

UMTA-MA-06-0025-83-2  
DOT-TSC-UMTA-83-1

# **Wheel/Rail Force Measurement at the Washington Metropolitan Area Transit Authority- Phase II Volume II Test Report**

P.J. Boyd  
J.P. Zaiko  
W.L. Jordan

ENSCO, Inc.  
Springfield VA 22151

June 1983  
Final Report

This document is available to the public  
through the National Technical Information  
Service, Springfield, Virginia 22161.



U.S. Department of Transportation  
**Urban Mass Transportation  
Administration**

Office of Technical Assistance  
Office of Systems Engineering  
Washington DC 20590

NOTICE

This document is disseminated under the sponsorship of the Department of Transportation in the interest of information exchange. The United States Government assumes no liability for its contents or use thereof.

NOTICE

The United States Government does not endorse products or manufacturers. Trade or manufacturers' names appear herein solely because they are considered essential to the object of this report.

# Technical Report Documentation Page

1. Report No. UMTA-MA-06-0025-83-2		2. Government Accession No.		3. Recipient's Catalog No.	
4. Title and Subtitle WHEEL/RAIL FORCE MEASUREMENT AT THE WASHINGTON METROPOLITAN AREA TRANSIT AUTHORITY - PHASE II VOLUME II TEST REPORT				5. Report Date June 1983	
				6. Performing Organization Code TSC/DTS-77	
				8. Performing Organization Report No. DOT-TSC-UMTA-83-1	
7. Author(s) P.J. Boyd, J.P. Zaiko, and W.L. Jordan				10. Work Unit No. (TRAIS) UM204/R2608	
9. Performing Organization Name and Address ENSCO, INC.* Transportation Technology Engineering Division 5400 Port Royal Road Springfield, VA 22151				11. Contract or Grant No. DTFR-53-80-C-00002	
				13. Type of Report and Period Covered Final Report May 1981-November 1981	
12. Sponsoring Agency Name and Address U.S. Department of Transportation Urban Mass Transportation Administration Office of Technical Assistance Washington, DC 20590				14. Sponsoring Agency Code URT-10	
15. Supplementary Notes *Under contract to: U.S. Department of Transportation Research and Special Programs Administration Transportation Systems Center Cambridge, MA 02142					
16. Abstract On the basis of preliminary studies, modification of the subway cars of the Washington Metropolitan Area Transit Authority for softer longitudinal primary suspension and tapered wheels was suggested to reduce excessive wheel and rail wear and rail fastener failure.  Force sensing instrumented wheels and other sensors were used to measure the effects of wheel taper and longitudinal stiffness of the primary suspension. Both changes reduced curving flange forces. Suspension compliance had the predominant effect at curves of less than 1000 ft radius, and wheel taper produced the greatest reduction of flange force at larger curves. The combination of both changes was complimentary. Reductions in lateral force of 60%-90% relative to the standard truck were achieved with 1:20 tapered wheels and softer suspension bushings.  No hunting was detected at 75 mph with up to 1:10 wheel taper for either the standard or soft primary longitudinal suspension.  Volume I of this Test Phase report carries the sub-title of Analysis Report.					
17. Key Words Wheel/Rail Forces Wheel/Rail Wear Urban Transit Vehicle Instrumented Wheels				18. Distribution Statement  DOCUMENT IS AVAILABLE TO THE PUBLIC THROUGH THE NATIONAL TECHNICAL INFORMATION SERVICE, SPRINGFIELD, VIRGINIA 22161	
19. Security Classif. (of this report) UNCLASSIFIED		20. Security Classif. (of this page) UNCLASSIFIED		21. No. of Pages 94	
				22. Price	



## PREFACE

In support of the Office of Rail and Construction Technology of the Urban Mass Transportation Administration (UMTA), the Transportation Systems Center (TSC) is conducting analytical and experimental studies to relate transit truck design characteristics, wheel/rail forces and wheel/rail wear rates, in order to provide options for reducing the wear rates of wheels and rails experienced by transit properties and minimizing system life cycle costs of vehicle and track components, while maintaining or improving equipment performance.

As part of this work, TSC planned and implemented a measurement program, in order to obtain onboard wheel/rail force measurements over a representative range of Washington Metropolitan Area Transit Authority (WMATA) operating conditions; obtain data to quantify the load environment on direct fixation fasteners and evaluate the influence of changes in fastener characteristics on fastener performance; evaluate the influence of taper and suspension modifications on high speed stability and to assess the feasibility of a retrofit to the WMATA truck to improve curving performance. These tests were conducted in the fall of 1981. The Analytic Sciences Corporation (TASC), under Contract DTRS-57-80-C-00062, provided support to TSC in these activities, by conducting analyses of the tradeoffs between curving performance and high speed stability, by definition and coordination of in-shop measurements to obtain engineering parameters for use in the analysis, by specification and procurement of the retrofit primary suspension element used in the truck tests, by comparison of measured data, with analytic predictions to assess measurement consistency and by recommending test program modifications to improve the accuracy and completeness of the results relating to the truck modifications. This report describes the work performed by TASC under this effort in support of this measurement program. The report Number is UMTA-MA-06-0025-83-.

Vehicle/truck instrumentation and data acquisition support for the truck tests was provided by ENSCO, Inc., while equipment and support personnel, for conducting selected vehicle and truck measurements in the WMATA shop, were provided by the Transportation Test Center (TTC), Pueblo, Colorado and are provided in this report. The modified primary suspension elements used in the (Phase II) test program were developed and fabricated by the BUDD Co., under sub-contract to and in accordance with design specifications developed by TASC. Vehicles, operators, track rights, shop facilities and shop test support were provided by WMATA.

Wayside instrumentation development, installation and calibration and data acquisition support for measurements of fastener loads was provided by Battelle Columbus Labs, under contractual arrangement with TSC. Results of these measurements will be documented in subsequent reports.

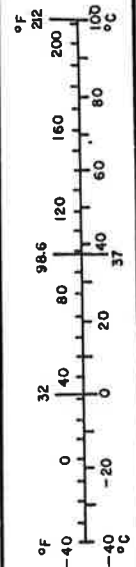
# METRIC CONVERSION FACTORS

## Approximate Conversions to Metric Measures

Symbol	When You Know	Multiply by	To Find	Symbol
<b>LENGTH</b>				
in	inches	2.5	centimeters	cm
ft	feet	30	centimeters	cm
yd	yards	0.9	meters	m
mi	miles	1.6	kilometers	km
<b>AREA</b>				
in <sup>2</sup>	square inches	6.5	square centimeters	cm <sup>2</sup>
ft <sup>2</sup>	square feet	0.09	square meters	m <sup>2</sup>
yd <sup>2</sup>	square yards	0.8	square meters	m <sup>2</sup>
mi <sup>2</sup>	square miles	2.6	square kilometers	km <sup>2</sup>
	acres	0.4	hectares	ha
<b>MASS (weight)</b>				
oz	ounces	28	grams	g
lb	pounds	0.45	kilograms	kg
	short tons (2000 lb)	0.9	tonnes	t
<b>VOLUME</b>				
tsp	teaspoons	5	milliliters	ml
Tbsp	tablespoons	15	milliliters	ml
fl oz	fluid ounces	30	milliliters	ml
c	cups	0.24	liters	l
pt	pints	0.47	liters	l
qt	quarts	0.95	liters	l
gal	gallons	3.8	liters	l
ft <sup>3</sup>	cubic feet	0.03	cubic meters	m <sup>3</sup>
yd <sup>3</sup>	cubic yards	0.76	cubic meters	m <sup>3</sup>
<b>TEMPERATURE (exact)</b>				
°F	Fahrenheit temperature	5/9 (after subtracting 32)	Celsius temperature	°C

## Approximate Conversions from Metric Measures

Symbol	When You Know	Multiply by	To Find	Symbol
<b>LENGTH</b>				
mm	millimeters	0.04	inches	in
cm	centimeters	0.4	inches	in
m	meters	3.3	feet	ft
m	meters	1.1	yards	yd
km	kilometers	0.6	miles	mi
<b>AREA</b>				
cm <sup>2</sup>	square centimeters	0.16	square inches	in <sup>2</sup>
m <sup>2</sup>	square meters	1.2	square yards	yd <sup>2</sup>
km <sup>2</sup>	square kilometers	0.4	square miles	mi <sup>2</sup>
ha	hectares (10,000 m <sup>2</sup> )	2.5	acres	ac
<b>MASS (weight)</b>				
g	grams	0.035	ounces	oz
kg	kilograms	2.2	pounds	lb
t	tonnes (1000 kg)	1.1	short tons	st
<b>VOLUME</b>				
ml	milliliters	0.03	fluid ounces	fl oz
l	liters	2.1	pints	pt
l	liters	1.06	quarts	qt
l	liters	0.26	gallons	gal
m <sup>3</sup>	cubic meters	36	cubic feet	ft <sup>3</sup>
m <sup>3</sup>	cubic meters	1.3	cubic yards	yd <sup>3</sup>
<b>TEMPERATURE (exact)</b>				
°C	Celsius temperature	9/5 (then add 32)	Fahrenheit temperature	°F



\*1 in = 2.54 (exact). For other exact conversions and more detailed tables, see NBS Misc. Publ. 286, Units of Weights and Measures, Price \$2.25, SD Catalog No. C13.10.286.

## TABLE OF CONTENTS

<u>Section</u>	<u>Title</u>	<u>Page</u>
1.0	INTRODUCTION	1
2.0	DESCRIPTION OF TESTS	3
	2.1 Steady Curving	3
	2.2 Acceleration in Curves	5
	2.3 Stability	5
	2.4 System Surveys	6
3.0	VEHICLE INSTRUMENTATION	7
	3.1 Transducers	7
	3.2 Instrumented Wheelset	10
	3.2.1 Description of Strain Gage Bridges	13
	3.2.2 Primary Sensitivity and Crosstalk	18
	3.2.3 Ripple	22
	3.2.4 Load Point Sensitivity	23
	3.2.5 Thermal and Centrifugal Effects and Other Sources of Drift	25
	3.2.6 Sensitivity to Longitudinal Force	25
	3.3 Data Acquisition System	27
	3.4 Data Reduction	29
4.0	TEST RESULTS	31
	4.1 Curving	31
	4.2 Stability	34

### Appendices

APPENDIX A - Steady State and Peak Lateral Wheel Forces on the High Rail vs. Test Speed	A-1
APPENDIX B - Tabulated Results by Test Series	B-1
APPENDIX C - Report of New Technology	C-1

## LIST OF ILLUSTRATIONS

<u>Figure Number</u>	<u>Title</u>	<u>Page</u>
2-1	Route Description	4
3-1	Sensor Placement on Lead Truck	8
3-2	Wheelset Data Flow	11
3-3	Vertical Force Measurement	14
3-4	Triangular Output and "A+B" Processing	15
3-5	Lateral Force Strain Distribution	17
3-6	Lateral Force Measurement Bridge	24
3-7	Longitudinal Force Strain Distribution	26
3-8	Data Acquisition System	28
4-1	Summary of Steady State Lateral Wheel Forces on the High Rail at Balance Speed	32
4-2	Summary of Peak Lateral Wheel Forces on the High Rail at Balance Speed	33
A-1	Curve 37 Steady State Lateral Wheel Forces	A-2
A-2	Curve 37 Peak Lateral Wheel Forces	A-3
A-3	Curve 49 Steady State Lateral Wheel Forces	A-4
A-4	Curve 49 Peak Lateral Wheel Forces	A-5
A-5	Curve 311 Steady State Lateral Wheel Forces	A-6
A-6	Curve 311 Peak Lateral Wheel Forces	A-7
A-7	Curve 3 Steady State Lateral Wheel Forces	A-8
A-8	Curve 3 Peak Lateral Wheel Forces	A-9
A-9	Curve 43 Steady State Lateral Wheel Forces	A-10
A-10	Curve 43 Peak Lateral Wheel Forces	A-11
A-11	Curve 157 Steady State Lateral Wheel Forces	A-12
A-12	Curve 157 Peak Lateral Wheel Forces	A-13

## LIST OF TABLES

<u>Table Number</u>	<u>Title</u>	<u>Page</u>
1-1	Test Series	2
2-1	Test Curves	5
3-1	Characteristics of WMATA Car Instrumented Wheels	21



## 1.0 INTRODUCTION

The Washington Metropolitan Area Transit Authority (WMATA) has experienced high rates of wear of wheels and rails and failures of rail fasteners. These problems are especially pronounced on curves having radii less than one thousand feet ( $5.7^{\circ}$ ). Preliminary studies conducted by the Transportation Systems Center (TSC) (ref ) and by Deleuw, Cather and Company (ref ) recommended several changes to the vehicles and track. This report describes a test program to evaluate the effect of the recommended changes to the WMATA car on curving flange forces and high speed stability. The participating agencies were the Urban Mass Transit Administration (UMTA), TSC, ENSCO, the Analytic Sciences Corporation, Battelle Columbus Laboratories, WMATA, and the Federal Railroad Administration (FRA).

The test directly addresses the classical trade off between curving and stability. The use of highly tapered wheels and longitudinally compliant primary suspension allows a vehicle to negotiate sharper curves without flange contact or excessive slip. Increasing the taper reduces the axle movement toward the high rail necessary to maintain pure rolling in a curve, and longitudinal compliance allows the axles to orient radially to the curve. However, both taper and compliance reduce the speed at which unstable "hunting" occurs. Wheel taper causes the varying lateral component of the contact force which drives the oscillation, and compliance partially uncouples the wheelsets from the truck frame creating a less stable system.

The possibility that the original truck design compromised curving to gain excessive stability, in view of the actual speed and curving requirements, motivated this test in which wheel taper and primary suspension stiffness were the principal variables. Tapers of zero (cylindrical wheel), 1:20, 1:10 and 1:5 were used. Testing with 1:5 taper was cancelled, however, because yard movements demonstrated the inability of the switch frogs to

handle steeply tapered wheels. The primary suspension of the Rockwell Trucks used at WMATA consisted of an elastomeric bushing with a longitudinal stiffness of 115,000 lb/in (vertical, 74,000 lb/in; lateral, 62,000 lb/in). An experimental bushing constructed for greater longitudinal compliance with a longitudinal stiffness of 29,700 lb/in (vertical, 116,00 lb/in; lateral, 37,000 lb/in) was also used to vary the radial self-steering property of the truck. The test was divided into series featuring different combinations of the two mechanical variables as shown in Table 1-1.

TABLE 1-1  
TEST SERIES

<u>Series</u>	<u>Wheel Taper</u>	<u>Primary Suspension Longitudinal Stiffness</u>
A	cyl.	115,00 lb/in
B	1:20	115,000 lb/in
C	1:10	115,000 lb/in
D*	1:5	115,000 lb/in
F	1:10	29,700 lb/in
G	1:20	29,700 lb/in
I**	1:20	115,000 lb/in
J	cyl.	29,700 lb/in

---

\* Cancelled due to insufficient switch frog clearance.

\*\*100 passes over curve 37 in Series B configuration to investigate the possibility of rail contamination.

## 2.0 DESCRIPTION OF TESTS

The Brentwood Shop facility served as the base of operations for all testing. Most of the tests were conducted on the Red Line (Van Ness-Silver Spring) which is adjacent to the Brentwood yard.

### 2.1 STEADY CURVING

The steady curving tests were performed by driving a married pair of vehicles from Farragut North to Brookland on the Red Line as indicated in Figure 2-1. The speed was held as constant as possible at six test curves along the route. Several passes over the entire route were made to provide a range of speed levels at each test curve. The speed was limited to the service speed at each curve by the automatic train control system. The instrumented wheelset was in the lead position of the first car for forward runs and in the trailing position of the last car for the reverse runs.

Table 2-1 identifies the test curves. They range in radii from 755 ft. to 2508 ft. with a radius of less than 1000 ft. for three test curves. All curves except 311 were turning left in the forward test direction. Superelevation typically was held to 4 inches in tunnels for clearance.

The listed service speeds were not exceeded during testing, but up to 4.5 inches of cant deficiency could be developed at representative sharp right and left curves. Curve 37 was equipped with force sensing instrumented rails to monitor revenue traffic as part of a simultaneous track oriented program also conducted by TSC. Very good agreement, especially at the heavily flange loaded high rail, was obtained between the independent wayside and onboard measurements.

The route evaluation runs listed in Appendix B were steady curving tests in which the speed was controlled by the autopilot rather than manual override. The maximum test speeds usually occurred during route evaluation runs.

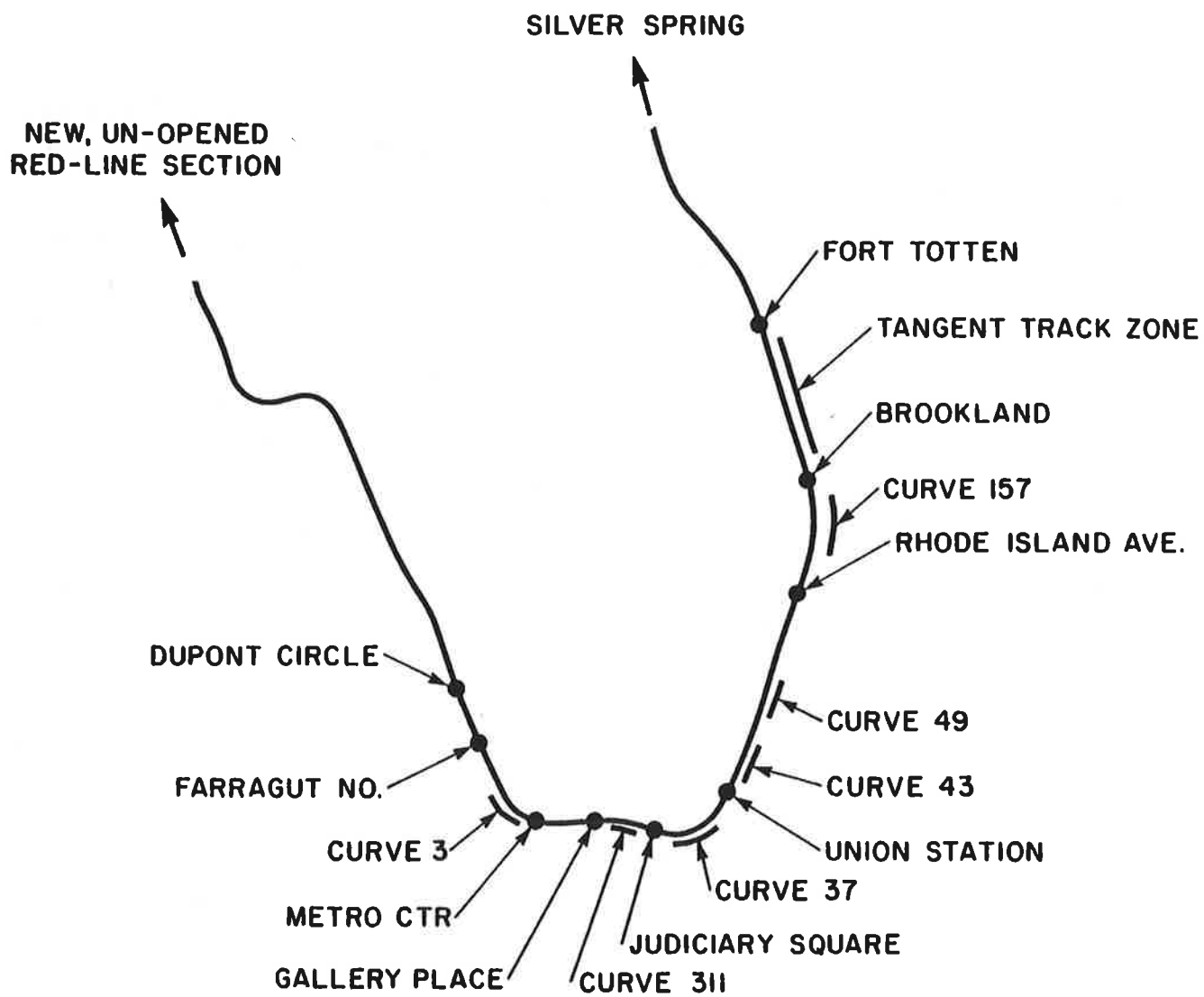


FIGURE 2-1. ROUTE DESCRIPTION

TABLE 2-1  
TEST CURVES

<u>No.</u>	<u>Length (ft)</u>	<u>Radius (ft)</u>	<u>Curvature (deg)</u>	<u>S.E. (in)</u>	<u>Service Speed (mph)</u>	<u>Cant Deficiency at Service Speed (in)</u>
3	490	1200	4.8	4	50	4.4
311	430	956	6.0	4	45	4.5
37	780	755	7.6	4	40	4.5
43	220	1750	3.3	6	65	3.7
49	310	800	7.2	6	45	4.2
157	680	2508	2.3	6	70	1.8

A special set of steady curving tests were performed at curve 6 on the Blue Line because its rails had substantially less wear than those at the older Red Line curves.

## 2.2 ACCELERATION IN CURVES

A mode of non-steady curving was also tested. The vehicle was stopped on tangent track before the test curve, and a traction motor current control setting was selected. The vehicle then accelerated through the curve with constant tractive effort. The test was repeated at the highest and next to highest current control levels. Curves 3, 37, and 157 were used as test sites.

## 2.3 STABILITY TESTS

A tangent track zone between Fort Totten and Brookland on the Red Line (Figure 2-1) was selected for stability tests. The maximum speed of the transit system, 75 mph, is permitted in this zone.

Each combination of taper and suspension bushing stiffness was run at 35, 55, 65 and 75 mph in both directions. Wheelset to truck frame yaw and lateral displacement, truck to carbody yaw and lateral displacement, and axle lateral acceleration were observed for signs of unstable oscillation.

#### 2.4 SYSTEM SURVEYS

In addition to the performance tests on the Red Line during out of service hours, the test cars were run over the entire system during Sunday revenue service. Both tracks of the Red, Orange and Blue Lines were traversed with the instrumented truck leading. The test train ran between scheduled trains under autopilot control.

The survey was performed twice. The standard configuration of stiff bushings and cylindrical wheels and an experimental configuration of soft bushings and 1:20 tapered wheels were used.

### 3.0 VEHICLE INSTRUMENTATION

The vehicle was equipped with instruments to measure, display and record the following parameters:

1. Vertical and lateral wheel/rail forces at each wheel on the lead axle.
2. Lateral acceleration of each axle on the lead truck and the lead axle of the trailing truck.
3. Longitudinal deflections of each primary suspension bushing of the lead truck.
4. Lateral deflection of each primary suspension bushing of the lead truck.
5. Vertical and lateral acceleration of a traction motor on the lead truck.
6. Yaw and lateral displacement of the lead truck with respect to the carbody.
7. Carbody lateral acceleration, vertical acceleration, and roll acceleration
8. Vehicle speed
9. Traction motor current of one truck
10. Simultaneously recorded track location reference marks

#### 3.1 TRANSDUCERS

Most of the transducers were mounted on the lead truck of the lead car. They are briefly described below in the same order in which the parameters they measure were listed. The force sensing wheelset was the most elaborate transducer and it is described in detail in the next section. Figure 3-1 locates the truck mounted transducers by the numbers of the headings of the following descriptions.

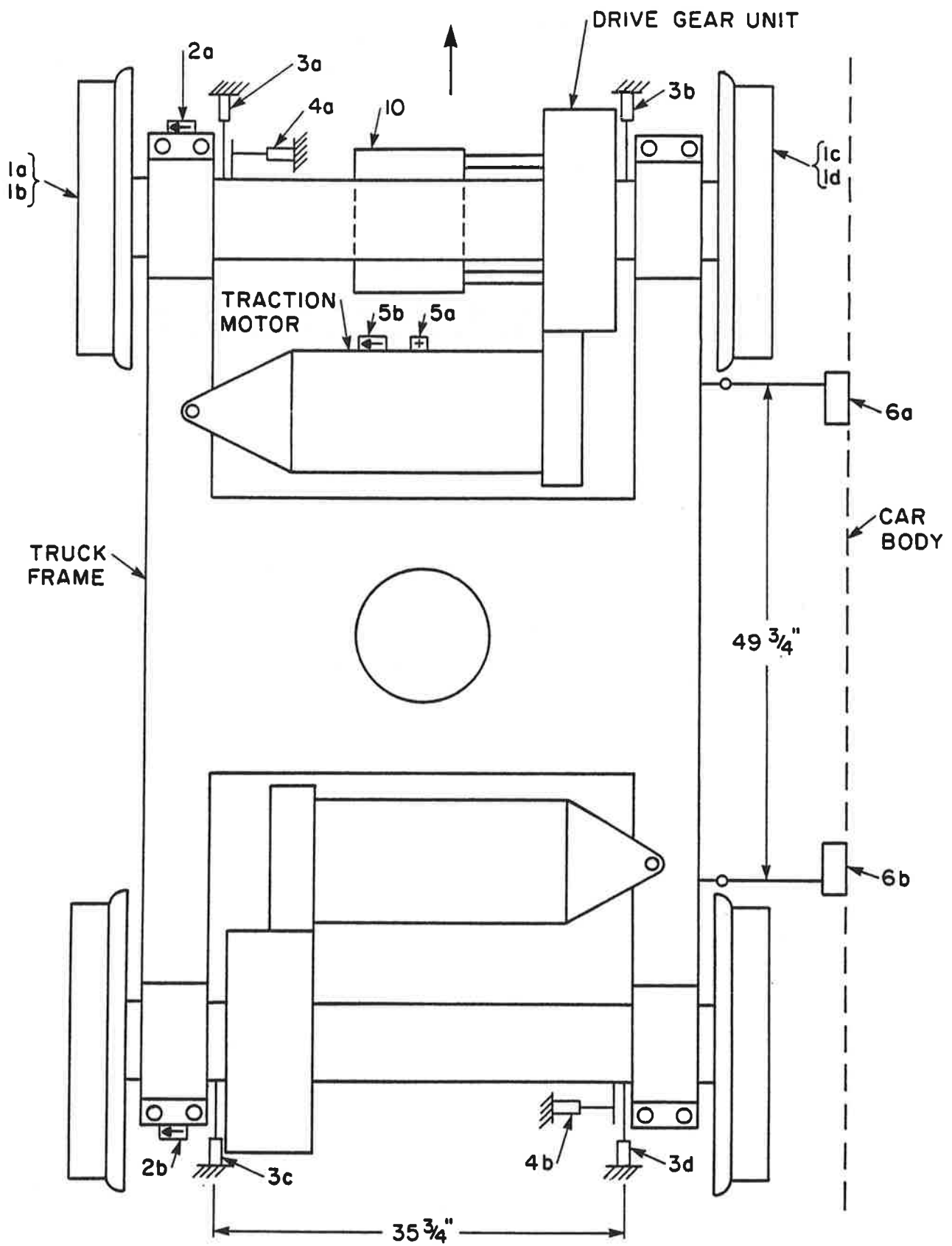


FIGURE 3-1. SENSOR PLACEMENT ON LEAD TRUCK

1. a) Left front wheel vertical force - a pair of strain gage bridges on the wheelplate producing triangular wave strain summations as the wheel rotates. A wheelset processor combines the output of the two out of phase bridges to create a single continuous signal proportional to vertical force. See Section 3.2 for details.
- b) Left front wheel lateral force - a similar pair of strain gage bridges on the wheelplate producing sinusoidal strain summations as the wheel rotates under a lateral load. A wheelset processor combines the output of the two out of phase bridges to create a single continuous signal proportional to lateral forces. See Section 3.2 for details.
- c) Right front wheel vertical force - same as left sensor
- d) Right front wheel lateral force - same as left sensor
2. a) Lead truck - lead axle lateral acceleration - 5g servo accelerometer mounted to the truck frame at the axle bearing clamp.
- b) Lead truck - trailing axle lateral acceleration - similar sensor
- c) Trailing truck - lead axle lateral acceleration - similar sensor
3. a), b), c), d) primary suspension longitudinal deflection at all four axle bearings on the lead truck - DC-DC LVDT displacement transducers (min  $\pm 1/2$ " range) acting between the bearing clamp of the truck frame and the outer race of the axle bearing.
4. a), b) Primary suspension lateral deflection for each axle of the lead truck - DC-DC LVDT displacement transducers (min  $\pm 1/2$ " range) acting between the bearing clamp of the truck frame and the outer race of one bearing on each axle.
5. a. Vertical acceleration of lead traction motor - 5g servo accelerometer mounted to the front face of the traction motor.
- b) Lateral acceleration of the lead traction motor - similar sensor.

6. a), b) String type ( $\pm 1\frac{1}{2}$ " range) displacement transducers between the right side of the body and the side of the truck frame. Truck yaw is the difference in output of the two transducers divided by their spacing; lateral translation is the average output of the sensors.
7. a) Carbody vertical acceleration - 1g servo accelerometer mounted on the car interior floor along the center line near the front truck.  
b) Carbody lateral acceleration - similar transducer and mounting  
c) Carbody roll acceleration - a second 1g vertical servo accelerometer offset laterally from 7a) provides the means in conjunction with 7a) for calculating roll acceleration.
8. Vehicle speed - a test point in the vehicle speed control circuit was used for an analog speed indication.
9. Traction motor current - a test point in the vehicle was used for an analog indication of the traction motor current for one truck.
10. Automatic Location Detection - a capacitive sensor mounted on the gear drive, cantilevered to the car centerline below the axle, produced spikes to indicate the presence of aluminum sheet metal targets placed at rail sensors or other track features. Permanent track features such as switch turnouts or electrical enclosures between the rails produced spikes to identify locations during survey runs.

### 3.2 INSTRUMENTED WHEELSET

The lead axle in the forward direction was equipped with wheels instrumented to sense vertical and lateral contact forces continuously. Figure 3-2 is a block diagram of the wheel/rail force measuring system. The essential elements are 1) pairs of strain gage bridges producing ac signals as the wheel rolls, proportional to lateral or vertical force, 2) slip rings to allow rotation of the electrical circuits, 3) high gain carrier amplifiers

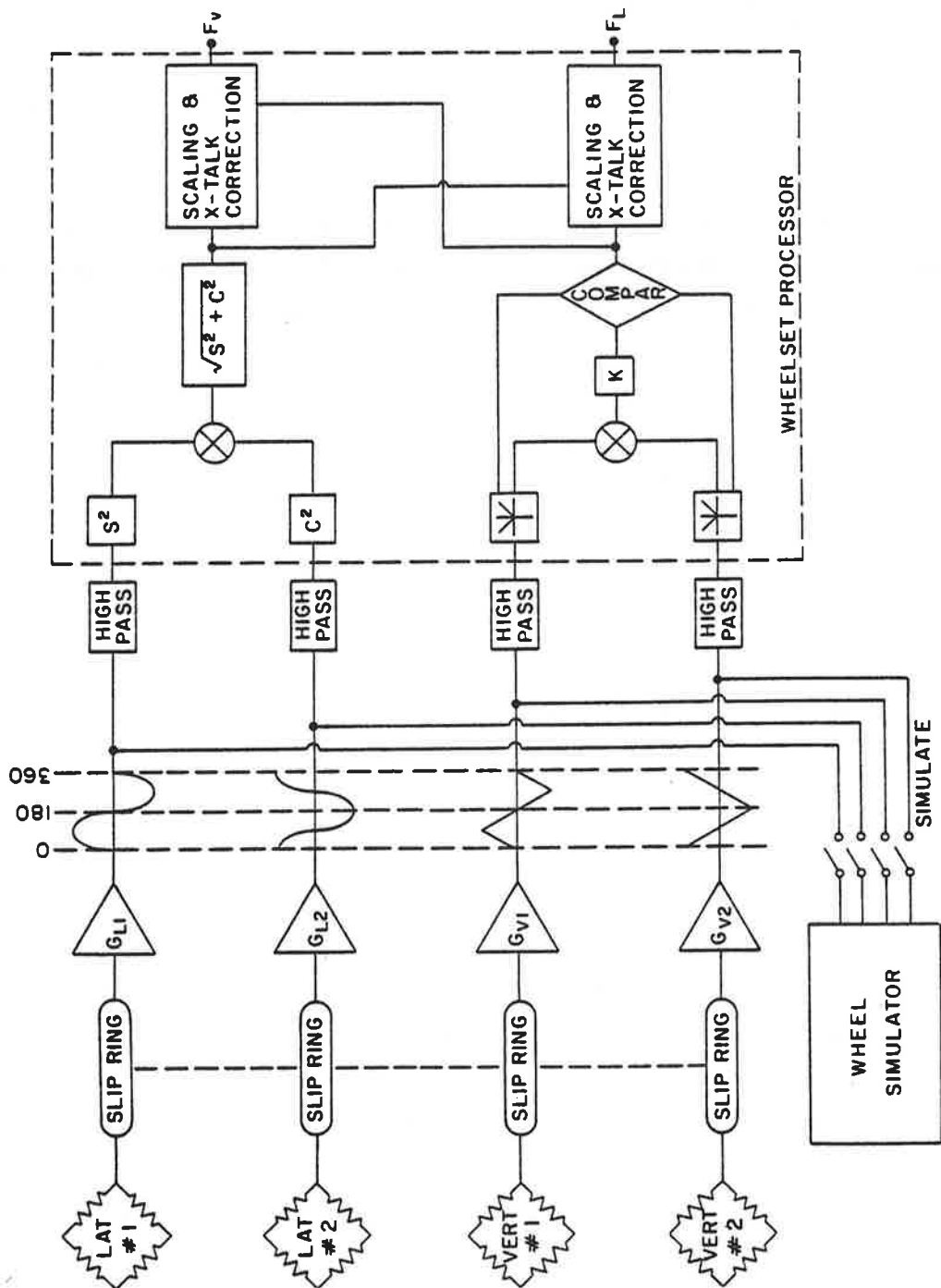


FIGURE 3-2. WHEELSET DATA FLOW

suited to bridge signals in the microvolt range, 4) high pass filters to achieve automatic zeroing, and 5) a processor to combine bridge signals into direct force indications and to perform crosstalk corrections. ENSCO has produced many wheelsets for FRA safety experiments to measure limiting values of flange force which required accuracy at high lateral forces. The WMATA car tests also required accuracy to very low lateral forces to evaluate the desired effects of wheel taper and suspension changes. Higher sensitivity of both lateral and vertical bridges and improved linearity of lateral bridges at very low force were achieved for the WMATA car, but the crosstalk was greater than for previous wheels. The crosstalk between vertical load and lateral force indication (and vice versa) were determined by calibration in a loading fixture after each change of taper, and thus accuracy was maintained through compensation performed by the processor. The basic sensitivity of the vertical and lateral bridges (microstrain per 1000 lb of load) did not change greatly during successive machinings of the wheel tread.

The basic objective of the design of force measuring wheels is to obtain adequate primary sensitivity for low signal/noise ratio and high resolution while controlling crosstalk, load point sensitivity, ripple, and the effects of heat, centrifugal force and longitudinal forces. The design philosophy was to choose strain gage bridge configurations which inherently minimized as many extraneous influences as possible and which were responsive to the general strain patterns expected in any rail wheel subjected to vertical and lateral forces. Such bridge configurations could be adapted to the standard production wheels of any test vehicle, eliminating problems of supply, mechanical compatibility, and possible alterations of vehicle behavior due to special wheels. The radial locations of the strain gages were optimized for each wheel size and shape while their angular locations were fixed by the chosen bridge configurations. Locomotive, passenger coach and freight car wheels, as well as small transit car wheels, have been instrumented successfully by ENSCO using the same general procedures.

### 3.2.1 DESCRIPTION OF STRAIN GAGE BRIDGES

The vertical force measuring bridges follow a concept used by ASEA/SJ. Each bridge consists of eight strain gages arranged in a Wheatstone bridge having two gages per leg. Each leg of the bridge has one strain gage on the field side and one strain gage on the gage side of the wheel. The four legs are evenly spaced  $90^\circ$  apart on the wheel as shown in Figure 3-3. The general strain distribution in a typical rail wheelplate due to a purely vertical load is characterized by maximum strains which are compressive and highly localized in the wheelplate above the point of rail contact. As the pair of gages in each leg of the bridge consecutively passes over the rail contact point, two negative and two positive peak bridge outputs occur per revolution. By correctly choosing the radial position of the gages, the bridge output as a function of rotational position of the wheel can be made to resemble a triangular waveform having two cycles per revolution. The purpose of having gages on both sides of the wheelplate in each leg is to cancel the effect of changes in the bending moments in the wheelplate due to lateral force and the change of axial tread/rail contact point.

When two triangular waveforms equal in amplitude and out of phase by one-fourth the wavelength, are rectified and added, the sum is a constant equal to the peak amplitude of the individual waveforms. In order to generate a strain signal proportional to vertical force and independent of wheel rotational position, the outputs of two identical vertical bridges out of phase by  $45^\circ$  of wheel arc are rectified and summed as shown in Figure 3-4. Since the bridge outputs do not have the sharp peaks of true triangular waveforms, the sum of one bridge peak and one bridge null is lower than that of two concurrent intermediate bridge outputs. In order to reduce the ripple or variation in force channel output with wheel rotation, the bridge sum is scaled down between the dips coinciding with the rounded bridge peaks. By taking as the force channel output the greatest of either individual bridge output or the scaled down sum of both bridges, the scaling down

# "A + B" TRIANGULAR OUTPUT (ASEA/SJ)

- TWO BRIDGES
- GAGES ON BOTH SIDES OF WHEELPLATE
- TRIANGULAR WAVEFORMS-2-CYCLES PER REVOLUTION
- $OUTPUT = MAX \{ |A|, |B|, K(|A| + |B|) \}$

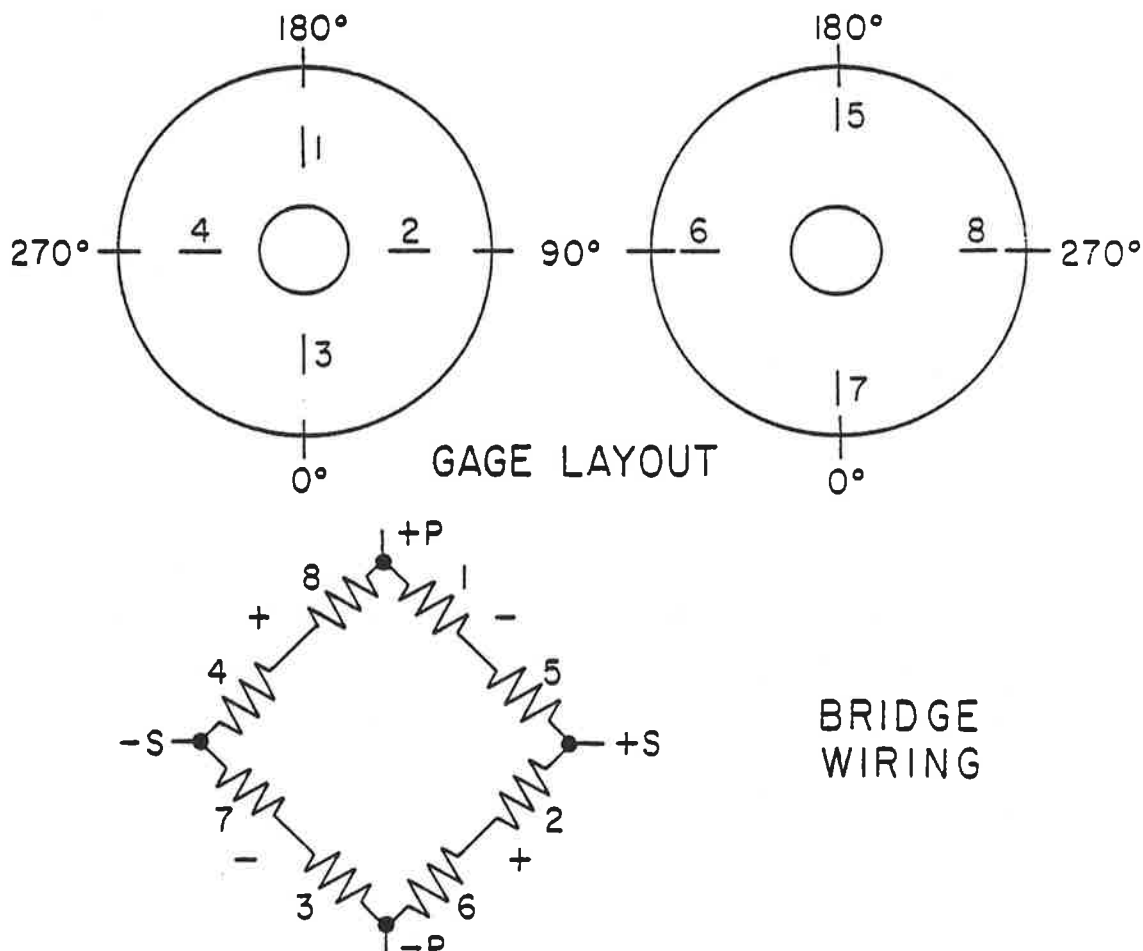


FIGURE 3-3. VERTICAL FORCE MEASUREMENT BRIDGE

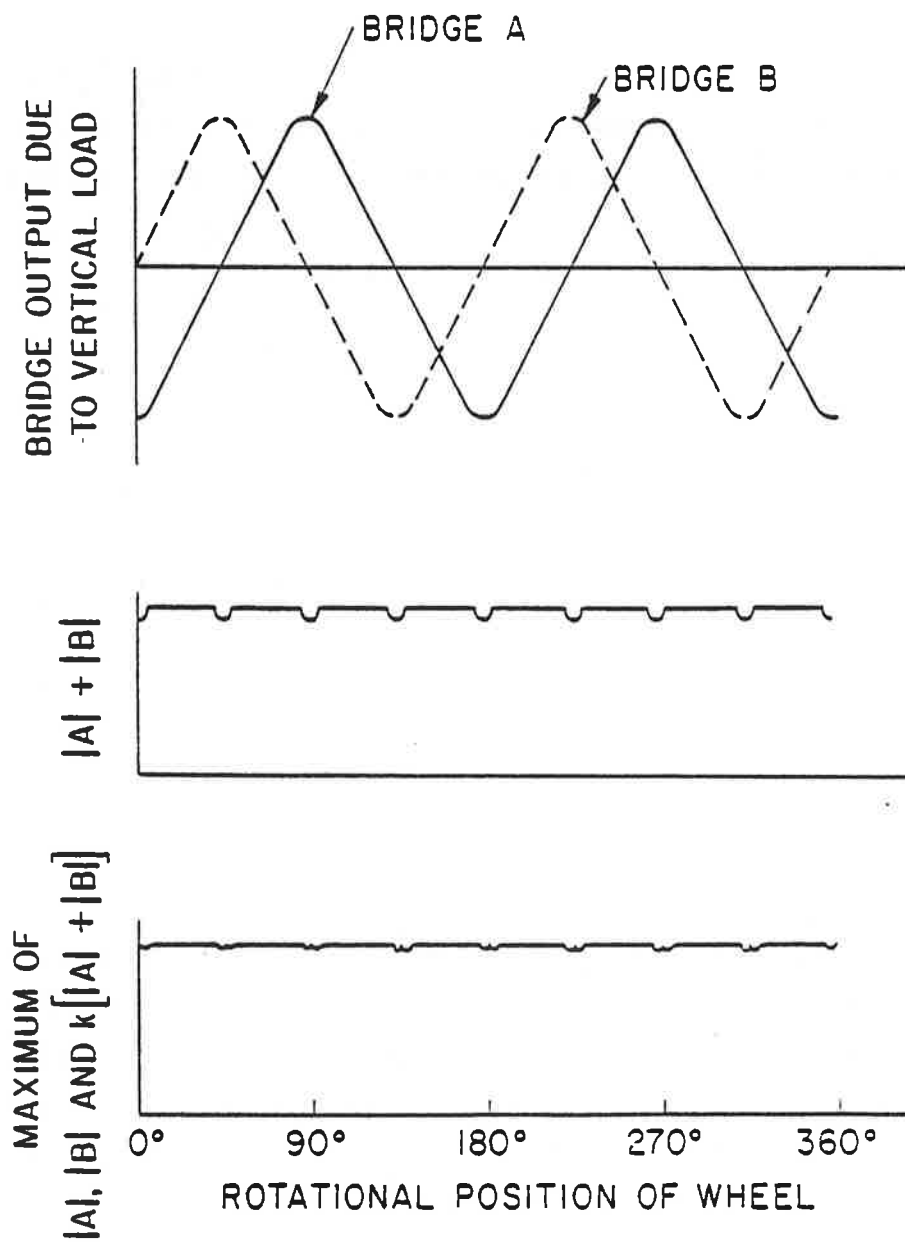


FIGURE 3-4. TRIANGULAR OUTPUT AND "A + B" PROCESSING

is applied selectively to the part of the force channel output between the dips as shown in Figure 3-4.

The general strain distribution of a typical rail wheelplate due to a purely lateral flange force is characterized by two components as shown in Figure 3-5. One component is a function of radius only because the wheelplate acts as a symmetric diaphragm in opposing the lateral force at the axle. The second component results from the moment about the hub caused by the flange force, and it tends to vary at a given radius with the cosine of the angular distance from the wheel/rail contact point. The strain distributions on the gage and field sides of the wheelplate are similar in magnitude but opposite in sign (compression or tension).

Lateral force measuring bridges which follow a concept advanced by EMD (Ref. 19) take advantage of the general strain distribution in a standard rail wheelplate. As shown in Figure 3-4, each bridge is composed of eight gages evenly spaced around the field side of the wheelplate at the same radius. The first four adjacent gages are placed in legs of the bridge that cause a positive bridge output for tensile strain and the next four gages are placed in legs causing a negative bridge output for tensile strain. The resulting bridge cancels out the strain due to the axial load because all eight gages are at the same radius with four causing positive and four causing negative bridge outputs. However, the bridge is very sensitive to the sinusoidal strain component associated with the hub moment due to the flange force because the tensile strains and the compressive strains above and below the axle are fully additive in bridge output twice each revolution (once as a positive peak and once as a negative peak). Radial gage locations may be chosen such that the bridge output varies sinusoidally with one cycle per wheel revolution. Two identical bridges  $90^\circ$  out of phase are used to obtain a force channel output independent of wheel rotational position as a consequence of the geometric identity:

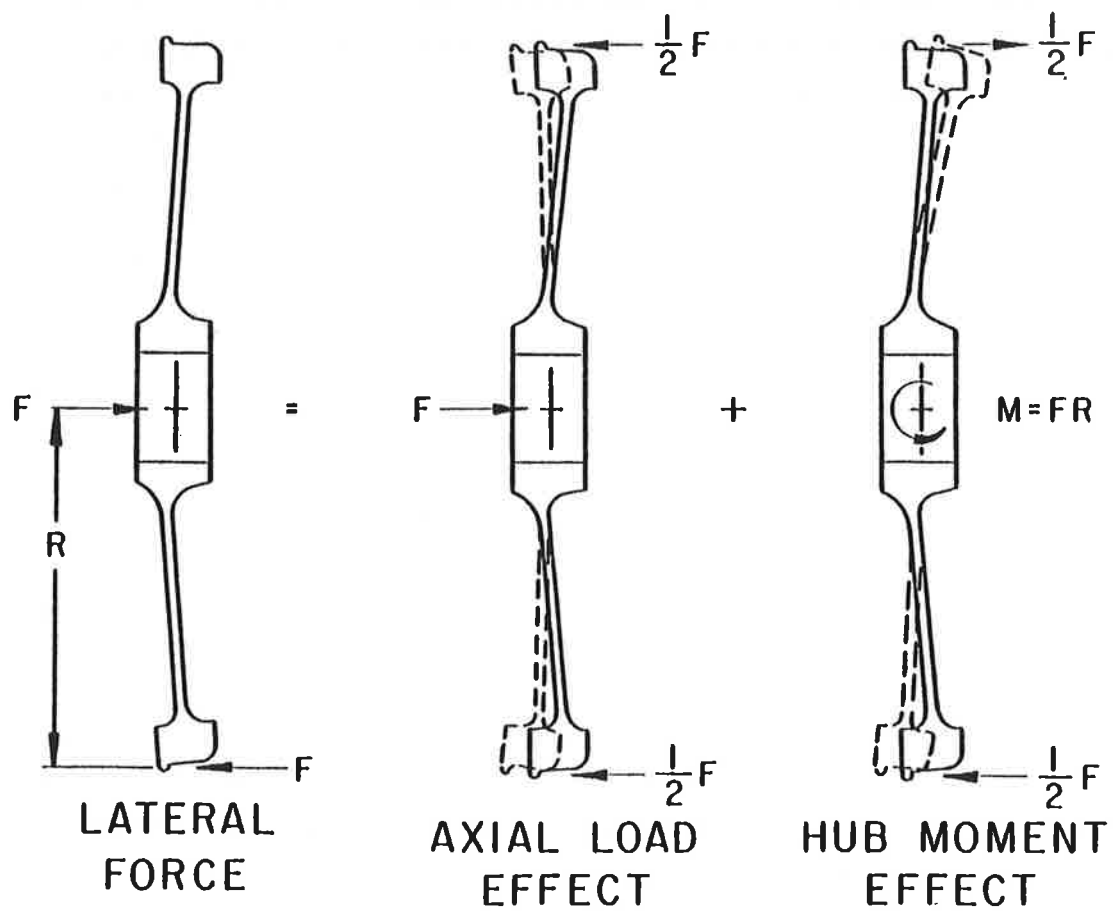


FIGURE 3-5. LATERAL FORCE STRAIN DISTRIBUTION

$$\sqrt{(L\sin\theta)^2 + (L\sin\{\theta+90^\circ\})^2} = |L| \text{ for any } \theta$$

### 3.2.2 PRIMARY SENSITIVITY AND CROSSTALK

The first step in the production of the instrumented wheels was the machining of both to an identical contour. The contour was dictated by the minimum allowable wheelplate thickness and by the production variation of the available sample of wheels. The machining contour is usually close to the original design shape but at a minimum thickness. The thinning of the wheelplate is the easiest step in maximizing sensitivity because it does not involve compromise with the other measurement properties of the wheel.

The most powerful tool in selecting the radial locations of the strain gages for the best compromise between primary sensitivity, crosstalk, ripple, and sensitivity to axial load point variation is a detailed empirical survey of the strains induced in the given wheelplate by the expected service loads. The use of wheels machined to an identical profile makes the empirical approach to wheelset instrumentation practical because the results of the strain survey may be applied to all wheels in the group. The calibration loads and the reference lateral position of the wheel on the rail should reflect the type of experiment in which the wheels will be used.

The WMATA car wheels were loaded both to the nominal static load of 10,000 lb and to 15,000 lb (to simulate load transfer) with the rail adjacent to the flange to determine the primary vertical sensitivity. Lateral loadings of 1,000 lbs., 2,000 lbs., 5,000 lbs. and 10,000 lbs. were made while maintaining the 15,000 lbs. vertical load. The normalized difference in lateral bridge strain between the highest combined load and the purely vertical

load is the lateral sensitivity, and the other loads were taken to verify linearity. The combined vertical and lateral loading at a high L/V ratio accomplished by forcing the wheelset laterally against a rail while maintaining a vertical load was used to select strain gage locations for minimal crosstalk. Vertical loadings at several points across the tread were taken to evaluate the sensitivity to axial load point.

In the strain survey conducted on the WMATA car wheels, strain gages were applied at intervals of one inch or less on both field and gage sides of the wheelplate along two radial lines separated by  $180^{\circ}$  of wheel arc. The calibration loads were repeated at every  $15^{\circ}$  of wheel arc until the strain along 24 equally spaced radial lines on both gage and field side was mapped for each load. This data was used in a computer program to predict the output of a force channel as a function of the radial locations of the gages in the companion bridges.

The vertical force measuring bridges of the WMATA car wheels have strain gages on both sides of the wheelplate. The simulation program allowed the rapid trial of many combinations of gage and field side radii as potential strain gage locations. The maximum sensitivity possible for a purely vertical load on a given wheel of a bridge actually producing the triangular waveform is rapidly revealed. The "triangularity" of the waveform of a candidate bridge can be tested by adding its output at each angular load position to that at a load position advanced by  $45^{\circ}$  of wheel arc. This test determines the ripple expected of a force channel composed of two out of phase candidate bridges.

A lateral force affects the vertical bridge both by directly changing the strain pattern in the wheelplate and by moving the point of vertical load contact with the rail toward the flange. By using as a measurement of crosstalk the difference in bridge output caused by adding a lateral load to an existing vertical load, correction factors may be chosen which compensate for net

lateral force crosstalk which includes direct lateral force crosstalk and the effect of slight vertical load point movement. It is desirable to identify vertical bridges in which the direct lateral force crosstalk and the effect of load point changes are opposed and yield a minimum net crosstalk for flange forces in service. The accuracy of the highly loaded flanged wheel is enhanced using a correction factor in processing based on the net lateral force crosstalk. Compromises in bridge selection are usually biased in favor of the flanged wheel because it generates the most vital data for vehicle dynamics or rail wear studies.

The primary sensitivities and crosstalk factors achieved for the WMATA car wheels are listed in Table 3-1. The vertical bridges were chosen from a detailed simulation with radial position increments of 0.1 inches on the basis of maximum primary sensitivity while holding the simulated ripple below 5% and minimizing crosstalk and sensitivity to axial load point. The primary sensitivity was observed to be linear within about 1% because the strains at each gage are low and the wheelplate behaves elastically. Primary vertical force sensitivity appears to be inversely proportional to tread diameter and wheelplate thickness for several wheelplate shapes which have been instrumented by ENSCO.

The lateral force measuring bridges of the WMATA car wheels have gages on only one side of the wheelplate and the trial simulation of bridges is used to determine the most advantageous side of the wheel and the radial gage position. Sensitivity was measured with combined vertical and lateral loads and crosstalk was determined from purely vertical loads. A very high sensitivity was achieved for good resolution, but at low lateral forces the linearity also required special consideration. The lateral force is computed from the sum of the squares of two bridge outputs causing all measurements to have a positive sign. The convenient determination of the direction of a lateral creep force requires

TABLE 3-1. CHARACTERISTICS OF WMATA CAR INSTRUMENTED WHEELS

A. Calibration Constants

<u>Wheel Description</u>	<u>Vertical Force Measurement</u>			<u>Lateral Force Measurement</u>	
	<u>Sensitivity</u>	<u>K</u>	<u>Raw Lat- eral Force Crosstalk*</u>	<u>Sensitivity</u>	<u>Raw Verti- cal Force Crosstalk*</u>
28" tread dia.; convex conical wheel plate, 3/4" min. thickness	7 1/2 µε/kip	.95	10-20%	60 µε/kip	7-17%

B. Uncorrected Variability

<u>Vertical Force Measurement</u>			<u>Lateral Force Measurement</u>	
<u>Sensitivity to Axial Load Point</u>	<u>Max.Ripple Vertical Load</u>	<u>Max.Ripple Combined Load</u>	<u>Sensitivity to Axial Point</u>	<u>Max. Ripple Combined Load</u>
+10%/inch	+5%	+7%	-3%/inch	+3%

---

\*Crosstalk varies with tread taper; the force indication is entirely compensated for crosstalk by the wheel processor.

a wheel rotational position sensor. (It can also be accomplished by careful examination of the sinusoidal output of a single bridge.) It is possible that a purely vertical load can cause a lateral bridge output having a sign opposite to that caused by lateral force, but the crosstalk would appear positive because of squaring. The first increment of lateral load would cause a reduction rather than an increase in the output of such a bridge

and bridge strains at low lateral forces would not be unique to a particular force. Although this would be of little concern in an experiment to measure high L/V ratios, low force measurements were vital to this test program. It is more important to optimize the sign of the vertical crosstalk than its magnitude where the lateral force measurement range extends to low forces. Raw vertical into lateral crosstalk of up to 17% (varying with taper) rather than the usual 1-3% was a consequence of optimizing the vital properties, but the processor compensation completely removed the crosstalk from the force indication.

### 3.2.3 RIPPLE

Ripple is caused by the failure of the bridges to produce the desired waveform and by deviation from the correct phase relationship between the companion bridges which are processed together as a force channel. The wheelplates are machined for uniformity to reduce ripple and a grid of radial and circumferential lines is scribed on the wheelplate to aid accurate gage placement. The massive computer aided simulation of trial bridges was used to determine gage locations of minimum inherent ripple. The ripple of the vertical force channel is reduced by attenuating the high bridge sums occurring between the rounded bridge peaks as shown in Figure 3-4. This method achieves a substantial reduction in ripple at a small cost in average sensitivity.

The lateral bridge output is inherently very sinusoidal. The requirement for two bridges at the same radius out of phase by  $90^{\circ}$  is in conflict with the  $45^{\circ}$  spacing between the gages in each bridge because theoretically both bridges should occupy the same space. Placing the gages side by side causes a deviation from the proper phase relationship which manifests itself as a ripple. Table 3-1 gives the maximum ripple for the WMATA car wheels. Larger wheels which have less phase deviation between lateral bridges also have less ripple. Greater ripple is measured at combined loads because crosstalk produces distortions of the waveforms.

Ripple does not create as much error as might be supposed. Even the peak wheel forces measured during vehicle dynamics testing frequently are averaged for about 50 milliseconds. A 28-inch wheel makes half a revolution in 50 milliseconds at 50 mph, substantially negating ripple in a 50 millisecond average wheel force since 2 cycles per revolution is a typical ripple frequency. A single instantaneous measurement is rarely sought and filtering has a mitigating influence on ripple similar to time averaging.

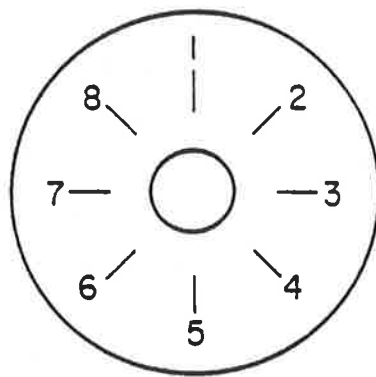
#### 3.2.4 LOAD POINT SENSITIVITY

Lateral changes of the point on the tread where the wheel contacts the rail affect the bridge strains in two ways. The change in hub moment is similar to the effect of a lateral force. It acts directly on the lateral bridge and through crosstalk on the vertical bridge. However, the failure of the tread to transmit the moment due to load point offset uniformly into the wheelplate probably has the major effect on the vertical bridge. Unsymmetric changes in the local intense compressive strains in the wheelplate above the rail contact can influence the vertical bridge sensitivity. Wheels with thin tread hoops are most susceptible in this regard. The load point sensitivity of the WMATA car wheels (Table 3-1) remained stable despite several retaperings, indicating sufficient remaining tread hoop stiffness.

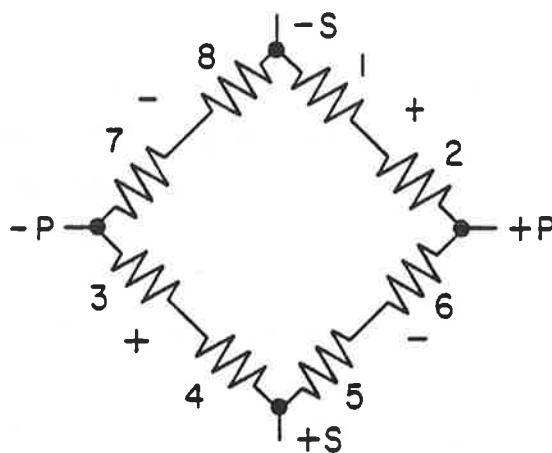
The effect of load point sensitivity on measurements taken with the WMATA car wheels on the high rail was negligible because the calibration was performed with the wheel flange adjacent to an actual rail. The true vertical load on the unflanged low rail wheel is about 15% less than the indicated. To correct the lateral force indication of the unflanged low rail wheel, its absolute value should be increased by about 5% of the corrected vertical if it is zero or positive (toward the flange) or decreased by the same amount if it is negative.

$\sqrt{\sin^2 + \cos^2}$  TECHNIQUE (EMD)

- TWO BRIDGES
- SINUSOIDAL OUTPUT
- 90° OUT OF-PHASE
- APPLIED AT SINGLE RADIUS TO ONE SIDE OF WHEELPLATE



GAGE  
LAYOUT



BRIDGE  
WIRING

FIGURE 3-6. LATERAL FORCE MEASUREMENT BRIDGE

### 3.2.5 THERMAL AND CENTRIFUGAL EFFECTS AND OTHER SOURCES OF DRIFT

The vertical and lateral bridges used on the WMATA car wheelsets are particularly immune to drift by virtue of strain gage location and instrumentation technique. Strains induced by thermal change and centrifugal force are radially symmetric on each side of the wheelplate. The lateral bridge consists of eight gages at the same radius on the same side of the wheelplate positioned in the bridge so that four add and four subtract. A radially symmetric strain field is cancelled by the additions and subtractions. Similarly, the vertical bridges have four gages at the same radius on each side of the wheelplate. On each side two gages add and two subtract.

Each bridge generates a triangular or sinusoidal waveform as the wheel rotates under load. High pass filtering of the amplified bridge signals at 0.2 Hz does not attenuate the oscillating part of the signal but it forces the signal to oscillate about zero. High pass filtering eliminates gradual drift that could occur from thermal effects on the wheelset wiring and wheel to amplifier cabling and zero drift of the strain gage bridge amplifiers. It would also suppress thermal and centrifugal effects in bridges which do not self cancel them.

### 3.2.6 SENSITIVITY TO LONGITUDINAL FORCE

Longitudinal forces involved in braking and driving are extraneous influences on the vertical and lateral force measurement bridges. Brakes on instrumented wheelsets are usually disabled to avoid sensor damage by overheating and to avoid accidental flatspotting. However, instrumented wheelsets on self propelled vehicles must cope with driving forces. Figure 3-7 shows the strain distribution in a driven wheel. The longitudinal force may be resolved into a torque about the axle and a horizontal force perpendicular to the axle. The similarity between this horizontal force component and the vertical force suggests an error source.

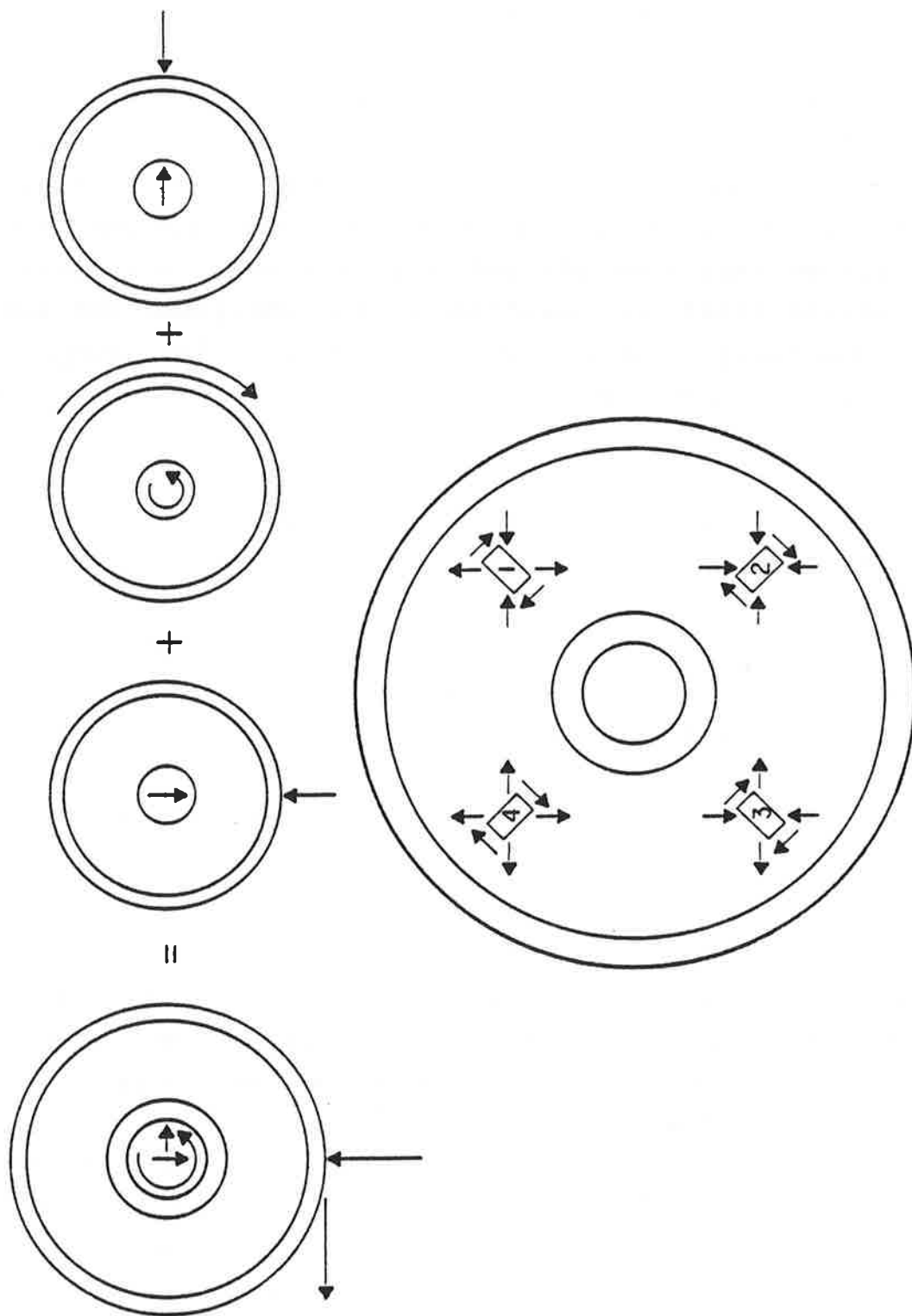


FIGURE 3-7. LONGITUDINAL FORCE STRAIN DISTRIBUTION

The vertical force measuring bridges on the WMATA car wheelsets are configured in such a way as to cancel the effect of longitudinal forces. Figure 3-7 shows the strain components at four gage positions on one side of the wheelplate due to vertical and driving forces. The bridge is shown in the vertical null output position. Gages at  $180^\circ$  spacing add together in their contribution to the bridge summation. The vertical, horizontal and shear components of strain are opposite in sense for gages spaced  $180^\circ$  apart and cancel each other out retaining the null bridge output. The longitudinal force does not create an intense local strain aligned with the sensitive axis of a strain gage which stimulates the vertical bridge in any rotational position. The insensitivity of the vertical bridges to longitudinal force has also been verified experimentally.

The lateral bridges used on the WMATA car wheels are also insensitive to longitudinal forces. The symmetric gage pattern limits the effect of the shear strains, but the horizontal force has the effect of adding vectorially to the vertical force to produce crosstalk. Since the longitudinal force is limited by friction to about  $1/4$  the vertical load, the vector sum of forces is only about 3% higher than the vertical force alone. An increase in crosstalk of 3% of 12% and (0.36%) is insignificant.

### 3.3 DATA ACQUISITION SYSTEM

Figure 3-8 is a block diagram of the data acquisition system.

The displacement and acceleration sensors described in the previous sections were cabled to junction boxes containing power supplies and to signal conditioning amplifiers where the full scale range of  $\pm 10$  Vdc was established. The outputs of the speed, traction motor current and location detection sensors were also scaled by the signal conditioning amplifiers.

The instrumented wheel strain gage bridge circuits were connected to the strain gage amplifiers by means of rotating sliprings at

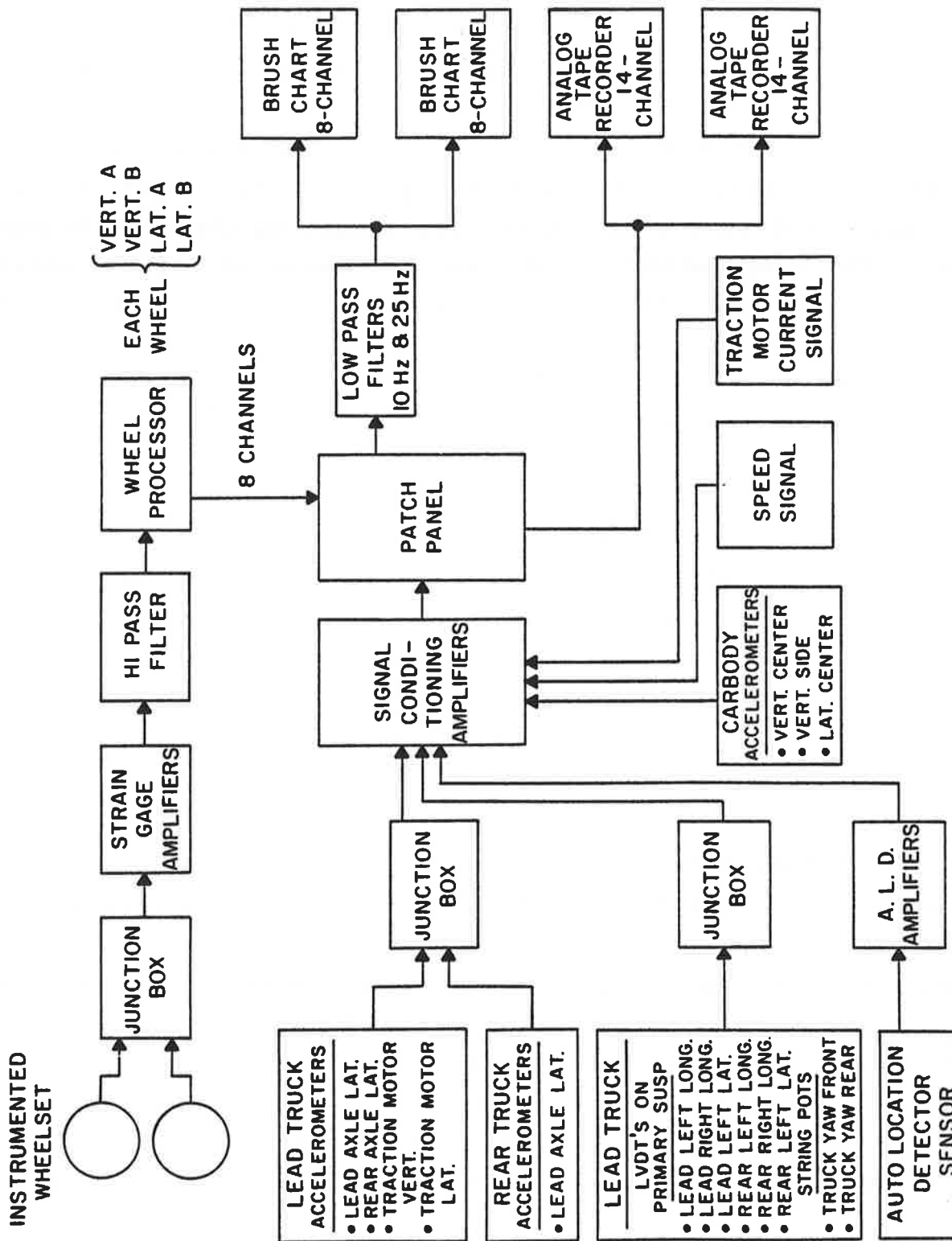


FIGURE 3-8. DATA ACQUISITION SYSTEM

each end of the axle and a junction box to join the wheel cables. The strain gage amplifiers supply ac carrier excitation to the bridges and demodulate the bridge responses to produce highly amplified dc signals proportional to the bridge strain summation. Carrier amplifiers provide superior noise immunity.

As the wheels rotate, the output of the individual bridges (two bridges are required for each force channel) under a constant force vary as sine waves or triangular waves. High pass filtering is applied to the rotating wheel bridge output to eliminate all sources of signal drift. Wheel rotation causes the bridge responses to oscillate at the wheel rotation frequency with drift resembling a change in dc level. High pass filtering strikes out the drift while preserving the ac waveform thus eliminating the need for rezeroing even during long test days.

The wheelplate processor combines the individual bridge signals to form continuous force measurements. It also performs cross-talk correction and scaling to a  $\pm 10$  Vdc range proportional to  $\pm 25,000$  lb. wheel/rail force.

The wheelset outputs and the conditioned outputs of all the other sensors are merged at a patch panel. All of the signals were recorded by the analog tape recorders, but the patch panel allowed sampling of the various sensors for real time observation with the strip chart recorders. The data signals were filtered before observation on the strip charts, but they were recorded on magnetic tape at the frequency response of the sensors.

### 3.4 DATA REDUCTION

The data presented in Appendices A and B were obtained by reading strip chart recordings. The data tapes were replayed to the strip chart recorders after the test was finished so that attention could be given to holding accurate zeros. The accelerometer channels were filtered at 10 Hz and the force and displacement channels were filtered at 25 Hz.

The steady state readings were averaged by inspection over the length of the constant radius part of each curve. The constant radius part of the curve was identified by targets placed on the ties for detection by the ALD sensor and by the character of the high rail lateral force indication.

The peak readings were simply the highest filtered level attained at the spirals or body of each curve. The filtering was used to eliminate events having such short time duration that they were insignificant to the dynamic behavior of the vehicle.

A second type of peak indication was also listed in the stability test data included in Appendix B. It was the level exceeded for a time of 100 milliseconds. It was used to eliminate more events on the basis of time duration to make hunting, should it occur, more obvious against the background of ordinary rail roughness.

## 4.0 TEST RESULTS

The principal data channels appropriate for each type of test run have been reduced and are tabulated in Appendix B. Auxilliary data channels have not yet been reduced except for isolated runs to check for consistency. The peak and steady state lead wheel lateral forces at each test curve have been plotted versus vehicle speed in Appendix A. A detailed analysis of the test results is presented in Part I of this report, and only a few general observations are offered below.

### 4.1 CURVING

Figure 4-1 summarizes the steady state lateral force measurements by plotting the data (interpolated to balance speed) versus curve radius for combinations of suspension bushing stiffness and wheel taper (1:20 and cylindrical). Both changing from stiff to soft bushings and changing from cylindrical to tapered wheels produced consistent reductions in flanging forces. Suspension compliance appeared to be the more effective change at sharp curves of less than 1000 ft. radius, and wheel taper was the more beneficial change at curves of larger radius.

The combination of changes was especially effective. Reductions of lateral force between 60% and 90% were achieved, and the effect of the combined changes was greater than the sum of the effects of the individual changes.

Curve 311 was the only right curve of the six test curves. Measurements of truck geometry indicated a slight skewing of the axles that could bias it against curving to the right, and this bias may account for the high force measurements at curve 311. The combination of soft bushings and wheel taper appeared to reduce the sensitivity of the truck to production variations.

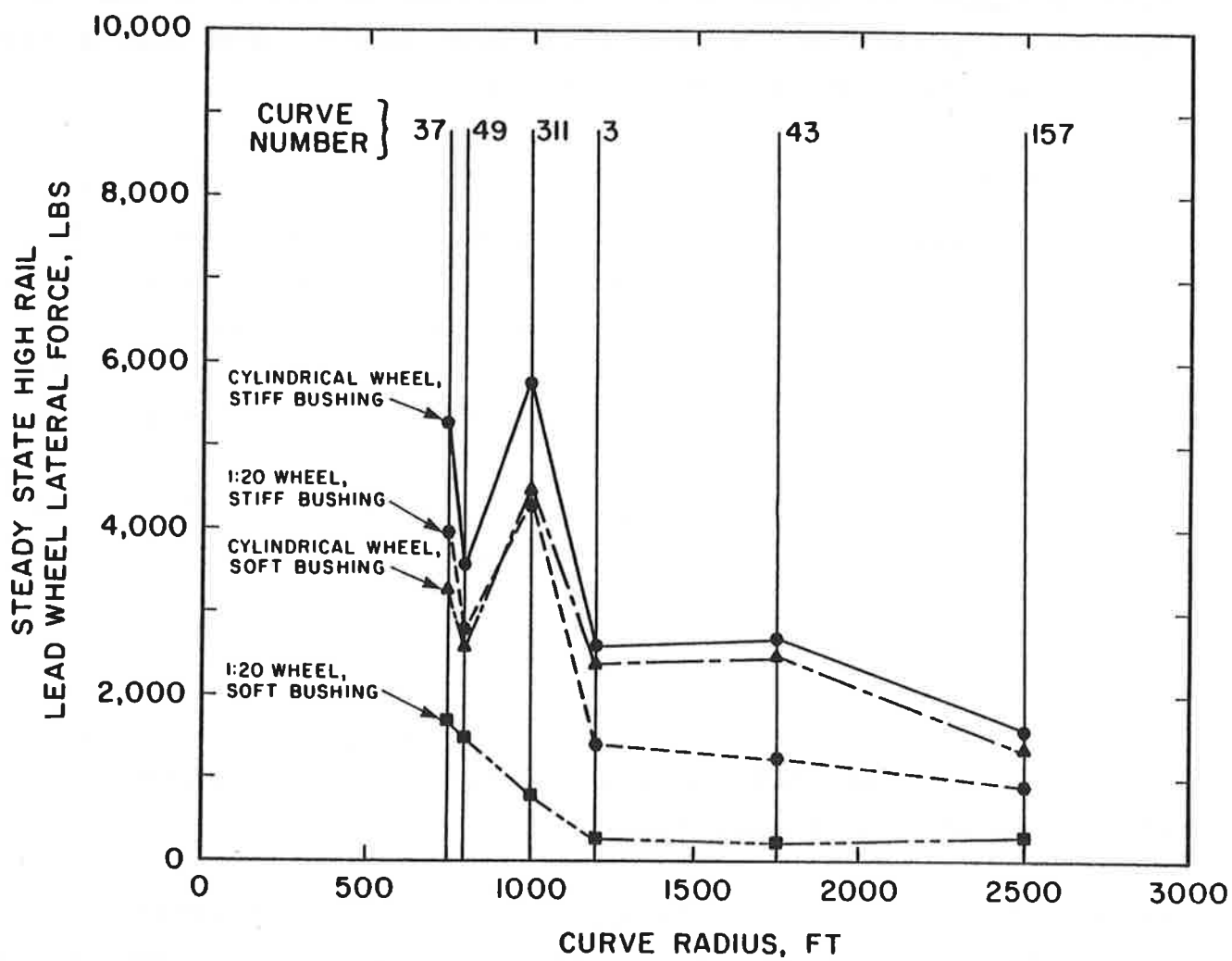


FIGURE 4-1. SUMMARY OF STEADY STATE LATERAL WHEEL FORCES ON THE HIGH RAIL AT BALANCE SPEED

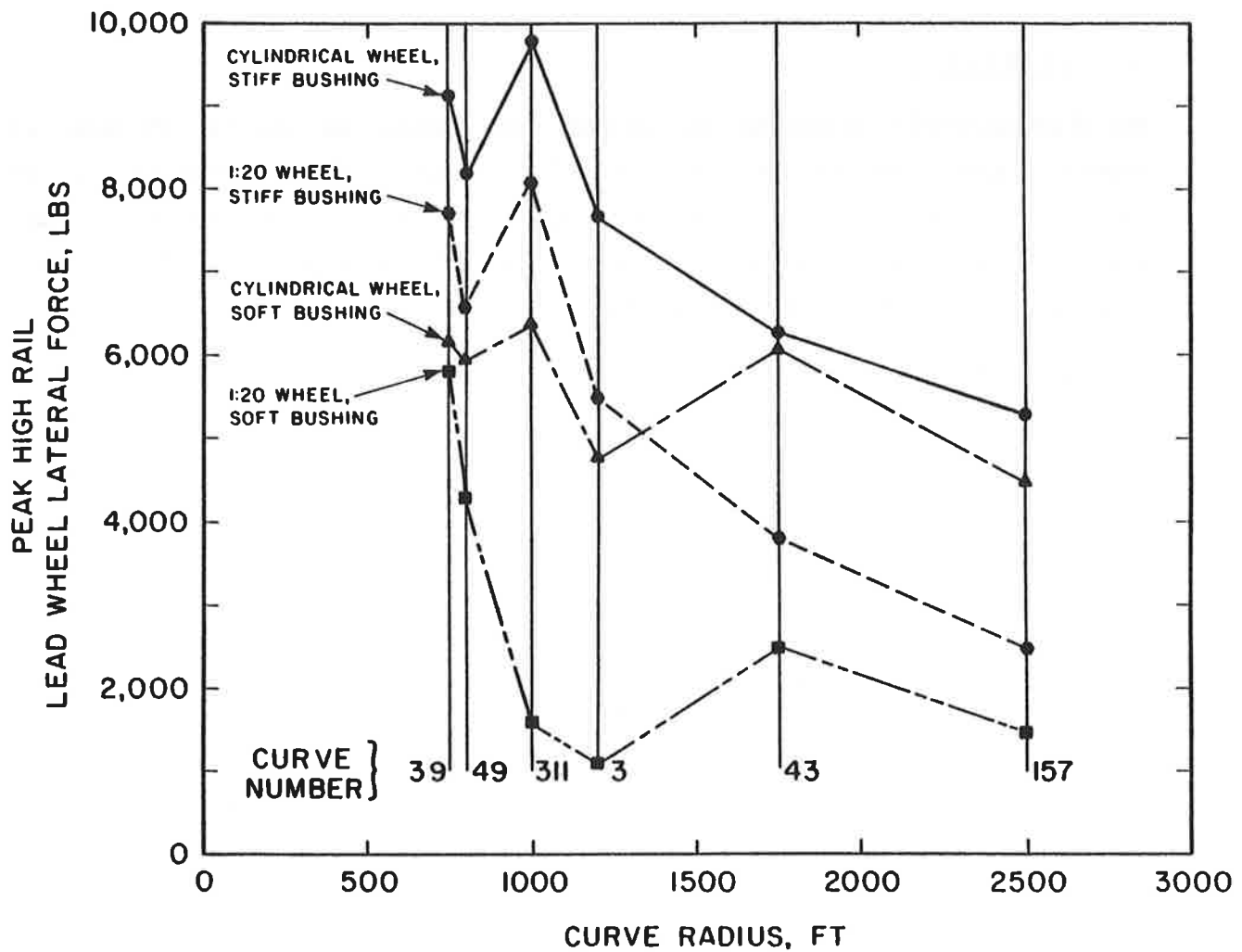


FIGURE 4-2. SUMMARY OF PEAK LATERAL WHEEL FORCES ON THE HIGH RAIL AT BALANCE SPEED

The same observations can be made for the peak measurements presented similarly in Figure 4-2. The change to greater longitudinal suspension compliance seemed the more effective for curves up to about 1300 ft. radius, but the combination of changes was extremely effective in reducing peak lateral flange forces in all cases.

#### 4.2 STABILITY

No discernable hunting occurred for tests at up to 75 mph with wheel taper as great as 1:10 for either the standard or soft longitudinal bushing. The attempt to test a greater wheel taper was frustrated because of inability of highly tapered wheels to negotiate switch frogs smoothly.

APPENDIX A: STEADY-STATE AND PEAK LATERAL WHEEL FORCES  
ON THE HIGH RAIL VS. TEST SPEED DATA PLOTS

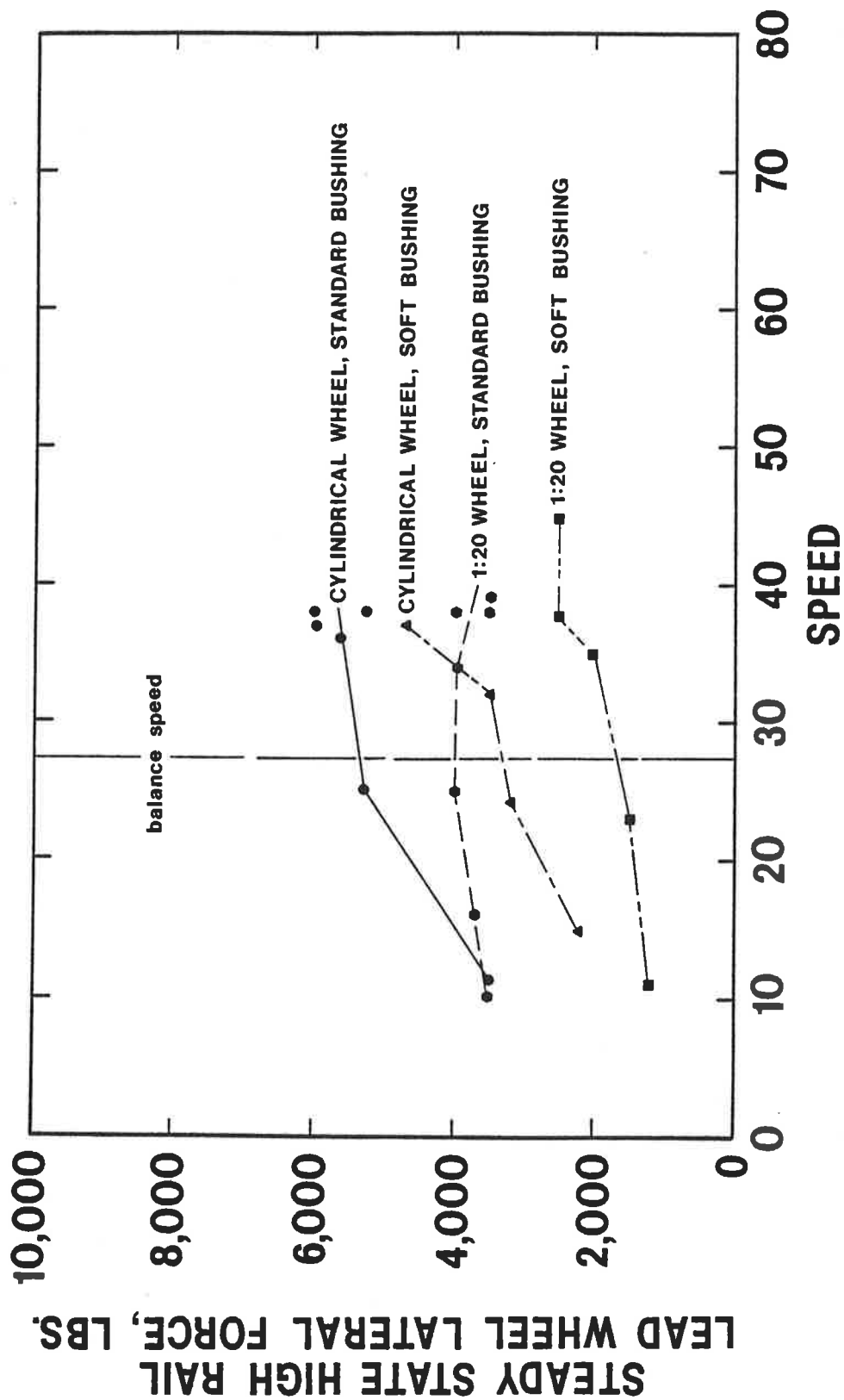


FIGURE A-1. CURVE 37 STEADY STATE LATERAL WHEEL FORCES

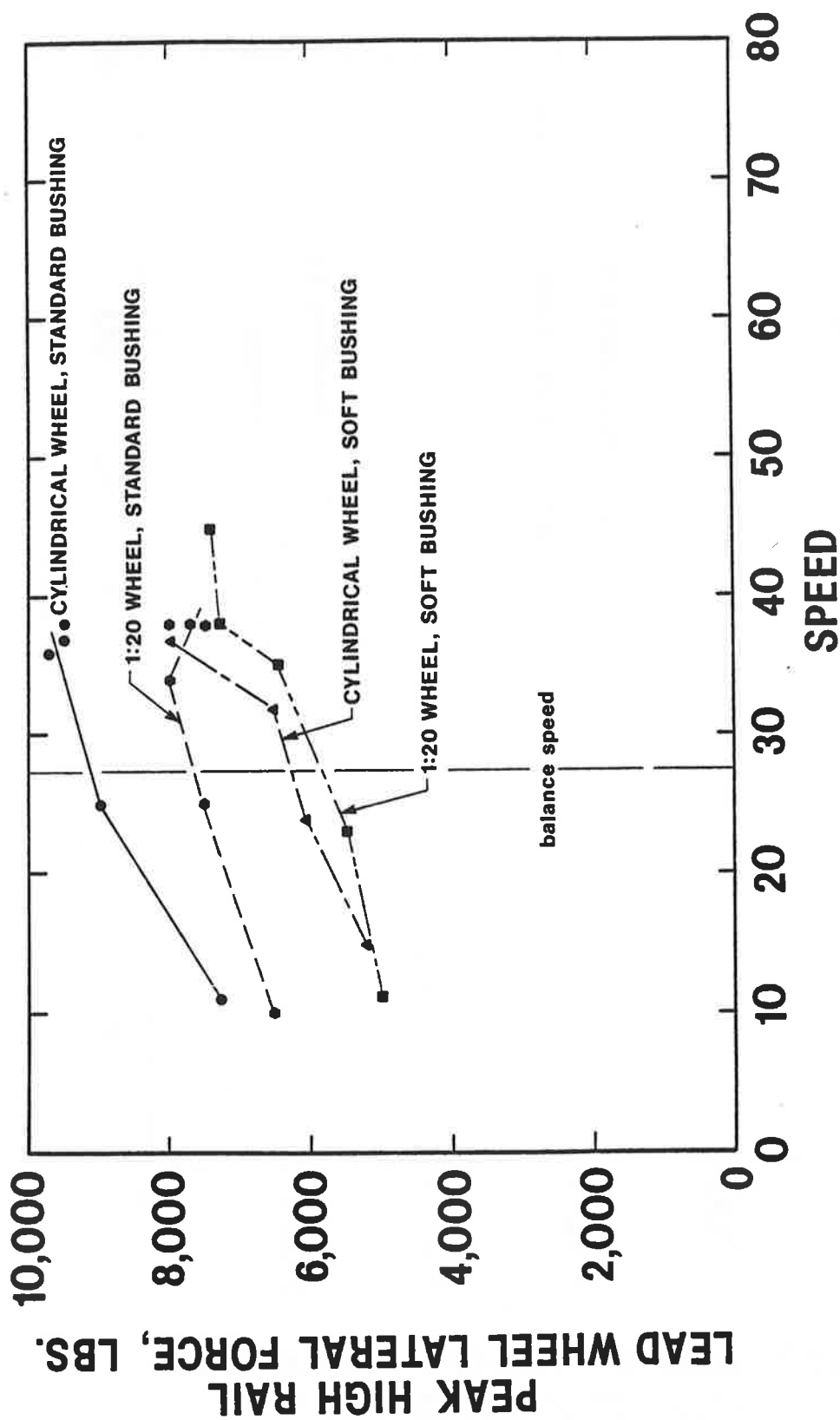


FIGURE A-2. CURVE 37 PEAK LATERAL WHEEL FORCES

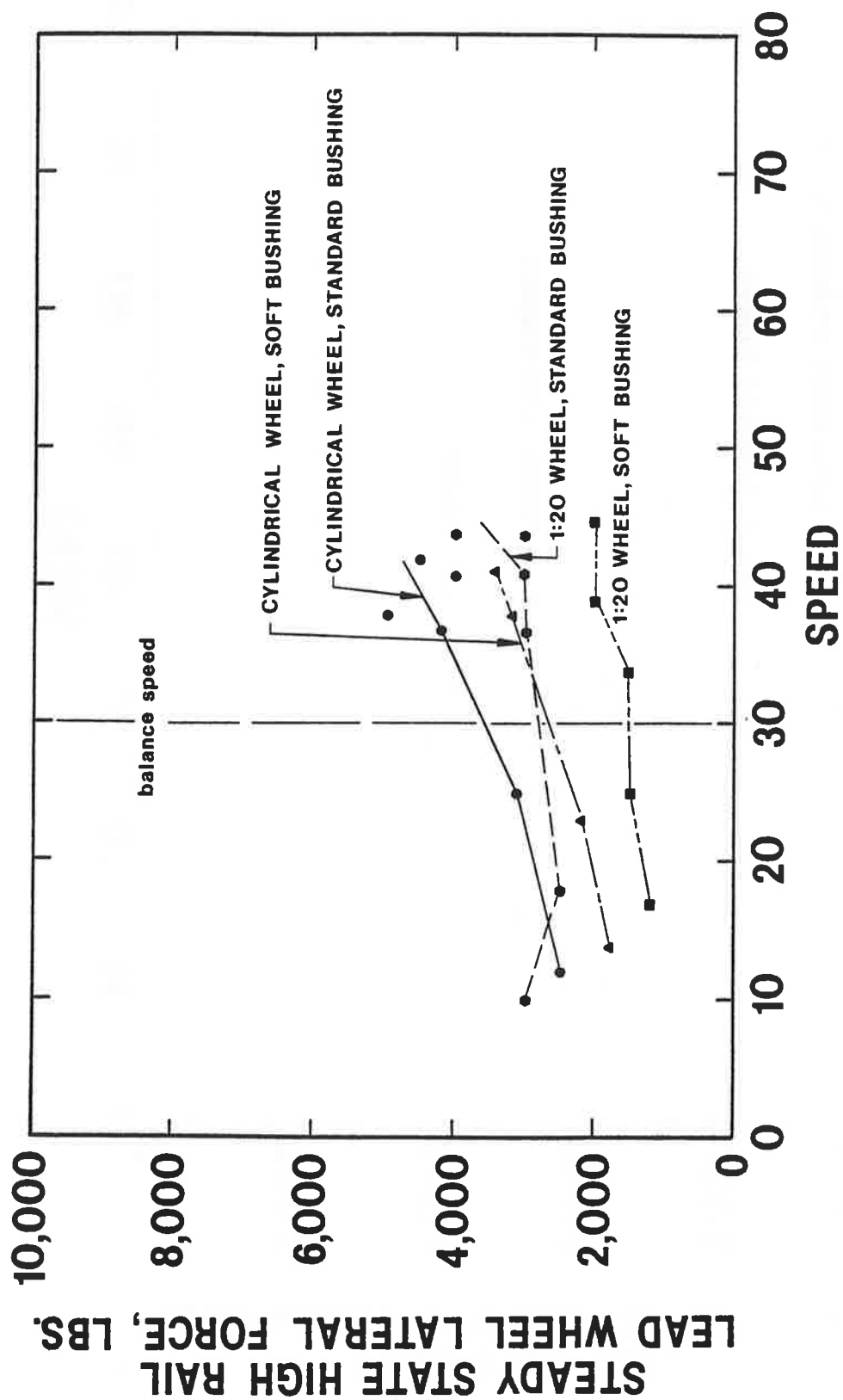


FIGURE A-3. CURVE 49 STEADY STATE LATERAL WHEEL FORCES

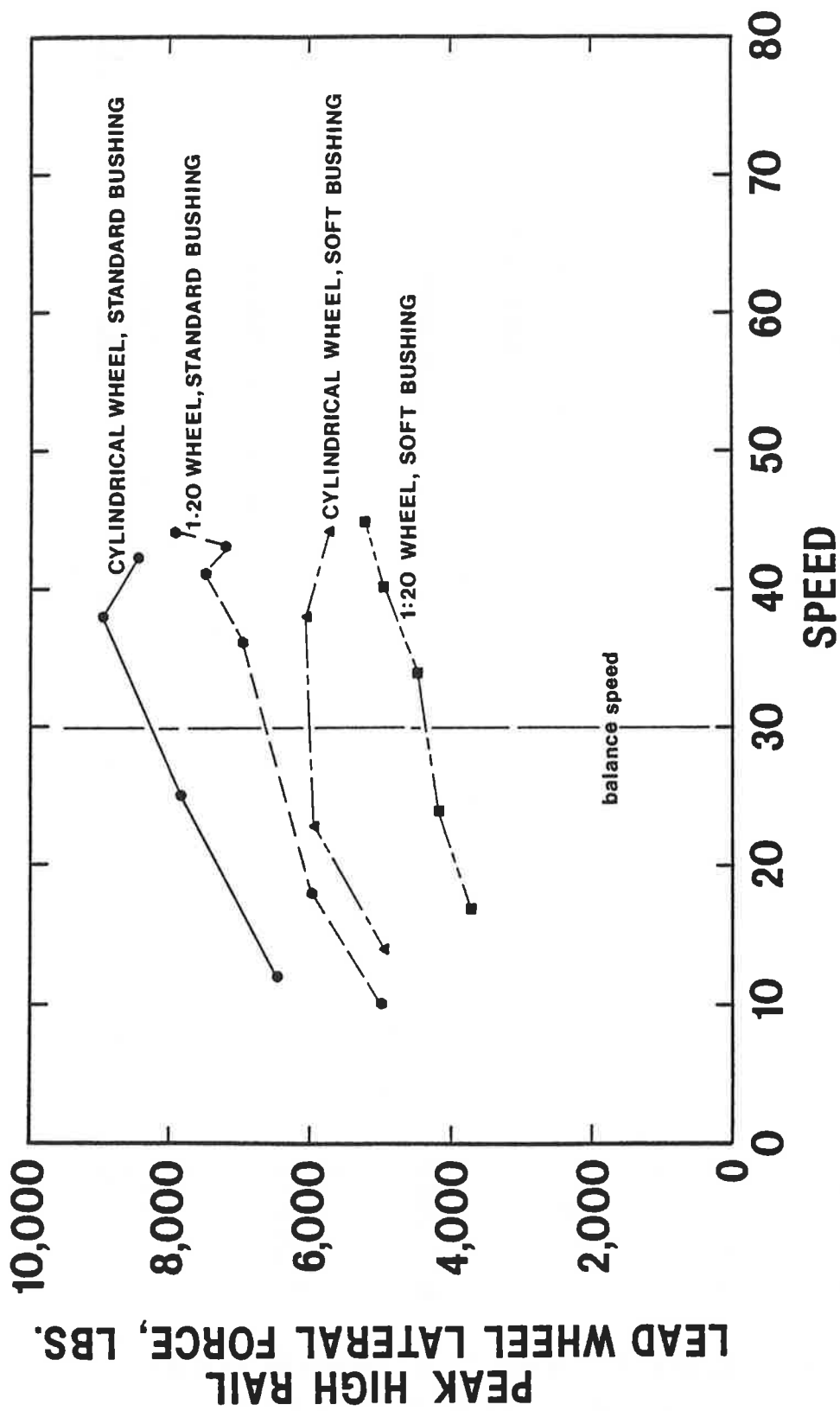


FIGURE A-4. CURVE 49 PEAK LATERAL WHEEL FORCES

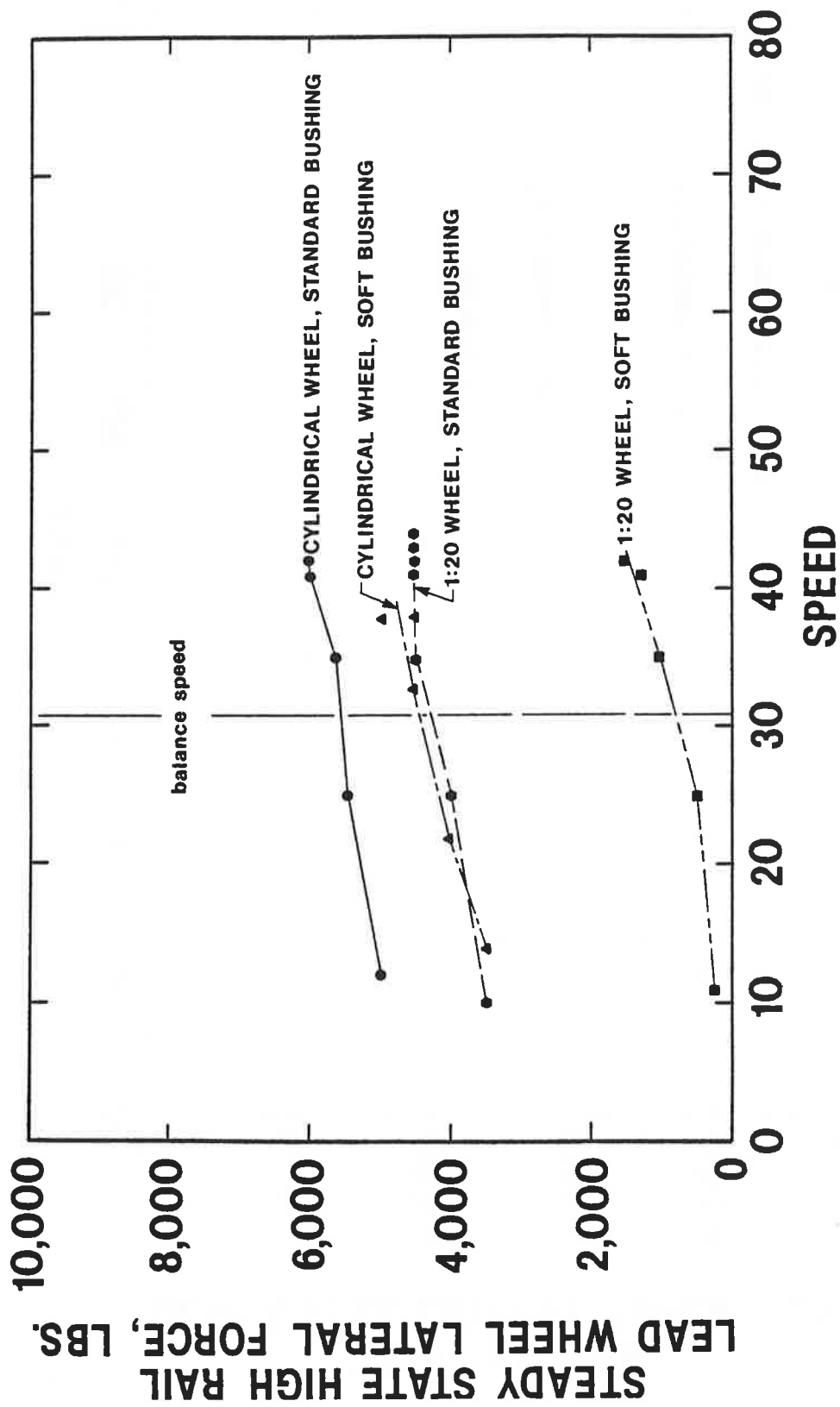


FIGURE A-5. CURVE 311 STEADY STATE LATERAL WHEEL FORCES

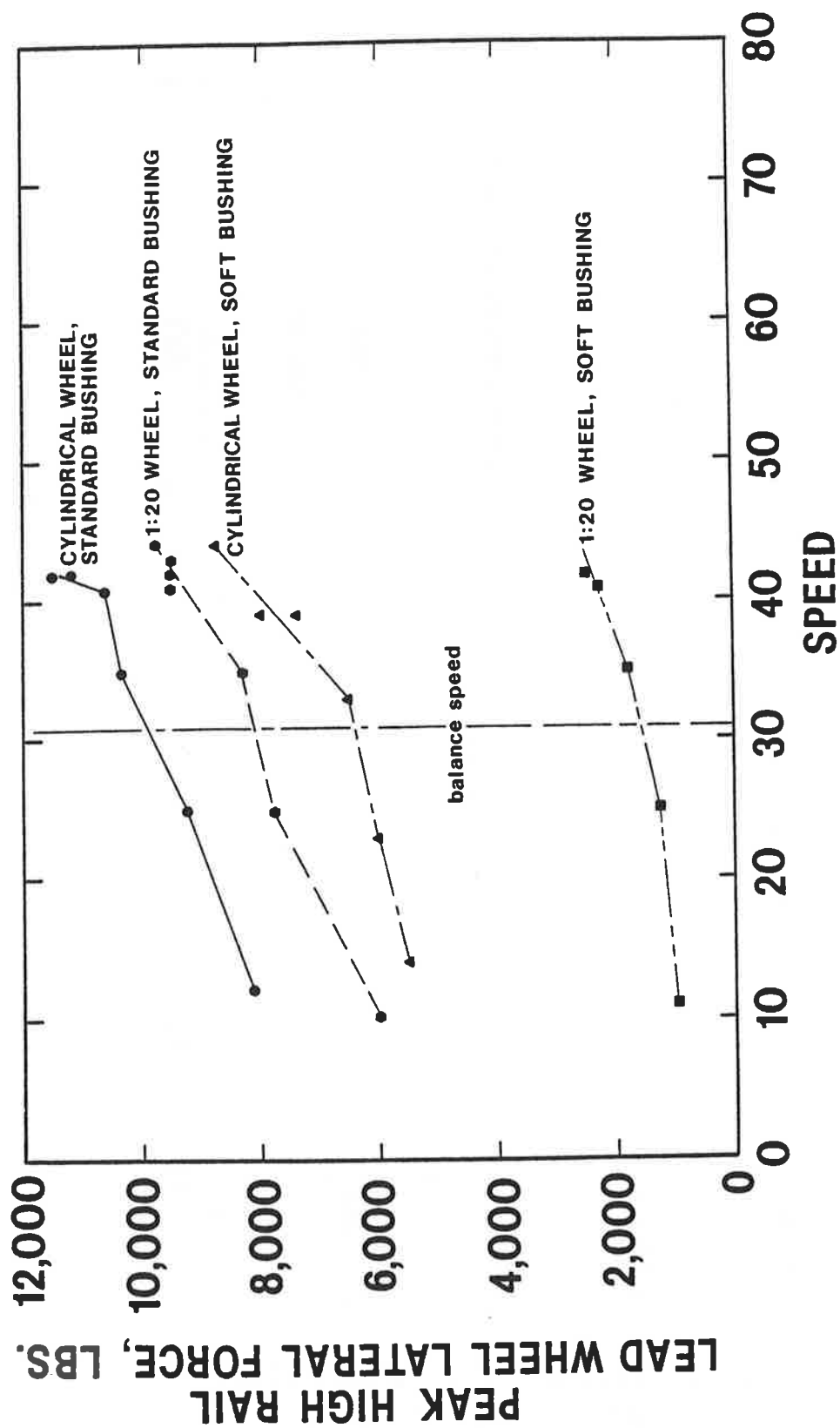


FIGURE A-6. CURVE 311 PEAK LATERAL WHEEL FORCES

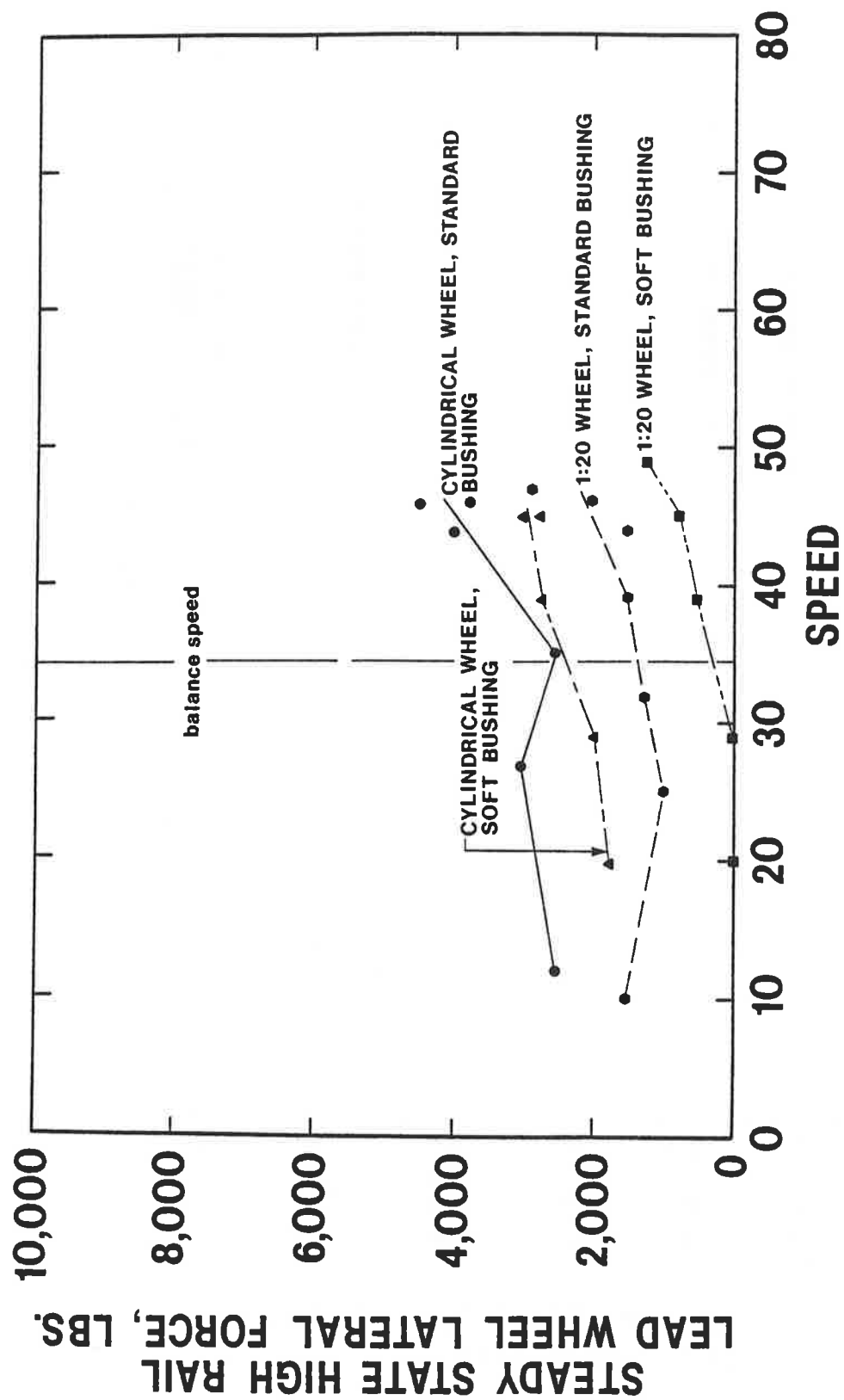


FIGURE A-7. CURVE 3 STEADY STATE LATERAL WHEEL FORCES

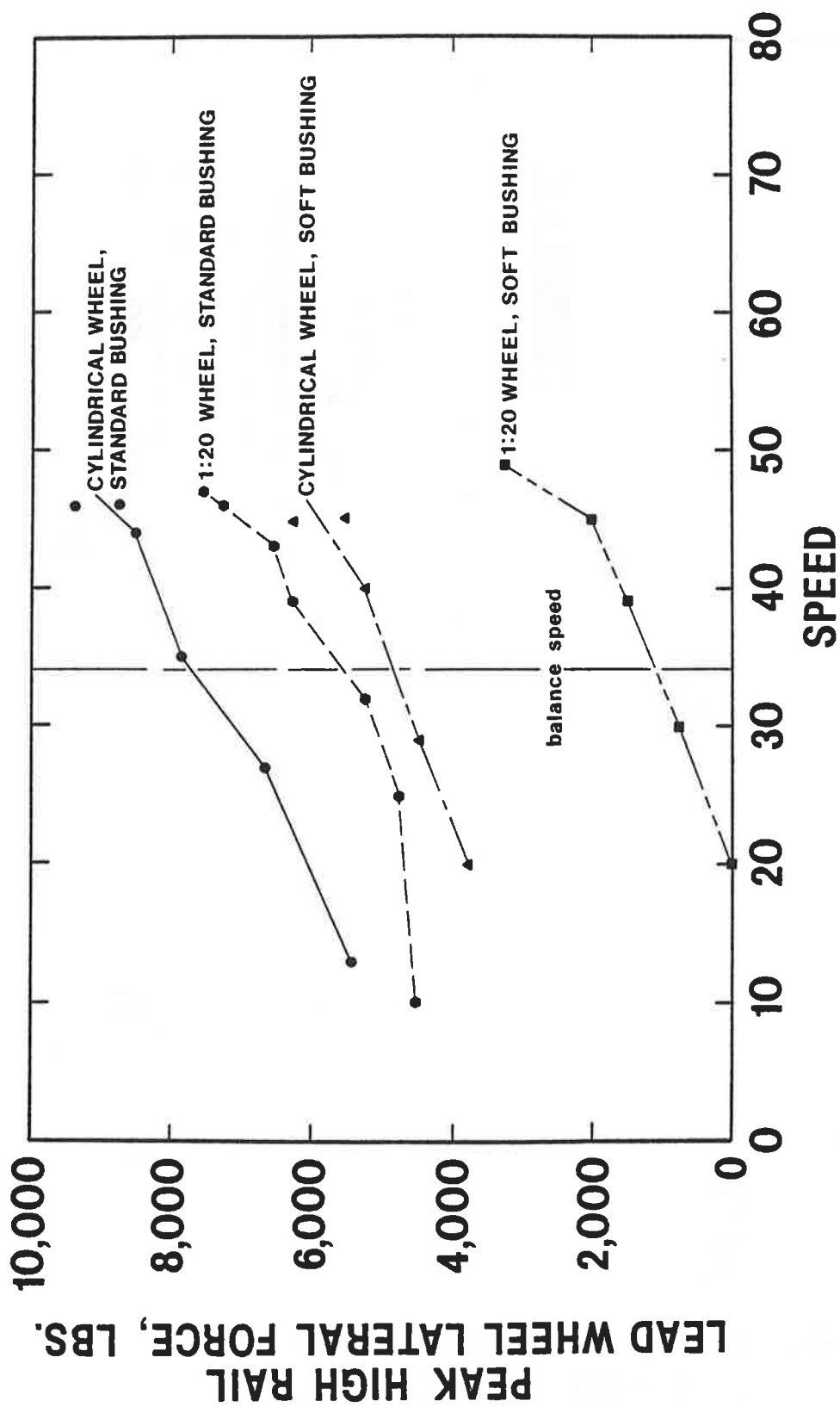


FIGURE A-8. CURVE 3 PEAK LATERAL WHEEL FORCES

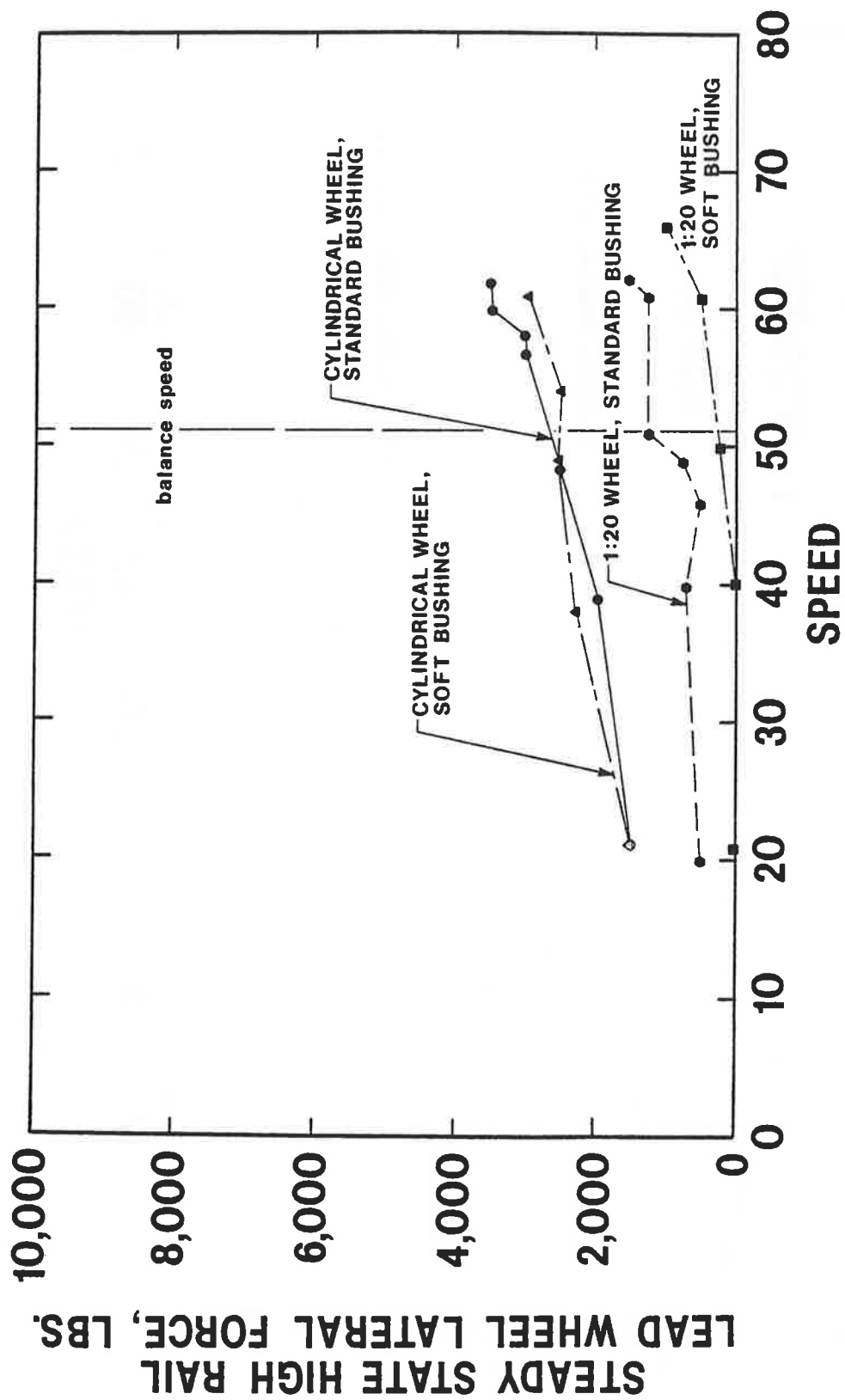


FIGURE A-9. CURVE 43 STEADY STATE LATERAL WHEEL FORCES

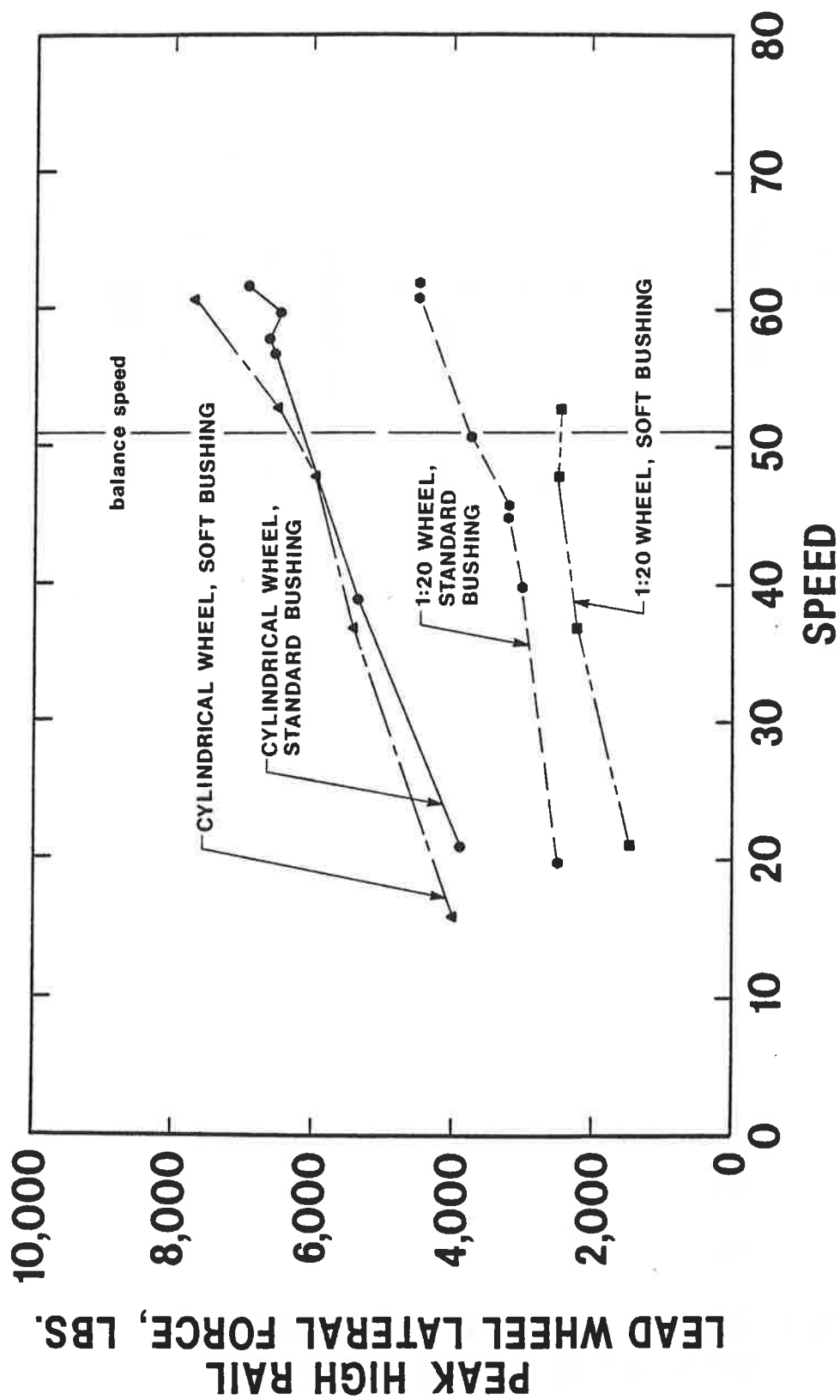


FIGURE A-10. CURVE 43 PEAK LATERAL WHEEL FORCES

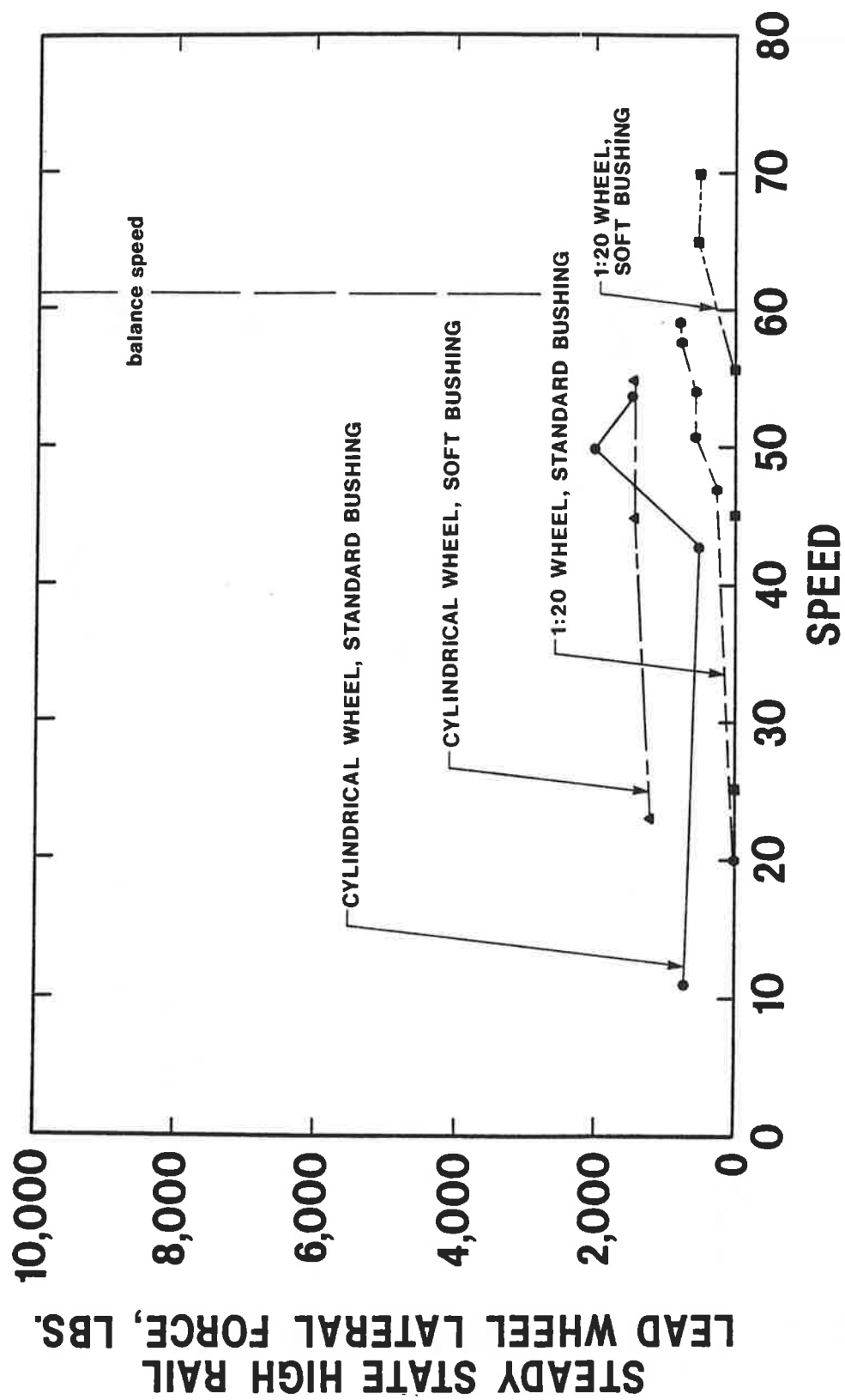


FIGURE A-11. CURVE 157 STEADY STATE LATERAL WHEEL FORCES

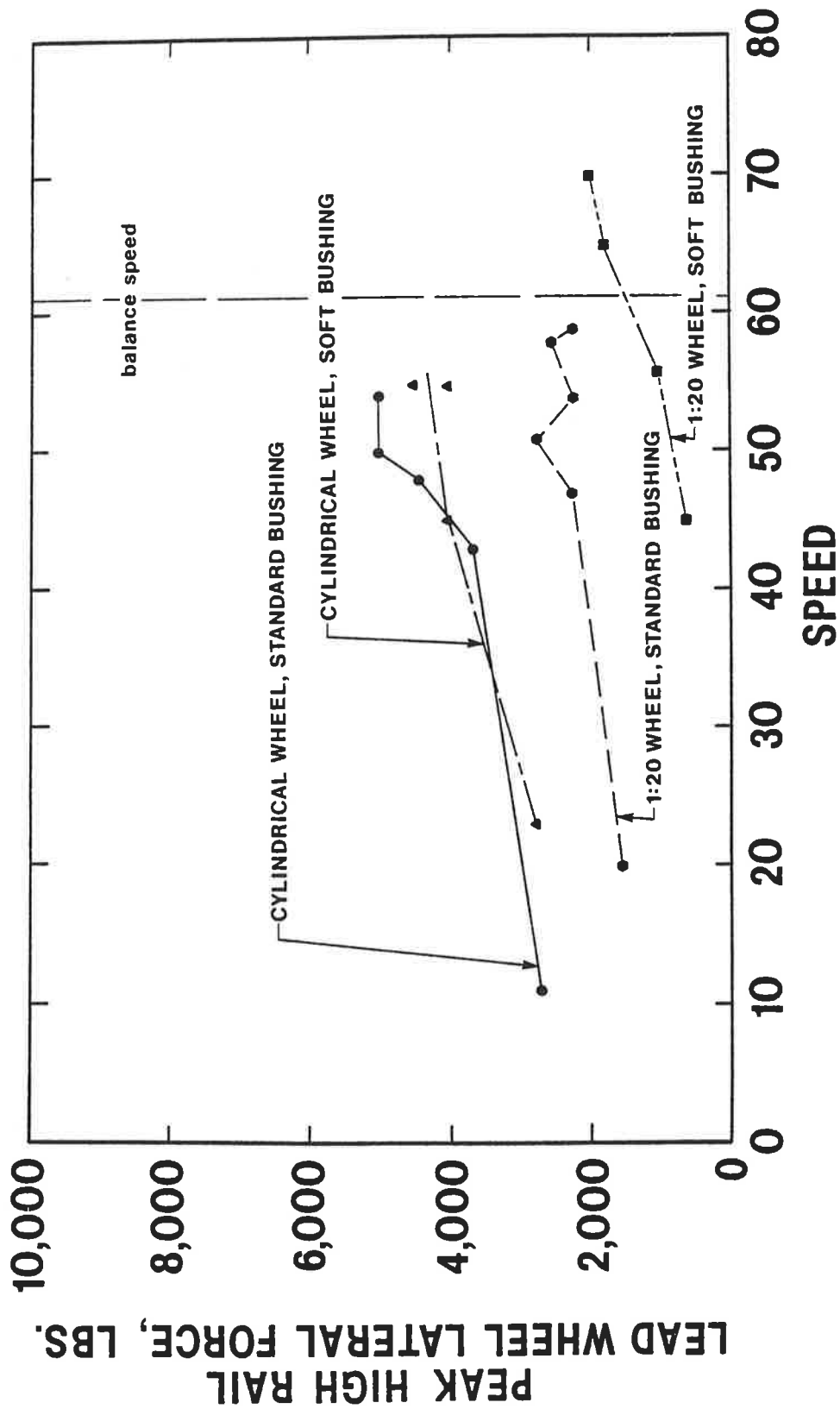


FIGURE A-12. CURVE 157 PEAK LATERAL WHEEL FORCES



## APPENDIX B: TABULATED RESULTS BY TEST SERIES

### LEGEND OF DATA ABBREVIATIONS

#### A. RUN CODING

example: 25 - G - SC (1R)

Day of the month ←

Test Series ←

A - cyl; stiff  
B - 1:20; stiff  
C - 1:10; stiff  
F - 1:10; soft  
G - 1:20; soft  
I - 1:20; stiff  
J - cyl; soft

Test Type ←

SC - steady curving  
RE - route evaluation  
ST - stability  
TR - acceleration in curve

Sequence Number ←

1 is the first run of a test type  
2 is the second run of a test type  
3 ...

Direction ←

R is reverse run  
F or blank is forward run  
(instrumented wheelset leading  
consist)

#### B. MEASUREMENTS

$L_L$  - Left wheel lateral force, lbs.

$L_R$  - Right wheel lateral force, lbs.

$\Delta Y_{LF}$  - Front axle primary lateral displacement, measured  
at left, in.

- $\Delta Y_{RR}$  - Rear axle primary lateral displacement, measured at right, in.
- $\Delta X_{LF}$  - Left front primary longitudinal displacement, in.
- $\Delta X_{RF}$  - Right front primary longitudinal displacement, in.
- $\Delta X_{LR}$  - Left rear primary longitudinal displacement, in.
- $a_{yA-1}$  - Lateral acceleration; lead axle, lead truck, g's
- $a_{yA-2}$  - Lateral acceleration; trailing axle, lead truck, g's
- $a_{yA-3}$  - Lateral acceleration; lead axle, trailing truck, g's

# SERIES A - STEADY CURVING, FORWARD

Curve Number	SPEED		$\Delta L_L$		$\Delta L_R$		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{LR}$	
	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	13	12	2750	1500	5400	2500	-.016	-.004	-.017	-.008	-.044	-.029	-.031	-.025
	32	27	2950	1475	6650	3000	-.014	.001	-.018	-.004	-.047	-.030	-.040	-.026
	44	35	2150	900	7850	2000	.015	.001	-.023	-.007	-.050	-.038	-.029	-.020
	47	46	1400	750	8750	3750	.016	.004	-.020	-.008	-.040	-.030	-.042	-.030
311	12	11	8150	5000	4750	3650	.025	.013	.015	.004	.041	.035	.056	.049
	25	24	9200	5475	4950	3800	.033	.011	.014	.003	.035	.028	.046	.039
	36	33	10400	5650	4700	3500	.030	.009	.017	.006	.033	.026	.047	.042
	42	40.5	10600	6000	3900	3000	.035	.009	.015	.006	.029	.022	.027	.023
37	12	10	4450	3000	7250	4500	-.019	-.007	-.023	-.015	-.047	-.042	-.050	-.045
	25	24	3950	3000	9000	5300	.017	.005	-.023	-.009	-.044	-.036	-.052	-.045
	39	34	3600	2475	9750	4575	.021	.007	-.024	-.006	-.054	-.043	-.056	-.048
	39	37	3150	2000	9500	5250	.019	.007	-.023	-.008	-.047	-.039	-.049	-.044
43	21	20	3100	1550	3850	1500	-.015	-.004	-.007	.001	-.045	-.028	-.023	-.015
	40	38	2975	1500	5400	2000	-.023	-.001	-.017	-.001	-.035	-.016	-.027	-.018
	57	56	2575	1100	6600	3000	.013	.004	-.017	-.003	-.047	-.025	-.024	-.014
	58	57.5	2450	1000	6650	3000	.013	.002	-.013	-.002	-.043	-.025	-.034	-.023
49	13	11	3450	1550	6500	2600	-.015	-.001	-.014	-.004	-.049	-.037	-.040	-.031
	26	24	3650	1400	7900	3100	-.013	.003	-.019	-.008	-.038	-.024	-.036	-.019
	37	36	2100	900	6400	4250	.020	.007	-.015	-.007	-.047	-.031	-.028	-.020
	41.5	40	1900	850	6200	4000	.017	.009	-.013	-.003	-.038	-.029	-.025	-.016
157	11	11	2450	1100	2750	750	-.022	-.009	-.014	-.006	-.028	-.008	-.019	-.009
	45	40	3800	870	3700	500	.019	.003	-.018	-.001	.016	.002	.014	.010
	56	52	2350	900	5000	1500	.015	.001	-.023	-.002	-.033	-.013	-.024	-.010
	52	44	2400	1150	4450	1600	-.015	.004	-.013	-.002	-.030	-.014	-.025	-.012

SERIES A - STEADY CURVING, REVERSE

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{LR}$	
	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3 22 ASC(1R) 22 ASC(2R) 22 ASC(3R) 22 ASC(4R) 22 ASC(5R)	12	11	900	100	800	250	-.015	-.005	-.011	-.002	-.038	-.031	-.045	-.034
	32	31	-300	-150	1400	400	-.014	-.002	-.013	-.001	-.038	-.030	-.047	-.034
	40	36	-500	-100	2400	700	-.012	-.001	-.013	-.001	-.044	-.034	-.048	-.030
	45	45	1300	500	3000	1300	.018	.002	-.020	-.003	-.041	-.030	-.064	-.048
311 22 ASC(1R) 22 ASC(2R) 22 ASC(3R) 22 ASC(4R) 22 ASC(5R)	12	11	-600	-500	1250	250	-.015	-.001	.007	.000	-.041	.036	.032	.027
	27	26	-400	-50	1200	300	-.015	.000	.011	.003	.041	.037	.033	.030
	36	35	900	150	1150	50	-.020	-.002	.022	.005	.038	.034	.034	.028
	42	42	1600	600	800	50	-.018	-.003	.027	.007	.036	.029	.022	.015
37 22 ASC(1R) 22 ASC(2R) 22 ASC(3R) 22 ASC(4R) 22 ASC(5R)	13	12	900	0.0	500	250	-.016	-.007	-.020	-.006	-.044	-.037	-.054	-.050
	32	28	-500	-200	2400	500	-.009	-.001	-.019	-.004	-.049	-.038	-.057	-.049
	41	36	-250	-200	2200	1000	.018	.003	-.023	-.004	-.048	-.042	-.073	-.058
	42	38	-200	-100	2300	1250	.019	.002	-.022	-.004	-.053	-.045	-.087	-.071
43 22 ASC(1R) 22 ASC(2R) 22 ASC(3R) 22 ASC(4R) 22 ASC(5R)	21	20	1100	650	1100	150	-.014	-.022	-.016	.000	-.028	-.020	-.034	-.021
	42	42	1400	550	2200	250	-.021	-.001	-.017	.000	-.027	-.019	-.041	-.019
	53	52	1600	500	3300	1400	.021	.002	-.018	-.001	-.029	-.019	-.041	-.018
	57	54	900	250	2950	750	.021	.003	-.023	-.004	-.023	-.014	-.064	-.038
49 22 ASC(1R) 22 ASC(2R) 22 ASC(3R) 22 ASC(4R) 22 ASC(5R)	13	12	1200	200	1900	500	.018	.001	.015	.003	-.031	-.027	-.034	-.026
	25	24.5	-400	-100	800	100	.015	.000	.015	.004	-.034	-.028	-.045	-.029
	38	37.5	-250	-200	1350	500	.021	.003	.017	.001	-.037	-.029	-.039	-.027
	40	38.5	1300	500	3600	1200	.020	.002	.017	.001	-.037	-.030	-.056	-.038
157 22 ASC(1R) 22 ASC(2R) 22 ASC(3R) 22 ASC(4R) 22 ASC(5R)	48	45	-300	-200	900	010	.014	.001	-.019	-.003	.011	.002	-.013	-.003

SERIES A - ROUTE EVALUATION, FORWARD - INSTRUMENTED AXLE IN LEAD

Curve Number	SPEED	L <sub>L</sub>		L <sub>R</sub>		ΔY <sub>LF</sub>		ΔY <sub>RR</sub>		ΔX <sub>LF</sub>		ΔX <sub>LR</sub>	
		Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	22 ARE (F)		700	8500	4000								
	22 ARE (EB)	44	2500	9400	4500								
311	22 ARE (F)		6000	3500	3000								
	22 ARE (EB)	42	11500	6000	3000								
37	22 ARE (F)		2500	9500	6000								
	22 ARE (EB)	37	3000	2500	6000								
43	22 ARE (F)		1000	6500	3500								
	22 ARE (EB)	60	2500	7000	35000								
49	22 ARE (F)		2000	8500	4500								
	22 ARE (EB)	42	3000	9000	5000								
157	22 ARE (F)		1250	5000	2000								
	22 ARE (EB)	50	2000	5000	2000								

SERIES A - ACCELERATION IN CURVES

Run Number	Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
		Max.	Min.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
23 ATR3P3	3	32	21	3000	1450	6750	3000	.008	-.001	-.016	-.008	-.030	-.020	-.034	-.028
23 ATR3P4	3	43	14	3150	1450	8300	3200	.012	.001	-.017	-.005	-.043	-.019	-.059	-.046
23 ATR37P3	37	28	8	4100	2850	8350	5000	.017	.003	-.015	-.010	-.054	-.041	-.045	-.036
23 ATR37P4	37	41	16	3800	2500	9400	5600	.018	.008	-.019	-.011	-.062	-.042	-.070	-.051
23 ATR157P3	157	40	10	2500	1400	3800	1300	.027	.008	.020	.009	-.034	-.009	-.017	-.008
23 ATR157P4	157	48	16	2450	1100	4200	1300	.028	.008	.021	.008	-.031	-.010	-.023	-.015

# SERIES A - STABILITY RUNS

Run Number	Speed (mph) min max	Peak Measurements - Absolute and Level Exceeded for 100 ms											
		L <sub>L</sub> (kips)		L <sub>R</sub> (kips)		a <sub>y</sub> A-1 (g's)		a <sub>y</sub> A-2 (g's)		a <sub>y</sub> A-3 (g's)		Δ <sub>y</sub> LF (in)	
		abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms
22AST35	30 36	3.5	3.5	2.5	1.5	.16	.08	.12	.04	.21	.08	.020	.012
22AST55	aborted												
22AST75	60 78	4.2	2.5	2.6	1.5	.38	.04	.22	.06	.41	.08	.018	.010
<u>Reverse Runs</u>													
22AST35R	32 36	.5	.5	.2	.2	-	-	.30	.04	.18	.04	.020	.014
22AST55R	56 56	1.0	.5	1.0	.2	.43	.05	.54	.06	.41	.05	.018	.010
22AST55R	62 74	1.0	.4	1.5	.2	.50	.06	.54	.09	.40	.10	.020	.012
												.014	.010

SERIES B - STEADY CURVING, FORWARD

Curve Number	SPEED		I <sub>L</sub>		I <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	9	10	3300	2300	4500	1500	+ .004	+ .001	+ .016	+ .008	-.012	-.004	+ .020	+ .008
	19	30	3800	1800	4750	1000	+ .022	+ .012	+ .016	+ .008	-.054	-.040	+ .058	+ .044
	24	40	3050	1800	5250	1250	+ .024	+ .012	+ .020	+ .012	-.056	-.044	+ .056	+ .044
	34	44	2550	1200	6250	1500	+ .028	+ .020	+ .016	+ .008	-.056	-.040	+ .054	+ .040
311	8	12	6000	3500	5800	3300	+ .020	+ .012	+ .012	+ .008	+ .020	+ .012	-.020	-.012
	24	26	7750	4000	5300	3800			+ .010	+ .004	+ .036	+ .028	-.028	-.020
	33	36	8250	4500	4800	3300			+ .012	+ .004	+ .036	+ .028	-.032	-.020
	39	42	9500	4500	3800	2800			+ .012	+ .004	+ .036	+ .024	-.036	-.024
37	8	12	5300	4300	6500	3500			+ .012	+ .006	-.036	-.032	+ .068	-.064
	24	26	4800	3800	7500	4000			+ .012	+ .004	-.052	-.040	+ .060	+ .052
	32	36	4300	3300	8000	4000	+ .032	+ .020	+ .012	-.008	-.060	-.048	+ .060	+ .052
	36	40	3550	2300	8000	3500	+ .028	+ .020	+ .016	+ .008	-.060	-.048	-.060	+ .052
43	20	20	3300	1800	2500	500	+ .016	+ .008	+ .012	+ .004	-.040	-.028	+ .056	+ .040
	40	40	3050	1300	3000	700	+ .024	+ .012	+ .020	+ .008	-.048	-.028	+ .056	+ .040
	48	41	2550	1300	3250	500	+ .024	+ .008	+ .020	+ .008	-.052	-.032	+ .052	+ .032
	49	40	4800	130	3250	500	+ .010	+ .008	+ .020	+ .012	-.056	-.036	+ .040	+ .024
49	8	12	5300	3800	5000	3000			+ .010	+ .008	-.025	-.016	+ .048	+ .054
	17	18	4300	3300	6000	2500	+ .020	+ .016	+ .016	+ .012	-.048	-.036	+ .032	+ .028
	36	37	3050	2300	2000	3000	+ .024	+ .016	+ .012	+ .006	-.052	-.040	+ .046	+ .040
	40	42	2800	2050	7500	3000			+ .010	+ .006	-.052	-.040	+ .046	+ .036
157	20	20	3050	1800	1500	0			+ .012	+ .008	-.036	-.024	+ .048	+ .030
	44	50	2800	1550	2250	250	+ .020	+ .012	-.016	+ .008	-.034	-.020	+ .052	+ .032
	52	56	2300	1000	2250	700	+ .020	+ .012	+ .012	+ .004	-.040	-.024	+ .048	+ .032
	50	52	2550	1300	2750	700			+ .008	+ .004	-.048	-.024	+ .048	+ .028

SERIES B - STEADY CURVING, REVERSE

Curve Number	SPEED		I <sub>L</sub>		I <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	10	12	1000	0	550	250			-.024	-.018	-.026	-.022	+.032	+.020
	29	32	250	0	1000	550	-.020	-.012	-.024	-.020	-.030	-.024	+.039	+.020
	39	40	0	0	1500	800	-.020	-.012	-.026	-.020	-.028	-.020	+.028	+.016
	46	47	0	0	2250	1000	-.016	-.006	-.028	-.020	-.028	-.018	+.024	+.014
311	10	11	700	250	1550	500			+.018	+.010	+.028	+.024	-.024	-.018
	22	28	1000	0	800	0			+.024	+.014	+.026	+.012	-.024	-.016
37														
	10	12	800	250	800	500	-.022	-.012	-.028	-.018	-.028	-.024	+.044	+.038
	35	37	250	0	1500	800	-.014	-.012	-.022	-.020	-.030	-.026	+.040	+.032
	33	38	250	0	2250	1000	-.022	-.010	-.032	-.020	-.034	-.026	+.036	+.028
	37	41	250	0	2500	1500	-.024	-.008	-.032	-.020	-.034	-.024	+.036	+.028
43														
	20	21	750	0	550	250	-.018	-.014	-.022	-.016	-.016	-.012	+.024	+.016
	41	43	250	0	1000	550	-.022	-.014	-.024	-.016	-.024	-.016	+.018	+.010
	48	53	0	0	1750	800	-.024	-.012	-.020	-.016	-.026	-.020	+.024	+.012
	48	54	250	0	1500	800	-.020	-.010	-.020	-.016	-.026	-.020	+.010	+.008
49														
	9	11		500	800	250			-.024	-.016	-.034	-.030	+.040	+.028
	25	26	1050	250	1050	500			-.018	-.012	-.040	-.032	+.040	+.028
	36	36	250	0	1750	800			-.020	-.016	-.034	-.024	+.028	+.020
	40	41	0	0	2250	1000			-.020	-.020	-.028	-.016	+.032	+.024
157														
	19	22	3250	1000	550	0			-.016	-.008	-.028	-.016	+.030	+.016
	45	46	2000	550	1050	500	+.012	+.002	-.016	-.008	-.028	-.016	+.026	+.012
	54	57	1750	500	1500	750			-.012	-.008	-.028	-.018	+.026	+.012
	54	58	1250	500	1250	750			-.016	-.010	-.030	-.020	+.028	+.008

SERIES B - ROUTE EVALUATION, FORWARD

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		Y <sub>LF</sub>		ΔY <sub>RR</sub>		ΔX <sub>LF</sub>		ΔX <sub>RF</sub>		
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	
3	10 BRE (F)	42	46	1800	800	6500	1500	.028	.018	.016	.007	-.068	-.056	.052	.044
	10 BRE (FB)	6	34	4300	2800	6750	2000	+.028	+.010	+.012	+.004	-.040	-.028	+.060	+.052
	12 BRE (F)	46	48	2050	800	7500	3000			+.028	+.012	-.076	-.064	+.052	+.040
	12 BRE (FB)	44	47	2300	1050	7250	2000			+.014	+.006	+.046	-.032	+.048	+.040
311	10 BRE (F)	42	45	8750	4000	2800	1800	.016	-.006	.016	.008	.044	.036	-.040	-.036
	10 BRE (FB)	42	44	9500	5600	3050	1800			+.012	+.004	+.032	+.024	-.052	-.044
	12 BRE (F)	43	44	9750	4500	3300	2300			+.018	+.008	+.032	+.024	-.052	-.044
	12 BRE (FB)	40	44	9500	4500	3300	2050			+.012	+.004	+.036	+.028	-.032	-.020
37	10 BRE (F)	37	40	3300	2300	7250	3500	+.040	+.012	+.016	+.004	-.064	-.048	+.068	+.060
	10 BRE (FB)	36	37	3550	3400	9750	5000	+.032	+.024	+.016	+.006	-.056	-.044	+.056	+.048
	12 BRE (F)	36	40	3800	3300	7250	4000			+.008	+.004	-.076	-.064	+.072	+.060
	12 BRE (FB)	37	40	3550	2800	7250	5000			+.010	+.004	-.052	-.040	+.060	+.048
43	10 BRE (F)	48	54	1800	800	3750	1000	+.028	+.020	+.020	+.012	-.068	-.042	+.040	+.020
	10 BRE (FB)	45	54	1800	800	3500	800	+.018	+.012	+.010	+.010	-.052	-.035	+.040	+.028
	12 BRE (F)	60	63	1550	550	4500	1500			+.020	+.008	-.056	-.036	+.056	+.040
	12 BRE (FB)	60	62	1800	800	4500	1000			+.016	+.008	-.046	-.028	+.048	+.032
49	10 BRE (F)														
	10 BRE (FB)														
	12 BRE (F)	43	44	2800	1800	7250	4000			+.012	+.004	-.048	-.040	+.064	+.056
	12 BRE (FB)	43	44	3300	1800	8000	3000			+.010	+.006	-.056	-.044	+.052	+.044
157	10 BRE (F)														
	10 BRE (FB)														
	12 BRE (F)	55	64	2300	800	2250	800			+.012	+.006	-.040	-.028	+.050	+.028
	12 BRE (FB)	54	62	2300	1300	2500	800			+.012	+.006	-.042	-.024	+.052	+.032

SERIES B - ROUTE EVALUATION, REVERSE

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
10 BRE (R)	48	49	0	0	2800	1250	-.020	-.010	-.032	-.024	-.038	-.026	+.024	+.016
12 BRE (R)	48	49	250	0	2750	1800			-.024	-.016	-.022	-.012	+.028	+.014
10 BRE (R)	36	44	1800	800	500	0	-.010	-.006	+.034	+.020	+.030	+.022	-.024	-.020
12 BRE (R)	11	14	750	250	2000	750			+.014	+.008	+.038	+.030	-.012	-.010
10 BRE (R)	38	39	0	0	2500	1500	-.016	-.012	-.028	-.024	-.044	-.036	+.034	+.028
12 BRE (R)	37	40	250	0	2500	1500			-.022	-.018	-.026	-.018	+.020	+.012
10 BRE (R)	52	56	0	0	1500	800	-.020	-.014	-.036	-.020	-.016	-.012	+.028	+.020
12 BRE (R)	56	61	250	0	2000	1000			-.024	-.018	-.014	-.004	+.028	+.020
12 BRE (R)	42	44	500	0	2750	1500			-.028	-.020	-.018	-.014	+.030	+.020
12 BRE (R)	56	62	1250	500	1750	800			-.018	-.012	-.024	-.012	+.016	+.012

SERIES B - ACCELERATION IN CURVES

Run Number	Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
		Max.	Min.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
10-BTR3P3	3	34	-	4300	2800	6000	1500	+0.012	+0.008	+0.012	+0.008	-0.040	-0.028	+0.060	+0.052
10-BTR3P4	3	44	-	4300	2800	6000	2000	+0.028	+0.020	+0.016	+0.008	-0.040	-0.028	+0.060	+0.052
10-BTR37P3	37	32	-	5800	4300	7000	4500	+0.028	+0.016	+0.012	+0.004	-0.048	-0.036	+0.060	+0.052
10-BTR37P4	37	42	-	5550	4050	8000	4500	+0.028	+0.020	+0.012	+0.004	-0.060	-0.032	+0.000	+0.052
12-BTRL57P3	157	44	-	3300	1800	2000	500			+0.004	+0.002	-0.032	-0.020	+0.048	+0.032
12-BTRL57P4	157	54	-	3300	1800	2500	800			+0.012	+0.004	-0.032	-0.016	+0.048	+0.032

# SERIES B - STABILITY RUNS

Run Number	Speed (mph)		Peak Measurements - Absolute and Level Exceeded for 100 ms											
			L <sub>L</sub> (kips)		I <sub>R</sub> (kips)		a <sub>y</sub> A-1 (g's)		a <sub>y</sub> A-2 (g's)		a <sub>y</sub> A-3 (g's)		Δ <sub>y</sub> LF (in)	
	min	max	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms
12BST(35)	35	37	2250	2000	2000	1250	.14	.04	.14	.04	.20	.08	.004	.008
12BST(55)	55	57	2250	1000	2250	1000	.24	.12	.26	.08	.28	.08	.006	.012
12BST(75)	56	69	2750	1250	2000	1000	.28	.10	.32	.08	.028	.08	.010	.012

## Reverse Runs

12BST (35R)	35	36	1200	1250	1750	750	.12	.04	.12	.06	.12	.04	.010	.006
12BST (55R)	56	58	1250	1000	1750	750	.30	.08	.30	.08	.20	.04	.008	.010
12BST (75R)	72	77	1500	1000	1700	500	.32	.10	.44	.08	.34	.06	.010	.014

SERIES C - STEADY CURVING, FORWARD

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	10	15	2500	2000	4000	1500	+0.24	+0.016	+0.012	+0.010	-0.032	-0.028	+0.036	+0.028
	25	26	2000	1500	4250	1250	+0.20	+0.012	+0.012	+0.008	-0.032	-0.024	+0.036	+0.032
	42	44	1750	1000	6250	2000	+0.016	+0.008	+0.008	+0.004	-0.042	-0.032	-0.020	+0.018
	46	48	1500	1000	6500	2500	+0.016	+0.008	+0.012	+0.006	-0.040	-0.032	+0.028	+0.020
311	8	10	5000	2500	3750	2500	-0.014	-0.004	-0.016	-0.004	+0.028	+0.024	-0.016	-0.012
	22	25	6000	3500	3500	2000	-0.016	-0.008	-0.010	-0.008	+0.028	+0.020	-0.016	-0.004
	33	37	6750	4000	3000	2250	-0.022	-0.010	-0.018	-0.008	+0.020	+0.014	-0.024	-0.020
	40	41	7500	4000	2500	1750	-0.024	-0.012	-0.014	-0.006	+0.016	+0.008	-0.028	-0.024
37	8	12	3500	2500	4750	2500	+0.020	+0.012	+0.010	+0.008	-0.032	-0.028	+0.032	+0.028
	24	27	3500	2250	6000	3250	+0.028	+0.016	+0.008	+0.006	-0.036	-0.028	+0.048	+0.042
	33	36	3500	2500	7000	3500	+0.024	+0.012	+0.004	+0.002	-0.040	-0.032	+0.044	+0.040
	38	41	3000	2000	7250	3500	+0.024	+0.012	+0.004	+0.002	-0.048	-0.040	+0.040	+0.032
43	19	20	2500	1500	2250	500	+0.016	+0.008	+0.014	+0.010	-0.032	-0.024	+0.036	+0.028
	42	42	2500	1750	2750	1000	+0.020	+0.012	+0.014	+0.010	-0.032	-0.020	+0.036	+0.030
	50	51	2500	1500	3000	1000	+0.018	+0.008	+0.010	+0.008	-0.036	-0.024	+0.034	+0.026
	63	65	2000	750	4500	1500	+0.016	+0.008	+0.016	+0.010	-0.042	-0.032	+0.026	+0.020
49	9	10	3250	2000	3500	2000	+0.020	+0.012	+0.012	+0.010	-0.028	-0.020	+0.032	+0.028
	25	27	3250	2250	6000	2500	+0.020	+0.012	+0.012	+0.010	-0.032	-0.024	+0.032	+0.028
	47	49	3000	2500	6750	3000	+0.020	+0.010	+0.008	+0.004	-0.036	-0.028	+0.032	+0.028
	42	42	2750	1500	6750	3000	+0.022	+0.012	+0.012	+0.008	-0.042	-0.036	+0.030	+0.026
157	18	25	3000	1500	2000	0	+0.016	+0.004	+0.014	+0.012	-0.020	-0.016	+0.030	+0.022
	44	45	2500	1500	2250	500	+0.016	+0.004	+0.012	+0.006	-0.034	-0.024	+0.028	+0.020
	58	59	2500	1000	2500	500	+0.014	+0.004	+0.014	+0.006	-0.034	-0.016	+0.034	+0.020
	65	67	2000	1000	3000	1000	+0.024	+0.004	+0.020	+0.008	-0.044	-0.024	+0.030	+0.014

SERIES C - STEADY CURVING, REVERSE

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	8	14	1250	0	500	0	-.024	-.008	-.008	-.008	-.016	-.016	.016	.014
	28	28	500	0	750	250	-.020	-.008	-.016	-.016	-.016	-.012	.012	.012
	39	41	500	0	1500	500	-.022	-.008	-.018	-.016	-.020	-.012	.016	.012
	41	45	500	0	1500	1000	-.020	-.008	-.018	-.016	-.016	-.012	.016	.014
311	8	18	500	0	1500	500	.016	.008	-.010	.002	.020	.018	-.016	-.014
	20	28	500	0	500	0	.028	.012	.012	.008	.022	.020	-.020	-.016
	33	34	750	250	1000	250	.020	.004	.016	.010	.018	.014	-.018	-.016
	32	39	1000	250	1000	500	-.016	.002	.020	.012	.024	.018	.012	.010
37	8	14	1250	500	500	0	-.032	-.012	-.012	-.012	-.020	-.016	.024	.024
	24	28	750	0	1000	250	-.028	-.012	-.014	-.012	-.028	-.024	.024	.020
	32	37	500	0	1500	500	-.032	-.008	-.016	-.016	-.024	-.022	.022	.018
	38	42	250	0	2000	1000	-.024	-.012	-.024	-.020	-.034	-.020	.016	.016
43	20	20	500	0	500	0	.026	.016	-.016	-.016	-.016	-.014	.008	.008
	40	40	500	0	1250	500	-.022	-.012	-.020	-.018	-.014	-.004	.008	.006
	49	50	250	0	1500	500	-.024	-.010	-.018	-.016	-.016	-.010	.012	.008
	58	58	0	0	2000	1000	-.022	-.006	-.020	-.016	-.014	-.004	.012	.008
49	8	12	2000	500	500	0	-.028	-.016	-.016	-.014	-.020	-.016	.016	.014
	24	26	1000	0	1000	0	-.024	-.012	-.020	-.014	-.028	-.026	.020	.016
	33	35	500	0	1500	500	-.026	-.012	-.022	-.014	-.032	-.022	.020	.016
	34	40	500	0	2000	750	-.032	-.012	-.020	-.016	-.036	-.024	.020	.014
157	18	20	2000	500	1000	0	-.034	-.012	-.016	-.008	-.012	-.004	.008	.004
	45	46	1000	0	750	0	-.024	-.008	-.016	-.012	-.016	-.008	.012	.006
	54	56	500	0	1000	250	-.028	-.008	-.020	-.006	-.016	-.008	.016	.012
	60	65	500	0	1500	500	-.020	-.008	-.016	-.008	-.014	-.008	.012	.008

SERIES C - ROUTE EVALUATION, FORWARD

Curve Number		SPEED		L <sub>T</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
		Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
19 CRE (F)	3	38	43	2500	1500	5250	2000	+ .020	+ .012	+ .004	.002	-.040	-.030	+ .032	+ .020
19 CRE (F)	311	32	32	6500	4000	3250	2000	-.022	-.008	-.022	-.008	+ .022	+ .020	-.020	-.018
19 CRE (F)	37	32	33	3250	2500	6500	3000	+ .024	+ .012	+ .004	+ .002	-.040	-.036	+ .040	+ .032
19 CRE (F)	43	41	43	3000	2000	3500	750	+ .018	+ .008	+ .014	+ .008	-.040	-.024	+ .036	+ .030
19 CRE (F)	49	41	42	2750	2000	6750	3000	+ .024	+ .014	+ .008	+ .004	-.044	-.036	+ .028	+ .024
19 CRE (F)	157	52	54	2500	1500	2000	500	+ .010	+ .004	+ .010	+ .004	-.042	-.028	+ .024	+ .012

SERIES C - ROUTE EVALUATION, REVERSE

Curve Number	19 CRE (R)	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
		Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
	3	31	32	500	0	1000	0	-.024	-.012	-.016	-.14	-.020	-.016	.016	.012
	311	0	25	500	0	1500	250	.028	.012	.020	.012	.024	.022	-.016	-.012
	37	36	38	500	0	1750	750	-.020	-.008	-.018	-.016	-.026	-.018	.028	.024
	43	50	50	500	0	1250	500	-.024	-.012	-.018	-.016	-.020	-.012	.018	.014
	49	42	44	500	0	2500	1000	-.036	-.008	-.028	-.016	-.040	-.032	.030	.024
	157	53	58	500	0	1000	250	-.034	-.008	-.026	-.008	-.018	-.008	.012	.006

SERIES C - (BALLASTED) STEADY CURVING, FORWARD

Curve Number	SPEED		I <sub>L</sub>		I <sub>R</sub>		Δ Y <sub>LF</sub>		Δ Y <sub>RR</sub>		Δ X <sub>LF</sub>		Δ X <sub>RF</sub>		
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	
3	20 CSC (1)	19	20	2750	1750	4000	1000	+ .026	+ .012	+ .018	+ .014	- .036	- .028	+ .046	+ .042
	20 CSC (2)	29	30	2250	1250	4500	1500	+ .022	+ .012	+ .018	+ .012	- .040	- .030	+ .038	+ .032
	20 CSC (3)	40	41	1750	1000	6000	1750	+ .024	+ .016	+ .014	+ .008	- .040	- .028	+ .030	+ .028
	20 CSC (4)	43	46	1500	1000	6500	2000			+ .010	+ .006	- .044	- .036	+ .016	+ .012
	20 CSC (5)	48	50	1250	750	8250	2500			+ .016	+ .008	- .044	- .036	+ .024	+ .020
311	20 CSC (1)	12	14	6500	4000	4500	3000	- .016	- .002	- .016	- .008	+ .036	+ .030	- .022	- .018
	20 CSC (2)	22	24	6500	4000	4000	3000	- .008	- .002	- .014	- .008	+ .030	+ .024	- .024	- .022
	20 CSC (3)	32	35	8000	4000	3500	2000	- .018	- .006	- .012	- .004	+ .028	+ .020	- .024	- .020
	20 CSC (4)	38	39	9000	5000	3500	2000			+ .014	+ .004	+ .024	+ .014	- .022	- .016
	20 CSC (5)	42	44	9200	5500	3250	2000			+ .012	+ .004	+ .024	+ .010	- .030	- .024
37	20 CSC (1)	10	14	5000	3500	6000	3000	+ .026	+ .014	+ .014	+ .012	- .028	- .024	+ .028	+ .026
	20 CSC (2)	22	25	5000	3500	6250	3250	+ .028	+ .018	+ .010	+ .008	- .046	- .044	+ .050	+ .048
	20 CSC (3)	32	35	3500	2500	7750	3000	+ .036	+ .020	+ .012	+ .006	- .052	- .042	+ .056	+ .044
	20 CSC (4)	36	40	3600	2400	8000	3400			+ .010	+ .004	- .052	- .044	+ .056	+ .046
	20 CSC (5)	39	44	3250	2250	7700	3500			+ .016	+ .006	- .050	- .046	+ .048	+ .044
43	20 CSC (1)	18	20	3600	2000	2500	250	+ .022	+ .012	+ .016	+ .016	- .032	- .026	+ .048	+ .042
	20 CSC (2)	35	38	3250	2000	3250	250	+ .022	+ .012	+ .020	+ .012	- .040	- .028	+ .040	+ .034
	20 CSC (3)	48	49	2500	1250	3000	750	+ .022	+ .016	+ .014	+ .012	- .040	- .028	+ .032	+ .028
	20 CSC (4)	57	58	2000	1000	3750	1000			+ .020	+ .012	- .040	- .030	+ .034	+ .028
	20 CSC (5)	61	62	1500	500	4250	1000			+ .020	+ .010	- .044	- .032	+ .032	+ .024
49	20 CSC (1)	13	15	3500	2250	4250	1750	+ .020	+ .010	+ .016	+ .014	- .034	- .028	+ .046	+ .042
	20 CSC (2)	23	25	3500	2250	5000	2000	+ .020	+ .012	+ .014	+ .010	- .040	- .032	+ .036	+ .034
	20 CSC (3)	34	35	3000	1750	6000	2500	+ .024	+ .018	+ .012	+ .010	- .040	- .032	+ .032	+ .030
	20 CSC (4)	38	42	2600	1500	7100	3000			+ .012	+ .008	- .040	- .032	+ .036	+ .032
	20 CSC (5)	44	46	2250	1000	8500	3500			+ .020	+ .008	- .040	- .032	+ .032	+ .028
157	20 CSC (1)	24	25	3250	1500	1500	0	+ .018	+ .004	+ .016	+ .012	- .026	- .022	+ .044	+ .032
	20 CSC (2)	42	46	2500	1250	2000	250	+ .018	+ .004	+ .016	+ .010	- .036	- .022	+ .036	+ .024
	20 CSC (3)	64	66	2500	1250	2000	250	+ .018	+ .004	+ .020	+ .012	- .038	- .024	+ .036	+ .022
	20 CSC (4)	62	66	2000	1000	2750	500			+ .016	+ .008	- .040	- .024	+ .044	+ .028
	20 CSC (5)	60	66	2000	750	2500	500			+ .020	+ .008	- .040	- .020	+ .040	+ .020

SERIES C - (BALLASTED) STEADY CURVING, REVERSE

Curve Number	SPEED		I <sub>L</sub>		I <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	19	20	1250	250	750	250	-.028	-.016	-.012	-.012	-.018	-.010	.018	.016
	28	30	500	0	750	250	-.020	-.016	-.016	-.012	-.016	.008	.016	.014
	38	40	500	0	1750	750	-.022	-.012	-.020	-.018	-.020	-.008	.012	.010
	42	46	500	0	2250	1000			-.024	-.020	-.016	-.008	.016	.012
	48	50	500	0	2500	1500			-.024	-.020	-.018	-.012	-.008	-.006
311	14	14	1000	250	1500	250	.028	.008	.015	.008	.028	.024	-.008	-.006
	24	24	1500	500	1000	0	.020	.008	.016	.010	.028	.024	-.010	-.008
	33	35	1500	750	1000	0	.028	.004	.022	.016	.020	.016	-.022	-.020
	40	40	2000	1000	500	0			.026	.016	.022	.016	-.018	-.012
	44	45	2500	1500	500	0			.028	.020	.018	.012	-.020	-.016
37	15	16	1500	500	1250	250	.040	.024	-.014	-.012	-.020	-.012	.024	.020
	22	27	1000	0	1200	250	-.038	-.016	-.020	-.018	-.020	-.012	.030	.028
	32	38	500	0	2000	1000	-.040	-.012	-.028	-.020	-.024	-.012	.022	.020
	32	42	250	0	2500	1500			-.034	-.020	-.028	-.020	.024	.020
	34	46	500	0	3500	2000			-.032	-.016	-.028	-.020	.024	.020
43	20	20	1500	250	250	0	-.028	-.012	-.020	-.014	-.020	-.016	.012	.008
	40	40	500	0	1500	500	-.026	-.016	-.016	-.012	-.014	-.010	.010	.009
	50	50	500	0	1500	500	-.034	-.016	-.024	-.016	-.012	-.004	.010	.008
	60	60	250	0	2000	1500			-.020	-.014	-.014	-.006	-.008	-.002
	62	64	250	0	3000	1500			-.022	-.016	-.016	-.008	.014	.010
49	15	15	2000	750	750	0	-.016	-.008	.004	.002	-.015	-.009	.016	.012
	23	25	1250	0	1000	250	-.030	-.016	-.018	-.008	-.020	-.014	.028	.012
	30	34	500	0	1500	500	-.038	-.020	-.016	-.012	-.020	-.012	.020	.012
	38	40	500	0	2000	1000			-.014	-.008	-.020	-.008	.022	.016
	43	44	500	0	3000	1500			-.028	-.020	-.026	-.012	.020	.012
157	24	25	2750	750	0	0	-.028	-.016	-.004	-.004	-.018	-.014	.020	.012
	44	46	1750	250	500	0	-.028	-.012	-.010	-.004	-.020	-.012	.010	.004
	49	55	1000	250	1000	250	-.038	-.016	-.020	-.004	-.020	-.010	.016	.008
	55	60	1000	250	1250	500			-.016	-.012	-.020	-.012	.016	.008
	60	62	1000	250	1500	500			-.018	-.008	-.012	-.004	.016	.008

# SERIES C - STABILITY RUNS

Run Number	Speed (mph)	Peak Measurements - Absolute and Level Exceeded for 100 ms											
		L <sub>L</sub> (kips)		L <sub>R</sub> (kips)		a <sub>y</sub> A-1 (g's)		a <sub>y</sub> A-2 (g's)		a <sub>y</sub> A-3 (g's)		Δ <sub>y</sub> L <sub>F</sub> (in)	
		abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms
19C ST35	34	2.2	1.5	1.5	1.2	.03	.12	.04	.20	.05	.016	.012	.008
19C ST55	56	2.5	1.5	1.6	1.0	.05	.25	.06	.36	.04	.016	.008	.012
19C ST65	68	3.4	1.5	1.6	1.0	.05	.30	.07	.44	.06	.016	.008	.012
(rain)													
19C ST75	54	3.0	1.0	1.7	1.0	.04	.28	.06	.38	.08	.014	.006	.016
Reverse Runs													
19C ST35R	38	.5	.5	.5	.3	.16	.04	.06	.13	.03	.06	.008	.008
19C ST55R	56	1.0	.5	1.0	.3	.30	.04	.06	.20	.04	.016	.008	.010
19C ST65R	67	1.0	.5	1.0	.3	.32	.06	.08	.22	.04	.015	.006	.010
19C ST75R	72	1.0	.5	1.0	.3	.32	.06	.07	.22	.06	.016	.006	.020

# SERIES C (BALLASTED) - STABILITY RUNS

Run Number	Speed (mph) min max	Peak Measurements - Absolute and Level Exceeded for 100 ms											
		L <sub>L</sub> (klps)		L <sub>R</sub> (klps)		a <sub>Y</sub> A-1 (g's)		a <sub>Y</sub> A-2 (g's)		a <sub>Y</sub> A-3 (g's)		Δ y <sub>LF</sub> (in)	
		abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms
20C ST35	35	2.5	1.5	1.5	1.0	.14	.02	.12	.02	.18	.03	.014	.008
20C ST55	44	3.0	1.2	2.0	1.0	.21	.03	.18	.03	.35	.04	.014	.006
20C ST65	64	2.7	1.0	2.0	1.0	.20	.05	.20	.04	.32	.05	.014	.006
20C ST75	64	3.0	1.0	2.0	1.0	.34	.05	.20	.04	.40	.06	.014	.006
Reverse Runs													
20C ST35R	37	.75	.5	.75	.5	.12	.04	.28	.04	.12	.02	.014	.004
20C ST55R	54	1.0	.5	1.0	.5	.32	.04	.44	.04	.26	.04	.014	.004
20C ST55R	58	1.5	.5	1.0	.2	.38	.06	.44	.04	.32	.06	.014	.004
20C ST75R	53	2.0	.5	1.0	.5	.34	.08	.44	.06	.34	.08	.014	.006

# SERIES F - STEADY CURVING

Curve Number	SPEED		I <sub>L</sub>		I <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3														
311														
17 FSC(1)	14	18	3000	1700	4200	1000	+ .032	+ .018	+ .024	+ .008	-.144	-.112	+ .180	+ .148
17 FSC(1R)	16	18	1500	500	500	0	-.024	-.010	-.046	-.032	-.104	-.064	+ .128	+ .096
17 FSC(2)	37	38	2000	600	5500	1500	+ .032	+ .016	+ .032	+ .020	-.176	-.120	+ .232	+ .148
43														
49														
157														

# SERIES F - STABILITY RUNS

Run Number	Speed (mph)		L <sub>L</sub> (kips)		L <sub>R</sub> (kips)		Peak Measurements - Absolute and Level Exceeded for 100 ms											
	min	max	abs	100ms	abs	100ms	a <sub>y</sub> A-1 (g's)	abs	100ms	a <sub>y</sub> A-2 (g's)	abs	100ms	a <sub>y</sub> A-3 (g's)	abs	100ms	Δ <sub>y</sub> L <sub>F</sub> (in)	abs	100ms
17F ST35	35	36	1500	500	500	500	.14	.06	.12	.04	.20	.04	.04	.20	.04	.016	.010	.010
17F ST55	34	58	1500	500	500	500	.16	.02	.20	.04	.24	.06	.06	.24	.06	.016	.016	.012
17F ST65	35	68	1500	500	500	500	.30	.04	.20	.08	.26	.08	.08	.26	.08	.020	.014	.012
17F ST75	34	65	1500	500	500	500	.30	.02	.16	.04	.26	.04	.04	.26	.04	.016	.016	.012
<u>Reverse Runs</u>																		
17F ST35R	34	36	1200	500	500	500	.12	.02	.22	.06	.10	.02	.02	.10	.02	.018	.016	.012
17F ST55R	32	55	1000	500	1000	200	.22	.04	.32	.04	.16	.04	.04	.16	.04	.020	.030	.020
17F ST65R	32	75	1200	500	500	200	.20	.06	.32	.06	.20	.04	.04	.20	.04	.020	.028	.020
17F ST75R	34	75	1200	500	1000	200	.24	.06	.26	.06	.20	.04	.04	.20	.04	.020	.020	.016

SERIES G - STEADY CURVING, FORWARD

Curve	SPEED	L <sub>L</sub>	L <sub>R</sub>	ΔY <sub>LF</sub>	ΔY <sub>RR</sub>	ΔX <sub>LF</sub>	ΔX <sub>RF</sub>			
3	22	2000	500	2500	250	+0.020 -0.010	+0.004	-1.70	+0.160	+0.090
	28	1500	500	3000	500	+0.020 +0.008	+0.024	+0.006	-0.176	+0.104
	38	1500	500	4500	1000		+0.020	+0.004	-0.176	+0.172
	48	1250	250	5600	1500	+0.028 +0.012	+0.024	+0.008	-0.172	+0.096
	48	1250	100	7000	1500		+0.032	+0.012	-0.168	+0.176
311	10	2250	750	1500	500	-0.028 -0.022	-0.008	-0.002	+0.185	+0.140
	22	3100	1000	1250	250	-0.032 -0.016	+0.008	0	+0.176	+0.128
	32	3600	1250	1000	0		+0.008	+0.004	+0.192	+0.144
	37	5250	2000	750	0		-0.020	-0.008	+0.124	+0.128
	42	7600	2000	750	0		-0.014	-0.004	+0.180	+0.088
37	8	4000	2000	5000	1250	+0.020 +0.008	+0.020	+0.008	-0.170	+0.195
	22	3000	1250	5500	1500	+0.020 +0.008	+0.020	+0.008	-0.184	+0.208
	32	2500	1000	6250	2000	+0.028 +0.012	+0.028	+0.020	+0.196	+0.168
	36	2500	750	7250	2500		+0.030	+0.016	-0.200	+0.212
	44	2000	500	7400	2500	-0.024 +0.008	+0.042	+0.024	-0.228	+0.212
43	20	2500	1000	2000	0	+0.020 +0.010	+0.012	+0.004	-0.130	+0.150
	37	2100	750	2600	250	+0.020 +0.008	+0.008	0	-0.152	+0.184
	48	1500	250	3000	250	+0.020 +0.010	+0.016	+0.008	-0.172	+0.180
	58	1000	250	4250	500		+0.020	+0.004	-0.172	+0.192
	58	1000	250	4000	500	+0.024 +0.008	+0.020	+0.006	-0.160	+0.164
49	16	3500	2000	3750	1250	+0.016 +0.008	-0.032	-0.004	-0.150	+0.160
	23	3000	1500	4250	1500		+0.032	+0.004	-0.176	+0.176
	33	2750	1000	4500	1500	+0.036 +0.012	+0.040	+0.016	-0.168	+0.164
	38	2250	750	5000	2000		+0.026	+0.008	-0.176	+0.192
	44	2000	500	5250	2000	+0.032 +0.012	+0.048	+0.012	-0.196	+0.168
157	22	200	1250	500	0	+0.014 +0.008	-0.024	-0.008	-0.115	+0.110
	45	2500	750	1600	0		+0.012	0	-0.152	+0.136
	56	1500	750	1750	0	+0.024 +0.010	+0.020	+0.004	-0.132	+0.164
	63	1600	500	2250	500		+0.024	+0.004	-0.128	+0.188
	60	1500	500	2250	750	+0.022 +0.008	+0.026	+0.004	-0.116	+0.204

SERIES G - (BALLASTED) STEADY CURVING, REVERSE

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	25 GSC(1R)	19	20	1250	250	0	-0.16	-0.008	-0.32	-0.016	-0.130	-0.060	+0.140	+0.070
	25 GSC(2R)	28	30	1000	250	0	-0.16	-0.008	-0.32	-0.020	-0.124	-0.056	+0.148	+0.056
	25 GSC(3R)	38	40	1000	0	1500	500	500	-0.38	-0.016	-0.128	-0.056	+0.140	+0.056
	25 GSC(4R)	44	45	750	0	2000	750	750	-0.44	-0.016	-0.124	-0.056	+0.128	+0.040
	25 GSC(5R)	48	50	600	0	3250	1250	1250	-0.56	-0.016	-0.104	-0.040	+0.132	+0.040
311	25 GSC(1R)	10	12	1000	250	250	-0.24	-0.008	+0.24	+0.016	+0.130	+0.090	-0.125	-0.090
	25 GSC(2R)	24	26	1250	500	0	+0.16	+0.004	+0.30	+0.020	+0.104	+0.056	-0.128	-0.080
	25 GSC(3R)	34	36	1750	1000	0	0	0	+0.40	+0.024	+0.128	+0.080	-0.128	-0.088
	25 GSC(4R)	39	42	2250	1250	0	+0.24	+0.012	+0.40	+0.024	+0.112	+0.072	-0.128	-0.088
	25 GSC(5R)	40	44	2500	1500	250	0	0	+0.62	+0.032	+0.125	+0.064	-0.136	-0.088
37	25 GSC(1R)	10	12	1000	0	250	-0.24	-0.012	-0.56	-0.032	-0.120	-0.060	+0.050	+0.110
	25 GSC(2R)	25	28	250	0	1250	500	500	-0.48	-0.028	-0.136	-0.088	+0.148	+0.080
	25 GSC(3R)	34	38	0	0	2000	1000	1000	-0.40	-0.024	-0.144	-0.080	+0.144	+0.080
	25 GSC(4R)	35	46	250	0	4500	2000	2000	-0.42	-0.024	-0.164	-0.080	+0.104	+0.064
	25 GSC(5R)	36	49	500	0	4750	2000	2000	-0.44	-0.012	-0.144	-0.064	+0.168	+0.080
43	25 GSC(1R)	20	20	2000	1000	0	+0.20	+0.008	-0.36	-0.016	-0.080	-0.505	+0.090	+0.060
	25 GSC(2R)	40	40	1500	500	500	0	0	-0.30	-0.012	-0.088	-0.40	+0.104	+0.048
	25 GSC(3R)	50	51	750	250	1250	250	250	-0.26	-0.010	-0.104	-0.048	+0.112	+0.048
	25 GSC(4R)	60	61	500	0	2750	500	500	-0.26	-0.012	-0.096	-0.40	+0.108	+0.048
	25 GSC(5R)	64	67	1000	250	3750	1000	1000	-0.24	-0.008	-0.104	-0.064	+0.092	+0.048
49	25 GSC(1R)	15	17	1250	250	0	-0.12	-0.006	-0.40	-0.032	-0.110	-0.070	+0.110	+0.080
	25 GSC(2R)	24	27	1000	0	750	0	0	-0.50	-0.032	-0.124	-0.072	+0.136	+0.080
	25 GSC(3R)	44	44	500	0	1500	500	500	-0.52	-0.032	-0.152	-0.096	+0.128	+0.056
	25 GSC(4R)	38	40	250	0	1750	750	750	-0.48	-0.028	-0.108	-0.056	+0.136	+0.072
	25 GSC(5R)	46	47	250	0	3250	1500	1500	-0.56	-0.028	-0.120	-0.064	+0.104	+0.056
157	25 GSC(1R)	24	27	2500	1500	0	+0.20	+0.008	+0.24	+0.004	-0.050	-0.030	+0.070	+0.035
	25 GSC(2R)	44	47	2000	1250	600	-0.22	-0.004	-0.32	-0.012	-0.080	-0.040	+0.064	+0.032
	25 GSC(3R)	52	58	2100	750	1000	0	0	-0.32	-0.012	-0.092	-0.048	+0.046	+0.016
	25 GSC(4R)	64	68	1750	500	1750	500	500	-0.38	-0.010	-0.092	-0.056	+0.052	+0.008
	25 GSC(5R)	69	72	1250	500	2000	500	500	-0.34	-0.012	-0.076	-0.048	+0.056	+0.016

SERIES G - ACCELERATION IN CURVES

Run Number	Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
		Max.	Min.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
25-G-TR3P3	3	42	0	2600	750	4000	250	+ .022	+ .012	+ .024	+ .006	-.160	-.070	+ .200	+ .120
25-G-TR3P4	3	46	0	2100	1000	4500	500	+ .026	+ .008	+ .028	+ .008	-.165	-.060	+ .230	+ .150
25-G-TR37P3	37	34	0	3500	1750	4500	1500	+ .022	+ .008	+ .020	+ .008	-.170	-.100	+ .210	+ .170
25-G-TR37P4	37	46	0	4000	1750	5250	2000	+ .026	+ .008	+ .038	+ .012	-.190	-.900	+ .230	+ .200

# SERIES G - STABILITY RUNS

Run Number	Peak Measurements - Absolute and Level Exceeded for 100 ms															
	Speed (mph)		L <sub>L</sub> (kips)		L <sub>R</sub> (kips)		a <sub>y</sub> A-1 (g's)		a <sub>y</sub> A-2 (g's)		a <sub>y</sub> A-3 (g's)		Δ <sub>y</sub> LF (in)		Δ <sub>y</sub> RR (in)	
			abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms
	min	max														
25GST (35)	33	36	1000	600	2000	500	-0.14	-0.04	+0.14	+0.04	+0.12	+0.06			+0.016	+0.012
25GST (55)	55	56	1250	1000	2500	500	-0.30	-0.14	+0.28	+0.08	-0.34	-0.10			+0.016	+0.008
25GST (65)	64	67	1500	750	2750	250	-0.40	-0.16	+0.32	-0.16	-0.42	+0.14			+0.010	+0.008
25GST (75)	75	78	1750	750	3250	250	-0.52	-0.12	+0.40	-0.16	-0.56	-0.18			-0.020	-0.012
25GST (35R)	32	37	1100	750	600	500	+0.12	+0.04	+0.20	+0.04	+0.12	+0.04			+0.018	+0.014
25GST (65R)	66	66	1500	500	600	0	-0.28	+0.08	-0.34	-0.06	-0.28	+0.08			-0.016	+0.008
25GST (75R)	76	78	1500	1000	750	0	-0.36	-0.08	-0.42	-0.10	-0.42	+0.10			+0.280	+0.008

SERIES I - STEADY CURVING, FORWARD

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		Δ Y <sub>LF</sub>		ΔY <sub>RR</sub>		ΔX <sub>LF</sub>		ΔX <sub>RF</sub>	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	31 ISC (1F)	14	22	3250	1750	4250	1250	+ .026 +.014	+ .022 +.012	- .032 -.026	+ .048 +.040			
	31 ISC (2F)	22	23	3000	1500	4250	1000	+ .028 +.016	+ .022 +.012	- .036 -.028	+ .050 +.040			
	31 ISC (3F)	40	46	2600	1500	6000	2000	+ .032 +.020	+ .024 +.012	- .040 -.032	+ .048 +.040			
	31 ISC (4F)	36	48	2750	1250	6250	1750	+ .018 +.012	+ .028 +.012	- .048 -.036	+ .040 +.032			
	31 IRE (F)	47	49	2500	1250	7100	2250	+ .028 +.018	+ .028 +.014	- .044 -.034	+ .038 +.032			
311	31 ISC (1F)	15	17	6400	3500	4000	3000	- .014 -.004	- .020 -.010	+ .036 +.028	- .034 -.028			
	31 ISC (2F)	25	29	7500	3000	3750	2250	- .022 -.008	- .020 -.012	+ .034 +.028	- .040 -.032			
	31 ISC (3F)	38	40	8000	4000	3750	2500	- .022 -.008	- .010 -.008	+ .032 +.024	- .048 -.038			
	31 ISC (4F)	41	43	9000	4000	3000	2000	- .018 -.008	- .020 -.008	+ .032 +.024	- .046 -.040			
	31 IRE (F)	44	45	9250	3750	2750	1750	- .020 -.010	- .018 -.010	+ .024 +.020	+ .052 +.044			
37	31 ISC (1F)	14	19	5000	3250	7250	4000	+ .020 +.012	+ .020 +.008	- .036 -.028	+ .052 +.048			
	31 ISC (2F)	26	27	4400	3000	7000	4000	+ .024 +.016	+ .018 +.008	- .044 -.036	+ .052 +.046			
	31 ISC (3F)	36	40	4000	2750	8000	4500	+ .032 +.024	+ .024 +.012	- .048 -.038	+ .052 +.046			
	31 ISC (4F)	38	42	4000	2750	8500	4000	+ .024 +.016	+ .020 +.008	- .044 -.036	+ .050 +.044			
	31 IRE (F)	37	40	4000	2750	8500	4500	+ .026 +.020	+ .018 +.006	- .050 -.036	+ .044 +.038			
43	31 ISC (1F)	20	22	3500	2000	3500	750	+ .020 +.012	+ .018 +.010	- .028 -.020	+ .044 +.038			
	31 ISC (2F)	40	41	2750	1750	4500	1000	+ .020 +.012	+ .020 +.012	- .038 -.028	+ .040 +.032			
	31 ISC (3F)	53	54	3000	1500	5750	1250	+ .018 +.008	+ .022 +.012	- .036 -.028	+ .040 +.034			
	31 ISC (4F)	64	66	2250	1250	6500	1000	+ .022 +.012	+ .020 +.012	- .040 -.028	+ .042 +.032			
	31 IRE (F)	65	66	2250	1000	6750	1250	+ .026 +.018	+ .020 +.010	- .044 -.028	+ .036 -.028			
49	31 ISC (1F)	14	18	4000	2500	5000	2500	+ .024 +.008	+ .014 +.004	- .032 -.024	+ .044 +.036			
	31 ISC (2F)	34	36	3750	2250	5250	2750	+ .046 +.012	+ .032 +.008	- .040 -.032	+ .040 +.032			
	31 ISC (3F)	38	40	3400	2000	6600	3500	+ .040 +.016	+ .036 +.012	- .044 -.030	+ .044 +.034			
	31 ISC (4F)	43	46	3000	2000	7000	3750	+ .040 +.012	+ .034 +.008	- .048 -.032	+ .048 +.036			
	31 IRE (F)	45	45	3000	2000	7000	3500	+ .048 +.012	+ .032 +.004	- .050 -.036	+ .044 +.032			
157	31 ISC (1F)	26	27	3500	2000	2250	500	+ .020 +.008	+ .018 +.008	- .032 -.020	+ .048 +.036			
	31 ISC (2F)	40	57	3000	1500	2500	500	+ .024 +.008	+ .022 +.008	- .040 -.024	+ .046 +.028			
	31 ISC (3F)	43	62	3400	1750	2000	1000	+ .020 +.012	+ .024 +.008	- .035 -.020	+ .048 +.036			
	31 ISC (4F)	40	62	3600	1750	3250	1000	+ .018 +.006	+ .018 +.006	- .032 -.016	+ .050 +.038			
	31 IRE (F)	42	61	3600	1750	3400	750	+ .026 +.010	+ .016 +.004	- .034 -.018	+ .042 +.032			

LVDT STICKING

# SERIES I - STEADY CURVING, REVERSE

Curve Number	SPEED		L <sub>L</sub>		I <sub>R</sub>		ΔY <sub>LF</sub>		ΔY <sub>RR</sub>		ΔX <sub>LF</sub>		ΔX <sub>RF</sub>		
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	
3	31 ISC (1R)	19	21	1000	0	750	0	-0.22	-0.016	-0.018	-0.008	-0.016	-0.010	+0.014	+0.010
	31 ISC (2R)	30	32	750	0	1250	250	-0.016	-0.014	-0.018	-0.008	-0.016	-0.010	+0.018	+0.008
	31 ISC (3R)	42	44	500	0	2100	1000	-0.010	-0.004	-0.018	-0.008	-0.016	-0.008	+0.014	+0.008
	31 ISC (4R)	47	49	400	0	2400	1250	+0.012	+0.004	-0.022	-0.010	-0.018	-0.010	+0.012	+0.006
	31 IRE (5R)	51	51	250	0	2750	1500	-0.008	.000	-0.024	-0.012	-0.016	-0.010	+0.012	+0.004
311	31 ISC (1R)	15	16	1000	250	750	0	+0.012	+0.004	+0.016	+0.006	+0.024	+0.022	-0.032	-0.030
	31 ISC (2R)	26	27	1500	500	250	0	+0.016	+0.006	+0.018	+0.008	+0.024	+0.022	-0.032	-0.032
	31 ISC (3R)	38	39	2250	1250	600	0	+0.020	+0.008	+0.026	+0.010	+0.028	+0.024	-0.030	-0.024
	31 ISC (4R)	44	45	2500	1500	0	0	+0.020	+0.008	+0.024	+0.012	+0.024	+0.020	-0.032	-0.030
	31 IRE (5R)	42	45	2600	1500	0	0	+0.010	+0.004	+0.022	+0.008	+0.028	+0.024	-0.032	-0.028
37	31 ISC (1R)	14	18	1600	250	750	100	-0.016	-0.012	-0.014	-0.010	-0.020	-0.016	+0.028	+0.024
	31 ISC (2R)	26	27	1000	0	1100	250	-0.014	-0.008	-0.016	-0.008	-0.022	-0.016	+0.024	+0.022
	31 ISC (3R)	38	40	600	0	2250	1250	-0.012	.000	-0.020	-0.010	-0.030	-0.020	+0.020	+0.016
	21 ISC (4R)	40	42	600	0	2100	1500	-0.018	-0.004	-0.030	-0.008	-0.024	-0.018	+0.028	+0.020
	31 IRE (5R)	32	41	900	0	2250	1000	+0.012	+0.004	-0.020	-0.008	-0.026	-0.018	+0.022	+0.014
43	31 ISC (1R)	22	22	1250	250	100	0	-0.024	-0.016	-0.014	-0.008	-0.010	-0.006	+0.014	+0.010
	31 ISC (2R)	42	42	500	0	750	250	-0.020	-0.012	-0.020	-0.012	-0.008	-0.004	+0.012	+0.010
	31 ISC (3R)	53	53	400	0	1600	750	-0.008	-0.002	-0.016	-0.010	-0.012	-0.004	+0.010	+0.006
	31 ISC (4R)	52	53	500	0	1500	750	-0.008	.000	-0.018	-0.012	-0.008	-0.004	+0.018	+0.012
	31 IRE (5R)	53	58	250	0	1500	750	-0.008	-0.004	-0.020	-0.012	-0.006	.000	+0.016	+0.008
49	31 ISC (1R)	14	16	2100	1000	0	0	-0.032	-0.020	-0.032	-0.012	-0.024	-0.016	+0.038	+0.032
	31 ISC (2R)	24	26	1500	500	500	0	-0.024	-0.004	-0.024	.000	-0.020	-0.012	+0.024	+0.022
	31 ISC (3R)	38	38	1000	0	1400	500	-0.014	-0.004	-0.018	-0.004	-0.020	-0.016	+0.022	+0.018
	31 ISC (4R)	42	43	750	0	2000	750	-0.026	-0.004	-0.034	-0.008	-0.022	-0.014	+0.028	+0.022
	31 IRE (5R)	44	46	600	0	2250	1250	-0.026	-0.004	-0.032	-0.006	-0.020	-0.014	+0.024	+0.016
157	31 ISC (1R)	27	29	2250	1000	250	0	-0.022	-0.012	-0.016	-0.008	-0.018	-0.008	+0.030	+0.020
	31 ISC (2R)	46	48	2000	500	500	0	-0.020	-0.008	-0.002	.000	-0.018	-0.008	+0.022	+0.012
	31 ISC (3R)	52	61	2100	500	1500	500	-0.018	-0.006	-0.024	-0.008	-0.016	-0.004	+0.024	+0.012
	31 ISC (4R)	60	65	2100	500	1750	750	-0.012	-0.004	-0.018	-0.006	-0.018	-0.008	+0.024	+0.012
	31 IRE (5R)	62	65	1750	500	1500	750	-0.010	-0.004	-0.018	-0.004	-0.012	-0.004	+0.024	+0.012
LVDT STICKING															

LVDT STICKING

# SERIES I - ACCELERATION IN CURVES

Run Number	Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		N <sub>LF</sub>		Δ Y <sub>RR</sub>		Δ X <sub>LF</sub>		Δ Y <sub>RF</sub>	
		Max.	Min.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
31-I-TR3P3	3	44	0	3750	2500	5600	2000	+016	+008	+022	+008	-030	-020	+042	+036
31-I-TR3P4	3	52	0	4250	2500	6000	2500	+010	+004	+020	+008	-030	.016	+044	+036
31-I-TR37P3	37	28	0	5250	4500	8250	4500	+026	+008	+014	+004	-030	-020	+038	+034
31-I-TR37P4	37	42	0	5000	4500	8100	4500	+008	+004	+016	+004	-044	-020	+042	+036

# SERIES I - STABILITY RUNS

Run Number	Speed (mph)		Peak Measurements - Absolute and Level Exceeded for 100 ms											
			L <sub>L</sub> (kips)		L <sub>R</sub> (kips)		a <sub>Y</sub> A-1 (g's)		a <sub>Y</sub> A-2 (g's)		a <sub>Y</sub> A-3 (g's)		Δ <sub>Y</sub> LF (in)	
	min	max	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms
31IST(35F)	36	38	1500	500	2250	500	+18	+04	+10	+04	+14	+04	+024	+016
31IST(55)	57	58	1600	1000	3250	250	+28	+06	+26	+04	+26	+04	+016	+008
31IST(65)	64	68	1750	1000	3100	250	.30	.06	.28	.06	.30	.08	.016	.008
31IST(75)	60	75	1900	750	3250	250	.32	.08	.32	.06	.32	.08	.020	.012
31IST(35R)	36	38	1750	1000	500	0	+12	+02	+16	+02	+14	+04	+018	+012
31IST(55R)	56	60	2000	1100	500	0	+20	-.04	.22	+02	.18	.06	+022	+008
31IST(65R)	58	70	2000	1000	750	0	.22	.04	.26	.06	.20	.04	.020	.008
31IST(75R)	63	74	1900	1000	750	0	.22	.04	.26	.04	.22	.04	.022	.008

SERIES J - STEADY CURVING, FORWARD

Curve Number	SPEED		L <sub>T</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	7 JSC (1)	20	3000	1250	3750	1750			-.028	-.010	-.180	-.080	+ .190	+ .090
	7 JSC (2)	30	2600	1000	4500	2000			-.024	-.008	-.185	-.080	+ .180	+ .060
	7 JSC (3)	41	2600	750	5250	2750			+ .022	.000	-.200	-.080	+ .175	+ .070
	7 JSC (4)	43	2250	1000	6250	3000			+ .036	+ .008	-.215	-.060	+ .195	+ .080
	7 JRE (F)	44	2000	500	5500	2750			+ .026	+ .006	-.155	-.050	+ .190	+ .080
311	7 JSC (1)	12	5500	3500	3750	2500			-.024	-.008	+ .170	+ .080	-.125	-.050
	7 JSC (2)	22	6000	4000	3250	2000			-.020	-.008	+ .145	+ .060	-.165	-.080
	7 JSC (3)	32	6500	4500	3000	1750			-.036	-.012	+ .155	+ .080	-.155	-.070
	7 JSC (4)	37	7400	4500	2600	1500			-.028	-.012	+ .120	+ .045	-.125	-.050
	7 JRE (F)	36	8250	5000	2750	1750			-.042	-.018	+ .165	+ .070	-.165	-.070
37	7 JSC (1)	12	3000	1400	5250	2500			-.024	-.008	-.170	-.090	+ .210	+ .140
	7 JSC (2)	23	3250	1500	6100	3250			+ .016	.000	-.175	-.100	+ .180	+ .100
	7 JSC (3)	31	2750	1000	6500	3500			+ .026	+ .004	-.170	-.080	+ .195	+ .120
	7 JSC (4)	12	3250	1500	7500	3250			+ .022	+ .006	-.230	-.090	+ .205	+ .130
	7 JRE (F)	35	2600	1000	8000	3750			+ .022	+ .006	-.190	-.090	+ .190	+ .120
43	7 JSC (1)	14	3750	2250	4000	2000			-.034	-.018	-.130	-.040	+ .165	+ .060
	7 JSC (2)	37	3000	1700	5500	2250			+ .014	+ .002	-.135	-.050	+ .145	+ .025
	7 JSC (3)	48	3000	1500	6000	2500			-.024	-.008	-.120	-.040	+ .145	+ .040
	7 JSC (4)	53	2750	1250	6600	2500			-.020	-.006	-.150	-.050	+ .210	+ .050
	7 JRE (F)	60	2250	1000	7750	3000			+ .044	+ .012	-.150	-.030	+ .190	+ .030
49	7 JSC (1)	12	3500	1250	5000	1750			-.028	-.010	-.160	-.110	+ .190	+ .120
	7 JSC (2)	22	3750	1250	6000	2250			-.032	-.006	-.180	-.110	+ .160	+ .090
	7 JSC (3)	44	3000	750	5750	2750			+ .026	+ .004	-.190	-.100	+ .180	+ .100
	7 JSC (4)	36	2900	1000	6100	3250			+ .022	+ .004	-.190	-.120	+ .210	+ .130
	7 JRE (F)	40	2750	750	7000	3500			+ .032	+ .008	-.200	-.100	+ .175	+ .100
157	7 JSC (1)	22	3500	2000	2750	1250			-.040	-.012	-.110	-.040	+ .080	+ .010
	7 JSC (2)	44	2750	1750	4000	1500			-.024	-.008	-.105	-.040	+ .075	.000
	7 JSC (3)	54	2500	1250	4000	1500			-.022	-.008	-.095	-.020	+ .080	+ .010
	7 JSC (4)	40	3000	1500	4500	2000			-.028	-.010	-.085	-.010	+ .140	+ .060
	7 JRE (F)	39	2750	1500	4000	1750			-.026	-.010	-.095	-.010	+ .110	+ .030

# SERIES J - STEADY CURVING, REVERSE

Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta X_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
3	17	19	2100	1000	1500	250			+0.42	+0.010	-0.150	-0.080	+0.175	+0.100
	27	28	2000	750	2000	5000			+0.46	+0.018	-0.130	-0.065	+0.165	-0.080
	38	38	1500	500	2500	1000			+0.52	+0.024	-0.145	-0.070	+0.150	+0.080
	44	44	1500	250	3600	1500			+0.46	+0.012	-0.140	-0.055	+0.125	+0.055
	48	49	1000	250	4400	1500			+0.48	+0.016	-0.160	-0.060	+0.100	+0.040
311	9	15	750	0	750	0			-0.28	-0.012	+0.225	+0.170	-0.200	-0.155
	24	24	1100	0	750	0			-0.44	-0.020	+0.220	+0.180	-0.195	-0.160
	34	36	2000	500	250	0			-0.44	-0.020	+0.220	+0.170	-0.210	-0.170
	39	41	2400	1000	200	0			-0.58	-0.032	+0.210	+0.160	-0.210	-0.160
	28	30	1250	250	600	0			-0.44	-0.020	+0.220	+0.170	-0.195	-0.160
37	14	16	2000	500	1750	250			+0.36	+0.012	-0.160	-0.100	+0.180	+0.120
	25	30	1500	250	2500	750			+0.46	+0.016	-0.160	-0.090	+0.195	+0.130
	32	39	1000	0	4200	1500			+0.60	+0.028	-0.160	-0.080	+0.185	+0.110
	30	37	1700	250	3900	1250			+0.52	+0.012	-0.190	-0.100	+0.105	+0.100
	36	37	1200	0	4000	1500			+0.50	+0.020	-0.190	-0.090	+0.165	+0.100
43	18	20	2750	1500	1000	0			+0.24	+0.006	-0.085	-0.055	+0.120	+0.050
	39	39	2500	1000	2750	500			+0.32	+0.010	-0.095	-0.045	+0.120	+0.050
	48	48	2250	1000	3000	1000			+0.32	+0.006	-0.105	-0.030	+0.110	+0.040
	44	46	2400	1000	2500	1000			+0.24	+0.008	-0.100	-0.045	+0.110	+0.035
	49	49	2500	750	3000	1000			+0.52	+0.008	-0.085	-0.035	+0.090	+0.040
49	13	15	2000	750	1500	0			+0.32	+0.008	-0.160	-0.100	+0.160	+0.110
	22	2	2000	500	2500	250			+0.36	+0.012	-0.170	-0.100	+0.175	+0.115
	32	36	1100	0	3200	750			+0.36	+0.012	-0.160	-0.090	+0.150	+0.090
	38	39	1000	0	3750	1250			+0.42	+0.012	-0.190	-0.100	+0.155	+0.090
	40	42	1000	0	4250	1250			+0.52	+0.032	-0.195	-0.090	+0.140	+0.070
157	24	28	2500	1250	1000	0			+0.24	+0.004	-0.070	-0.140	+0.060	+0.025
	46	46	2500	1000	1750	250			+0.46	+0.012	-0.060	-0.030	+0.045	+0.015
	48	56	2100	1000	2100	500			+0.44	+0.016	-0.050	-0.010	+0.070	+0.030
	58	61	2000	1000	3250	1000			+0.40	+0.008	-0.080	-0.040	+0.040	+0.010
	46	48	1500	750	3100	1500			+0.40	+0.016	-0.110	-0.040	+0.070	+0.025

SERIES J - STEADY CURVING AT AN UNWORN BLUELINE CURVE

Curve Number	SPEED		I <sub>L</sub>		I <sub>R</sub>		$\Delta Y_{LF}$		$\Delta Y_{RR}$		$\Delta Y_{LF}$		$\Delta X_{RF}$	
	Min.	Max.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
2 JCA(6A)1	35	37	2750	1250	7250	4000	-.016	-.004	-.022	-.006	-.153	-.080	+.145	+.090
2 JCA(6B)2	35	37	2750	1250	7500	4000	-.022	-.004	-.022	-.004	-.165	-.085	+.160	+.100
2 JCA(6C)3	35	38	2750	1250	7600	4000	-.022	-.006	*	*	-.150	-.080	+.165	+.100
2 JCA(6D)4	36	38	2750	1250	7000	4250	-.022	-.006			.160	-.090	+.165	+.090
5														

\*Not valid. The LVDT was loose, lost completely during next run.

# SERIES J - ACCELERATION IN CURVES

Run Number	Curve Number	SPEED		L <sub>L</sub>		L <sub>R</sub>		ΔY <sub>LF</sub>		ΔY <sub>RR</sub>		ΔX <sub>LF</sub>		ΔX <sub>RF</sub>	
		Max.	Min.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.	Peak	S.S.
7-J-TR3P3	3	40	0	3500	1500	5000	2500			-.024	-.006	-.140	-.040	+.195	+.100
7-J-TR3P4	3	48	0	3750	1500	6000	2500			-.024	-.008	-.080	-.020	+.220	+.130
7-J-TR37P3	37	32	0	3250	1500	5750	3000			-.020	-.004	-.160	-.070	+.220	+.140
7-J-TR37P4	37	40	0	4000	1500	6500	3000			-.024	-.004	-.200	-.040	+.225	+.170

# SERIES J - STABILITY RUNS

Run Number	Speed (mph)		Peak Measurements - Absolute and Level Exceeded for 100 ms															
			L <sub>L</sub> (klps)		L <sub>R</sub> (klps)		a <sub>y</sub> A-1 (g's)		a <sub>y</sub> A-2 (g's)		a <sub>y</sub> A-3 (g's)		Δ <sub>y</sub> LF (in)		Δ <sub>y</sub> RR (in)			
	min	max	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms	abs	100ms		
7-JST(35F)	35	37	3000	1750	4500	2000	.20	.08	.16	.02	.22	.06	+.016	+.012	+.016	+.012		
7-JST(55F)	48	58	2900	1250	4000	1750	.36	.04	.24	.02	.32	.04	+.014	+.008	+.020	+.012		
7-JST(65F)	52	59	3250	1250	5500	2000	.40	.12	.30	.04	.54	.06	+.016	+.008	+.024	+.010		
7-JST(75F)	64	75	4200	1250	5400	2000	.50	.04	.38	.04	.52	.08			+.022	+.010		
7-JST(35R)	34	38	1600	600	750	0	.16	0	.32	.04	.18	0	-.020	-.016	-.018	-.012		
7-JST(55R)	56	58	1500	500	1000	0	.42	.04	.56	.06	.34	.04	-.036	-.030	-.024	-.020		
7-JST(65R)	60	68	1250	500	1250	0	.52	.04	.62	.06	.34	.06	-.014	-.008	-.018	-.014		
7-JST(75R)	70	75	1500	500	750	0	.54	.04	.62	.08	.36	.04			-.022	-.012		

## APPENDIX C    REPORT OF NEW TECHNOLOGY

This report describes analyses and tests of wheel taper and primary suspension stiffness on wheel-rail interaction forces. The work described has not resulted in the development of any new or unique devices.





