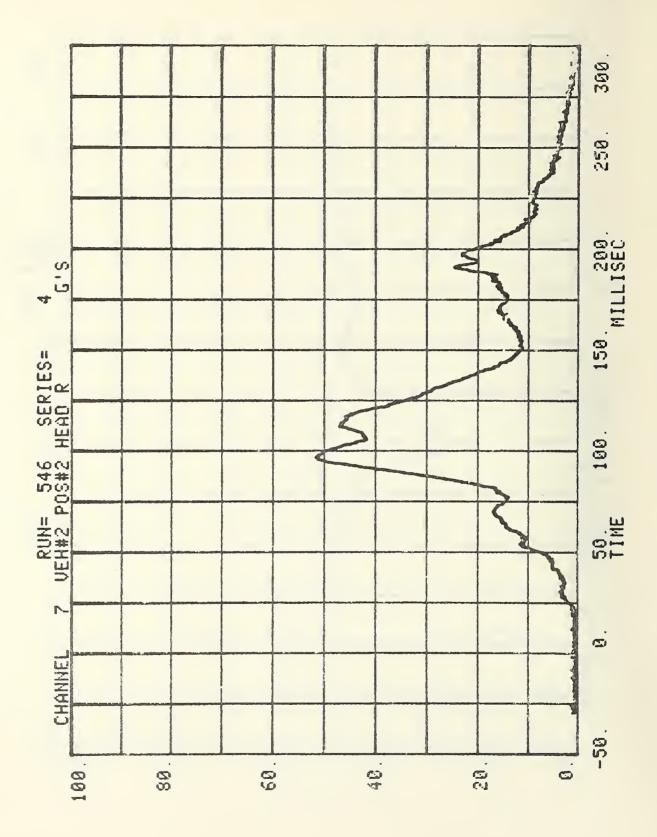
UEH#2 POS#2 HEAD R

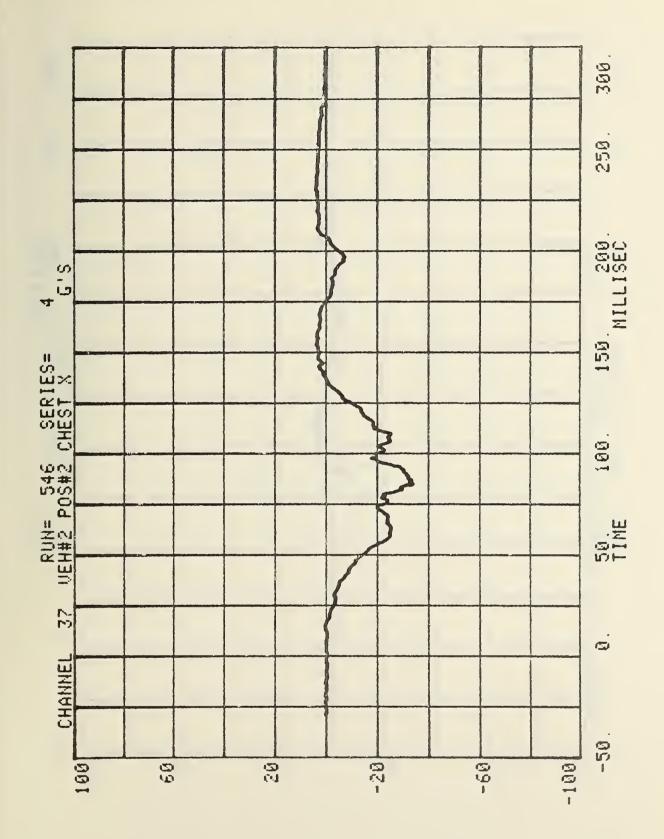
40.16'S TO T2= .13560 AVERAGE ACCELERATION BETWEEN T1 AND T2= FROM 71= .03556 MSEC SEVERITY INDEX= 673.2 EVENT TIME= 300.0 HIC= 510.4

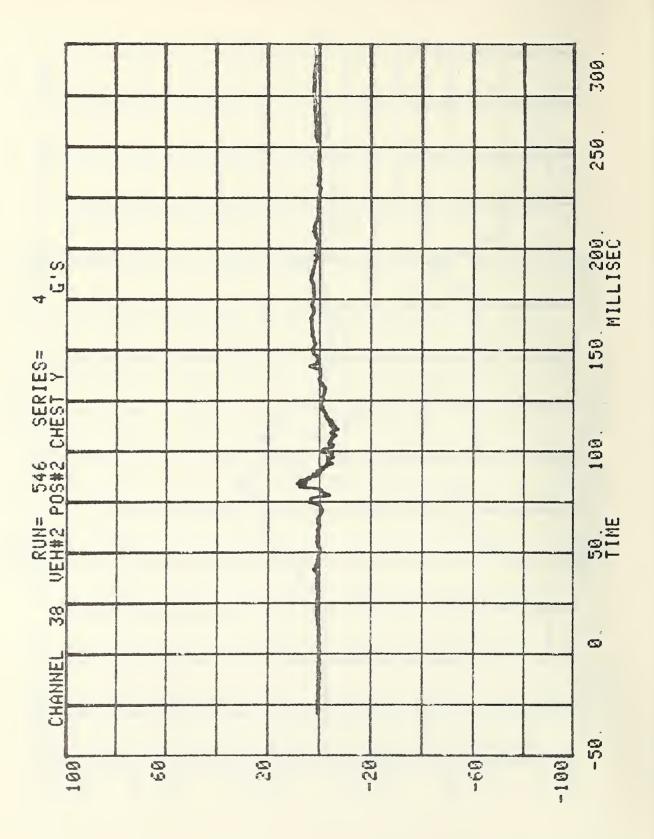


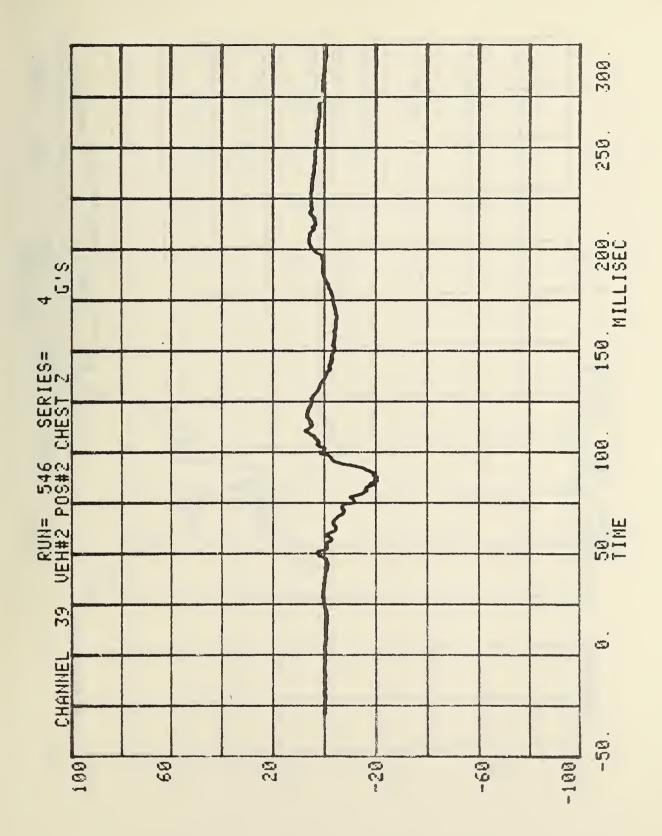


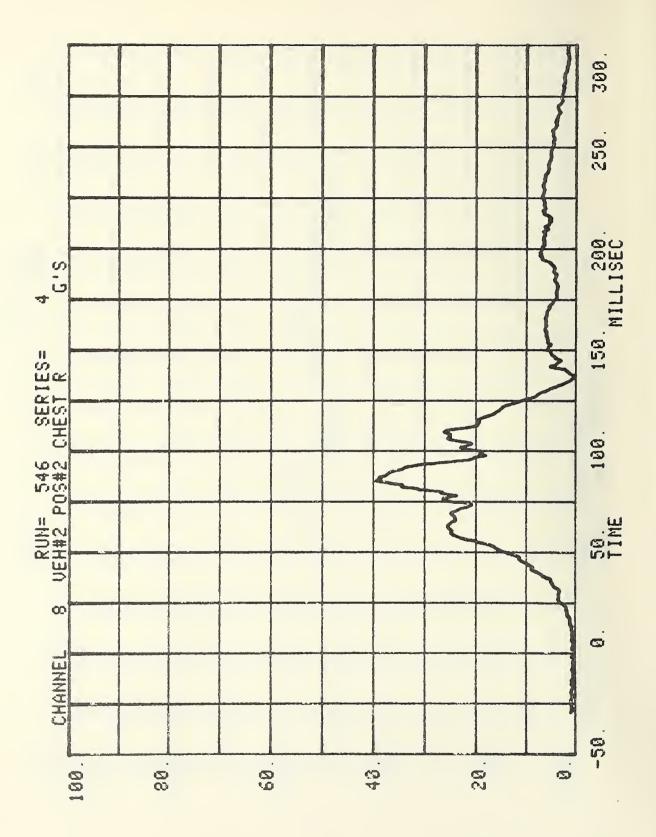


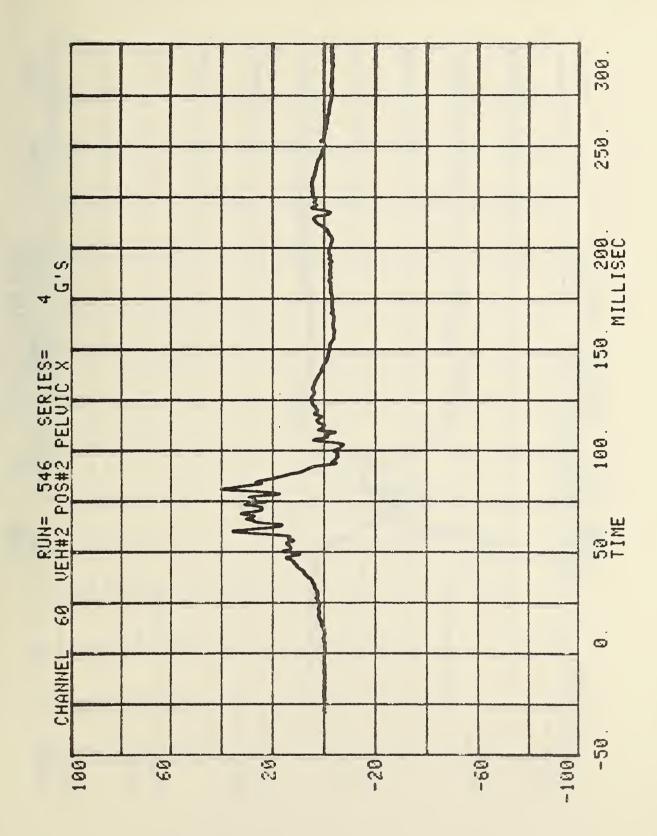


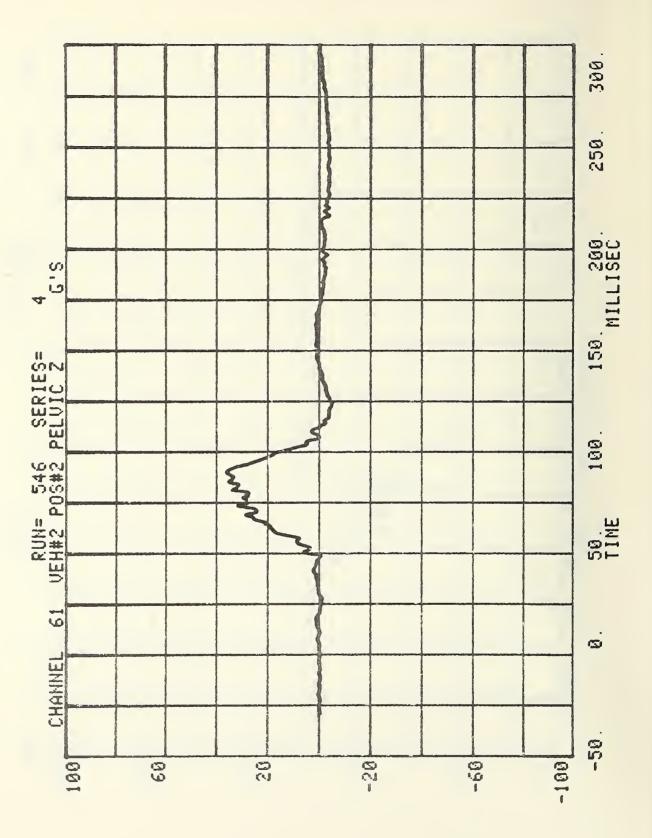


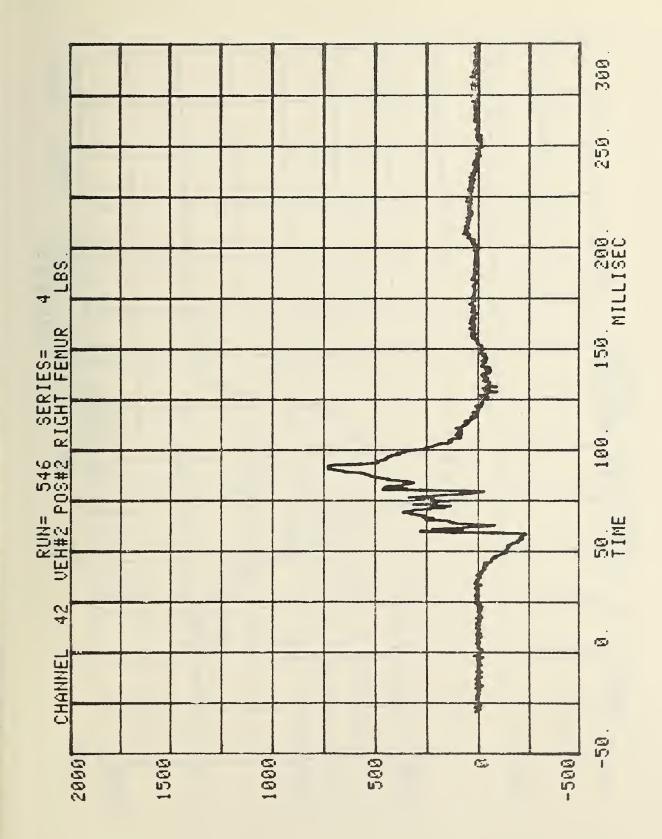


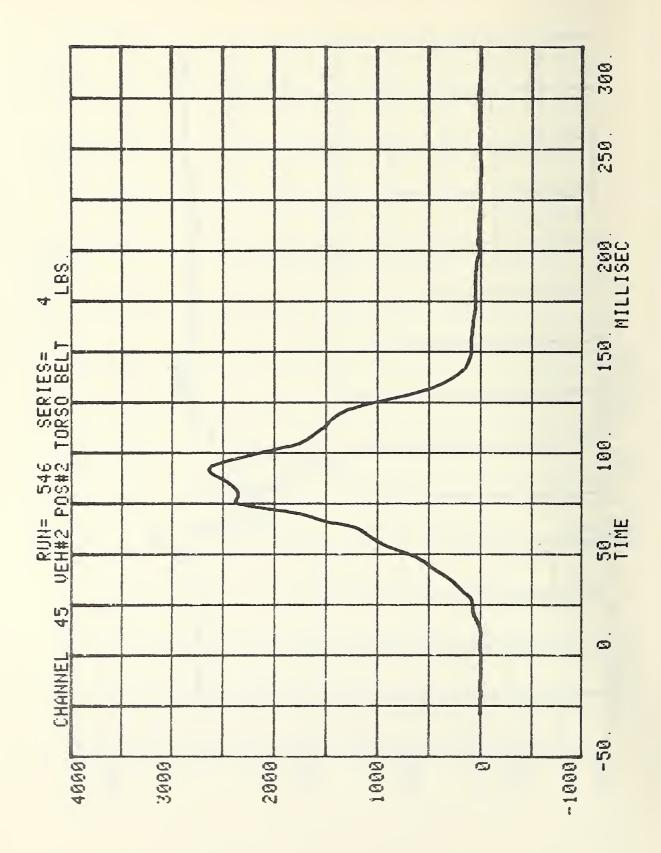












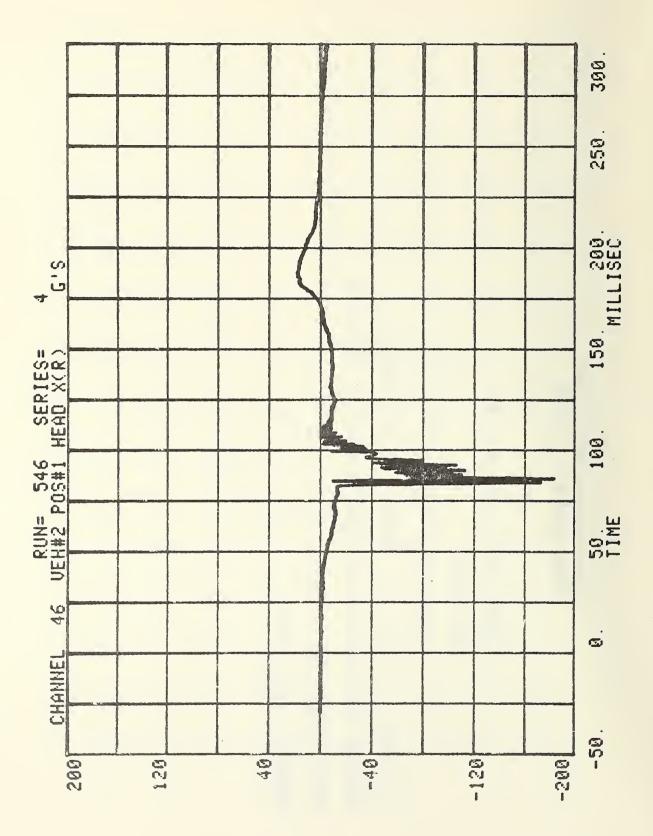
HEAD INJURY CRITERION HEAD SEVERITY INDEX

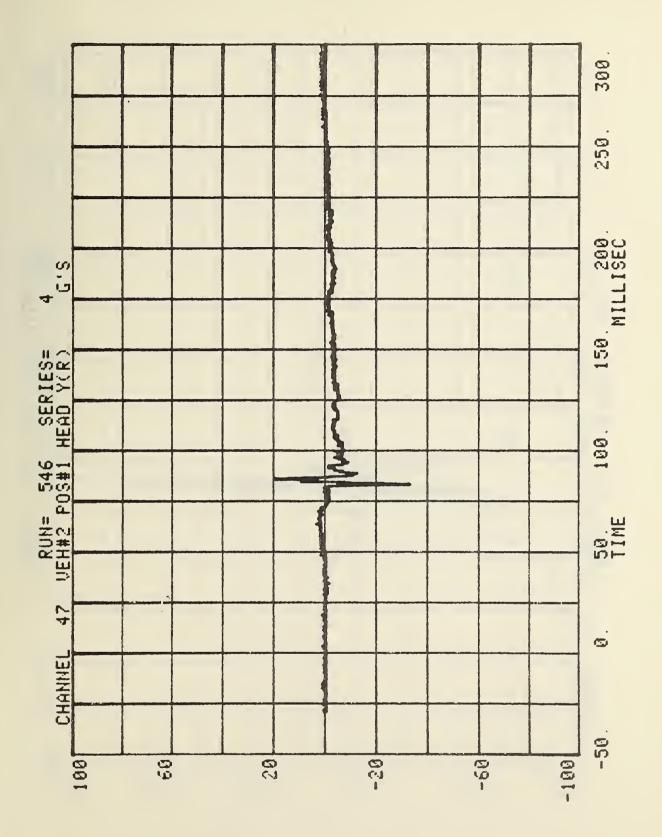
IBSA CAR-TO-CAR TEST #4 RUN= 546 UEH#2 POS#1 HEAD R(R)

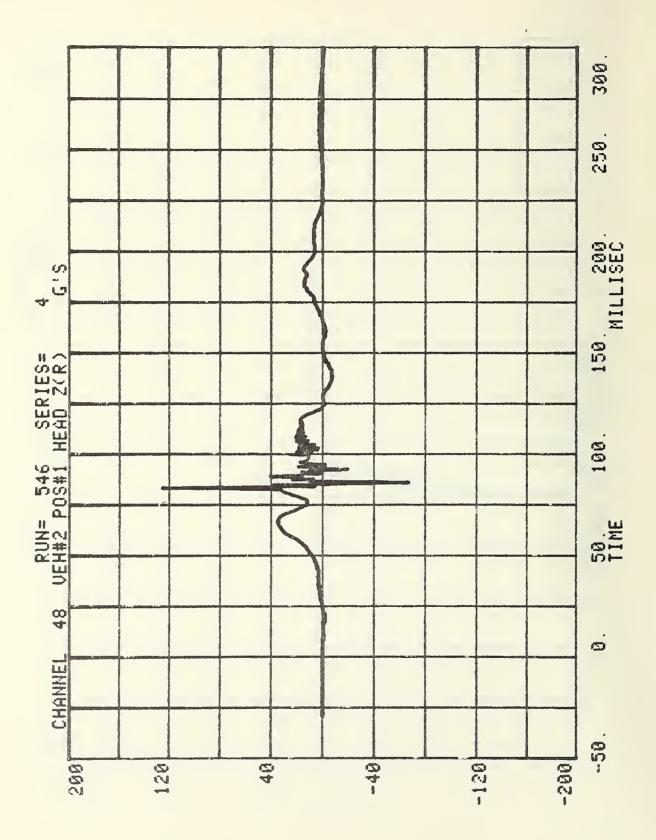
87.8G'S 10 12= .09600 AVERAGE ACCELERATION BETWEEN TI AND T2= FROM T1= .03280 HIC= 952.6

EVENT TIME= 300.0 MSEC

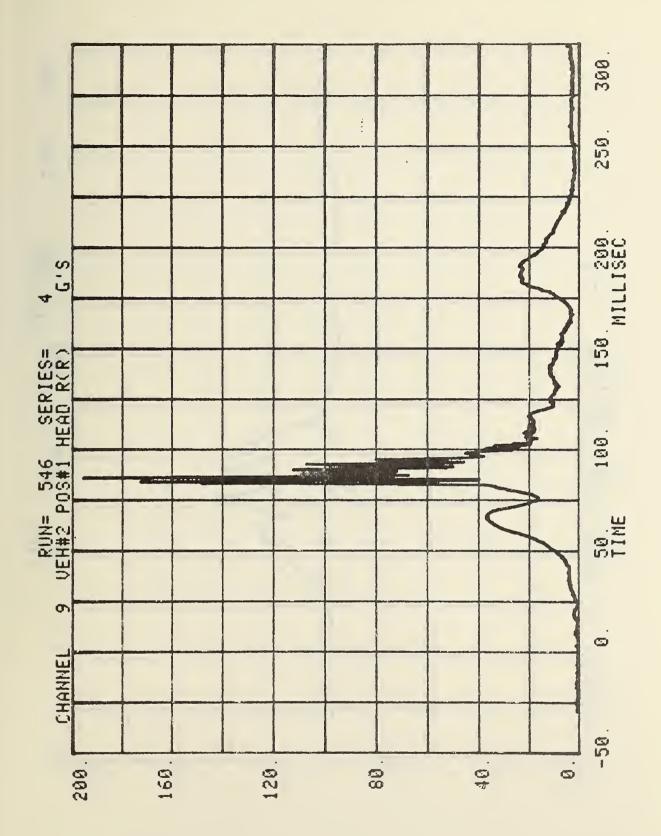
SEVERITY INDEX=1588.9

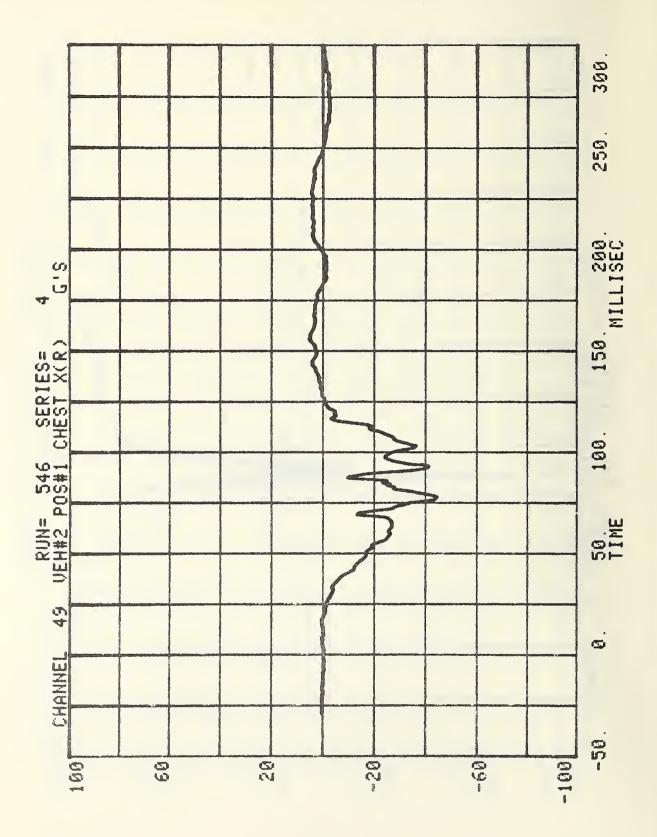


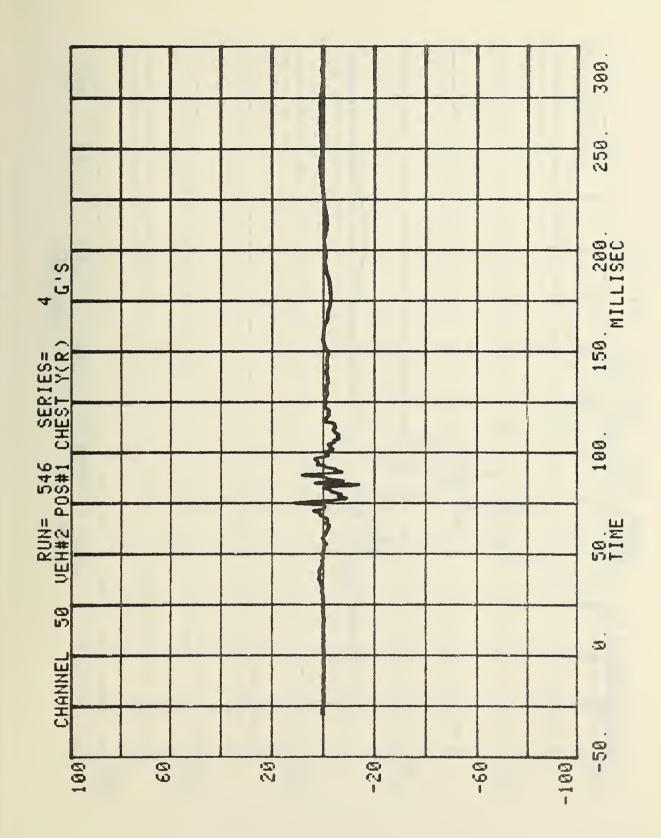


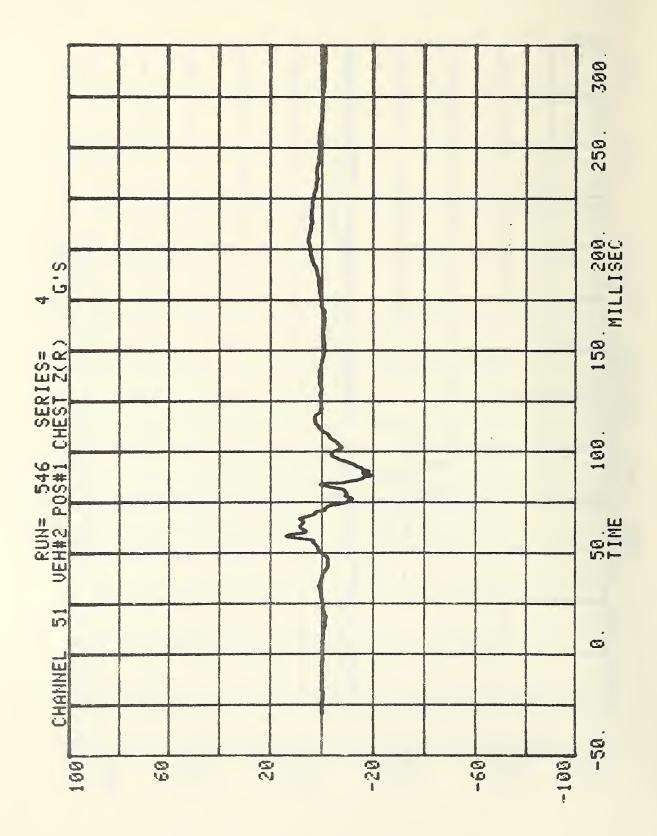


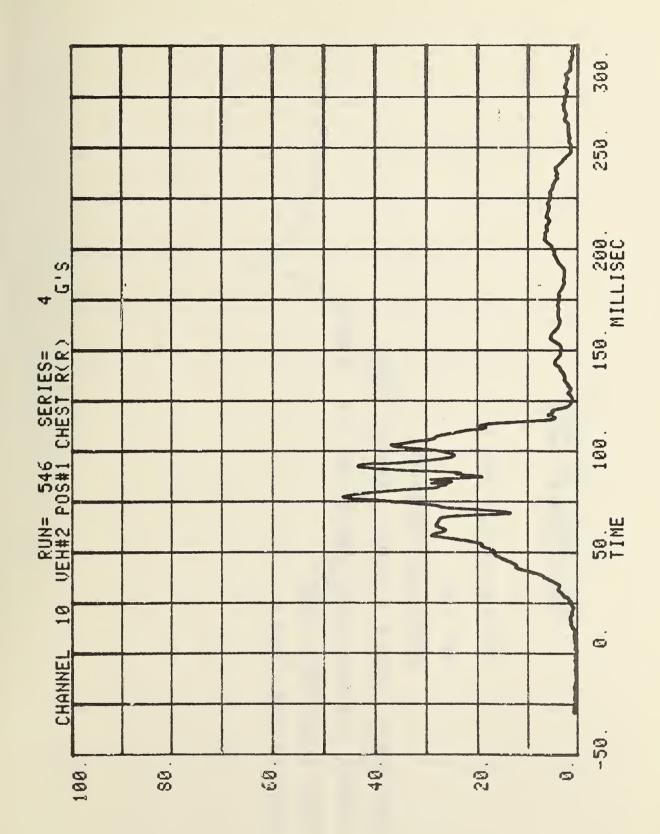
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## HEAD INJURY CRITERION HEAD SEVERITY INDEX

IBSA CAR-TO-CAR TEST #4

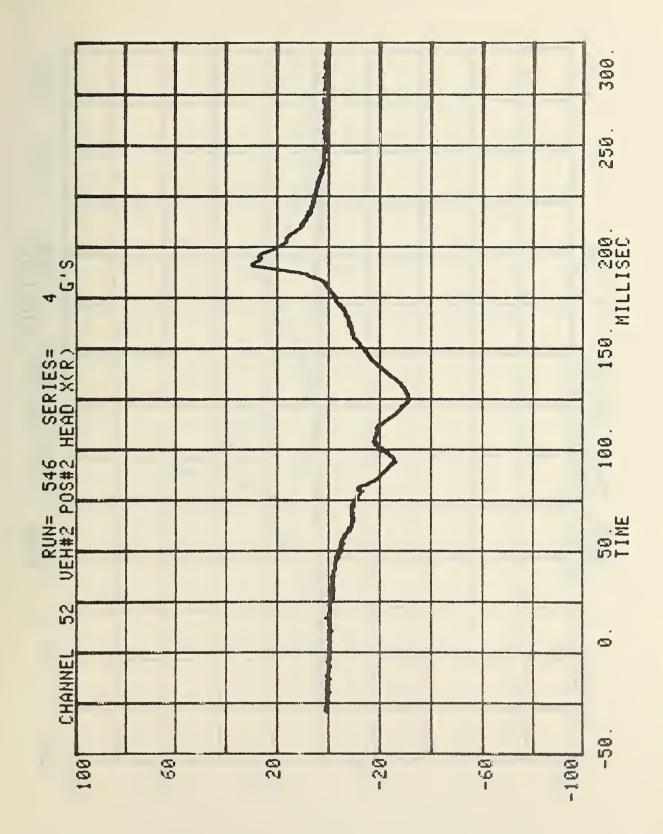
RUN= 546

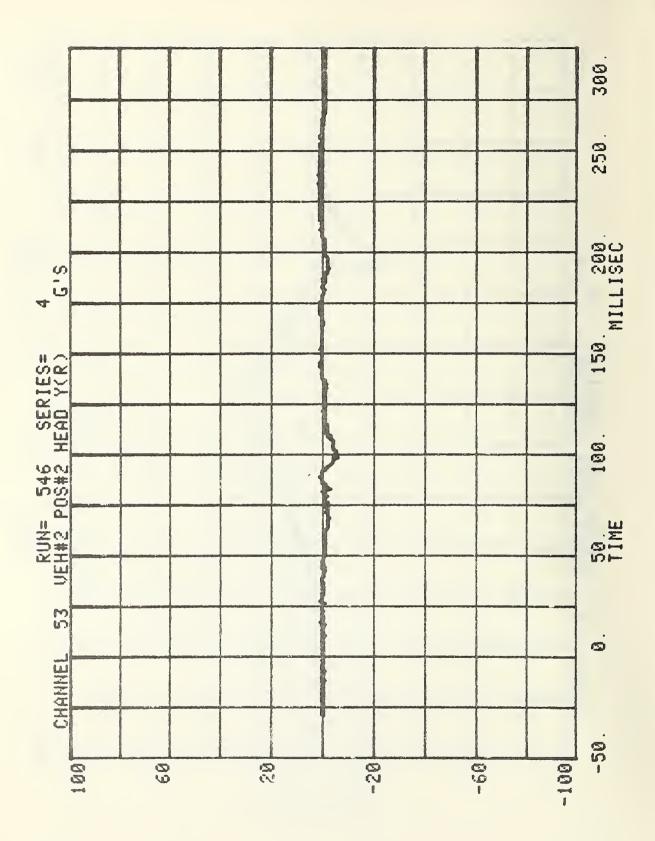
VEH#2 POS#2 HEAD R(R)

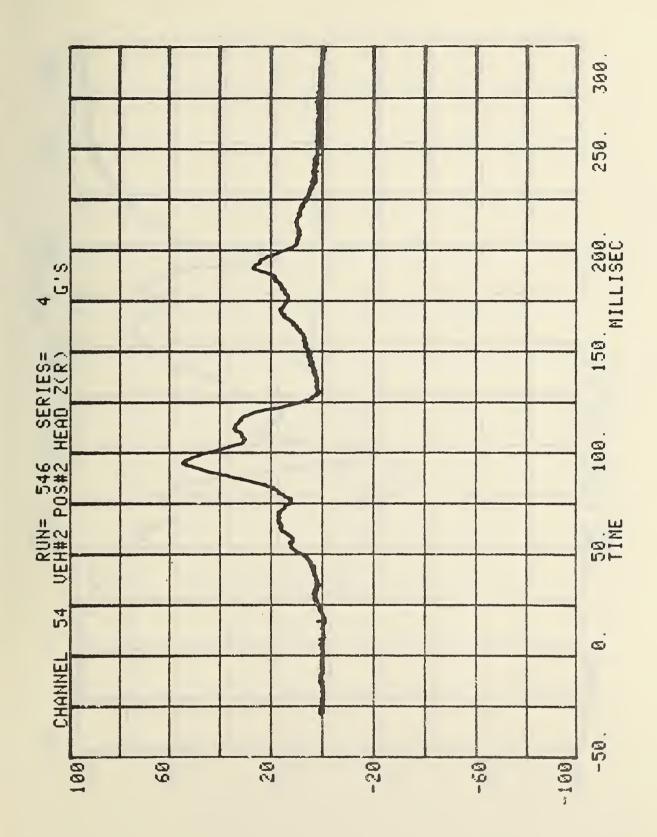
HIC= 540.5 FROM TI= .06030 TO T2= .20790

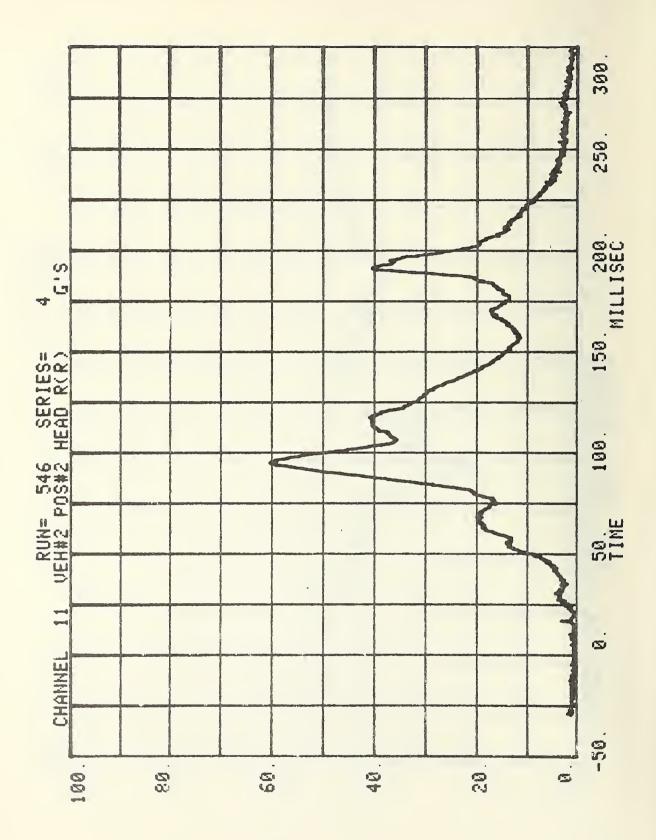
26.6615 AVERAGE ACCELERATION BETWEEN TI AND T2= EVENT TIME= 300.0 MSEC

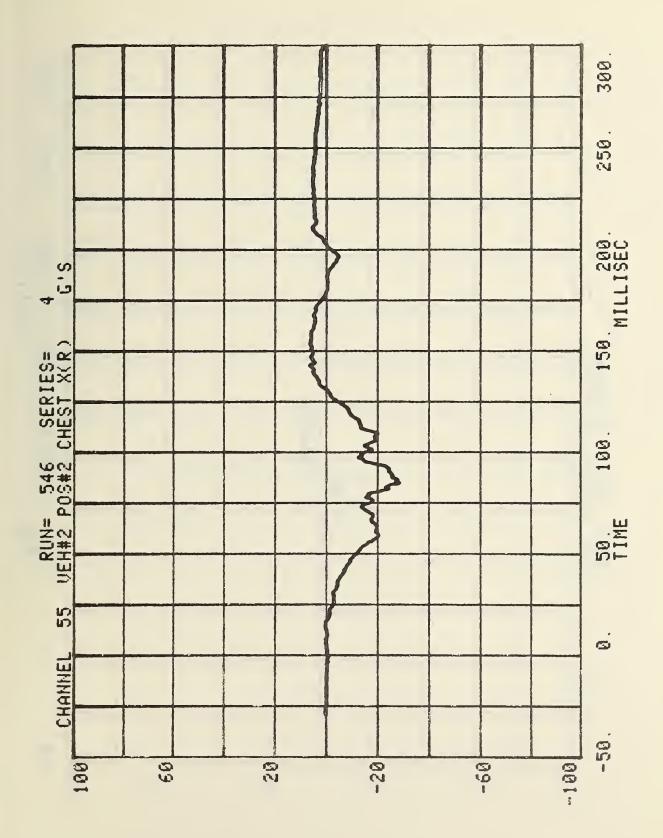
SEVERITY INDEX= 792.0

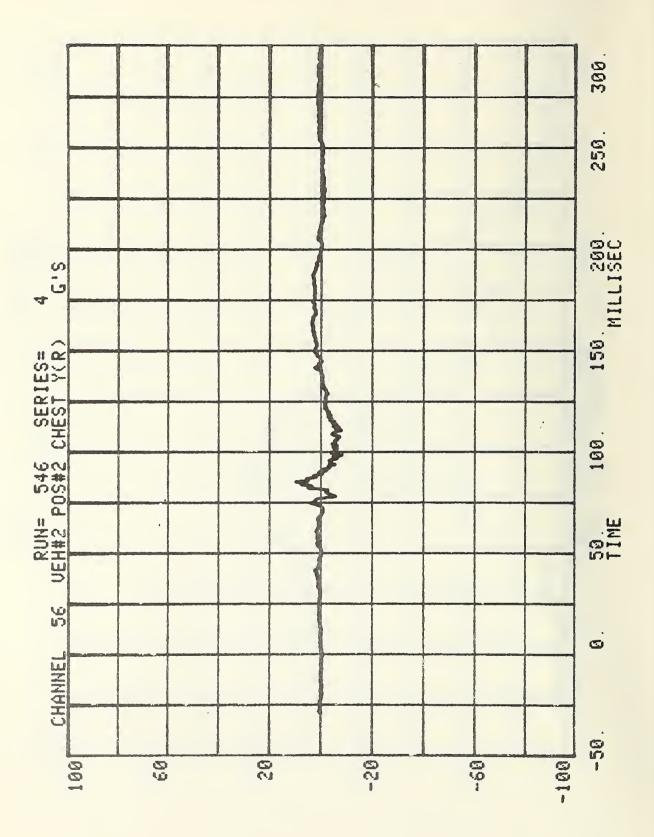


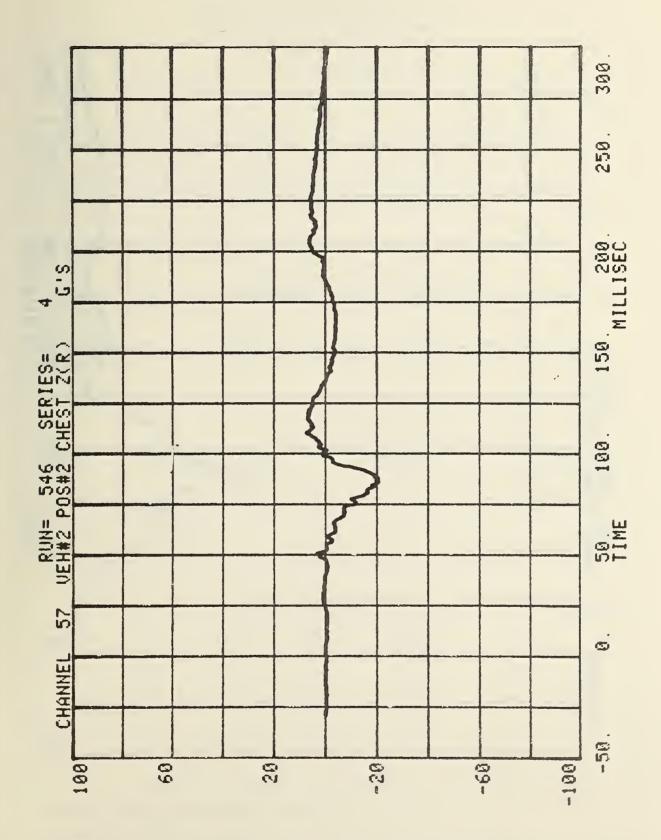


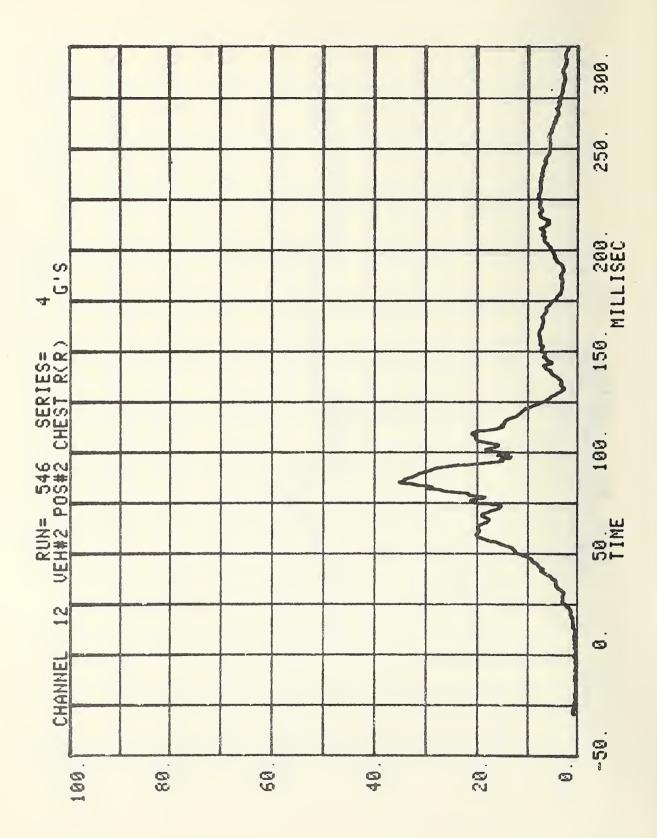












## PLYMOUTH HORIZON OCCUPANT AND RESTRAINT SYSTEM DATA SUMMARY

	MAXIMUM ACCELERATION (g)											
DUMMY	HEAD				CHEST <sup>1</sup>			PELVIS				
POSITION	Х	Y	Z	R	Х	Y	Z	R	Х	Y	Z	R
Driver $^{3}$ (1)	-144	21	53	153	-42	-7	15	44	60	-	45	-
	-184	-33	125	195	-42	-7	-16	43				
Passenger <sup>3</sup> (2)	-27	-5	49	52	-33	6	-20	38	40	-	36	-
	-31	-6	54	60	-26	-6	-20	33				

	MAXIMUM FORCE-FEMUR LOAD (LBS)					
DUMMY POSITION	RIGHT FEMUR	LEFT FEMUR				
Driver (1)	1850	630				
Passenger(1)	740	NA <sup>4</sup>				

	MAXIMUM FORCE-SEAT BELT LOADS (LBS)						
DUMMY POSITION	SHOULDER STRAP UPPER BELT LOAD	LAP STRAP RIGHT BELT LOAD	LAP STRAP LEFT BELT LOAD				
Driver (1)	2410						
Passenger(2)	2630						

		HEAD 'N	SEVERITY INDEX			
DUMMY POSITION	HIC	t <sub>1</sub> (SEC)	t <sub>2</sub> (SEC)	AVE. ACC. (g) t <sub>1</sub> <sup>TO t</sup> 2	HEAD	CHEST
Driver <sup>3</sup> (1)	727	0.083	0.097	77.4	1031	
	953	0.083	0.096	87.8	1589	
Passenger <sup>3</sup> (2)	510	0.086	0.136	40.1	673	
	541	0.060	0.208	26.6	792	

<sup>1</sup>DEFINED AS EXCEEDING 0.003 SEC. DURATION

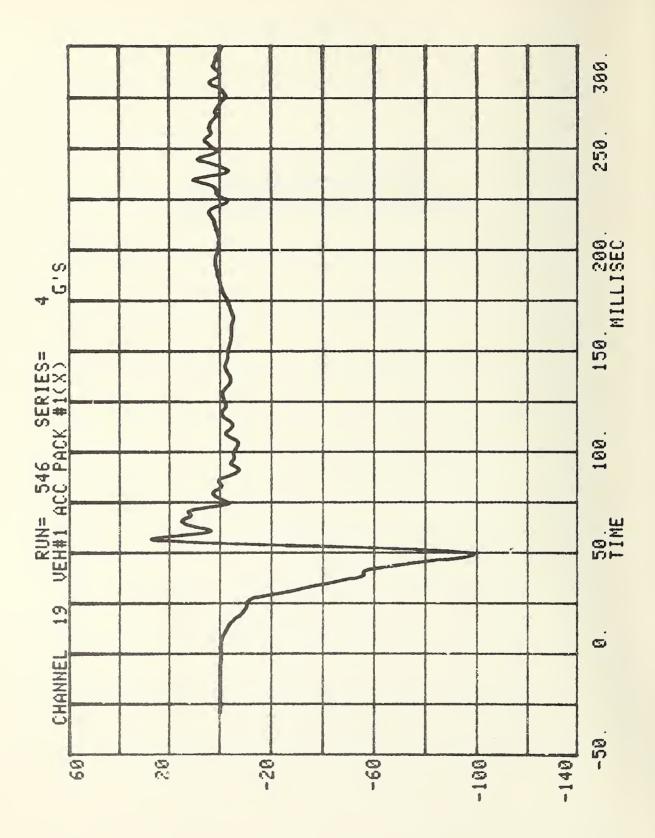
AS DEFINED IN FMVSS NO. 208

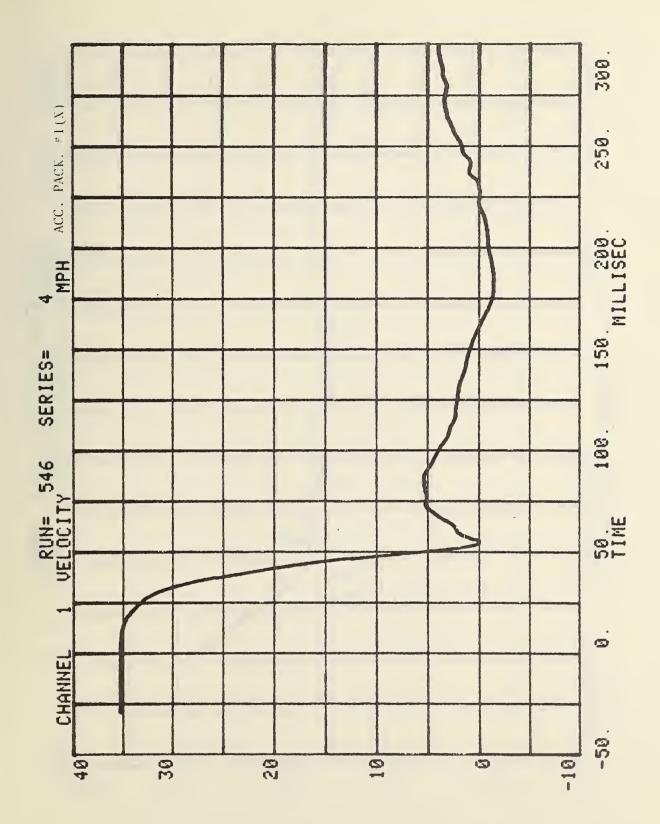
<sup>3</sup>VALUES ON FIRST AND SECOND LINES REFLECT PRIMARY AND REDUNDANT ACCELERATION MEASUREMENTS, PESPECTIVELY, FOR THE SAME DUMMY.

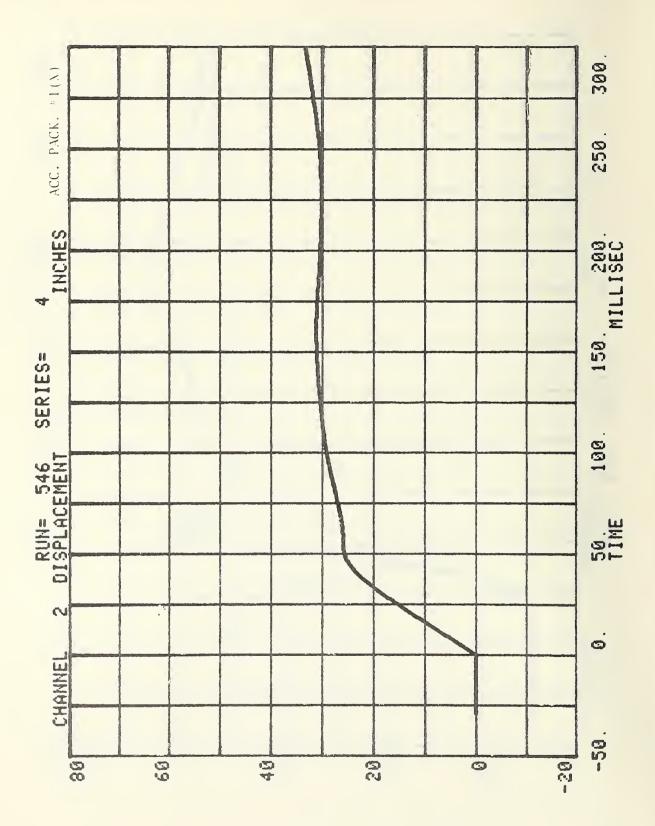
<sup>4</sup>NOT AVAILABLE: WIRES PULLED OUT OF LOAD CELL.

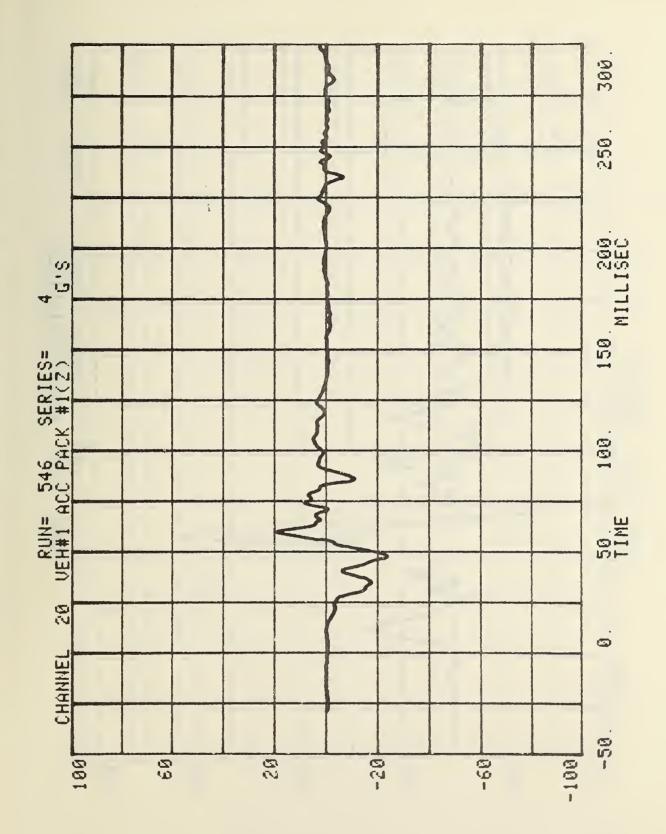


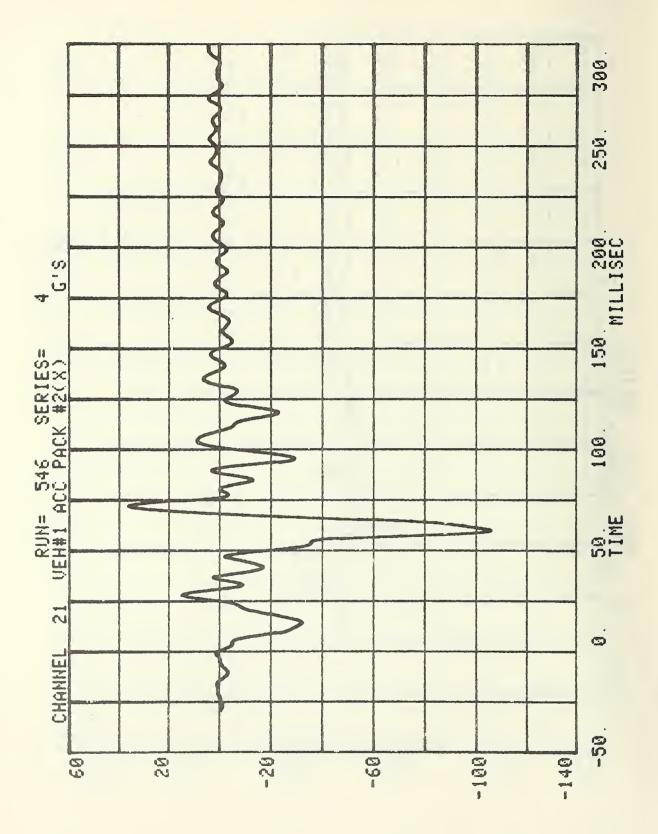
## APPENDIX F ELECTRONIC CRASH TEST DATA: FORD MUSTANG VEHICLE-MOUNTED SENSORS

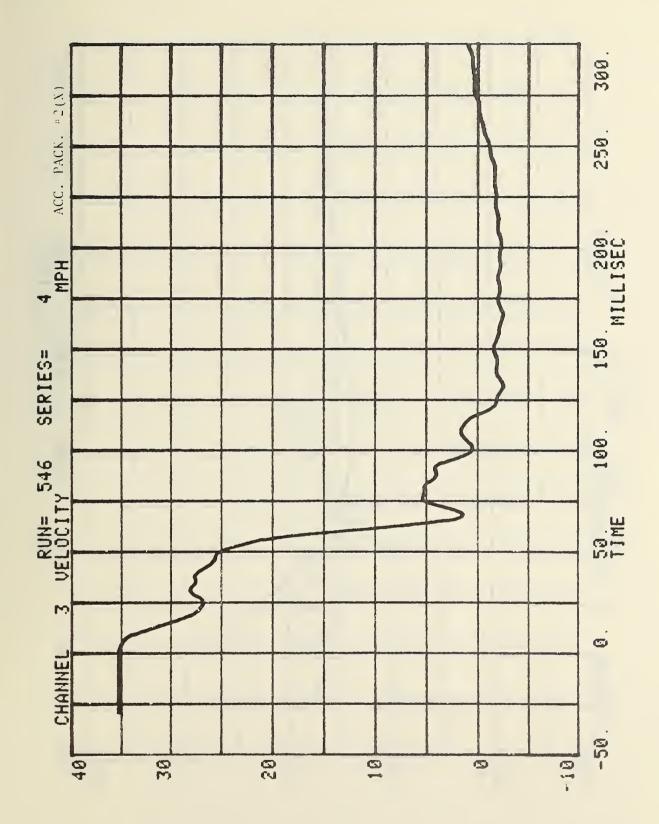


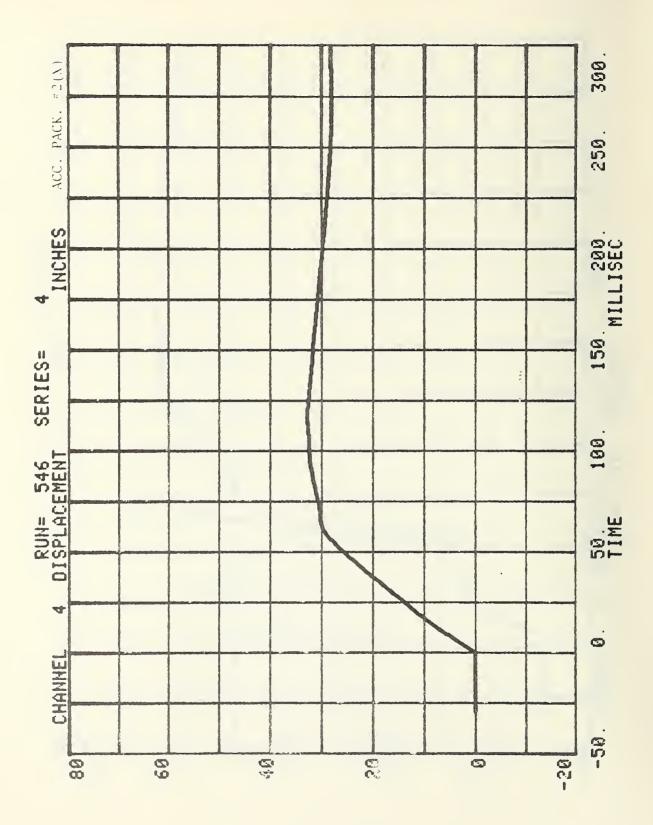


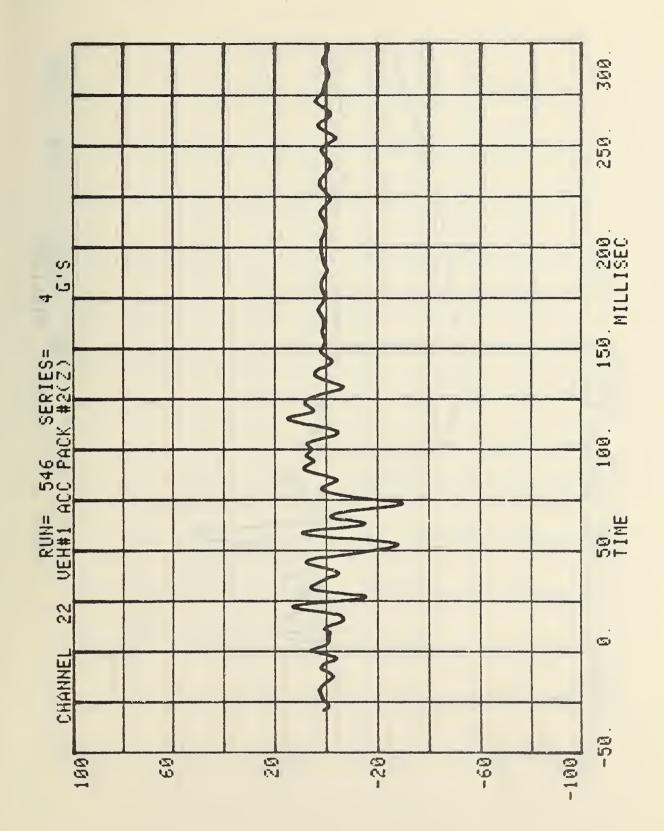


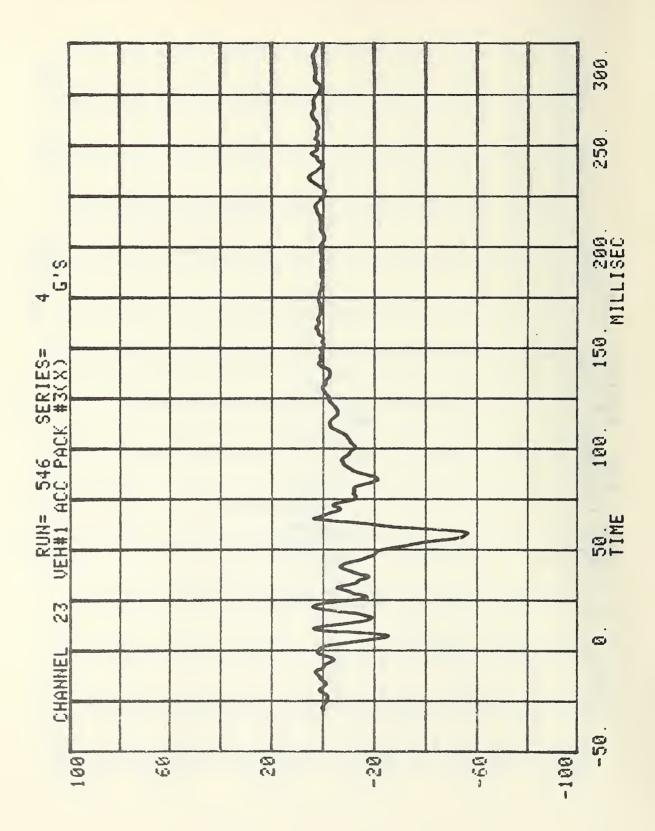




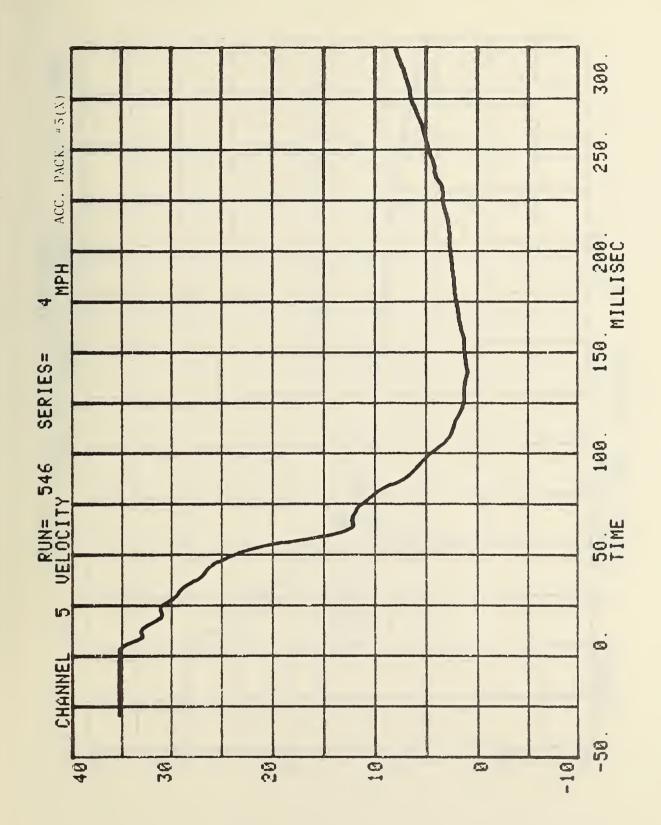


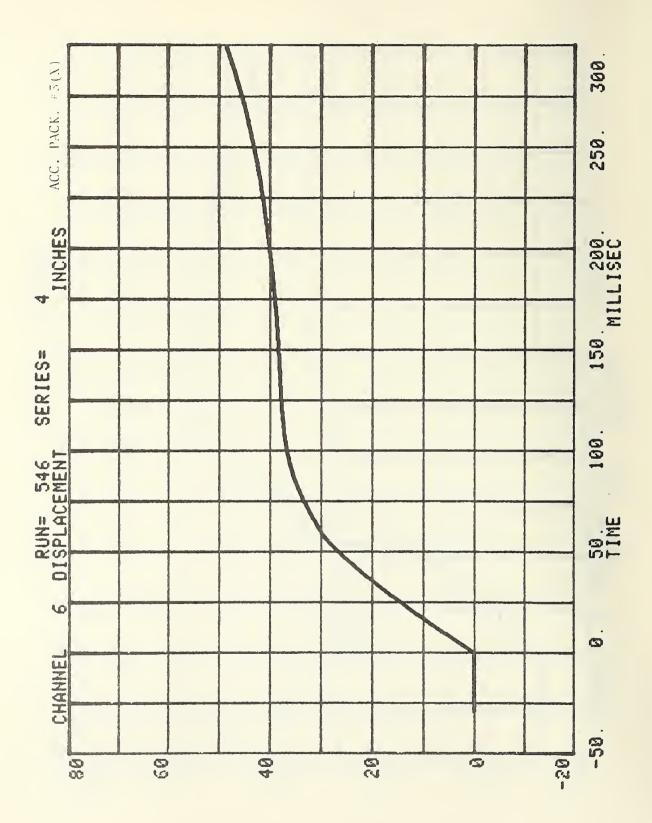


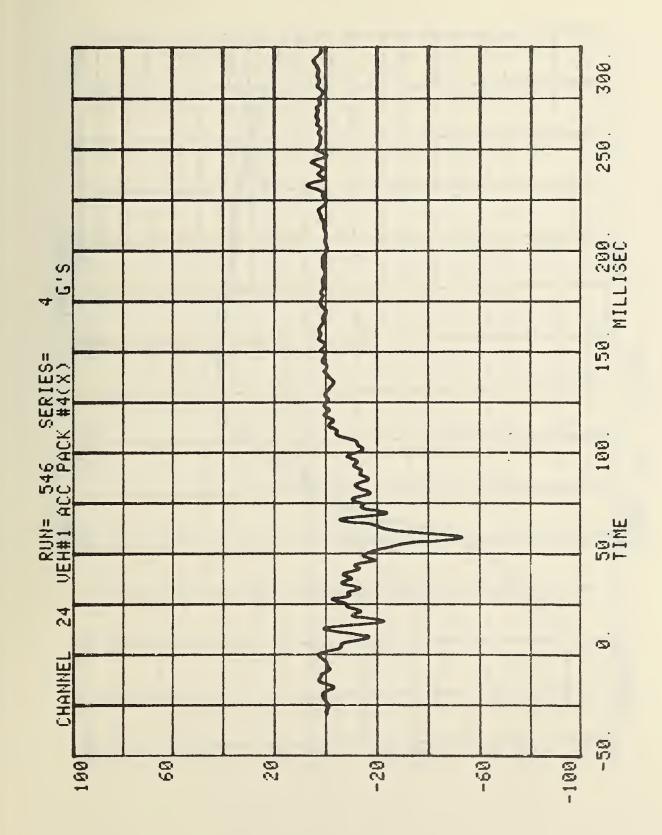


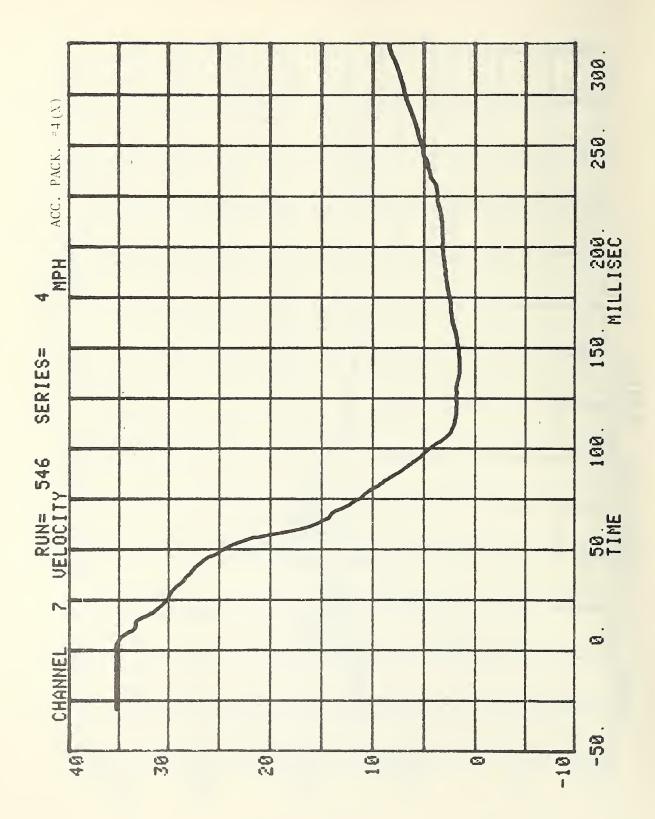


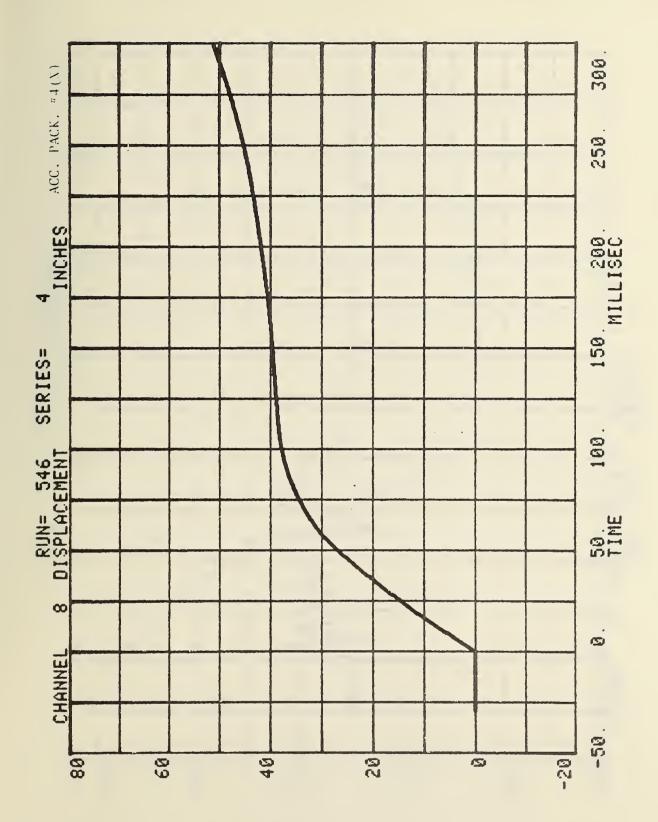
3329-V-8

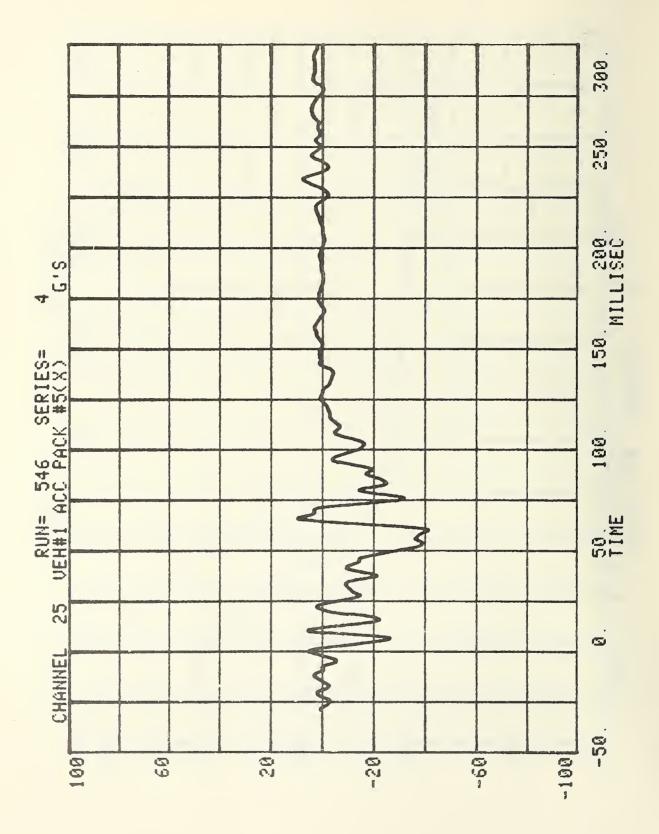


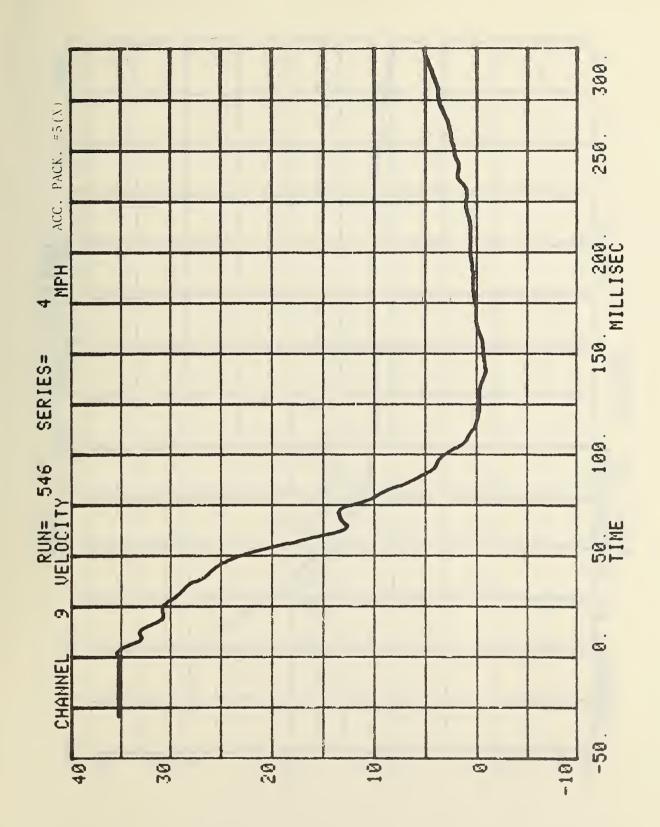


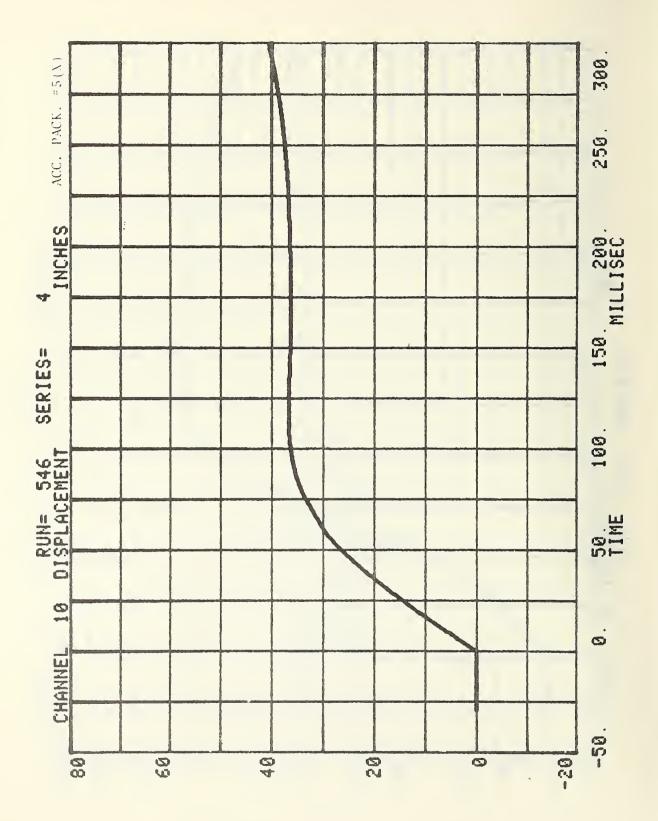


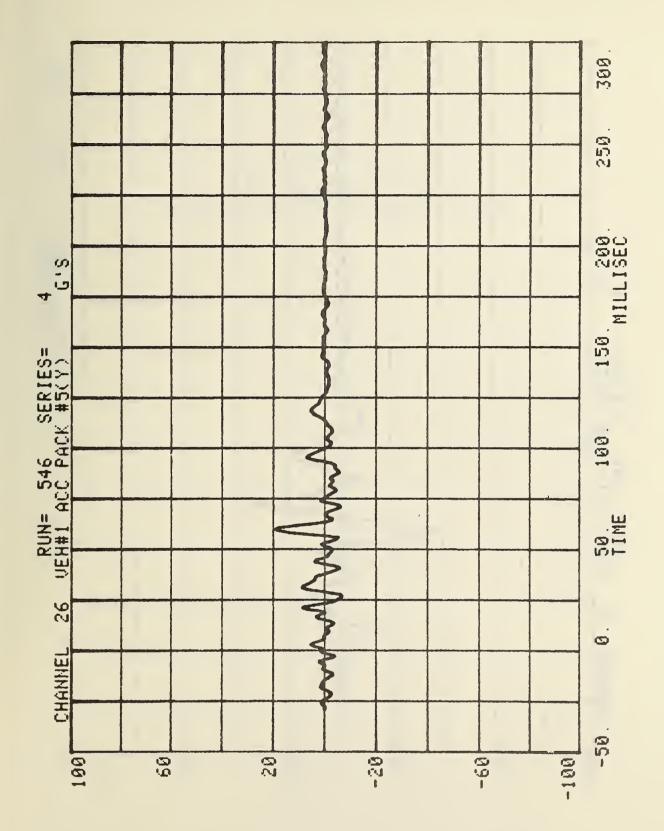


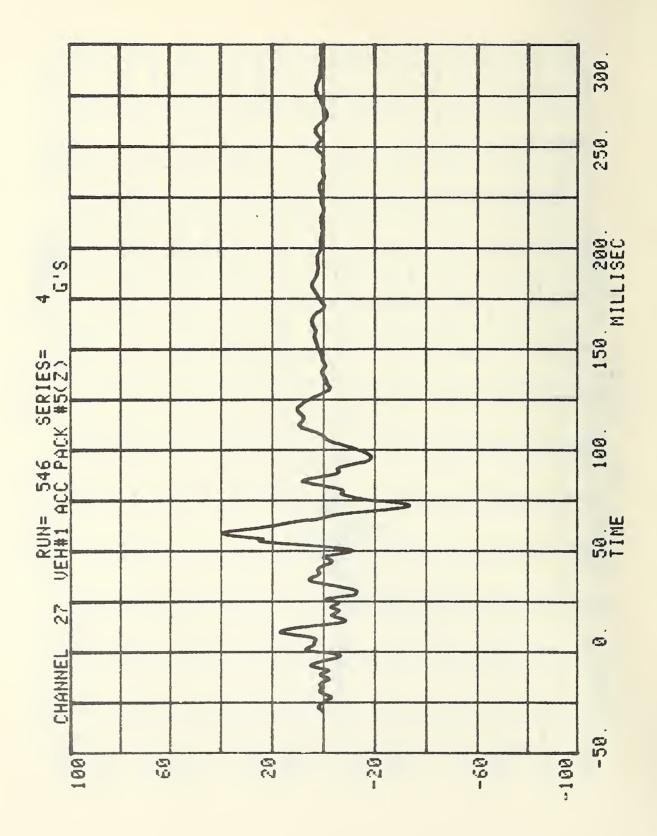












## APPENDIX G <u>ELECTRONIC CRASH TEST DATA:</u> FORD MUSTANG OCCUPANT AND RESTRAINT SYSTEM

## HEAD INJURY CRITERION HEAD SEVERITY INDEX

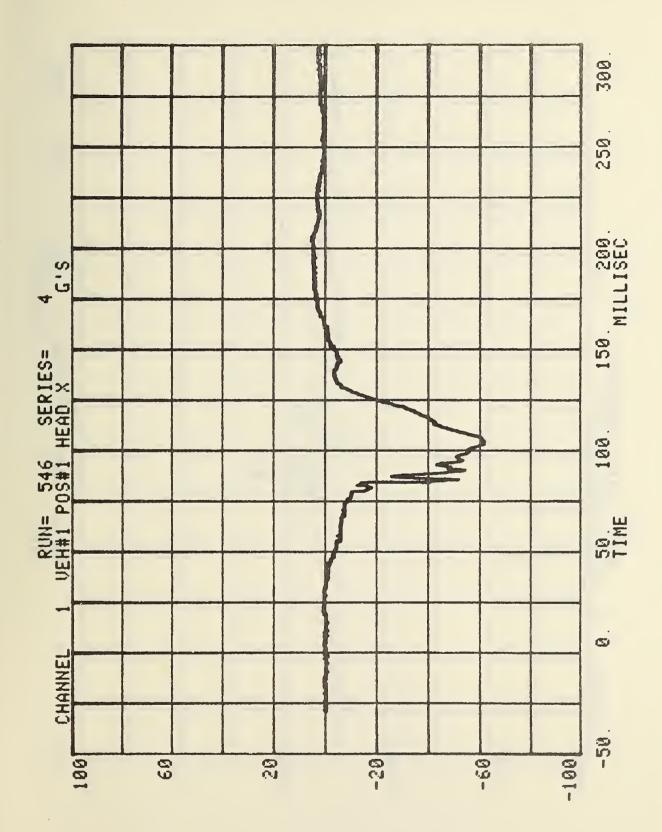
IBSA CAR-TO-CAR TEST #4

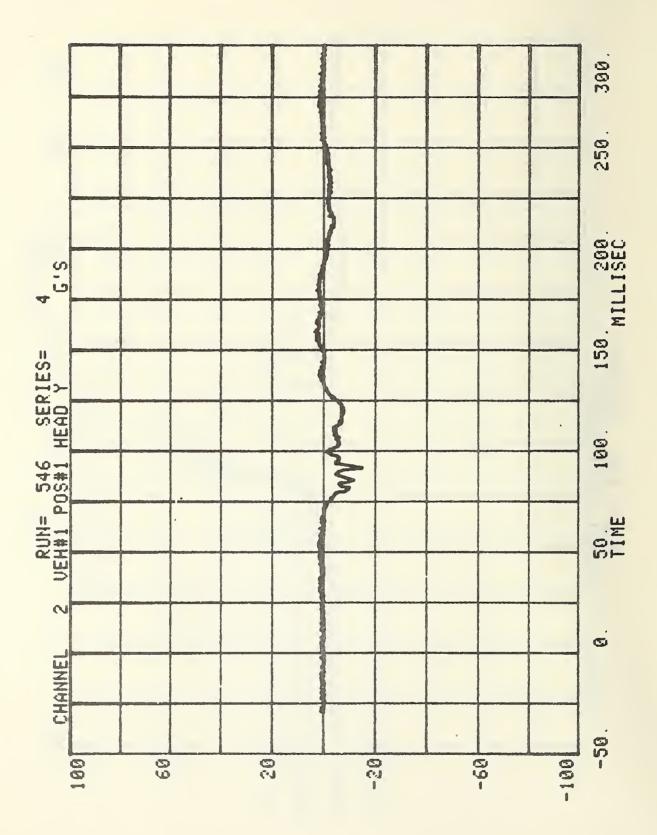
RUN = 546

UEH#1 POS#1 HEAD R

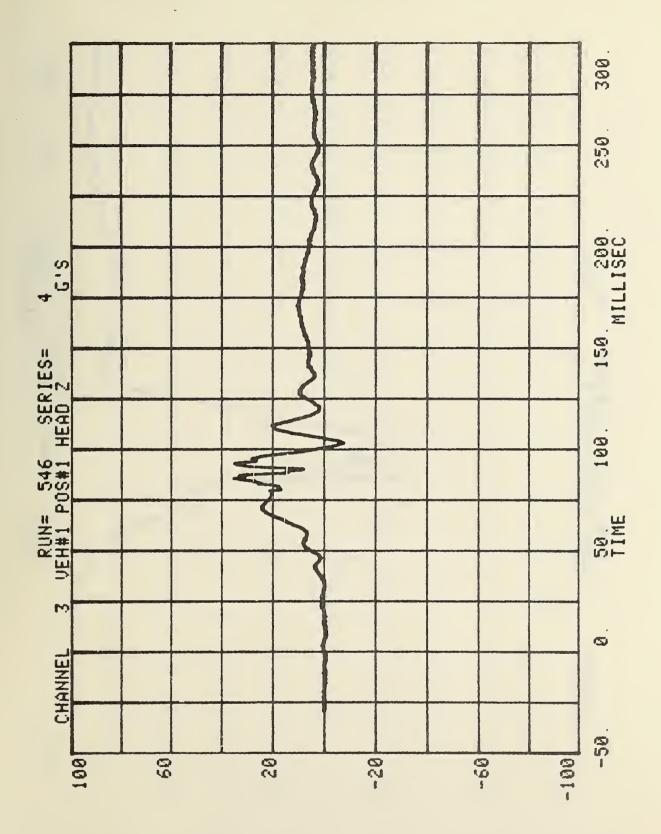
AUERAGE ACCELERATION BETWEEN TI AND T2= 51.0G'S FROM T1= .08400 T0 T2= .12210 EVENT TIME= 300.0 MSEC HIC= 709.3

SEVERITY INDEX= 850.9





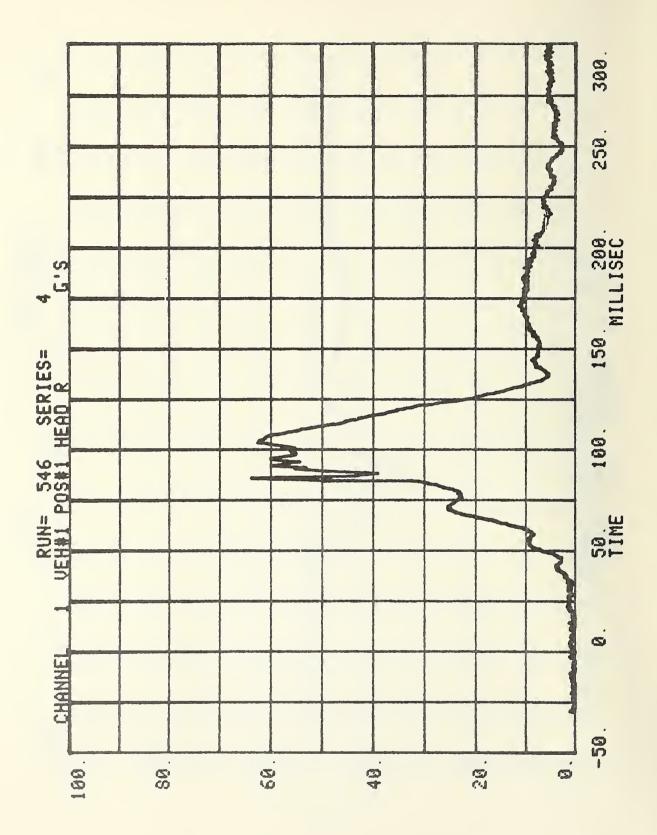
٠

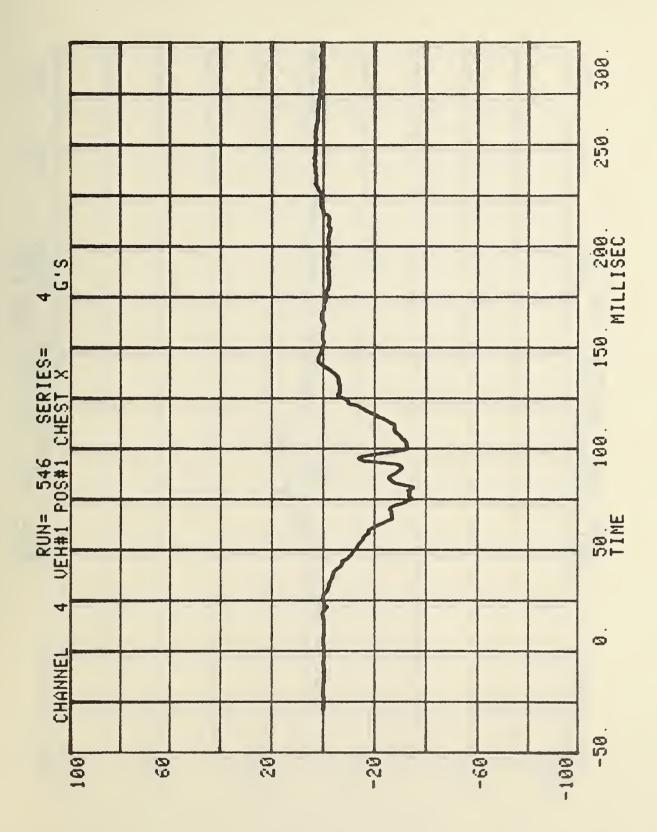


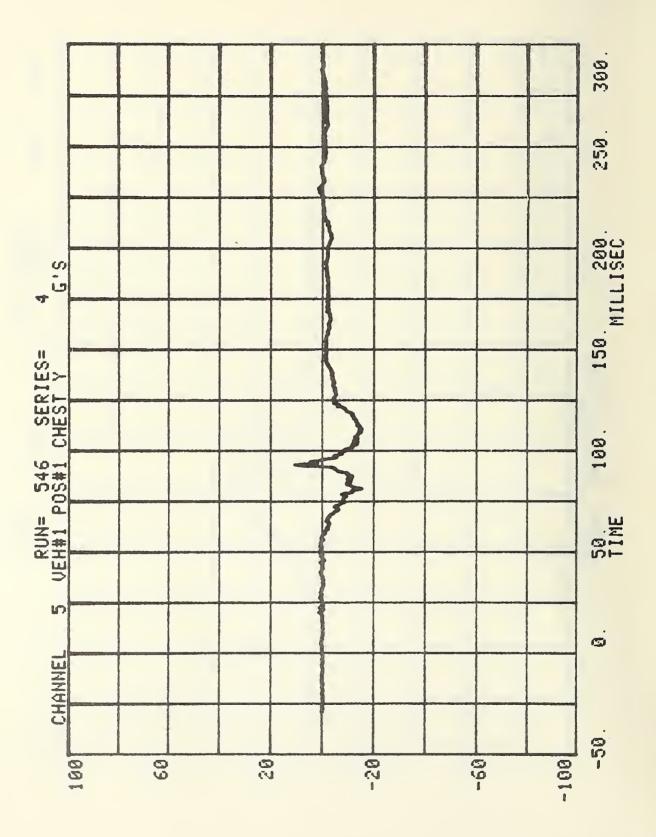
:

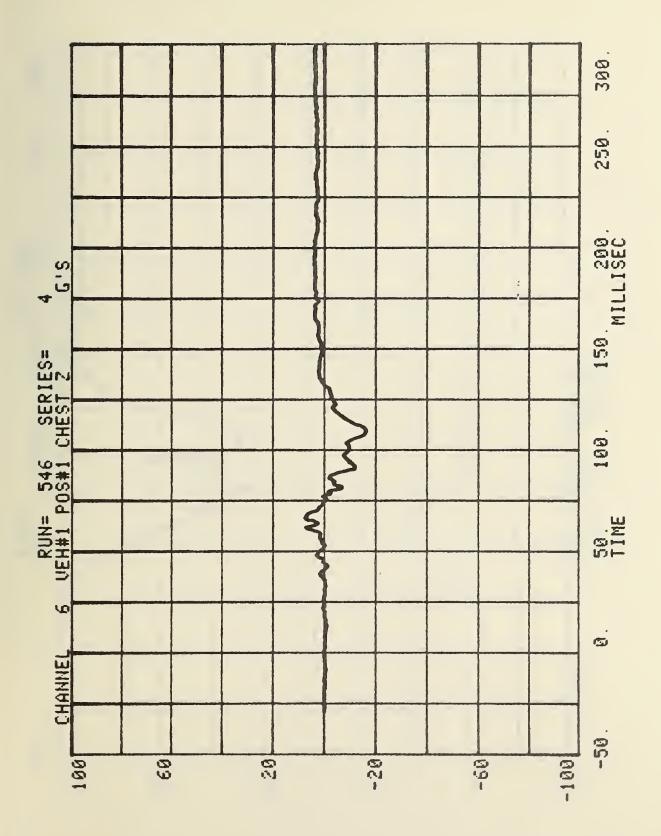
6829-V-8

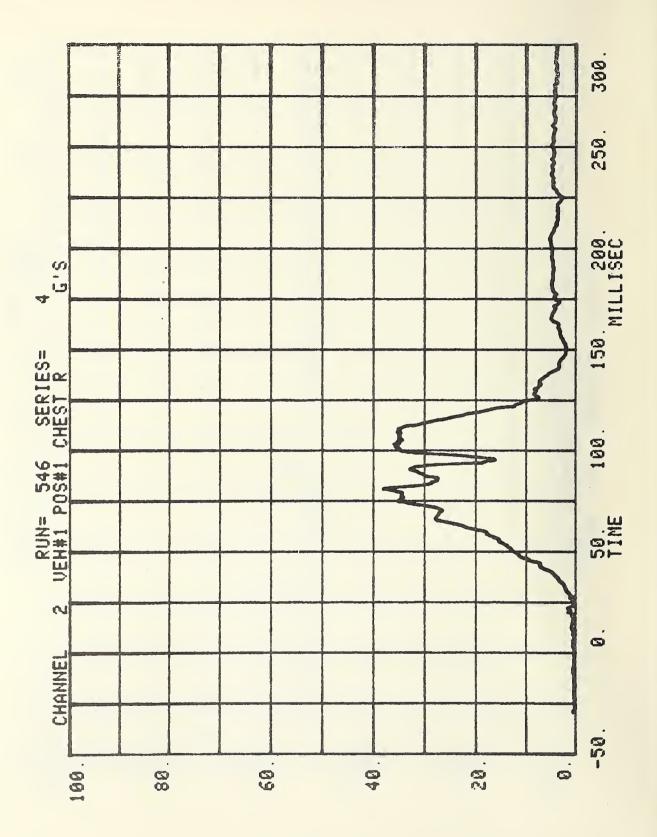
.

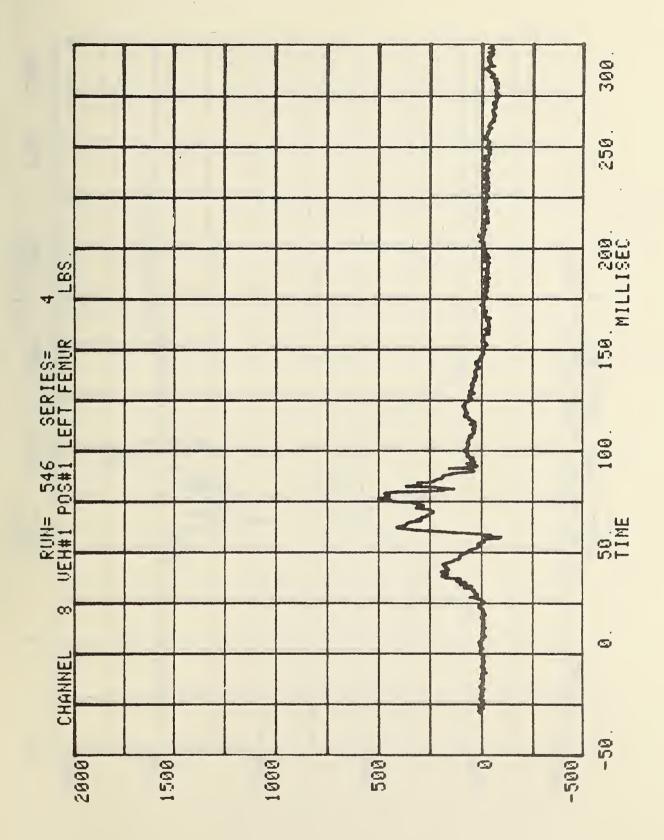


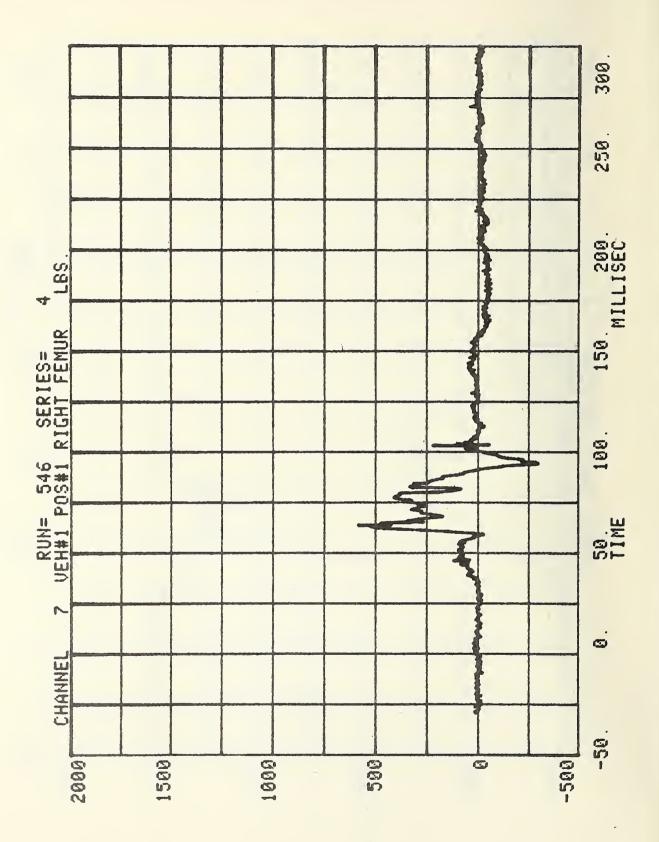


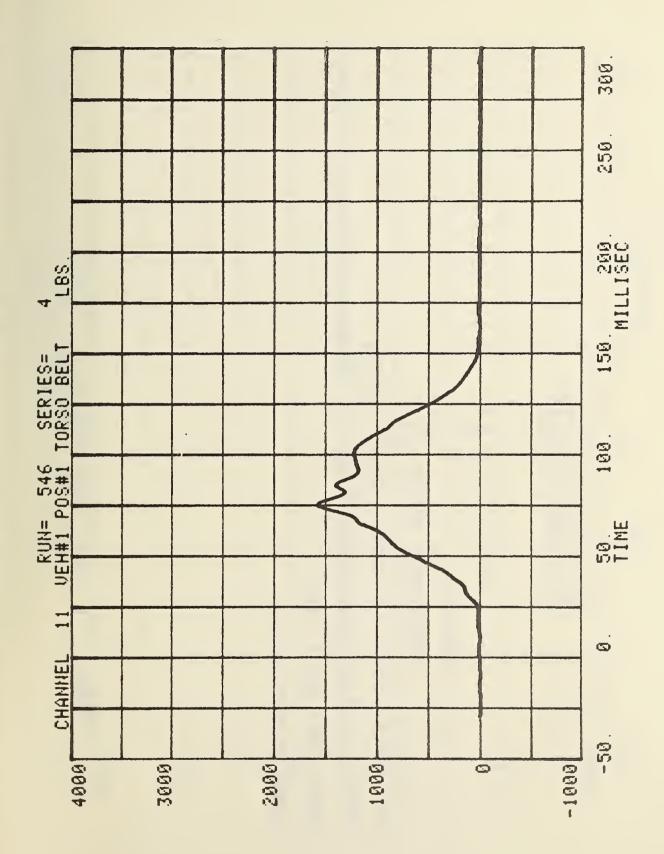












## HEAD INJURY CRITERION HEAD SEVERITY INDEX

IBSA CAR-TO-CAR TEST #4

RUN= 546

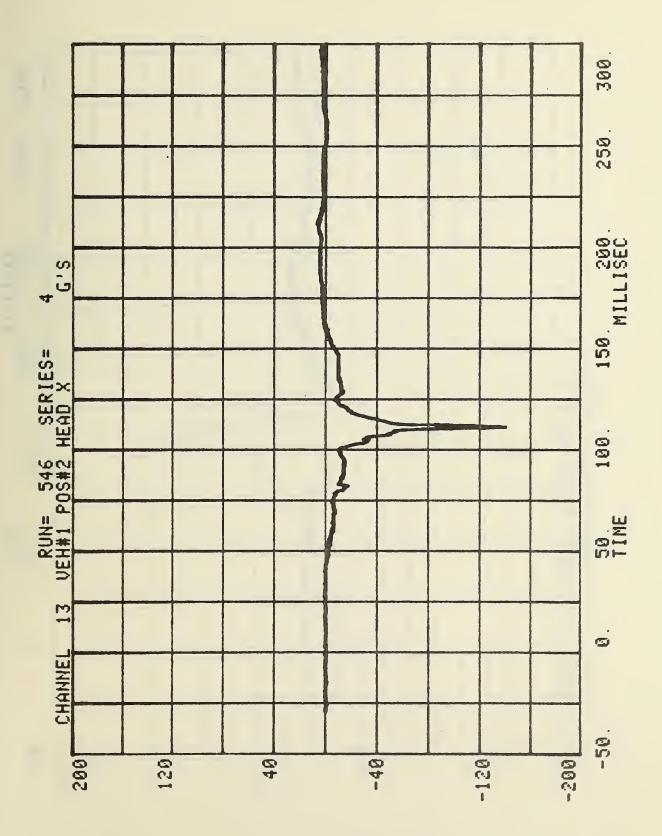
VEH#1 POS#2 HEAD R

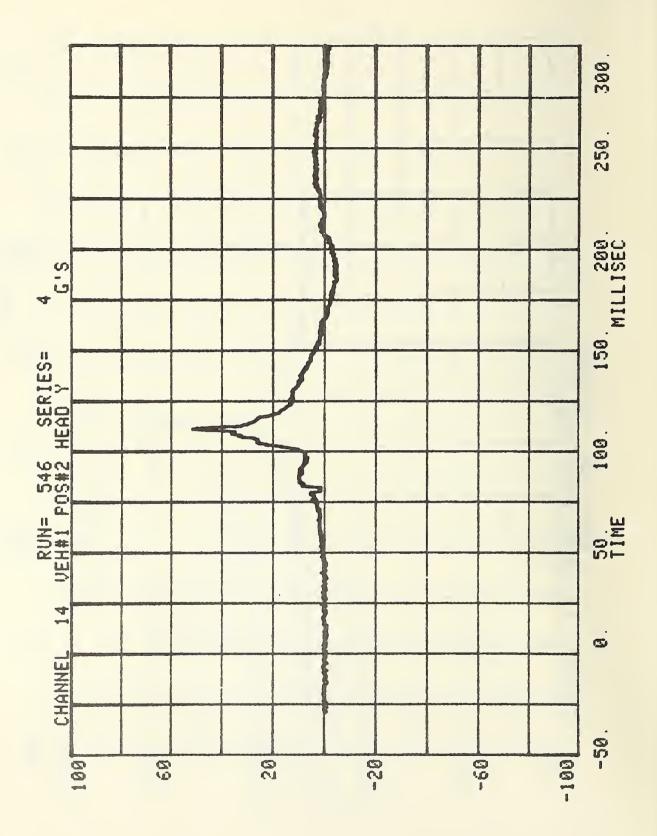
HIC= 896.0 FROM TI= .09510 TO T2= .12150

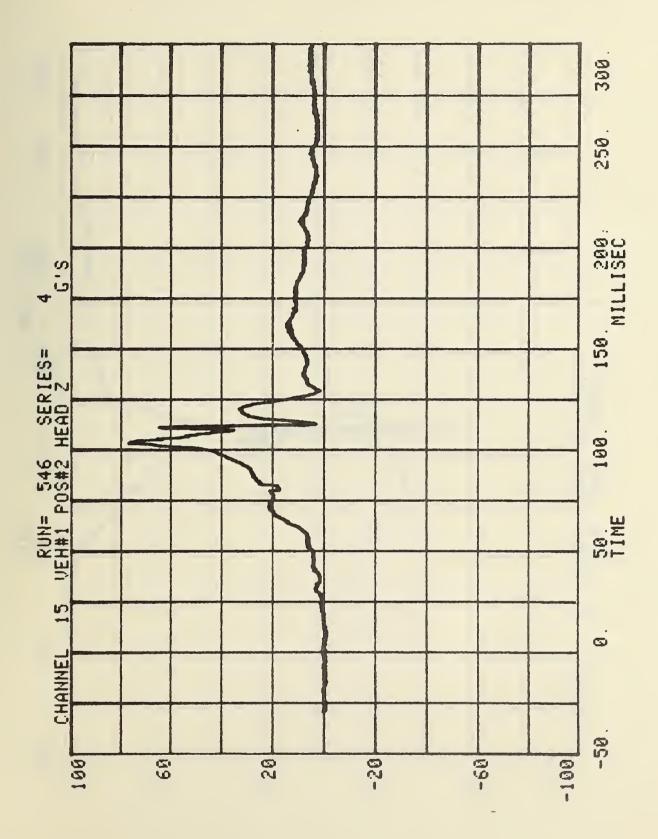
64.9C'S AVERAGE ACCELERATION BETWEEN T1 AND T2=

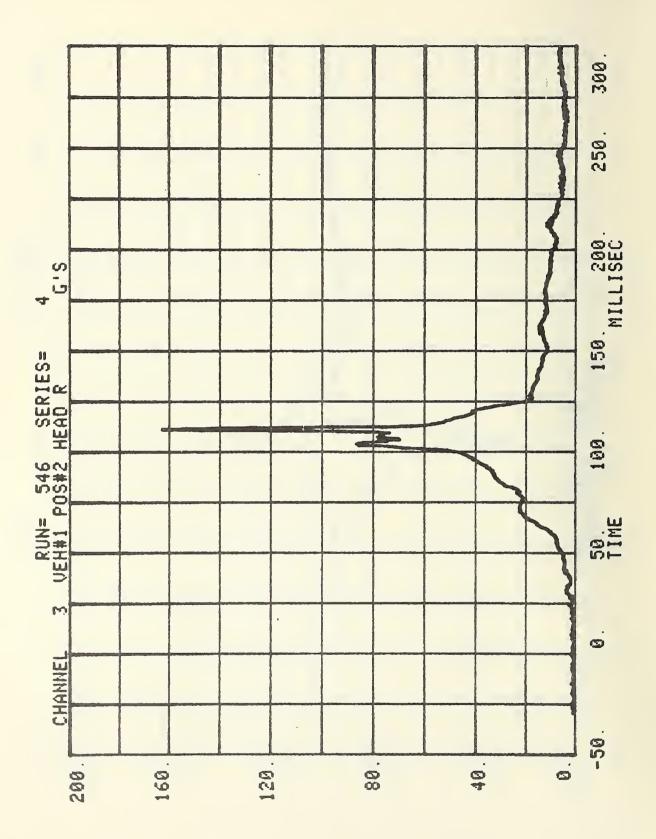
EVENT TIME= 300.0 MSEC

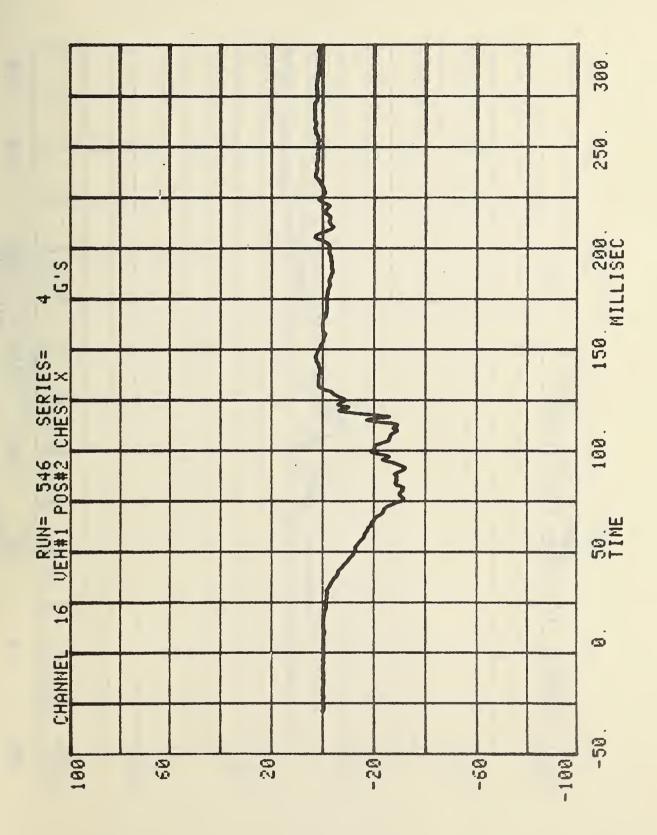
SEVERITY INDEX=1421.6

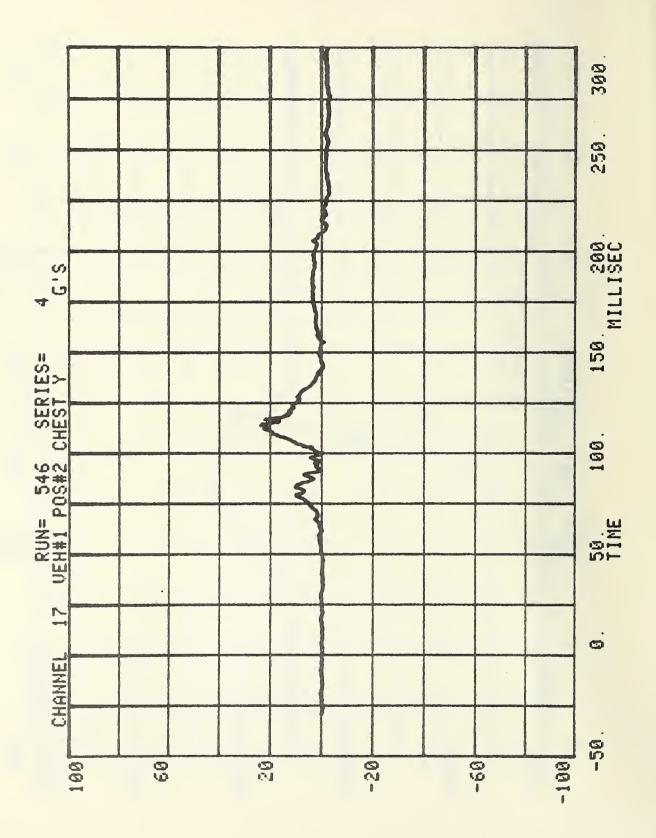


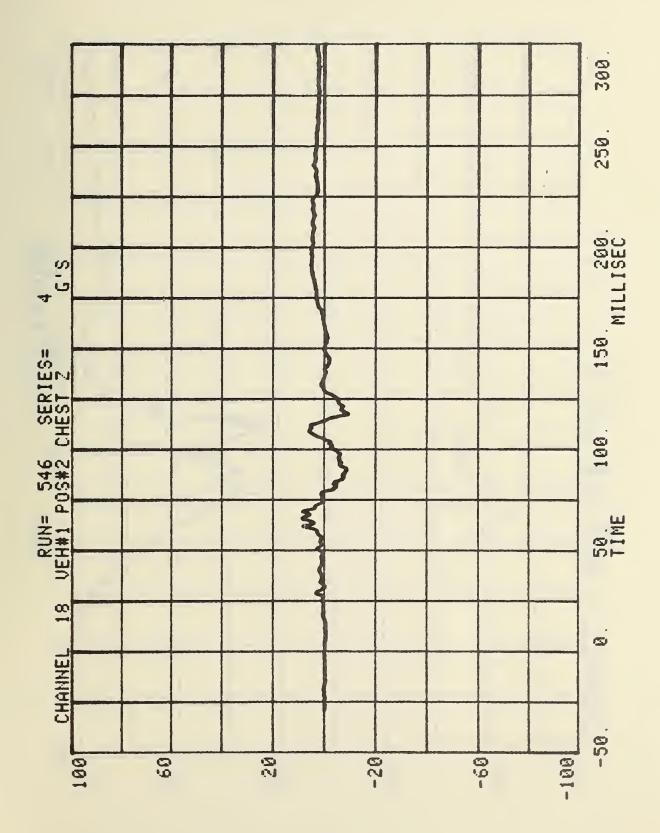


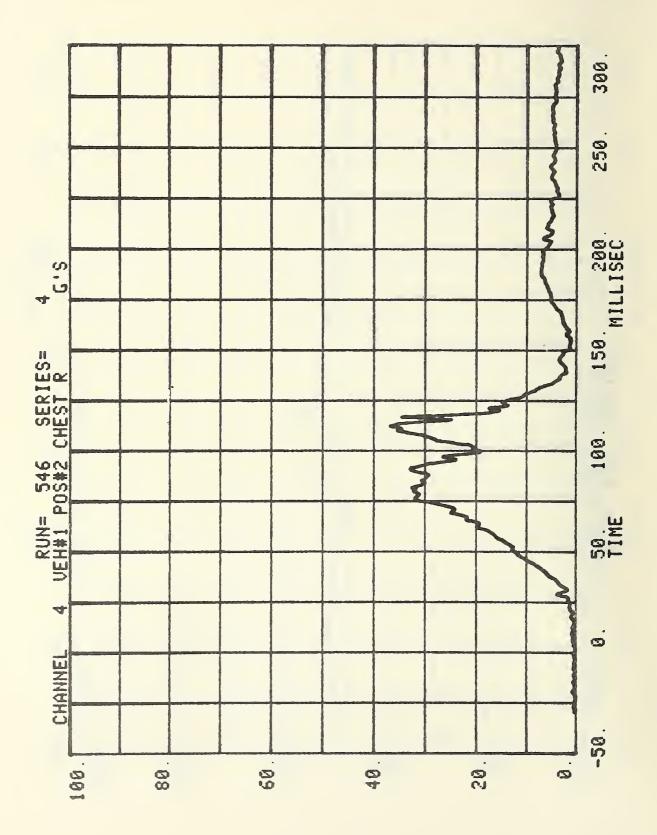


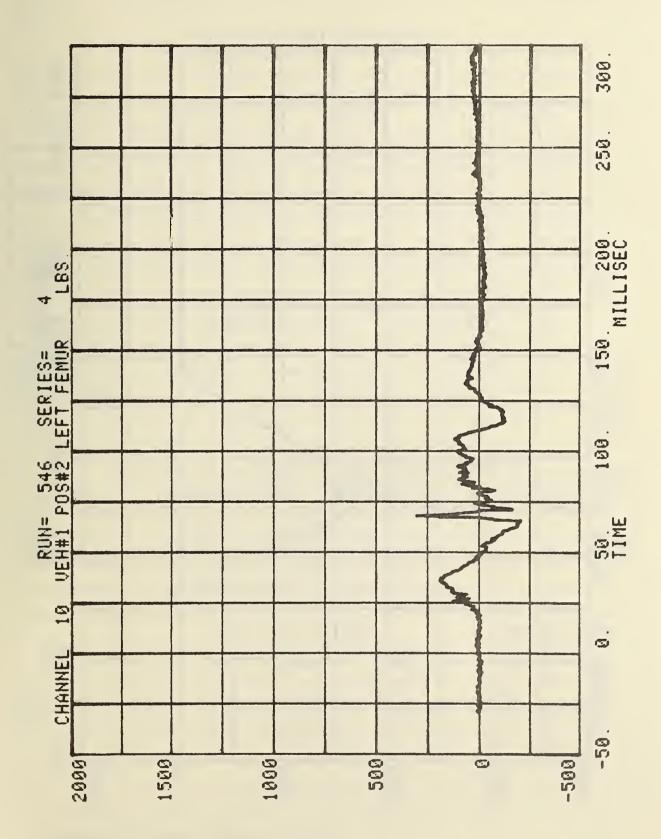




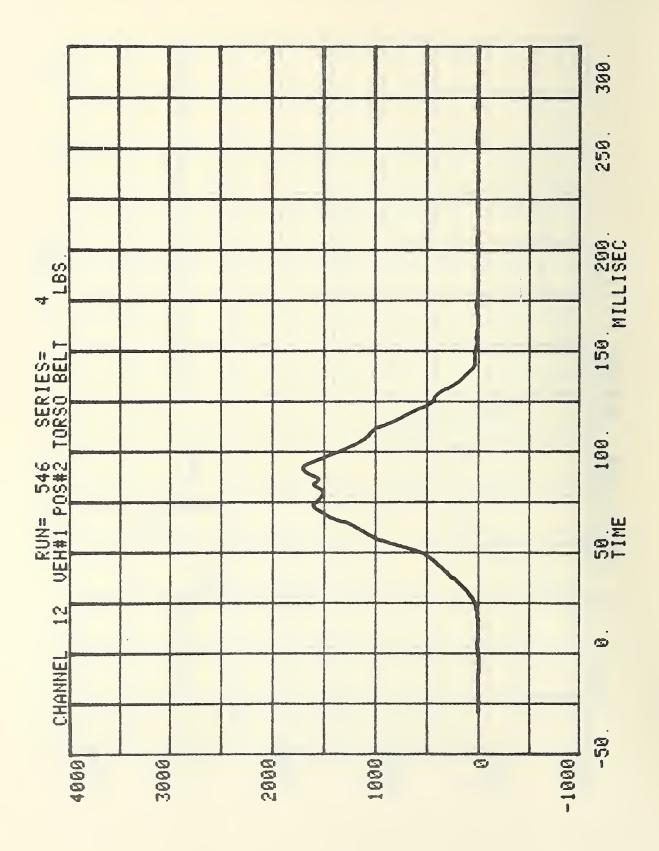








6829-V-8



6829-V-8

I.

## FORD MUSTANG OCCUPANT AND **RESTRAINT SYSTEM DATA SUMMARY**

	MAXIMUM ACCELERATION (g)											
DUMMY	HEAD				CHEST			PELVIS				
POSITION	Х	Y	Z	R	X	Y	Z	R	Х	Y	Z	R
Driver (1)	-61	-15	35	62	- 34	-15	-16	36				
Passenger(2)	-140	51	77	163	-31	22	-8	35				

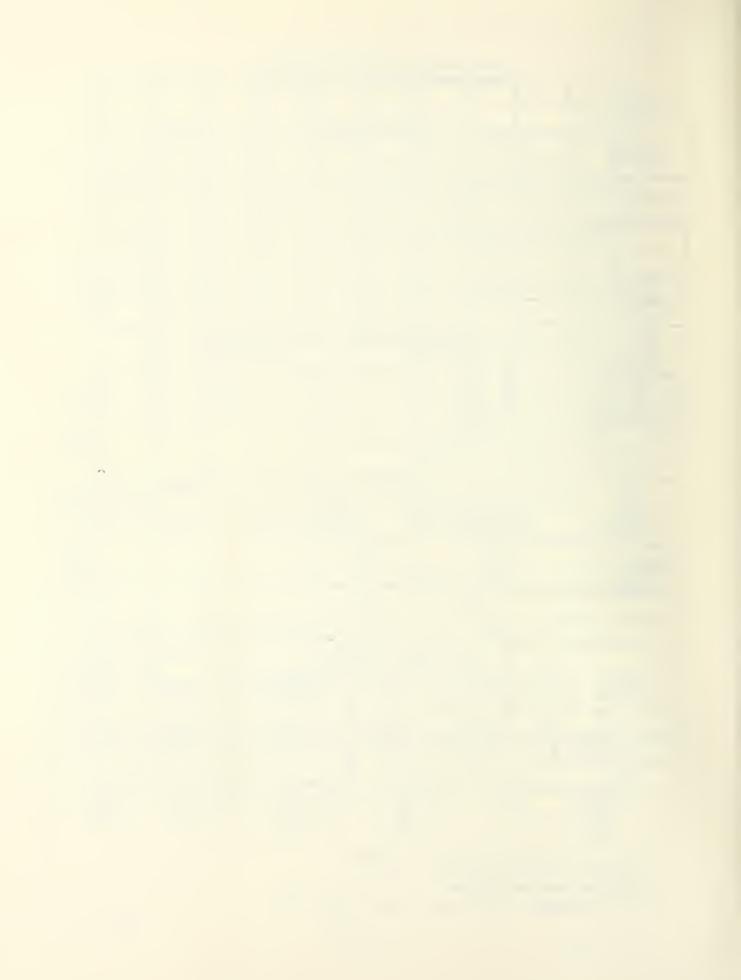
	MAXIMUM FORCE-FEMUR LOAD (LBS)					
DUMMY POSITION	RIGHT FEMUR	LEFT FEMUR				
Driver (1)	580	500				
Passenger(2)	NA <sup>3</sup>	300				

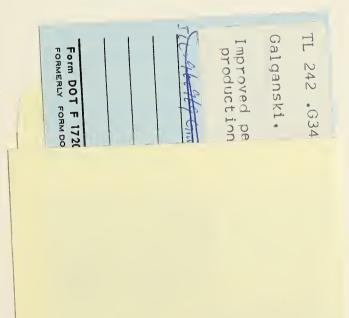
	MAXIMUM FORCE-SEAT BELT LOADS (LBS)						
DUMMY POSITION	SHOULDER STRAP UPPER BELT LOAD	LAP STRAP RIGHT BELT LOAD	LAP STRAP LEFT BELT LOAD				
Driver (1)	1580						
Passenger(2)	1700						

		HEADIN	SEVERITY INDEX			
DUMMY POSITION	HIC	t <sub>1</sub> (SEC)	t <sub>2</sub> (SEC)	AVE. ACC. (g) t <sub>1</sub> TO t <sub>2</sub>	HEAD	CHEST
Driver (1)	709	0.084	0.122	51.0	851	
Passenger(2)	896	0.095	0.122	64.9	1422	

DEFINED AS EXCEEDING 0.003 SEC. DURATION

<sup>2</sup>AS DEFINED IN FMVSS NO. 208 <sup>3</sup>NOT AVAILABLE: WIRES PULLED OUT OF LOAD CELL







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