## HVOSM STUDIES OF CROSS-SLOPE BREAKS ON HIGHWAY CURVES

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#### FOREWORD

This report summarizes the results of a research task directed toward investigation of the effects on vehicle operations of encroachments on cross-slope breaks at the outside edge of highway curves. Highway-Vehicle-Object Simulation Model (HVOSM) computer predictions of vehicle behavior were used to determine these effects. The reported research, which constitutes one part of a Federal Highway Administration program entitled "Effectiveness of Design Criteria for Geometric Elements," (Contract No. DOT-FH-11-9575) was performed by McHenry Consultants, Inc. for Research Engineers, Inc. under Subcontract to Jack E. Leisch & Associates. The opinions, findings and conclusions expressed in this report are those of the authors and not necessarily those of the Federal Highway Administration.

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### Introduction

One of the considerations in the design of highway cross section is the change in cross slope between the pavement and shoulder, referred to here as the cross-slope break.

AASHTO Policy (1,2) calls for a maximum cross-slope break of 0.07 m/m. This requirement has existed since 1954 and is consistent with the combination of the AASHTO pavement cross slope of 0.01 m/m for high-type surfaces and the maximum AASHTO shoulder cross slope of 0.08 m/m specified for turf shoulders. AASHTO states that although this maximum break is not desirable (for safety), it is tolerable.

When designing superelevated horizontal curves according to AASHTO, the cross-slope break requirement can constrain the shoulder cross-slope design on the outside of the curve. For example, with 0.06 m/m superelevation, the cross-slope break requirement limits the maximum negative shoulder cross slope to 0.01 m/m, which does not meet the AASHTO drainage requirements for even paved shoulders. The alternatives are to either design a positive shoulder slope or a rounded shoulder. The positive shoulder slope drains more runoff water across the pavement and creates problems with the melting of stored snow on the outside shoulder, and; the rounded shoulder design is more difficult to construct and maintain.

The research reported here is a limited study of the safety aspects of crossslope break to verify the adequacy of the AASHTO requirement. The primary research approach used the HVOSM computer simulation of vehicle traversals across various combinations of pavement and shoulder slope for a range of horizontal curvature.

### Criteria Development

One major purpose of shoulders is to provide a secondary recovery area for drivers who inadvertently drive onto the shoulder. Given that the designer expects this kind of traversal, the cross-slope break should be designed so it does not "cause" loss of control. This loss of control potential, of course, is most pronounced on horizontal curves where both correction paths and cross-slope breaks tend to be more severe.

The major adverse dynamic effect of cross-slope break traversals is lateral acceleration, which increases with speed, path curvature, cross-slope break, and negative shoulder slope. Assuming that the "design" event begins as a controllable traversal, the objective should be to limit lateral acceleration to a level which is stable at the tire-pavement interface and tolerable to the driver.

### Selection of Parameters for the Design Traversal

An inattentive driver can encroach on the shoulder at a horizontal curve in several ways:

- 1. A very shallow departure, in which the vehicle could be steered back to the pavement with minimal lateral displacement and a path curvature that is only slightly greater than the highway curve.
- 2. A moderate departure, in which the vehicle could be steered back to the pavement if the shoulder is wide enough and the cross-slope break and shoulder slope do not cause the vehicle to exceed available skid resistance, or result in intolerable centrifugal force on the driver.
- 3. A severe departure and/or out-of-control traversal, in which the vehicle cannot be steered back onto the pavement within the limits of the shoulder regardless of the amount of cross-slope break or shoulder slope.

A logical design is the cross-slope break which cannot effectively accommodate the most severe departures or out-of-control traversals but should

accommodate the other two kinds of traversals. Therefore, the moderate departure has been selected as the controlling event for design.

Although the moderate traversal of a vehicle onto the outside shoulder of a horizontal curve has an infinite number of paths, the most common shape seems clear. This nominal path would have an initial radius greater than the radius of the highway curve, would decrease in radius until the vehicle reached maximum lateral offset where the patheradius would be less than the highway curve radius, and would then increase in radius until the vehicle reached the highway curve radius within the normal travel lane. Because the variable dimensions and complexity of this path increase the difficulty of dynamic analysis, a simplified approach was used in this study to represent the criticality of the moderate traversal. The "design" path (see Figure 1) selected for this study would be circular, of smaller radius than the highway curve, and tangent to a concentric arc that is 0.5 m inside of the outside edge of shoulder. The path radius as a function of the highway curve radius was represented by the 95th percentile transient path measured in highway operational studies conducted by Glennon and Weaver (3). This relationship which was originally expressed in degrees of curve (English) translates to the following radius relationship.

$$R_V = \frac{19,825 \text{ R}}{R + 23,096}$$

 $R_{_{\rm U}}$  = 95th percentile vehicle path radius (meters)

R = highway curve radius (meters)

One other hypothesis in developing the design traversal for cross-slope breaks relates to speed. The driver was assumed to be driving at design speed just prior to the initial encounter with the cross-slope break. At the first point of encounter with the cross-slope break, the driver was assumed to remove his foot from the accelerator, initiating a 0.1 g engine braking.

### Performance Criteria for Lateral Friction Demand

As stated earlier, the major adverse effect of cross-slope break traversals is lateral acceleration. If lateral acceleration is great enough, vehicle loss of control can occur either directly because of vehicle skidding or indirectly because intolerable centrifugal forces on the driver cause him to take actions (braking, increased steer angle, decreased steer angle) that lead to loss of control. Setting a "design" level for lateral acceleration at the tire pavement interface requires answers to the following:

- Level of available skid resistance reasonably expected on highway shoulders;
- 2. Consideration of dry or wet shoulder surface;
- 3. Margin of safety required between the "design" lateral acceleration and the expected level of available friction.

The answer to the third point is the easiest to rationalize. Given that the moderate shoulder traversal is a recovery from an infrequent event, a much lesser safety factor is needed than used for, say, highway curve design or stopping sight distance design which both involve maneuvering on the traveled way. When examining the (critical) design event, the required skid resistance need only be as high as the lateral acceleration demands.

Whether the design case should consider a wet shoulder surface is not clear. Paved shoulders, because they usually are not worn by traffic, should exhibit reasonably high wet pavement skid resistance. Gravel shoulders have nearly equal skid resistance for dry and wet surfaces. Turf shoulders, on the other hand, exhibit adequate skid resistance when dry but very low skid resistance when wet. It is probably reasonable to expect a skid resistance (coefficient of friction at the tire-pavement interface) of about 0.40 for paved and gravel shoulders with wet surfaces and for dry turf shoulders. A more appropriate expectation of skid resistance for wet turf shoulders would be about 0.25.

### Performance Criteria for Driver Discomfort

Although the study by Weaver and Glennon (3) showed that the selected shoulder traversal is entirely manageable without <u>adverse</u> cross slope, it would put the driver on the threshold of control loss if, with adverse cross slope, the level of discomfort (centrifugal force) causes him to brake or change his steering. If he flattens his path, he will run off the shoulder and encounter the usually more severe cross-slope break at the outside edge of the shoulder. If he sharpens his path, the higher lateral friction demand may exceed available skid resistance. And, if he brakes, the resultant of both braking

friction demand and cornering friction demand may exceed available skid resistance. Therefore, the appropriate criterion is that level of discomfort below which most drivers could handle the selected shoulder traversal without performing one or more of the loss-of-control maneuvers described above. Figure 2 illustrates how cross slope affects driver discomfort.

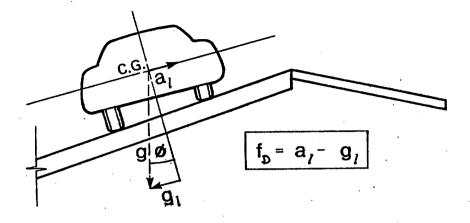
A 1974 Calspan study of driver performance on a test-track course gives some guidance on an appropriate driver discomfort threshold. The pertinent conclusion from that study is: (4)

"Under unfamiliar route conditions, the average driver utilizes lateral accelerations of about 0.3 g in the speed range of 25-40 mph." (Note: lateral acceleration was measured at the center of gravity of the vehicle.)

This result would be directly appropriate to the cross-slope break problem with five exceptions which probably tend to neutralize each other:

- 1. The Calspan tests cited above were performed on airport runways, which resulted in the drivers maneuvering around unsuperelevated curves. In such cases discomfort levels experienced by the drivers would be somewhat higher than the 0.3 g lateral acceleration measured at the c.g. of the vehicle. Thus, a slightly higher discomfort level for design could be inferred from these tests.
- 2. To be consistent with the safety-conservative design philosophy generally employed by AASHTO, a discomfort threshold lower than the average (say, 15th percentile) may be appropriate.
- 3. An even more appropriate design threshold would consider the relationship between driver discomfort and speed. Drivers such as those observed by Calspan who tolerated lower discomfort levels probably represent those drivers that would generate the lower end of the speed distribution under actual highway conditions. A design threshold selected for consistency with the concept of design speed would reflect the higher discomfort levels experienced by drivers who travel at or near design speed.
- 4. The distribution of discomfort levels on high-speed (100-120 km/h) highways would tend to reflect a lower overall threshold than measured on highways with moderate speeds, such as observed by Calspan.
  - 5. The relative unfrequency and involuntary nature of the design event justify consideration of higher discomfort levels than those experienced in normal steering associated with operations on a highway.

### **VEHICLE ON SUPERELEVATED CURVE**



$$\vec{f}_D$$
 = Discomfort Factor =  $\vec{a}_l + \vec{g}_l$ 

Where:

 $\vec{a}_I$  = Lateral Acceleration of Occupants

 $\vec{g}_l$  = Lateral Components of Gravity

Φ = Roll Angle

a<sub>1</sub>, g<sub>1</sub> in Vehicle Fixed Coordinate System

### VEHICLE ON SHOULDER WITH ADVERSE SLOPE

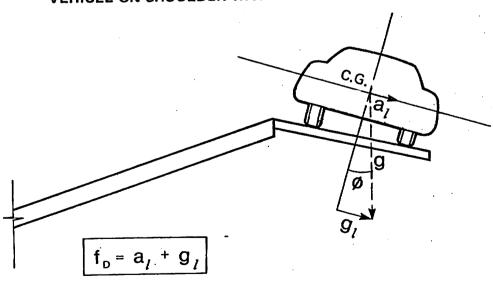


FIGURE 2 RELATIONSHIP BETWEEN DRIVER DISCOMFORT FACTOR AND COMBINATION OF ROLL ANGLE AND LATERAL ACCELERATION

Of the five points discussed above, three support selection of a greater than 0.3 discomfort level, and two support a lower threshold. Although there appears to be no strong justification for any specific discomfort level, a value of about 0.3 g would thus appear reasonable. It should be noted that this measure would only apply to that portion of drivers who would need most of the shoulder width for their corrective maneuver.

### **HVOSM Experiments**

The HVOSM (Highway-Vehicle-Object Simulation Model) is a computerized mathematical model originally developed at Cornell Aeronautical Laboratories and subsequently refined by Calspan Corporation (7). The HVOSM is capable of simulating the dynamic response of a vehicle traversing a three-dimensional terrain configuration. The vehicle is composed of four rigid masses; viz., sprung mass, unsprung masses of the left and right independent suspensions of the front wheels, and an unsprung mass representing a solid rear-axle assembly.

This study used the Roadside Design version of HVOSM that is currently available from FHWA. Certain modifications were necessary to perform the cross-slope break traversals and to interpret the appropriate dynamic response. These modifications, which included the following, are described in more detail in Appendix A.

- 1. Ground Contact Point Interpolation
- 2. Effective Range Angled Boundary Option (ERABO)
- 3. Driver discomfort factor output
- 4. Friction demand output
- 5. Terrain Table Generator
- 6. Driver Model inputs (damping, steer velocity, steer initialization)

The objective of the HVOSM experiments was to evaluate the dynamic effects of the cross-slope breaks associated with outside shoulder traversals on highway curves. Table 1 lists the general conditions and specifications for the HVOSM runs, which are described more fully below.

#### Basic Test Conditions

Because of FHWA contractual requirements, this study was conducted using metric units. Since the most critical highway curve conditions are the AASHTO controlling curves for design, the AASHTO criteria relating design speed and design "f" were used to develop the metric geometrics (rounded) of controlling highway curves for 20 km/h design-speed increments. The criterion curve used was the one developed for inclusion in the current draft

### Table 1

### HVOSM TEST CONDITIONS

Condition	Specification
Highway Curve Radius	Metricated AASHTO Controlling Curves (meters)
Superelevation	Metricated AASHTO Controlling Curves (.02 to .10 m/m)
Shoulder Width	2.7 meters
Shoulder Cross Slope	02 to06 m/m
Available Friction at Interface	0.8
Vehicle	1971 Dodge Coronet
Initial Vehicle Speed	Metricated Design Speed(kph)
Vehicle Deceleration	Engine Braking @ 0.1 g
Vehicle Path Radius	95th percentile measured by Glennon and Weaver
Vehicle Path Radius Tangent Point	2.2 meters from edge of traveled way

version for the upcoming edition of AASHTO Geometric Design Policy. These metricated controlling design curves are shown in Table 2.

Table 2

METRIC AASHTO CONTROLLING HORIZONTAL CURVES

		Hori:	zontal	Curve R	adius (	Meters)
Design Speed(kph)	<u>Design f</u>	St	perele	vation	Rate (m	/m)
		.02	.04	<u>.06</u>	.08	.10
120	.092	1020	870	750	670	600
100	.116	_	510	450	410	370
80	.140	-	280	260	230	210
60	.152	-	150	140	130	120
40	.164	-	65	60	55	50

As previously described, the design shoulder traversal would have a circular radius that represents the 95th percentile path relative to each highway curve radius. Using the equation shown earlier, Table 3 gives the radius of vehicular traversals for each metricated AASHTO highway curve.

Table 3

ASSUMED MAXIMUM PATH CURVATURE FOR CONTROLLING CURVES

Design Speed(kph)	Sı		Radius (		
	.02	.04	.06	.08	.10
120	586	525	472	435	400
100	-	351	318	294	270
80	-	212	198	178	164
60	-	120	113	105	97
40	-	54	50	46	42

A full-width shoulder of 2.7 m with negative cross slopes of .02, .04 and .06 m/m was used in the basic HVOSM runs. The circular traversal path for these runs was, as previously described, tangent to a concentric arc at 2.2 m from

version for the upcoming edition of AASHTO Geometric Design Policy. These metricated controlling design curves are shown in Table 2.

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A full-width shoulder of 2.7 m with negative cross slopes of .02, .04 and .06 m/m was used in the basic HVOSM runs. The circular traversal path for these runs was, as previously described, tangent to a concentric arc at 2.2 m from

the edge of pavement. A small number of similar runs were made to evaluate the dynamics of both traversals on narrower shoulders and partial traversals on full-width shoulders.

Since the objective of the HVOSM test was to study the demands for various lateral acceleration components irrespective of available skid resistance, a high (0.8) available friction factor was used. A 1971 Dodge Coronet was used as the design vehicle, since it seemed to best represent the current population of passenger cars among the vehicles that have been modeled for HVOSM application. Although there are some strong concerns about the dynamic effects of cross-slope breaks on articulated trucks, this HVOSM option was not available and would have been beyond the study scope to develop.

#### Preliminary HVOSM Runs

A series of initial HVOSM runs was made to study the dynamic differences between (1) 4-wheel and 2-wheel traversals onto the shoulder, and (2) entry to and exit from the shoulder. These runs were made at the most extreme test conditions as follows:

Condition	Specification
Speed	120 km/h
Highway Radius	600 m
Path Radius	400 m
Superelevation	.10 m/m
Negative Shoulder Slope	.06 m/m
Cross-slope Break	.16 m/m
Deceleration	None

The results of these runs indicated that the 4-wheel traversal and the entry to the cross-slope break produced the most extreme dynamic responses. For reasons of economy, therefore, the basic HVOSM experiment concentrated on full 4-wheel traversals over four seconds of real time (sufficient to measure dynamic responses).

### Basic HVOSM Experiments

The 21 controlling highway curve geometries with three shoulder cross slope dimensions (-.02, -.04, and -.06) combine to make 63 potential test conditions. However, the budget for this study would not allow testing all of these conditions. Table 4 shows the 14 test conditions that were selected for inclusion in the basic experiment. These include the three highest design speeds and cross-slope breaks ranging in .02 increments from .04 to .16 m/m.

Table 4 also shows the results from the basic HVOSM runs. A complete time trace of these dynamics is shown for one experiment in Appendix B. In general, results indicate that the dynamic effects are most sensitive to shoulder cross slope and exceed reasonable driver discomfort levels for the design conditions that produce the higher cross-slope breaks. The dynamic effects, however, seem fairly insensitive to cross-slope break within the range studied. The obvious relation between dynamic effects and cross-slope break is basically an indirect one that is a function of (1) the relation between negative shoulder slope and cross-slope break, and (2) the relation between highway curve (and path) radius and superelevation.

#### HVOSM Experiments to Test Sensitivities

Because the 14 basic HVOSM runs did not produce a universal relationship among all of the parameters of interest, three additional HVOSM runs were made. Two of these were identical to two of the basic runs with the exception that they involved only 2-wheel traversals with a lateral displacement of 0.8 m. A comparison of these runs with the 4-wheel traversals, as shown in Table 5, indicates that 2-wheel traversals (because of a less severe "effective" cross slope) have significantly less severe dynamic responses.

Table 4

HVOSM DYNAMIC RESPONSE RESULTS

TEST CONDITIONS						TEST RESULTS				
Speed (kph)	Highway Design Curve R(m)	Path R(m)	Super- elevation etw	Shoulder Slope sh	Cross-Slope Break	Max. Dis- Comfort Factor(g)	Max. Fric- tion Demand(g)	Max. Roll Angle(°)		
120	1020	586	.02	02	.04	. 24	.20	3.6		
120	1020	586	.02	06	.08	.30	.25	6.6		
120	870	525	.04	04	.08	.32	.27	5.8		
120	670	435	.08	04	.12	.35	.29	6.0		
120	600	400	.10	02	.12	.37	.30	5.0		
120	600	400	.10	06	.16	.43	.36	7.8		
100	510	351	.04	02	.06	.29	.25	4.2		
100	510	351	.04	06	.10	.35	.34	7.2		
100	450	318	.06	04	.10	.38	.31	6.3		
100	370	270	.10	04	.14	.38	.31	6.3		
80	280	212	.04	04	.08	.31	.26	5.5		
80	260	198	.06	02	.08	.28	.24	4.0		
80	260	198	.06	06	.12	.36	.30	7.1		
80	210	164	.10	-) 04	.14	.41	.38	6.6		

14

Table 5

COMPARISON OF FULL AND PARTIAL TRAVERSALS

•••	TEST	CONDI	TIONS			-	TEST RESULT	<u>s</u>
Speed (km/h)	Highway Design Curve R(m)	Path R(m)	e tw	e <sub>sh</sub>	Traversal Type	Max. Dis- Comfort Factor(g)		Max. Roll Angle(°)
120	870	525	.04	04	Full	.32	.27	5.8
120	870	525	.04	04	Partial	.25	.23	1.4
100	510	351	.04	06	Full	-35	.34	7.2
100	510	351	.04	06	Partial	.27	.23	4.8

Table 6 shows another sensitivity comparison wherein one of the basic test conditions was modified to run the vehicle at a speed 20 km/h higher than design speed. The extreme responses associated with overdriving a design condition are apparent.

Table 6

SPEED SENSITIVITY FOR FULL TRAVERSALS

TEST CONDITIONS					TEST RESULTS			
Speed (km/h)	Highway Design Curve R(m)	Path R(m)	e twi	e <sub>sh</sub>	Comfort	Max. Fric- tion Demand(g)	Max. Roll Angle(°)	
120	510	351	.04	06	.49	.42	8.5	
100	510	351	.04	06	.35	.34	7.2	

### Analysis of HVOSM Results

The basic HVOSM results presented in the prior section of this report indicate that the driver discomfort factor generally exceeds the lateral acceleration on the tires (the difference being a function of the roll-angle experienced on the negative shoulder slope). Therefore, the tentative performance criterion established for driver discomfort was the controlling threshold.

For comparison with the basic HVOSM test runs, Table 7 shows the nominal lateral acceleration for shoulder traversals computed with the standard centripetal force equation using the design speed, the shoulder cross slope, and the traversal path from Table 3. In comparing Tables 4 and 7, certain fairly distinct trends are apparent:

- 1. For a given curve design, the incremental dynamic effect varies directly at 1.5 times the increase in shoulder slope.
- 2. The incremental dynamic effect increases with decreasing horizontal curve radius for a given design speed.
- 3. The incremental dynamic effect increases slightly with design speed for any given combination of superelevation and shoulder slope.

Although there are some minor inconsistencies in the test results (due to minor flexibilities in the HVOSM path control algorithm), the noted trends allow a reasonable interpolation and extrapolation of the results as shown in Table 8.

It must be noted that Table 8 is for a full traversal onto a wider shoulder. For traversal onto narrower shoulders (less than 1.6 m) and for partial traversals on wider shoulders, the discomfort levels would be less because the effective (negative) cross slope is less. Because the net effect of shoulder slope is apparent from the HVOSM tests, it is possible to estimate the driver discomfort levels for partial traversals. Table 9 shows the driver discomfort levels when the vehicle is half on the superelevation and half on the negative shoulder slope (approximately 0.8 m beyond the cross-slope break).

#### Intrepretation of HVOSM Results

Based on the tentative criterion of a maximum 0.3 g for driver discomfort, Table 10 shows the tolerable shoulder cross slope designs for full shoulders (1.6 m or more). This result is very similar to the 1965 AASHTO single recommendation of 0.07 maximum cross-slope break.

Table 7

### NOMINAL CENTRIPETAL LATERAL ACCELERATION --FULL TRAVERSAL ON WIDE SHOULDERS

V	R		.02	•	.04	.06	.08	.10
Highway	Highway				SHOULDER S	LOPE FOR GIVEN SUPERE	LEVATIONe sh	,
Design Speed(km/h)	Design Radius(m)	- <u>.02</u> - <u>.0</u>	040608	0204	0608	02040608	02040608	02040608
120	1020 870 750 670 600	.21 .2	23 .25 .27	.24 .26	.28 .30	.26 .28 .30 .32	.28 .30 .32 .34	.30 .32 .34 .36
100	510 450 410 370			.24 .26	.28 .30	.27 .29 .31 .33	.29 .31 .33 .35	.31 .33 .35 .37
80	280 260 230 210			.26 .28	.30 .32	.27 .29 .31 .33	.30 .32 .34 .36	.33 .35 .37 .39
60	150 140 130 120			.26 .28	.30 .32	.27 .29 .31 .33	.29 .31 .33 .35	.31 .33 .35 .37
40	65 60 55 50			.25 .27	.29 .31	.27 .29 .31 .33	.29 .31 .33 .35	.32 .34 .36 .38

\* 
$$f = \frac{V^2}{127 \text{ R}} - e_{SH}$$
 ;  $R_V = \frac{19.825 \text{ R}}{R + 23,096}$ 

$$R_V = \frac{19,825 \text{ R}}{R + 23,096}$$

where:  $R_v = Radius of ''design'' path$ 

Tabl 8

# SMOOTHED RESULTS\* FOR DRIVER DISCOMFORT FACTOR--KULL TRAVERSAL ON WIDE SHOULDERS

SUPERELEVATION OF ADJOINING TRAVERED WAY--e<sub>tw</sub>
.02 .04 .06 .08

.10

Design	Design									LOPE F											
Speed(km/h)	Radius(m)	- <u>.02</u>	<u>04</u>	- <u>.06</u>	<u>08</u>	- <u>.02</u>	- <u>.04</u>	- <u>.06</u>	- <u>.08</u>	<u>02</u>	- <u>.04</u>	<u>06</u>	<u>08</u>	- <u>.02</u>	<u>04</u>	- <u>.06</u>	- <u>.08</u>	02	<u>04</u>	- <u>.06</u>	- <u>.08</u>
120	1020 870 750 670	.24	.27	.30	.33	.28	.31	.34	.37	.31	.34	.37	.40	.34	. 37	.40	.43				
	600									•								.37	.40	.43	.46
100	510 450					.28	.31	.34	.37	.31	.34	.37	.40	_		_	_				•
	410 370													.34	.37	.40	.43	.37	.40	.43	.46
80	280 260 230 210					.28	.31	.34	.37	.30	.33	.36	.39	.34	.37	.40	.43	.37	.40	.43	.46
								_			•				•			• ) /	.40	• • • •	• 10
60	150 140 130					.28	.31	.34	.37	.30	.33	.36	.39	.33	.36	.39	.42				
	120															•		.36	.39	.42	.45
40	65 60					.28	.31	.34	.37	.30	-33	.36	.39	22	26	-39	ho				
٠,	55, 50											•	1	•33	. ) 0	• 23	.42	.36	-39	.42	.45

<sup>\*</sup> Based on values from Tables 4 and 7

Tab 1

## SMOOTHED RESULTS FOR DRIVER DISCOMFORT FACTOR--PARTIAL TRAVERSAL ON WIDE SHOULDERS AND TRAVERSAL ON NARROW SHOULDERS

SUPERELEVATION OF ADJOINING TRAVELED WAY--e tw .08

	•		•	02	•		•	04			•	.06			•	08				.10	
Design	Design			- 4	-0					LOPE F						-					
Speed (km/h)	Radius(m)	<del>'02</del>	- <u>.04</u>	<u>06</u>	<u>08</u>	02	<u>04</u>	- <u>.06</u>	<u>08</u>	- <u>.02</u>	<u>04</u>	- <u>.06</u>	<u>08</u>	02	- <u>.04</u>	06	<u>08</u>	- <u>.02</u>	04	<u>06</u>	<u>08</u>
120	1020 870 750 670 600	.22	.23	.24	.25	.24	.25	.26	.27	.26	.27	.28	.29	.27	.28	.29	.30	.29	.30	.31	.32
100	510 450 410 370					.24	.25	.26	.27	.26	.27	.28	.29	.27	.28	.29	.30	.29	.30	.31	.32
80	280 260 230 210					.24	.25	.26	.27	.26	.27	.28	.29	.27	.28	.29	.30	.29	.30	.31	.32
60	150 140 130 120				·	.24	.25	.26	.27	.26	.27	.28	.29	.26	.27	.28	.29	.28	.29	.30	.31
40	65 60 55 50				•	.24	.25	.26	.27	.26	.27	.28	.29	.26	.27	.28	.29	.28	.29	.30	.31

Table 7

NOMINAL CENTRIPETAL LATERAL ACCELERATION --FULL TRAVERSAL ON WIDE SHOULDERS

			SUPERELEVA	TION OF ADJOINING TRA	VELED WAYe,	·
v	R	.02	.04	.06	.08	.10
Highway Design	Highway Design		SHOULDER S	LOPE FOR GIVEN SUPEREI	EVATIONe sh	
Speed(km/h)	Radius (m)	02040608	02040608	02040608	02040608	02040608
120	1020 870 750 670 600	.21 .23 .25 .27	.24 .26 .28 .30	.26 .28 .30 .32	.28 .30 .32 .34	.30 .32 .34 .36
100	510 450 410 370		.24 .26 .28 .30	.27 .29 .31 .33	.29 .31 .33 .35	.31 .33 .35 .37
80	280 260 230 210		.26 .28 .30 .32	.27 .29 .31 .33	.30 .32 .34 .36	.33 .35 .37 .39
60	150 140 130 120		.26 .28 .30 .32	.27 .29 .31 .33	.29 .31 .33 .35	.31 .33 .35 .37
40	65 60 55 50		.25 .27 .29 .31	.27 .29 .31 .33	.29 .31 .33 .35	.32 .34 .36 .38

\* 
$$f = \frac{V^2}{127 \text{ R}_V} - e_{SH}$$
 ;  $R_V = \frac{19,825 \text{ R}}{R + 23,096}$  where:  $R_V = \text{Radius of ''design'' path}$ 

Table 8

SMOOTHED RESULTS\* FOR DRIVER DISCOMFORT FACTOR-FULL TRAVERSAL ON WIDE SHOULDERS

			SUPERELEVA	TION OF ADJOINING TRA	VERED WAYe <sub>tw</sub>	
		.02	.04	.06	.08	:10
Design	Design		SHOULDER S	LOPE FOR GIVEN SUPERE	LEVATIONe <sub>sh</sub>	
Speed (km/h)		02040608		02040608	02040608	- <u>.02 = .040608</u>
120	1020 870 750 670 600	.24 .27 .30 .33	.28 .31 .34 .37	.31 .34 .37 .40	.34 .37 .40 .43	.37 .40 .43 .46
100	510 450 410 370	·	.28 .31 .34 .37	.31 .34 .37 .40	.34 .37 .40 .43	.37 .40 .43 .46
80	280 260 230 210		.28 .31 .34 .37	.30 .33 .36 .39	.34 .37 .40 .43	.37 .40 .43 .46
60	150 140 130 120		.28 .31 .34 .37	.30 .33 .36 .39	.33 .36 .39 .42	.36 .39 .42 .45
40	65 60 55, 50		.28 .31 .34 .37	.30 .33 .36 .39	.33 .36 .39 .42	.36 .39 .42 .45

<sup>\*</sup> Based on values from Tables 4 and 7

	·	SUPERELEVATION OF ADJOINING TRAVELED WAYe_									
		.02	.04	.06	. 08	.10					
Design	Design		SHOULDER SL								
Speed (km/h)	Radius (m)	02040608	•		02040608	02040608					
120	1020 870 750 670 600	.22 .23 .24 .25	.24 .25 .26 .27	.26 .27 .28 .29	.27 .28 .29 .30	.29 .30 .31 .32					
100	510 450 410 370		.24 .25 .26 .27	.26 .27 .28 .29	.27 .28 .29 .30	.29 .30 .31 .32					
80	280 260 230 210		.24 .25 .26 .27	.26 .27 .28 .29	.27 .28 .29 .30	.29 .30 .31 .32					
60	150 140 130 120		.24 .25 .26 .27	.26 .27 .28 .29	.26 .27 .28 .29	.28 .29 .30 .31					
40	65 60 55 50		.24 .25 .26 .27	.26 .27 .28 .29	.26 .27 .28 .29	.28 .29 .30 .31					

MAXIMUM NEGATIVE SHOULDER CROSS SLOPES
USING 0.30 g DISCOMFORT CRITERION

Superelevation of Highway Curve, m/m										
Design Speed (	km/h) .08	.07	.06	.05	<u>.04</u> ::	.03	02			
	<u>Max</u>	imum Ne	gative	Shoulde	r Cross	Slope,	m/m			
120	-	.00	.01	.02	.03	.04	.06			
100	-	.00	.01	.02	.03	.04	.06			
80	-	.00	.02	.02	.03	.04	.06			
60	.00	.01	.02	.02	. 03	.04	.06			
40	.00	.01	.02	.02	.03	.04	.06			

Inspection of the sensitivity of these design recommendations to the criterion for driver discomfort reveals considerable variance in the recommendations over a range of  $\pm$  .03 g in the discomfort threshold. Given this sensitivity; the uncertainty of the optimum level; the uncertainty of the distribution of lateral offset, speed and radius of actual shoulder traversals on highway curves; and the practicality of applying various results; consideration should be given to an 0.31 g threshold. With this threshold, an appropriate single recommendation for wider shoulders would be a 0.08 maximum cross-slope break. In other words, for those drivers who recovered from a full traversal onto the shoulder, only a few would have maximum discomfort levels above 0.31 g. On the other hand, for a partial traversal, which is probably the more frequent event, most drivers would not exceed a maximum discomfort level of about 0.26 g assuming the same traversal path with the less lateral offset.

#### Narrow Shoulder Design Considerations

Adoption of driver discomfort level as a basis for cross-slope break design has important implications in the treatment of narrow shoulders. When less than full-width shoulders are selected for design, an implicit decision has been made to not accommodate 4-wheel traversals with the designed shoulder. Traversals which are possible on narrow shoulders, and for which the cross-slope break should therefore be designed, include a range of 2-wheel traversals.

As has been demonstrated previously, the driver discomfort level is largely a function of negative shoulder slope. Figure 3 illustrates the sensitivity of lateral placement of the vehicle during a 2-wheel traversal on effective negative shoulder slope. For increasingly wider shoulders, the maximum effective negative shoulder slope increases. It can be shown, therefore, that relatively large negative slopes are tolerable on very narrow shoulders. Conversely, as shoulder width increases, permissible shoulder slopes must decrease in order to maintain the established driver discomfort level.

Table II gives tolerable maximum cross-slope breaks for shoulders less than 1.6 m in width. It should be emphasized that cross-slope breaks employing values under those shown in Table II will produce an operationally superior (in terms of lower driver discomfort levels) design.

Table 11

MAXIMUM CROSS-SLOPE BREAKS
FOR NARROW (<1.6m) SHOULDERS

Shoulder Width(m)	Maximum Cross-Slope Break(m/m)					
< 0.6	.18					
0.8	.16					
1.0	.14					
1.2	.12					
1.4	.10					
1.6	.08					

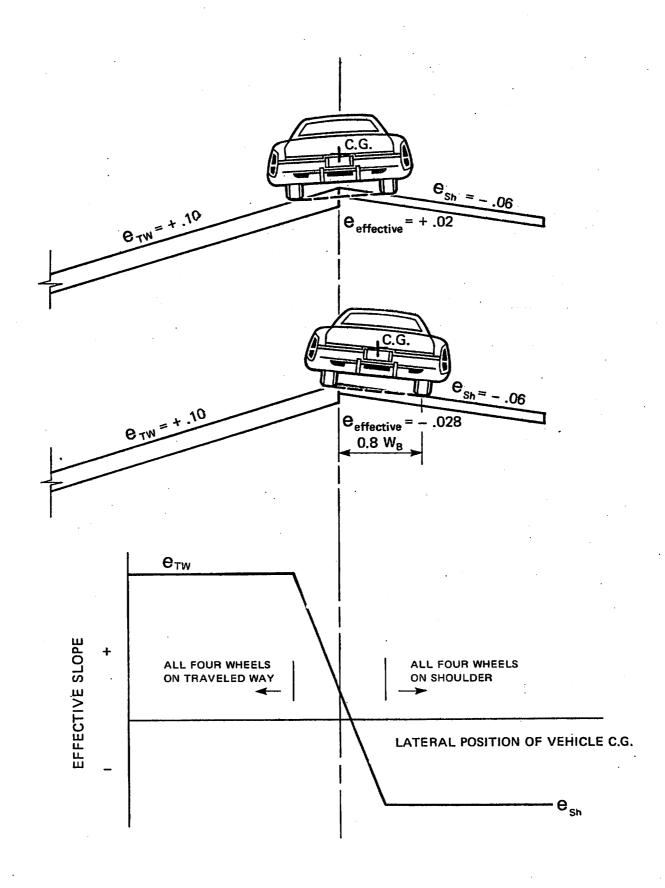


FIGURE 3 EFFECT OF VEHICLE LATERAL PLACEMENT ON EFFECTIVE ADVERSE SHOULDER SLOPE

### Conclusions and Design Implications

1. Shoulder Cross slope. -- The study results clearly show that the driver discomfort level (centrifugal acceleration) in negotiating shoulder traversals on curves is sensitive to speed, degree of curve, shoulder cross slope, and the lateral extent of movement onto the shoulder. For a given path and speed of shoulder traversal, therefore, the driver's discomfort mainly increases with shoulder slope and very little, if any, with the amount of cross-slope break. Thus, for a given design speed, and superelevation of a horizontal curve, the maximum tolerable cross-slope break is a function of the shoulder cross slope; or in other words, the shoulder slope rather than the cross-slope break is the controlling feature.

From the above discussion, as well as that dictated by logic and practical experience, the most important conclusion of this research is: where a negative shoulder cross slope is tolerable for a recovery maneuver, that shoulder cross slope should always be the minimum that is consistent with drainage requirements. This practice will minimize driver discomfort levels and maximize safety for drivers who need the shoulder as a secondary recovery area.

2. Cross-slope Break Requirements for Full Shoulders.—For paved or gravel shoulders with widths of 1.6 m or greater, where the shoulder cross slope is intended to accommodate up to a 4-wheel traversal onto the shoulder, research indicates a maximum tolerable cross-slope break of 0.08 m/m. For superelevation rates between 0.02 and 0.06 m/m, this criterion allows maximum (negative) shoulder cross slopes ranging from 0.06 to 0.02 m/m, respectively. For superelevation rates exceeding 0.06 m/m, a different kind of shoulder cross-slope design is required. The alternatives are to either carry the superelevation rate across the shoulder, or to continue this upward cross slope about half way across the shoulder and then break the remainder (or outer half) of the shoulder with a negative slope. Figure 4 illustrates this practice.

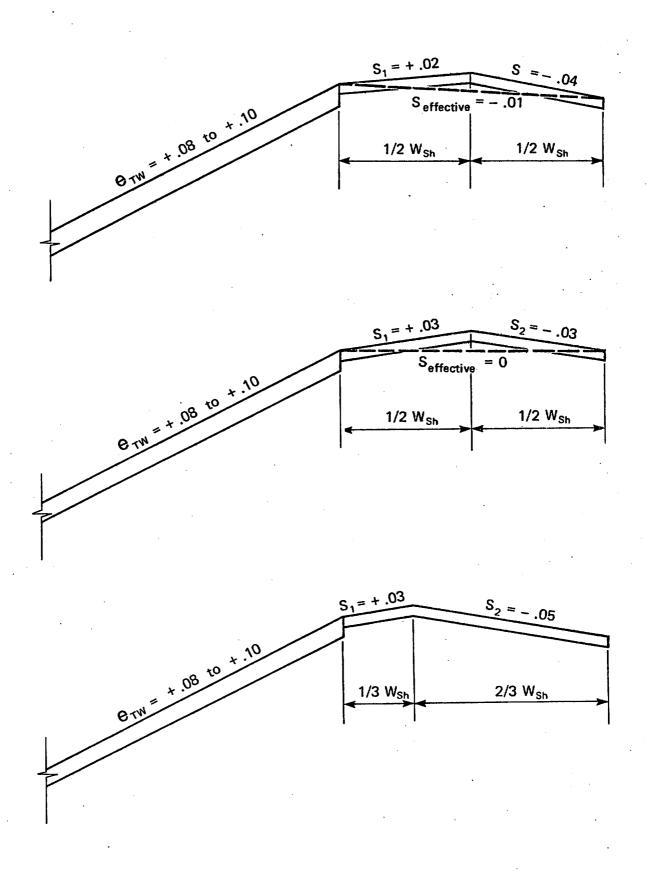


FIGURE 4 BROKEN SHOULDER DESIGNS FOR FULL (≥1.6 m) SHOULDERS

3. <u>Cross-slope Break Requirements for Narrow Shoulders</u>.--For paved or gravel shoulders with widths less than 1.6 m, which are designed to only accommodate 2-wheel traversals within the bounds of the shoulder, this research has demonstrated that the maximum tolerable cross-slope break varies as follows:

Shoulder Width(m)	Maximum Cross-slope Break(m/m)						
<u>&lt;</u> 0.6	0.18						
0.8	0.16						
1.0	0.14						
1.2	0.12						
1.4	0.10						
1.6	0.08						

Although these greater cross-slope breaks do not further compromise safety beyond the implicit decision of choosing the narrower shoulder, Conclusion I regarding minimizing (negative) shoulder cross slope consistent with drainage requirements should be strictly observed. Actually, if a smaller than maximum cross-slope break is feasible, it will tend to compensate for the reduced safety of the less than full shoulder width.

The conclusion of greater tolerable cross-slope breaks for narrower shoulders has important implications for rehabilitation projects where (1) narrow shoulders cannot be widened, (2) pavements are widened at the expense of shoulder width; and/or (3) superelevation rates are increased on roadways with narrow shoulders. In these extreme or isolated cases where the prior decision has been made to use a narrower shoulder, the greater tolerable cross-slope break designs can accommodate "safe" (i.e., 2-wheel encroachments within the shoulder) shoulder encroachments. In this case, the caveat expressed by Conclusion 1 regarding minimum possible shoulder cross slopes remains as the primary principle of shoulder design. Also, in establishing design criteria the narrower shoulders with greater tolerable cross-slope breaks should be weighed against the sensitivity of traffic operations, the probability of incidents, the distribution of lateral displacements for encroaching vehicles, and other conditions.

4. Special Considerations for Turf Shoulders.--Because of greater cross slopes and lower wet surface skid resistance, turf shoulders present a dilemma in satisfying the proposed cross-slope break requirements. Not only can the AASHTO shoulder cross slope of -0.08 m/m not be met using the 0.08 m/m cross-slope break recommendation for superelevated curves, but also for the path criterion used in this research, even a 0.02 m/m cross slope on a turf shoulder with a 0.25 wet coefficient of friction will produce skidding.

What this discussion suggests is that turf shoulders on the outside of highway curves must either have a positive cross slope or have paved or gravel surfaces with allowable negative cross slopes.

Another implication of this discussion is the unsuitability of turf shoulders on tangent sections of higher speed roadways. For example, on a 100 km/h roadway, a wet turf shoulder with a cross slope of -0.08 m/m and a coefficient of friction of 0.25 could only accommodate a 4-wheel traversal with a 600 m path radius without skidding. Since this kind of shoulder design may not satisfy the objective for secondary recovery, high-speed tangent sections should either have flatter turf shoulders (if possible), or have paved or gravel surfaces. The third option, for existing high-speed tangent sections, is to insure a safe traversable roadside with flat roadside slopes clear of fixed objects.

observed with HVOSM for negative shoulder cross slopes up to -0.06 m/m indicate the severity of vehicular traversals onto the roadsides of highway curves. For example, for a 100 km/h speed and 370 m radius of traversal path, the driver discomfort level would reach about 0.63 g on a 4:1 roadside slope. More important, the lateral friction demand would be close to 0.55 g and the roll angle might be severe enough to create overturning. This kind of relationship between highway curves and overturning accidents, particularly fatal accidents, seems to be partially substantiated by two recent research efforts (5,6). The implications for design might be to (1) design flatter than normal roadside slopes on highway curves (2) justify a greater need for guardrail related to embankment configurations on

highway curves than on tangent sections, and (3) provide wider than normal clear zones on highway curves.

- 6. Considerations for Underdesigned Existing Highway Curves. -- The one HVOSM comparison to test speed sensitivity indicates that the higher cross-slope breaks on existing highway curves where the design speed is at least 10 km/h less than the speed limit (expected operating speed) may cause loss-of-control for otherwise controllable shoulder traversals. Therefore, modifying the shoulder cross slope to carry the superelevation across the shoulder may be a worthwhile accident countermeasure at such locations.
- 7. Consideration of Trucks in Design. -- This study was constrained to the consideration of the dynamic responses of passenger vehicles in traversing shoulders on highway curves. Because of the higher centers of gravity and the fifth-wheel characteristics of tractor-semi-trailer truck combinations, the dynamic responses of these vehicles to similar traversals would probably be more severe than those observed for passenger vehicles. How much more severe these responses would be, however, cannot be estimated from the results of this research.

If trucks were found to be much more sensitive to cross-slope break traversals than passenger vehicles, two additional questions must be addressed for design recommendation. First, do professional truck drivers exhibit higher tolerable levels of driver discomfort? And, second, do shoulder traversals by trucks occur often enough to justify the truck as the "design" vehicle for cross-slope break recommendations? Although truck shoulder traversals may represent only a small portion of all such events and, therefore, trucks may not be the appropriate "design" vehicle, the application of Conclusion 1 will help to ameliorate any increased sensitivities exhibited by trucks. For special cases in which the truck is identified as the design vehicle, the use of a positive (upward) shoulder slope sooner than called for in Conclusion 2 may be appropriate.

## References

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- 8. McHenry, R.R., "The Astro Spiral Jump--An Automobile Stunt Designed via Simulation," SAE Paper No. 7600339, Automotive Engineering Congress and Manufacturing Meeting, October 18-33, 1976.
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#### APPENDIX A

### **HVOSM Modifications**

To perform this research, a number of refinements and revisions to the Highway-Vehicle-Object Simulation Model (HVOSM) program were required. These refinements and revisions included changes in the definition of the terrain, additional outputs of vehicle responses and revision of the Path-Following Driver Model. Additionally, two preprocessing programs were developed to simplify the interface between highway definition and HVOSM inputs.

### Ground Contact Point Interpolation

Prior to the present research effort, the FHWA distributed version of the HVOSM computer program contained the assumption that the terrain slopes under each wheel of the simulated vehicle remain constant within the terrain region that is covered by the combination of camber, pitch and steer angles. The elevations and slopes of the terrain under the individual wheel centers of the vehicle were obtained by interpolation of the terrain tables. A "ground plane" through the terrain point directly under the wheel center was used in the determination of the ground contact point (equation (44), page 192 of Reference 7; equation (61), page 104 of Reference 9).

Earlier simulation studies of ramp traversals (e.g., Ref. 7, 8) revealed a minor problem with erroneous extensions of the ends of ramps (see Figure A-1). In the present application to cross-slope breaks, the wheel centers and corresponding ground contact points can be on opposite sides of an interpolation boundary (see Figure A-2) and the erroneous terrain elevations can be sustained for a significant period of time. An alternate version of the HVOSM RD2 which was obtained from Calspan Corporation was found to contain changes dated 9/16/76 in Subroutine INTRP5(INDX) which corrected the interpolation problem related to a simulated transition across a pavement edge that includes a significant slope change.

The related changes were incorporated into the FHWA distributed version of the HVOSM RD2 which is being utilized for the present research effort as follows:

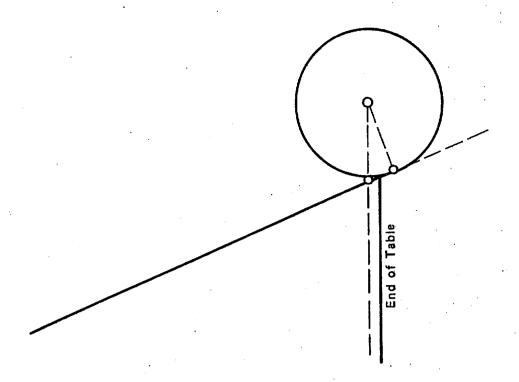


FIGURE A1 ILLUSTRATION OF ERRONEOUS EXTENSIONS OF ENDS OF RAMPS

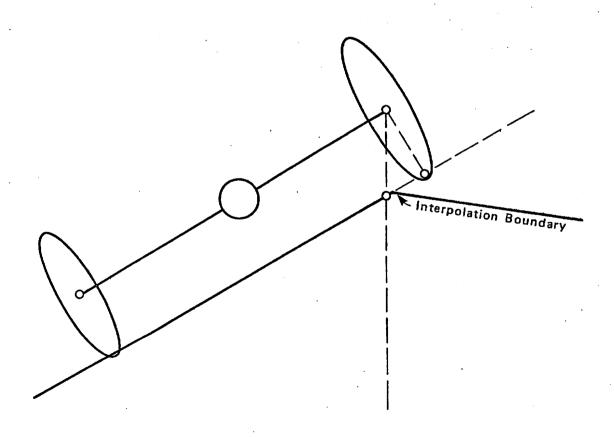


FIGURE A2 PROBLEMS WITH GROUND CONTACT POINT DETERMINATION
NEAR CROSS-SLOPE BREAKS

1. Prior to calculation in Subroutine INTPR5(INDX) of the pitch and slope of the terrain under each individual wheel, the tire contact point as determined from the previous rolling radius and current orientation is calculated. This contact point is then used for calculation of the pitch and camber of the terrain under the wheel. The code associated with this modification is as follows:

```
10 TCPH = COS (PHII(INDX))
TSPH = SIN (PHII(INDX))

BMTX13 = - AMTX(1,2) * TSPH + AMTX(1,3) * TCPH

BMTX23 = - AMTX(2,2) * TSPH + AMTX(2,3) * TCPH

XXX = XP(INDX) + BMTX13 * HI(INDX)

YYY = YP(INDX) + BMTX23 * HI(INDX)
```

where: PHII(INDX) = Camber Angle of wheel INDX relative to vehicle

XXX = X Coordinate of Ground Contact Point of wheel INDX

YYY = Y Coordinate of Ground Contact Point of wheel INDX

HI(INDX) = Previous time interval rolling radius for wheel INDX

- 2. Subroutine INTRP5 then calculates the pitch and camber of the terrain under wheel INDX as previously documented in Reference 7 and Reference 9.
- 3. Prior to the return from Subroutine INTRP5(INDX), the pitch, camber and elevation of the terrain under the ground contact point is used to calculate the corresponding elevation of the terrain under the wheel center for subsequent use in Subroutine GCP(I). The code associated with this is as follows:

```
= TCPG * SIN (THGI(INDX))
        TCPG
        TCB
                   = ~ SIN (PHGI(INDX))
        TCG
                   = COS (THGI(INDX)) * TCPG
                   = PX(INDX) - XXX
        XDF
                   = YP(INDX) - YYY
        YDF
        ZPGI(INDX) = ZPGI(INDX) - (TCA * XDF + TCB * YDF)/TCG
       THGI(INDX) = Pitch angle of terrain under wheel INDX with
where:
                     respect to the space-fixed axes
        PHGI(INDX) = Camber angle of terrain under wheel INDX with
                     respect to the space-fixed axes
        XP (INDX)
                   = X coordinate of the wheel center INDX with
                     respect to the space-fixed axes
        YP(INDX)
                   = Y coordinate of the wheel center INDX with
                     respect to the space-fixed axes
        ZPGI(INDX) = Z coordinate of the wheel center INDX with
```

respect to the space-fixed axes

# Effective Range Angled Boundary Option (ERABO)

The original purpose of the angled boundaries as documented in Reference 9 was to permit the simulation of abrupt slope changes and/or linear terrain irregularities such as ridges that intersect the roadway at angles substantially different from 90 degrees (e.g., edges or cracks in pavement, railroad tracks, etc.). The angled boundaries served to preclude the "rounding," by interpolation, of these profile changes. Up to four angled boundaries were available to the user, but the user was restricted by the requirement that there be a minimum of two tabular values between like boundaries (i.e., two angled boundaries or two Y' boundaries) or between a boundary and the beginning or end of a terrain table.

Within the present research effort, the angled boundaries have been used to approximate chords of a circular arc representing the edge of the pavement and separating a roadway curve from the shoulder. This utilization requires placement of the angled boundaries at close intervals not in keeping with the stated limitations of the original version.

The code in subroutine INTRP5(IND) of the HVOSM RD2 version uses the following interpolation procedure for choosing the appropriate angled boundary:

- 1. The highest number terrain table applicable to the wheel is determined.
- 2. The particular grid segment within which the wheel is located is determined.
- 3. The angled boundaries are scanned and the first angled boundary to pass through the grid segment in which the wheel is located is chosen.

Modification of the procedure to limit the ranges of the angled boundaries and, thereby, to permit their use to approximate a circular arc is the objective of the ERABO option. It gives the HVOSM user control over the X and/or Y range in which a specific angled boundary is used.

When used, the ERABO option performs additional tests to determine if the ground contact coordinates are within the effective range of a given angled boundary. If they are, the modified program will proceed with the interpolation procedure. If not, the modified program ignores the particular

angled boundary and continues the scan of other angled boundaries. Source modification of HVOSM included the following:

- 1. Modification of Subroutine BLKØ5 to include the inputs defining the ranges of boundaries.
- Modification of Subroutine INTRP5(IND) to include additional tests of the ranges of the angled boundaries.

Other related modifications were made in subroutine BLK05 and COMMON/INPT/ to permit the input of up to eight angled boundaries per table.

It was also found to be necessary to automate the generation of multiple angled boundaries and their corresponding effective ranges for the approximation of the successive chords of a circular arc representing the edge of the pavement and separating the roadway curve from its shoulder.

# Additional Outputs

Additional calculations and outputs of the existing HVOSM RD2 program were found to be required to enable the evaluation of the cross slope break study. The revisions were as follows:

1. "Discomfort Factor! -- The lateral acceleration output of HVOSM corresponds to measurements made with a "hard-mounted," or body-fixed accelerometer oriented laterally on the vehicle. During cornering, the lateral acceleration of the vehicle is, of course, directed toward the center of the turn. On a superelevated turn, the component of gravity that acts laterally on the vehicle is also directed toward the turn center. Thus, the lateral acceleration output is increased by superelevation.

Since the vehicle occupants respond to centrifugal force, their inertial reaction is toward the outside of a turn and therefore the component of gravity that acts laterally on them in a superelevated turn reduces the magnitude of the disturbance produced by cornering. A corresponding program output has been defined to evaluate occupant discomfort in turns.

The effects of a vehicle's roll angle and lateral acceleration on occupants are combined in a "discomfort factor" relationship which represents the net

lateral disturbance felt by the occupants (i.e., the occupants reaction to the combined effects of the lateral acceleration and roll angle).

The "discomfort factor" is coded in the following form:

DISCOMFORT FACTOR = - YLAT + 1.0 \* SIN Ø

where: DISCOMFORT FACTOR = G-units

YLAT = Vehicle Lateral Acceleration in vehicle-fixed

coordinate system, G units

Ø = Vehicle roll angle, radians.

Calculations related to the discomfort factor and corresponding outputs were incorporated into the HVOSM.

2. <u>Friction Demand.</u>—The friction demand is defined to be the ratio of the side force to the normal load at an individual tire. The friction demand is indicative of the friction being utilized by each individual tire.

The standard outputs of HVOSM include the side force and normal force for each tire. Coding changes were incorporated to calculate and print out the friction demand for each tire at each interval of time.

#### Terrain Table Generator

The primary research mode of Federal Highway Administration Research Contract DOT-FH-11-9575, "Effectiveness of Design Criteria for Geometric Elements," uses the HVOSM technology for analytical study of the dynamics of vehicle traversals of highway curves with widely varying combinations of geometrics.

The version of the HVOSM maintained by FHWA has the capability of accepting a 3-dimensional definition of the highway surface. The manual generation of these inputs to the HVOSM, however, is time consuming, and the nature and number of geometric configurations to be studied in the contract required automation of the procedure.

The Terrain Table Generator (TTG) was developed as an effective, cost-beneficial interface between standard roadway geometric descriptors and inputs to the HVOSM.

#### Driver Model

A recognized problem in the use of either simulation models or full-scale testing in relation to investigations of automobile dynamics is the manner of guiding and controlling the vehicle. Repeatability is essential, and the control inputs must be either representative of an average driver or optimized to achieve a selected maneuver without "hunting" or oscillation. In the present investigation of geometric features of highways, the transient portions of the vehicle responses constitute the justification for application of a complex computer simulation. The steady-state portions of the vehicle responses can be predicted by means of straightforward hand calculations. Thus, it is essential that the transient responses should not be contaminated by oscillatory steering control inputs.

The Driver model contained in the distributed version of the HVOSM Vehicle
Dynamics program was to be incorporated into the HVOSM Roadside Design version,
but it proved to be inadequate for the present research effort. Therefore,
new routines were written for the HVOSM Roadside Design program to accomplish
the following:

- 1. A "wagon-tongue" type of guidance algorithm to calculate path errors.
- 2. Interface within HVOSM to convert inputs of path descriptors to second-order polynomial definitions of the desired path.
- 3. Inclusion of a "neuro-muscular" filter within HVOSM to enable smooth driver steering activity.

The related revisions to the Driver model were incorporated into the FHWA distributed Roadside Design version of the HVOSM. However, the revised path-following algorithm was found to produce sustained oscillations about a specified path under some operating conditions. Since the extent of oscillation is dependent on the guidance system parameters as well as the vehicle speed and path curvature, it is possible to obtain peak values of transient response predictions that reflect an artifact of the guidance system rather than a real effect of the highway geometrics under investigation. For example, in Reference 10, comparisons are made between peak transient and steady-state response values which are believed to be more reflective of effects of the guidance system than of the simulated roadway geometrics.

Therefore, the following additional modifications were added to the Driver model:

## 1. Damping

A damping term was added to limit the extent of steering activity. Initial runs utilizing the damping term exhibited a reduction in the steering activity as expected. However, they were also found to contain an unexpected initial disturbance. This fact led to the discovery of an initialization problem in the path-following algorithm (see (3) below).

## 2. Steer Velocity

In addition to the damping term, an adjustable limit on the steering angle velocity was incorporated in the path-follower algorithm, enabling the user to limit the maximum instantaneous front wheel steer velocity to a selected value.

### 3. Steer Initialization

For runs such as those being performed in relation to the cross-slope break study, the starting point must be relatively close to the cross-slope break to achieve an economical use of computer time. Thus, the input of an initial steer angle to approximate steady-state steer was required. Previously, the path-follower algorithm was initialized to a steer angle of 0.0 degrees, regardless of the input value for the initial steer angle. Corresponding revisions were made to Subroutine DRIVER to enable input of an initial steer angle.

A revised listing of Subroutine DRIVER, including the cited modifications is presented in Figure A-10.

Table A-1 documents the values for probe length, PGAIN and QGAIN utilized to date for the reported research effort. The tables are presented as a guide for future utilization of the revised Driver model.

Table A-1
CROSS-SLOPE BREAK STUDY DRIVER INPUTS

Run No.	Speed Ft/Sec	Vehicle Path Ft	Path V <sup>2</sup> /R G-Units	Break %	PGAIN Deg/Ft	QGAIN Deg-Sec/Ft	Probe Length Ft
CSB1	109.0	1312.3	.28	12	.516	.034	103.
CSB2	109.0	1312.3	.28	16	.516	.034	103.
CSB3A	109.0	1427.2	.26	12	. 254	.027	54.2
CSB6A	109.0	1722.4	.21	8	.165	.017	102.7
CSB7	109.0	1922.6	.19	4	.516	.034	103.0
CSB8	109.0	1922.6	.19	8	.516	.034	103.0
CSB9B	91.0	885.8	.29	14	.413	.027	60.0
CSB12A	91.0	1043.3	.25	10	.523	.055	43.3
CSB13A	91.0	1151.6	.22	6	. 447	.034	60.0
CSB14C	91.0	1151.6	.22	10	. 447	.034	60.0
CSB14PA	109.0	1151.6	.32	10	.344	.034	78.3
CSB14PP	91.0	1151.6	.22	10	. 447	.034	60.0
CSB16A	73.0	538.6	.31	14	.825	.069	41.7
CSB18D3	73.0	649.6	.25	8	1.06	.138	40.0
CSB19D2	73.0	649.6	.25	12	1.06	.000	40.0
CSB20A	73.0	695.5	.24	8	.894	.089	41.7

### HVOSM Run Setup Procedure

Procedure for setup of a Cross-Slope Break (CSB) study run used in the present research effort:

- Analytically determine the extent of roadway required to meet the requirements of the particular run (i.e., roadway radius, vehicle path radius, etc.).
- 2. Perform an ERABO run to define the edge of roadway. Put the ERABO outputs in HVOSM form to define the angled boundaries and their effective ranges.
- 3. Perform two TTG runs, one with the shoulder slope, one with the roadway superelevation.
- 4. Determine, from TTG outputs, the shoulder and roadway points for each table.
- 5. Insert the corresponding points for the shoulder into the roadway tables.
- 6. Insert the roadway/shoulder tables into the HVOSM input deck.
- 7. Add the angled boundaries and their effective ranges to the HVOSM input deck.
- 8. Determine analytically the vehicle's heading, location and desired path inputs required to cross onto the shoulder from the roadway at approximately 0.7 sec after initial simulation time.
- Insure the vehicle is dynamically in equilibrium and perform the simulation run.

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CROSS-SLOPE BREAK STUDY: FH-11-9575
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24.50	0 20.0	3 15.23	10.04	4.54	-1.38	-7.68		-12.27	21 502
-10.80	-9.29	-7.67	-6.01	0.0	0.0	0.0	0.0		22 502
36.37			21.86						23 502
-13,12		-9.98	-8.31	0.0	0.0	0.0	0.0		24 502
48.2	5 43.7	5 38.89	33.67	28.10	22.13	15.75	9.11	1.97	25 502
-5.40	3 -13.2	7-12.29	-10.59	0.0	0.0	0.0	0.0	0.0	26 502
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4057.3954470.7974470.7974883.5664883.5665295.5785295.5785708.00 1.0
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5706.7116116.8366116.8366525.8286525.8286933.5626933.5627339.9222.0
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7339.9227744.7707744.7708147.9928147.9928549.4658549.4658950.0 2.0
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END OF DATA

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05710 C SUBROUTINE DRIVER FOR HVOSM RD-2
05720 C
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            DIMENSION AMTX(3,3),PPD(50),TPD(50)
05750
            COMMON/PATHD/IPATH, KLI, DI(10), RLI(10), NPTS, XINIT, YINIT,
                   PSA,DELL,X(100),Y(100),DX(100),DY(100),D(100)
05760
05770
            COMMON/WAGON/IWAGN, TPRB, DPRB, PLOTH, PMIN, PMAX, PGAIN, QGAIN, PSIFD
05780
            COMMON/FILT/ IFILT.TIL .TI .TNT .TAUF
05790
            COMMON/INTG/ NEQ ,T ,DT ,VAR(50),DER(50)
            COMMON/ACC/CMFCG, CMFA1, CMFA2
05800
05810
            DATA NPDMAX/50/,NPD/0/,DPSL/0.0/,N/0/
05820
            JJ = 0.
05830
            IF(IWAGN.EQ.0)GO TO 90
05840
            1 = الل
05850
            PSIA = PSI
05860
            DTP = DPRB
05870
            DPS = 0.0
05880
            DPSI = 0.0
05890
            IF(IFLAG.EQ.0)GO TO 90
05900
            IF(TPRB.GT.T + 0.1*DT)GO TO 10
05910 C
        COMPUTE NEW CHANGE IN STEER ANGLE
            TPRB = TPRB + DPRB
05920
            XP = VAR(18) + AMTX(1,1)*PLGTH
05930
05940
            YP = VAR(19) + AMTX(2,1)*PLGTH
05950
            CALL PROBE(XP, YP, NPTS, X, Y, DX, DY, D, IPRB, DIST, XX, YY)
        SELECTED POINT INDEX IPRB AND LOCATION OF CLOSEST POINT ON PATH XX, YY
05960 C
05970 C
        ARE NOT CURRENTLY USED
05980
            IF(DIST.EQ.O.O)GO TO 8
05990
            SGND=DIST/ABS(DIST)
            IF(T.NE.TPRB) DDIST = (DIST-DISTA)/DPRB
00000
            IF(ABS(DIST).GT.PMIN)DPS = -PGAIN*(ABS(DIST)-PMIN)*SGND
06010
       9
06020
                                        -QGAIN*DDIST
06030
            IF(ABS(DIST).LE.PMIN) DPS= -QGAIN*DDIST
06040
            IF(IFILT.EQ.0)G0 TO 55
            IF(NPD.EQ.NPDMAX)GO TO 10
06050
06060
            NPD = NPD + 1
06070
            PPD(NPD) = DPS - PSIA
            TPD(NPD) = T + TAUF
02080
06090
         10 IF(IFILT.EQ.0)GO TO 55
06100 C
06110 C FILTER
06120 C
06130
            IF(NPD.EQ.NPDMAX) GO TO 10
06140
            TPDTMP = TPD(N)
06150
            DO 20 NA = 1,NPD
06160
            N = NPD + 1 - NN
06170
         20 IF(T.GE.TPD(N))GO TO 30
06180
            GO TO 90
        30 IF(TPDTMP.LT.TPD(N)) DPSL = 0.0
06190
06200
             DPSI = PPD(N)*TMT*EXP(-(T - TPD(N))/TIL)/TIL
             DPSN = PPD(N) - TIL*DPSI
06210
06220
            DTP = 0.0
06230
            DPS = DPSN - DPSL
06240
             DPSL = DPSN
06250
             IF(NPD.EQ.1)G0 TO 50
06260 C
 06270 C
```

```
06280
       35
                = 1
06290
            DO 40 NN = N_1NPD
06300
            PPD(L) = PPD(NN)
06310
            TPD(L) = TPD(NN)
06320
         40 L = L + 1
06330
           NPD = L - 1
06340 C
06350
         50 PSI = PSIA + DPS
06360
           GO TO 58
06370
         55 PSI = DPS
06380
         58 CONTINUE
06390 C CHECK PREVIOUS TIME INTERVAL CONFORT FACTOR (SEE SUBROUTINE OUTPUT)
06400 C
            IF GREATER THAN PHAX ALLOW ONLY REDUCTION IN STEER ANGLE
06410
           IF((PMAX.GT.O.O).AND.(ABS(CMFA1).LT.PMAX))GD TO 60
06420
           IF(ABS(PSI).GT.ABS(PSIA)) PSI=PSIA
06430
       60 CONTINUE
06440 C CHECK MAX STEER ANGLE
06450
           IF((OMGPS.GT.O.O).AND.(ARS(PSI) .GT.
06460
               PSI = SIGN(OMGPS,PSI)
06470
           IF(DTP.NE.O.O)DPSI = (PSI-PSIA)/DTP
06480 C###
            1/16/81 MCI
                            *****************************
06490
           DPS0 = DPS*57.2958
06500
           PSIA0 = PSIA*57.2958
06510
           PSIO = PSI*57.2958
06520
           DELPSI = PSIO- PSIAO
           XPFT = XP/12.0
06530
06540
           YPFT = YP/12.0
06550
           XXFT = XX/12.0
06560
           YYFT = YY/12.0
06570 C
           IF(FKD.EQ.1.0) GO TO 90
06580
           IF(KPAGE.LE.50.AND.T.NE.0.0000) GO TO 110
06590
           WRITE(50,100)
00890
       100 FORMAT(
06610
          A1H1,33X,37HPROBE CODRDINATES
                                          PATH COORDINATES, 5X, 3HPSI, 6X,
06620
          B3HDPS.6X,4HPSIA,2X,7HDPSI
                                      ,2X,7HDPSN
                                                   ,5HIFLAG,2X,4HIPRB/
06630
          C31H TIME
                         DELTA PSIF
                                      ERROR ,6X,1HX,9X,1HY,10X,1HX,8X,1HY/
06640
                           (DEG)
                                       (IN) ,4X,4H(FT),6X,4H(FT),7X,
          D31H (SEC)
06650
          E4H(FT),5X,4H(FT)/)
06660
           KPAGE = 0
06670
       110 WRITE(50,120) T, DELPSI, DIST, XPFT, YPFT, XXFT, YYFT, PSIO, DPSO,
08680
                         PSIAD, DPSI, DPSN, IFLAG, IPRB
          A
06690
       120 FORMAT(1H ,F7.3,2(4X,F7.3),2(3X,F7.1),2X,2(2X,F7.1),3(2X,F7.4),
06700
                    2X,F7.5,2X,F7.5,2X,13,2X,12)
           KPAGE = KPAGE + 1
06710
06720
        90 RETURN
06740
           END
```

```
133100 C SUBROUTINE PATH: PATHM.FOR
133200 C
         PATH GENERATOR HVOSM RD-2
133300
            SUBROUTINE PATH
133400
            COMMON/PATHD/IPATH ,KLI ,DI(10),RLI(10),
133500
                       NPTS, XINIT, YINIT, PSA, DELL,
                       X(100),Y(100),DX(100),DY(100),D(100)
133600
           2
133700 C LIMIT ARRAY SIZES
133800
            IF(KLI.GT.10)KLI = 10
133900
            IF(NPTS.GT.100)NPTS = 100
134000
           CALL SETD(KLI, DI, RLI, NPTS, DELL, D)
134100 C SETD WAS MODIFIED ON 30 DEC 1980 TO PRODUCE SPIRAL
134200 C INITIALIZE FIRST POINT AND TANGENT
134300
           X(1) = XINIT
134400
           Y(1) = YINIT
134500
           BX(1) = COS(PSA)
134600
           DY(1) = SIN(PSA)
134700 C
           CALL PATHG(NPTS, DELL, X, Y, D, DX, DY)
134800
134900 C
135000
           RETURN
135200
           END
```

```
135400 C PATHG
135500 C
          PATH GENERATOR, SUBROUTINE PATHS
135600 C
135700
             SUBROUTINE PATHS (NPTS, DELL, X, Y, D, DX, DY)
135800
             DIMENSION X(1), Y(1), DX(1), DY(1), D(1)
135900
             DATA RAD/0.017453292519943296/
136000 C INITIALIZE
136100
             CONS = DELL*RAD/200.0
136200 C*
136300
                  = DELL*DX(1)
            DXX
136400
            DYY
                  = DELL*DY(1)
136500 C#
136600
            DSI
                  = 0.0
136700
            DC1
                  = 1.0
136800 C START LOOP
136900
            DO 20 I = 2, NPTS
137000 COMPUTE SINE AND COSINE OF HALF SECTOR ANGLE
137100
                 = CONS#D(I-1)
            DS2
137200
            DC2
                 = SQRT((1.0-DS2)*(1.0+DS2))
137300 C**
137400 COMPUTE SINE AND COSINE OF SECTOR ANGLE
137500
            SP
                  = 2.0*DS2*DC2
137600
            œ
                  = 1.0 - 2.0*DS2**2
137700 C UPDATE TANGENT VECTOR
            DX(I) = CP*DX(I-1) - SP*DY(I-1)
137800
137900
            DY(I) = SP*DX(I-1) + CP*DY(I-1)
138000 C**
138100 COMPUTE SINE AND COSINE OF AVERAGE SECTOR ANGLE
138200
            SP
                         DS1*DC2 + DC1*DS2
                  =
138300
            CP
                         DC1*DC2 - DS1*DS2
138400 COMPUTE NEW INCREMENTS
138500
            DXS
                  = DXX
138600
            DXX
                  = DXS*CP - DYY*SP
                  = DXS*SP + DYY*CP
138700
            DYY
138800 C UPDATE POSITION
138900
            X(I) = X(I-1) + DXX
139000
            Y(I) = Y(I-1) + DYY
139100 C SAVE SINE AND COSINE OF HALF SECTOR ANGLE FOR NEXT I
139200
            DS1
                 = DS2
139300
         20 DC1
                  = DC2
139400
            RETURN
139500 C
139600 C
139700 C
139710
            END
```

```
140800 C PROBE
140900 C SUBROUTINE PROBE: CALCULATES DISTANCE OF A POINT FROM CENTERLINE
141000 C
             SUBROUTINE PROBE(XP, YP, M, X, Y, DX, DY, D, I, DIST, XX, YY)
141100
141200
            DIMENSION X(1), Y(1), DX(1), DY(1), D(1)
            DATA RAD/0.017453292519943296/, ILAST/1/
141300
141400 C INITIALIZE
141500
                 = ILAST
141600
             TEST = DX(I)*(XP-X(I))+DY(I)*(YP-Y(I))
141700
            TSAV = SIGN(1.0, TEST)
            GO TO 15
141800
141900 C
142000 C START SEARCH
142100 C
142200
          7I = I + 1
142300
            IF(I.LE.M)GO TO 10
142400
             IF(TSAV.LT.0.0)G0 TO 20
142500
            I = M
142600
            GO TO 25
142700
         10 TEST = DX(I)*(XP-X(I))+DY(I)*(YP-Y(I))
142800
            IF(TEST*TSAV.LE.0.0)GO TO 25
142900
         15 IF(TEST)20,25,7
143000
         20 I = I - 1
143100
            IF(I.GE.1)60 TO 10
143200
             IF(TSAV.GT.0.0)60 TO 7
143300
            I = 1
143400 C
143500 C FINISH SEARCH
143600
         25 IF((TEST.LT.0.0).AND.(I.GT.1)) I=I-1
143700
             ILAST = I
143BOO C FINISH OF DETERMINATION OF I
143900 C
144000 C
144100 C
144200 C
144300 CALCULATE DISTANCE
             ZDN = -DY(I)*(XP-X(I))+DX(I)*(YP-Y(I))
144400
             CONS = D(I)*RAD*0.005
144500
             ZDZ = ((XP-X(I))**2+(YP-Y(I))**2)*CONS
144600
             DIST = (ZDN-ZDZ)/(0.5+SQRT(0.25-CONS*(ZDN-ZDZ)))
144700
144800 C
144900 CALCULATE POSITION OF CLOSEST APPROACH POINT ON ARC
145000 C THE FOLLOWING CODE MAY BE DELETED AND THE REFERENCES TO XX AND YY TAKE
145100 C OUT OF THE CALL IF THE POINT OF CLOSEST APPROACH ON THE ARC IS NOT NEE
145200 C
145300
             DEN = 1.0-2.0*DIST*CONS
145400 C
145500
             IF(DEN.GT.0.0)GO TO 30
145600
             WRITE(6,26)I, XP, YP, DIST, DEN
          26 FORMAT(' SUBROUTINE PROBE HAS NEGATIVE OR ZERO DENOMINATOR'/
145700
            X' IN POSITION FORMULA: IMPLIES POINT NOT IN SECTOR'/16,4F10.4)
145800
            STOP
145900
146000 C THIS STOP SHOULD NEVER OCCUR IN NORMAL USAGE
146100 C
146200
          30 XX = (XP-X(I)+DIST*DY(I))/DEN + X(I)
             YY = (YP-Y(I)-DIST*DX(I))/DEN + Y(I)
146300
          35 RETURN
146400
146600
             END
```

```
146900 C SUBROUTINE SETD FOR HVDSM RD-2
146900 C
           ROUTINE TO SET DEGREE OF CURVATURE FROM DI'S
147000 C
147100
             SUBROUTINE SETD(KLI, DI, RLI, NPTS, DELL, D)
147200
             DIMENSION DI(1), RLI(1), D(1)
147300 C INITIALIZE
147400
            L = 1
147500
             Z
                = 0.0
147600
             DELL2 = DELL*0.5
147700 C START LOOP
147800
             DO 10 N = 1,NPTS
147900
            D(N) = DI(L)
148000
             IF(L.EQ.KLI)GO TO 10
148100
             IF(Z+DELL2.LT.RLI(L))GO TO 10
148200
            D(N) = D(N) + (DI(L+1) - DI(L))*(Z - RLI(L) + DELL)
148300
            1
                                     /(RLI(L+1) - RLI(L) + DELL)
148400
            IF(Z+DELL2.GT.RLI(L+1))L = L + 1
148500
          10 Z = DELL*FLOAT(N)
148600
            RETURN
             END
148700
```

# APPENDIX B

The following pages document representative output from the 13 HVOSM cross-slope break simulations. The following parameters apply:

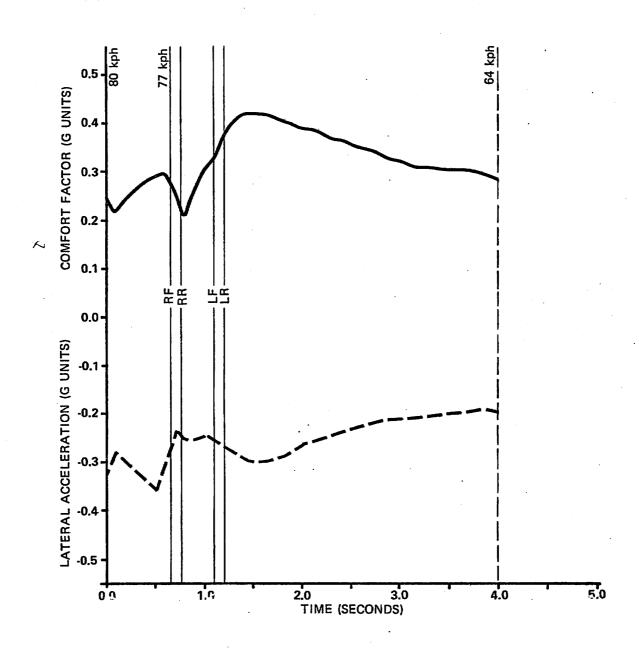
Run CSB-16A

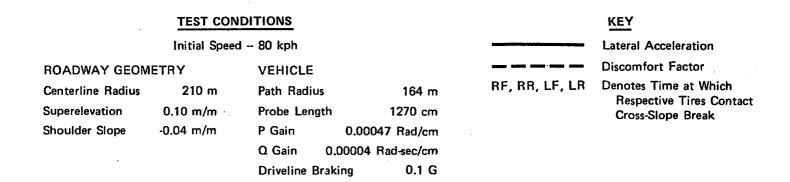
Initial Speed: 80 km/h
Roadway Radius: 210 m

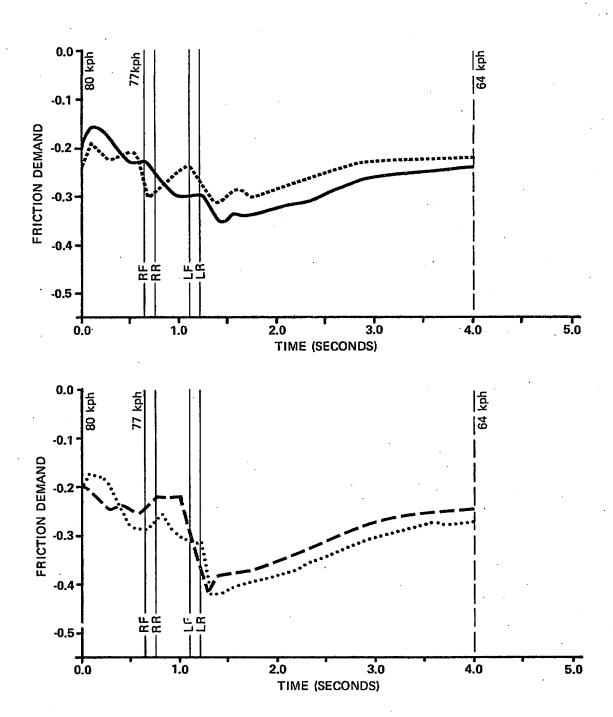
Roadway Superelevation: .10

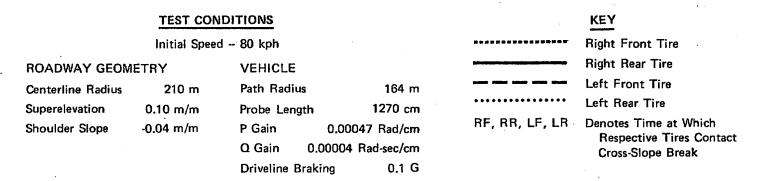
Shoulder Slope: -.04

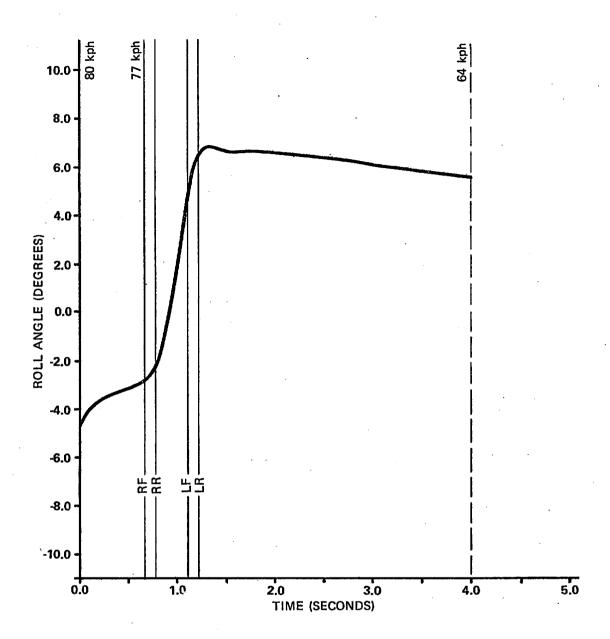
Vehicle Path Radius: 164 m -- four-wheel excursion











# **TEST CONDITIONS**

Initial Speed -- 80 kph

**ROADWAY GEOMETRY** 

210 m

Centerline Radius Superelevation

Shoulder Slope

0.10 m/m

-0.04 m/m

P Gain

**VEHICLE** 

Path Radius

Probe Length

0.00047 Rad/cm

Q Gain

0.00004 Rad-sec/cm

**Driveline Braking** 

0.1 G

164 m

1270 cm

KEY.

RF, RR, LF, LR

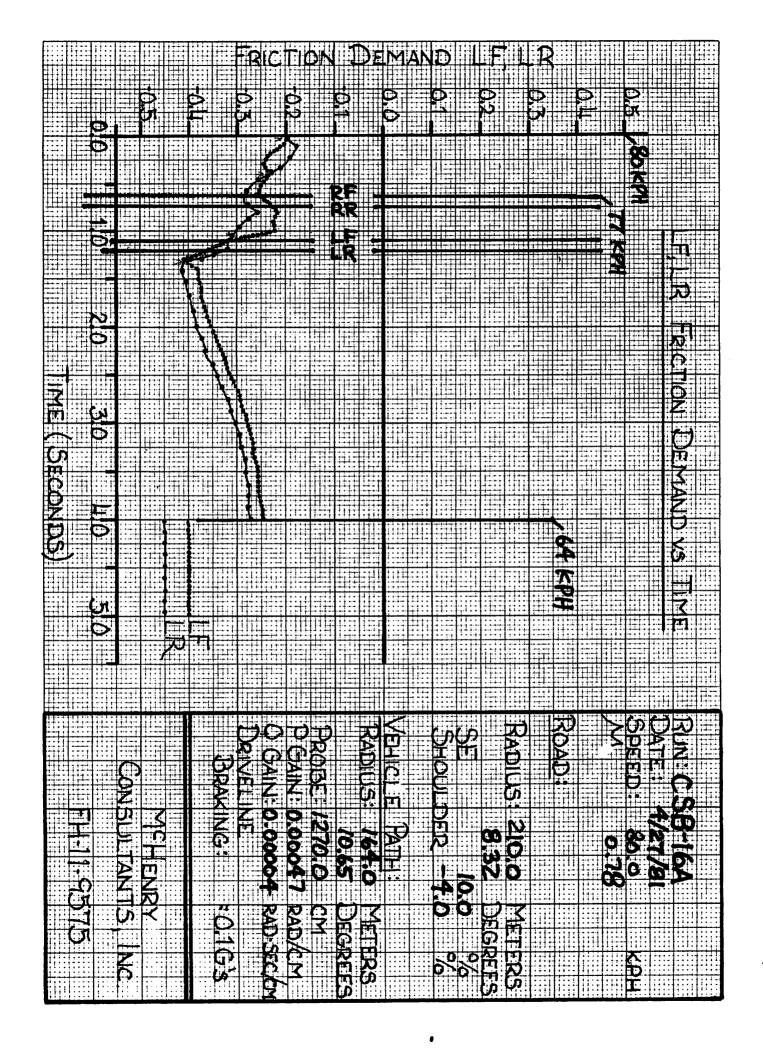
Denotes Time at Which Respective Tires Contact

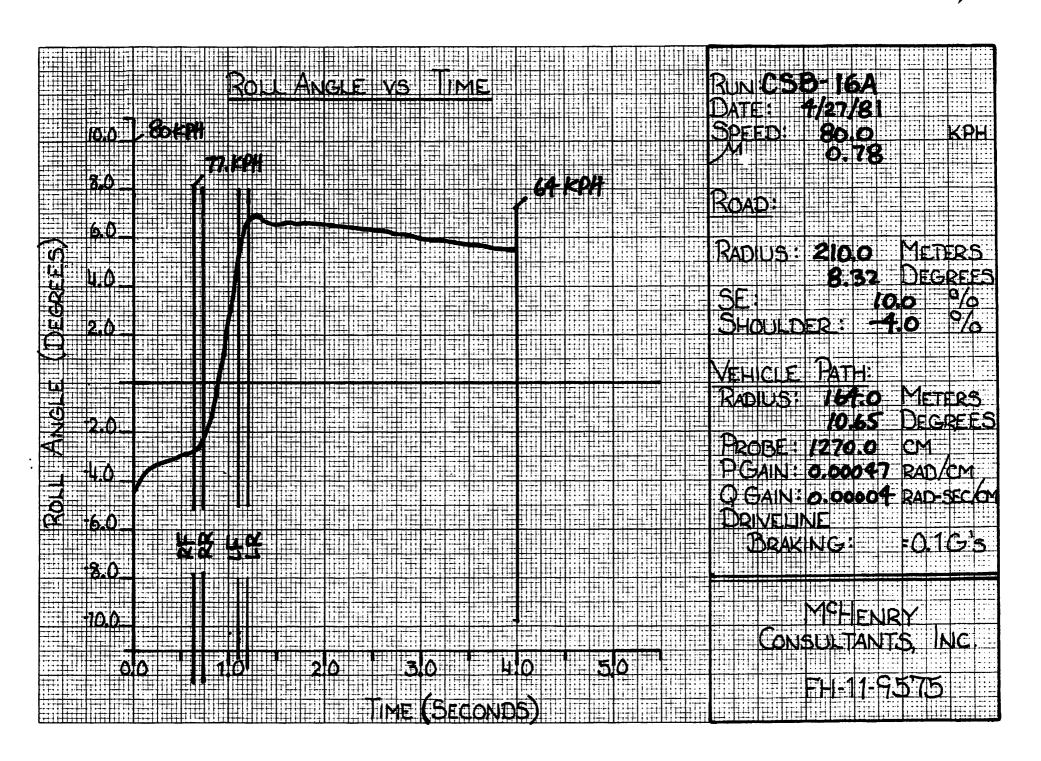
Cross-Slope Break

FIGURE B3 EXAMPLE HVOSM OUTPUT FOR STUDIES OF CROSS-SLOPE BREAKS ---**ROLL ANGLE VS. TIME** 

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list csb#16 nonum DSNAME='MCIJEL.CSB#16' 0 100 CROSS SLOPE BREAK STUDY: DOT-FH-11-9575 0 101 4.00 0.010 0.010 70.0 30.0 30.0 0.08 1.0 0.0 0.0 0.0 0 102 0 103 1 0 104 1 1 1 1 1 1 1 0 200 1971 DODGE CORONET 4-DOOR SEDAN 3760.0 23000.0 23300.0 530.0 550.0 0 201 8.43 0.51 0.82 49.3 68.7 59.8 61.8 0.0 47.0 0 202 0.203 14.0 0.0 10.82 10.68 0.0-14.00.0 0.0 588.0 0 204 105.0 189.0 600.0 600.0 0.50 -2.402.1 0 205 120.0 324.0 600.0 864.0 600.0 0.50 -4.40 3.6 7.48 0 206 6.85 40.0 0.10 38.0 0.10 0 207 40400.0 -5100. 0.02 300.0 1000.0 0.523 100000. 0.010 1.50 0 208 0 209 -3.03.0 1.0 -0.95 -1.22 1 209 -1.26-0.98-0.41 0.0 -0.430 300 FIRESTONE RADIAL VI 0 301 6.0 0.25 1.0 1.0 1.0 .58 1 301 1450.0 5.0 10.0 -37.013.2 3043. 91435. 1.0 0 302 .78 13.2 164 M PATH: LH TURN N/FILTER & DAMPING 0 400 0 401 0.40 0.050 0.0 0.0 1.0 -199.0 -199.0 -199.0 -199.0 -199.0 -199.0 1 401 1.0 1.0 1.0 0.05 0.00905 0.0 0.0 0 402 0 403 1.7366 120. 75.0 0.0 1320. 4.0 -.8875 3000.0 -.8875 6000.0 -.8875 9000.0 0 404 -.8875 0.0 0 405 0.10 500.0 0.0 1.0 500.0 0.0012 .0001 CSB#16: 210 M RADIUS, 10%SE,-4% SHOULDER 0 500 0 501 -600.00 360.00 120.00 1200.00 2400.00 240.00 8.0 -137.76 -140.32 -145.45 -153.15 -163.43 -176.32 -191.82 -209.97 1 501 2 501 89.5 38.5 87.5 86.5 85.5 84.5 83.5 82.5 4.70 4.85 5.24 5.88 6.79 7.95 0.0 0.0 0.0 3 501 -0.09 0.05 2.02 3.19 0.0 0.0 0.0 4 501 0.44 1.10 5 501 -4.88 -4.75 -4.35 -3.68 -2.75 -1.560.00.0 0.0 -9.55 -9.14 -8.47 -7.52 -6.31 0.00.0 0.0 6 501 -9.67 -11.99 -12.34-13.37 -13.25 -12.29 -11.07 0.0 0.0 0.0 7 501 -0.00 -0.35 -1.39 -3.13 -5.56 -8.670.0 0.0 0.0 8 501 8.82 6.36 3.21 24.00 23.64 22.57 9 501 12.00 11.65 10.59 36.00 23.64 22.57 20.78 18.28 15.08 35.64 34.55 10 501 24.00 35.64 34.55 32.73 30.19 26.94 0.0 0.0 0.0 11 501 36.00 -480.00 480.00 120.00 2369.88 3569.88 240.00 8.0 0 502 -55.95 -58.55 -63.76 -71.62 -82.17 -95.42 -111.43 -130.25 1 502 2 502 80.5 79.5 78.5 77.5 76.5 75.5 74.5 81.5 10.14 12.53 0.0 3 502 7.99 0.0 0.0 3.03 4.43 6.09 -1.73 -0.311.37 3.30 5.48 7.90 0.0 0.0 0.0 4 502 0.0 0.0 5 502 -5.04-3.34 -1.39 0.82 3.28 0.0 -11.24 -9.78 -8.06 -6.07 -3.83 -1.350.0 0.0 0.0 6 502 -8.24 -11.94-12.76 -10.75 -8.49 -5.967 502 0.00.0 0.0 -0.11 -4.55 -9.65-13.13 -10.57 0.0 0.0 0.0 8 502 3.64 2.04 -3.79 -10.28 27.39 23.53 18.96 15.52 11.71 7.21 9 502 27.39 23.53 18.96 13.72 7.80 1.23 39.26 35.34 30.71 10 502 35.34 30.71 25.38 19.38 12.71 0.0 0.0 0.0 11 502 39.26 -240.00 840.00 120.00 3516.84 4716.84 240.00 8.0 0.503 187.85 185.16 179.74 171.55 160.51 146.51 129.69 109.76 1 503 69.5 2 503 73.5 72.5 71.5 70.5 68.5 67.5 66.5

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-6.54		-0.87		5.75	9.40	0.0	0.0	0.0	5 503
-11.16		-5.41		1.29	4.99	0.0	0.0	0.0	6 503
			-6.67			0.0	0.0	0.0	7 503
2.7			2-11.15				0.0	0.0	8 503
14.2				4-12.02			0.0	0.0	9 503
25.7			6 2.0		1-12.55		0.0	0.0	10 503
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48.6		1 33.0			1 4.9		0.0	0.0	
			0 4618.8						0 504
588.92	586.08		571.60				504.92		1 504
65.5			62.5		60.5	59.5	58.5		2 504
3.45	7.27		15.56	20.0	24.66	0.0	0.0	0.0	3 504
-0.97	2.90	7.00	11.28	15.78	20.49	0.0	0.0	0.0	4 504
-5.38	-1.46	2.68	7.02	11.59	16.34	0.0	0.0	0.0	5 504
-9.78	-5.81	-1.62	2.78	7.39	12.17	0.0	0.0	0.0	6 504
-12.7	9-10.14	-5.90	-1.46	3.22	8.05	0.0	0.0	0.0	7 504
-1.8	4 -12.0	5-10.18	-5.66	-0.94	3.97	0.0	0.0	0.0	8 504
9.0	7 -1.2	6 -12.1	1-9.86	-5.11	-0.11	0.0	0.0	0.0	9 504
19.9			1 -13.13				0.0	0.0	10 504
30.7	9 20.20		5 -2.69			0.0	0.0	0.0	11 504
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1200.	1346.70	1346.70	1493.35	1493.35	1639.91	1639.91	1786.34	1.0	0 515
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