

Baseline Fuel Economy and Emissions
Tests of a Chrysler 1978, 225 CID Engine

(U.S.) Transportation Systems Center
Cambridge, MA

Prepared for

National Highway Traffic Safety Administration
Washington, DC

Sep 80

REPORT NO. DOT-TSC-NHTSA-80-15

DOT-HS-805 488

BASELINE
FUEL ECONOMY AND EMISSIONS TESTS
OF A
CHRYSLER 1978, 225CID ENGINE

G. GAGNE, M. BELL, AND R. WALTER

U.S. DEPARTMENT OF TRANSPORTATION
RESEARCH AND SPECIAL PROGRAMS ADMINISTRATION
TRANSPORTATION SYSTEMS CENTER
CAMBRIDGE MA 02142



FINAL REPORT
SEPTEMBER 1980

DOCUMENT IS AVAILABLE TO THE U.S. PUBLIC
THROUGH THE NATIONAL TECHNICAL
INFORMATION SERVICE, SPRINGFIELD
VIRGINIA 22161

U.S. DEPARTMENT OF TRANSPORTATION
NATIONAL HIGHWAY TRAFFIC SAFETY ADMINISTRATION
OFFICE OF RESEARCH AND DEVELOPMENT
WASHINGTON DC 20590

Technical Report Documentation Page

1. Report No. DOT-HS-805 488	2. Government Accession No.	3. Recipient's Catalog No.	
4. Title and Sublice Baseline Fuel Economy and Emissions Tests of a Chrysler 1978, 225 CID Engine		5. Report Date September 1980	
6. Author(s) C.Gagne, M.Bell, and R.Walter		7. Performing Organization Code DOT/TSC	
8. Performing Organization Name and Address U.S. Department of Transportation Research and Special Programs Administration Transportation Systems Center Cambridge MA 02142		9. Work Unit No. (TRAIS)	
10. Contract or Grant No. HS055/R0420		11. Type of Report and Period Covered Final Report	
12. Sponsoring Agency Name and Address U.S. Department of Transportation National Highway Traffic Safety Administration Office of Research and Development Washington DC 20590		13. Sponsoring Agency Code DOT/NHTSA	
14. Supplementary Notes			
15. Abstract			
16. Key Words engine mapping emissions negative torques EGR			
17. Distribution Statement DOCUMENT IS AVAILABLE TO THE U.S. PUBLIC THROUGH THE NATIONAL TECHNICAL INFORMATION SERVICE, SPRINGFIELD, VIRGINIA 22161			
18. Security Classif. (of this report) Unclassified	19. Security Classif. (of this page) Unclassified	20. No. of Pages 46	21. Price

PREFACE

This report presents the baseline, steady-state test results of a 1978 Chrysler 225 CID engine. This work was performed in the Automotive Research Laboratory at the Transportation Systems Center of the U.S. Department of Transportation. This work is in support of the Automotive Fuel Economy Research and Analysis Support Program for NHTSA's Technology Assessment Division, Office of Passenger Vehicle Research. The authors gratefully acknowledge the technical support of Ralph Colello, Dr. Thomas Trella, Russell Zub, and Norman Deserres.

Metric Conversion Factors

Approximate Conversion from Metric Measures													
From	To	Volume	Weight	Length	Area	Mass	Volume	Weight	Length	Area	Mass	Volume	Weight
Volume	Volume	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Weight	Weight	1 kg = 1000 g	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Length	Length	1 m = 100 cm	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Area	Area	1 ha = 10,000 m ²	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Mass	Mass	1 kg = 1000 g	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Temperature	Temperature	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F
Volume	Volume	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Weight	Weight	1 kg = 1000 g	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Length	Length	1 m = 100 cm	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Area	Area	1 ha = 10,000 m ²	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Mass	Mass	1 kg = 1000 g	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g	1 m = 100 cm	1 ha = 10,000 m ²	1 kg = 1000 g	1 L = 1000 mL	1 kg = 1000 g
Temperature	Temperature	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F	1°C = 33.8°F

TABLE OF CONTENTS

<u>Section</u>		<u>Page</u>
1. INTRODUCTION.....	1	
2. EXPERIMENTAL DESIGN.....	2	
2.1 Engine.....	2	
2.2 Test Configuration.....	3	
2.2.1 Instrumentation.....	7	
2.2.2 Data Acquisition.....	7	
2.3 Baseline Test Matrix.....	7	
3. TEST PROCEDURES.....	12	
4. DATA REDUCTION.....	13	
5. TEST RESULTS.....	14	
APPENDIX A - DATA REDUCTION EQUATIONS.....	A-1	
APPENDIX B - CHRYSLER 225 CID TEST DATA AND REDUCED TEST DATA.....	B-1	

LIST OF ILLUSTRATIONS

<u>Figure</u>		<u>Page</u>
1. ELECTRONIC FLOW DIAGRAM.....	4	
2. CELL NO. 2 TEST ENGINE FLUID CONTROL.....	5	
3. TEST CELL NO. 2 LAYOUT.....	6	
4. DISTRIBUTOR ADVANCE CHARACTERISTICS.....	9	
5. MEASURED VACUUM ACTUATOR.....	10	
6. CORRECTED BHP, TORQUE, AND BSFC VS WIDE-OPEN THROTTLE.....	15	
7. FUEL CONSUMPTION VS POWER.....	16	
8. AIR-FUEL RATIO VS POWER.....	17	
9. EGR VS POWER.....	18	

LIST OF ILLUSTRATIONS (CONTINUED)

<u>Figure</u>		<u>Page</u>
10.	CO VS POWER.....	19
11.	HC VS POWER.....	20
12.	NO _x VS POWER.....	21

LIST OF TABLES

<u>Table</u>		<u>Page</u>
1.	ENGINE CHARACTERISTICS.....	2
2.	CHRYSLER 225 CID ENGINE BREAK-IN SCHEDULE.....	3
3.	FUEL SPECIFICATIONS.....	3
4.	ANALOG INSTRUMENTATION.....	8
5.	STEADY-STATE BASELINE MAPPING SPEED-TORQUE VALUES..	11
B-1.	CHRYSLER 225 CID TEST DATA.....	B-2
B-2.	CHRYSLER 225 CID REDUCED TEST DATA.....	B-13

1. INTRODUCTION

The work reported details the results of baseline, steady-state testing of a 1978 Chrysler, 6-cylinder, 225 CID engine. These tests measured the fuel rate and gaseous emissions while the engine was operated on a dynamometer at 57 speed/load points.

The objectives of this engine test are as follows:

- o Characterize the fuel economy and emissions of present and prototype automotive engines;
- o Provide an independent data base for vehicle performance modeling and fuel economy assessment.

2. EXPERIMENTAL DESIGN

This section briefly reviews the test design and highlights salient features of the Chrysler 225 CID baseline testing.

2.1 ENGINE

The manufacturers specifications for the Chrysler 225 CID engine are given in Table 1.

TABLE 1. ENGINE CHARACTERISTICS

Model Year	1978
Manufacturer	Chrysler Corp.
No. of cylinders	6
Displacement	225 CID (3.7L)
Bore	3.40 inches (86 mm)
Stroke	4.125 inches (105 mm)
Compression ratio	8.4 to 1
Max. Rated HP	109 @ 3600 rpm
Max. Rated Torque	182 @ 2000 rpm
Calibration	49 State, automatic
Transmission type	automatic
Engine weight*	564 lbs (256 kg)

This mean-tolerance engine was broken-in with the test schedule shown in Table 2. The engine came equipped with an oxidation catalyst and Exhaust Gas Recirculation (EGR) for emissions control. The engine was equipped with a manual transmission flywheel for mounting to the dynamometer.

*Includes starter, alternator, fan, manual transmission flywheel, bell housing, wire harness, battery cables, vacuum lines, air inlet system. Does not include radiator, water hoses, exhaust pipes, muffler, engine coolant, oil.

TABLE 2. CHRYSLER 225 CID ENGINE BREAK-IN SCHEDULE

<u>Period (hr)</u>	<u>Speed (rpm)</u>	<u>Torque (lb-ft)</u>
1	1200	64.0
1	1600	94.5
1	2000	108.5
2	2400	122.5
2	2800	133.0
2	3200	136.5
2	3600	138.3(1)
2	4000	WOT (1)
1/2	4400	WOT (2)

(1) Cycle 4 min. at load, 1 min. at 1600 rpm no load

(2) Check wide-open throttle (WOT) friction from 4400 rpm down

A single batch of unleaded Indolene test fuel was used for engine break-in and testing. The gasoline specifications are shown in Table 3.

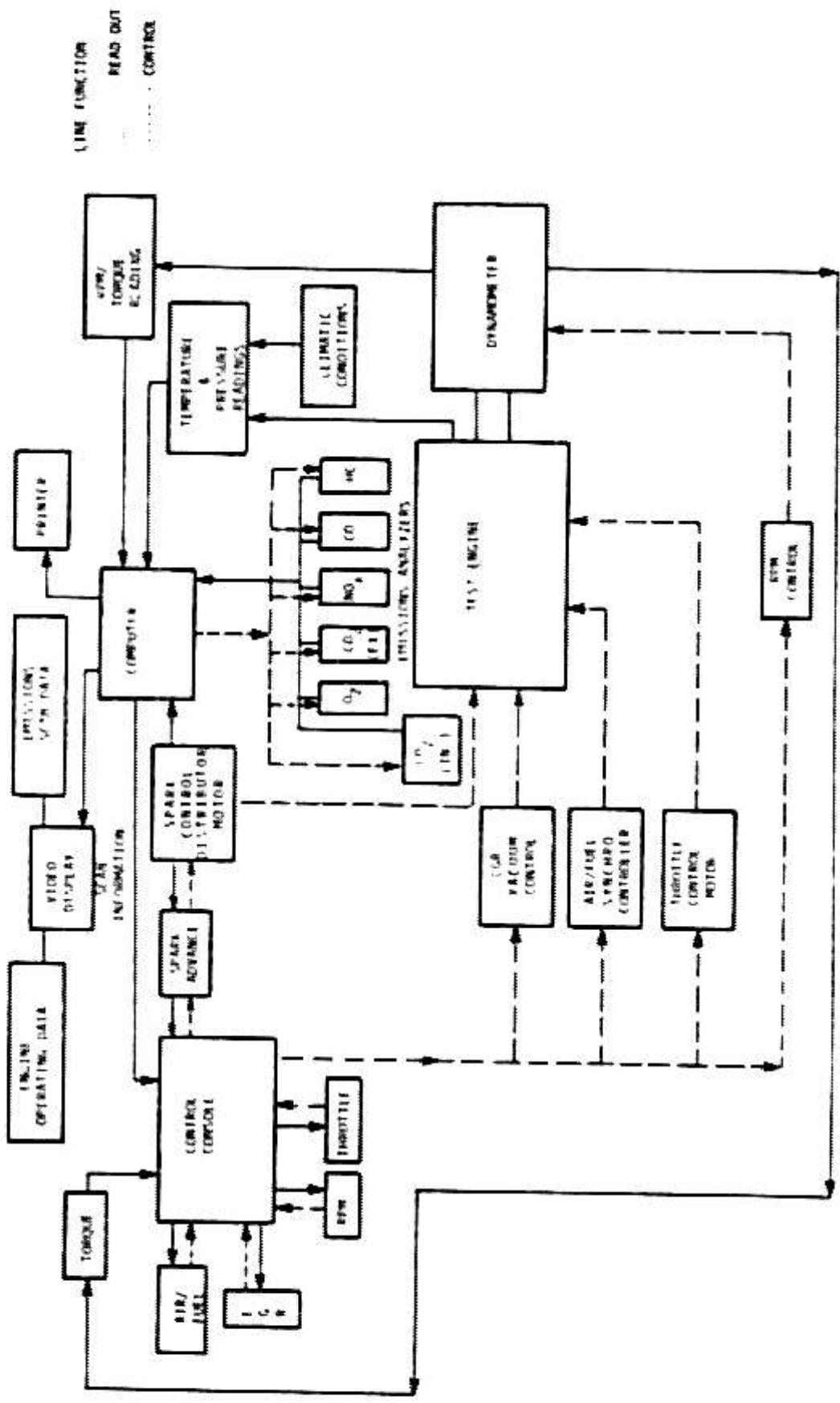
TABLE 3. FUEL SPECIFICATIONS

Amoco Unleaded Indolene
 Specific Gravity ... 0.7455@60°F
 Percent Carbon... 83.39
 Percent Hydrogen... 12.93
 H/C Atomic ratio... 1.85
 Upper heating value... 19,733 BTU/lb

2.2 TEST CONFIGURATION

The engine was installed in the Automotive Research Laboratory test cell No. 2. This cell has a DC, programmable dynamometer for power absorption and motoring tests. Figures 1 and 2 show the electronic and fluid - flow configurations for the engine mounted in the cell for testing. Figure 3 is an illustration of the engine installed in the cell.

FIGURE 1. ELECTRONIC FLOW DIAGRAM



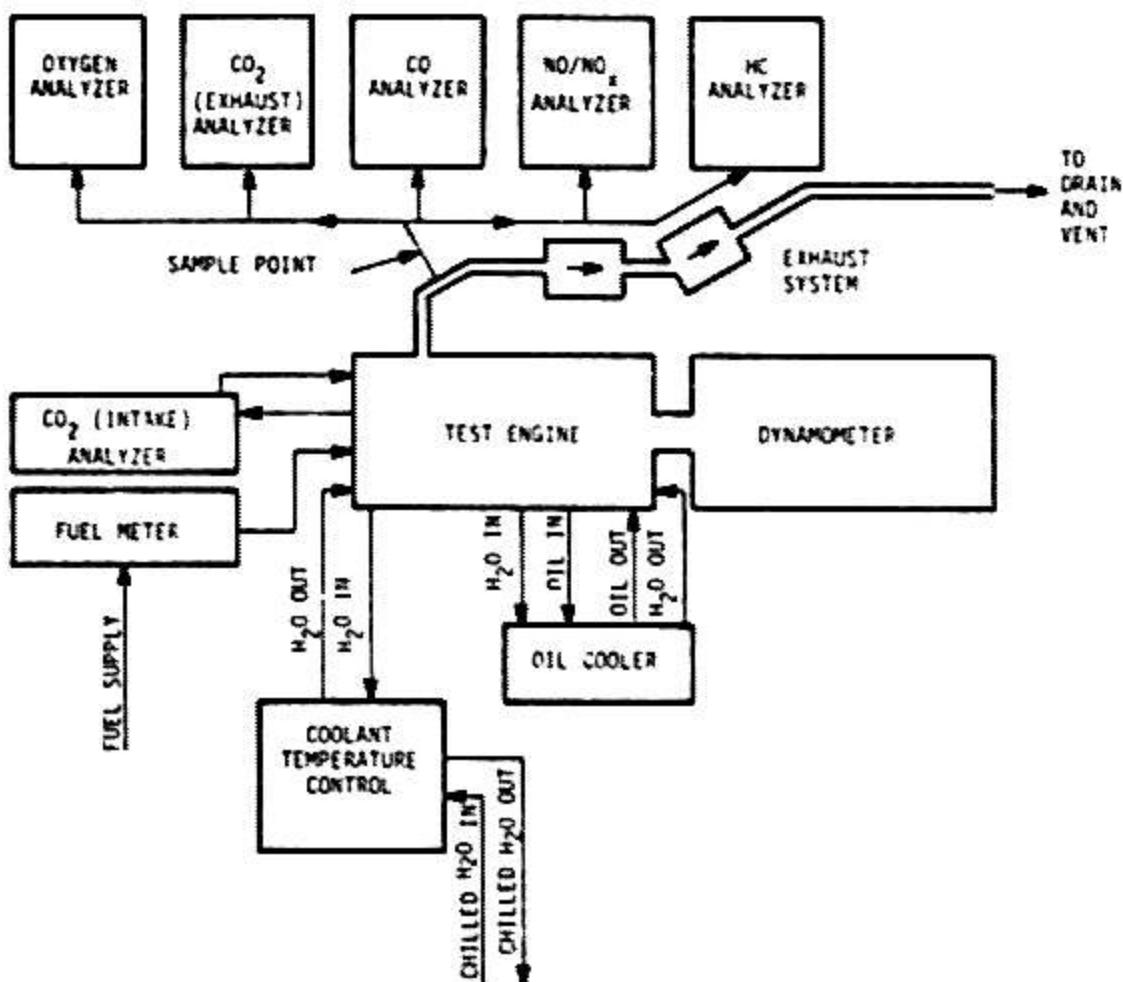


FIGURE 2. CELL NO. 2 TEST ENGINE FLUID FLOW

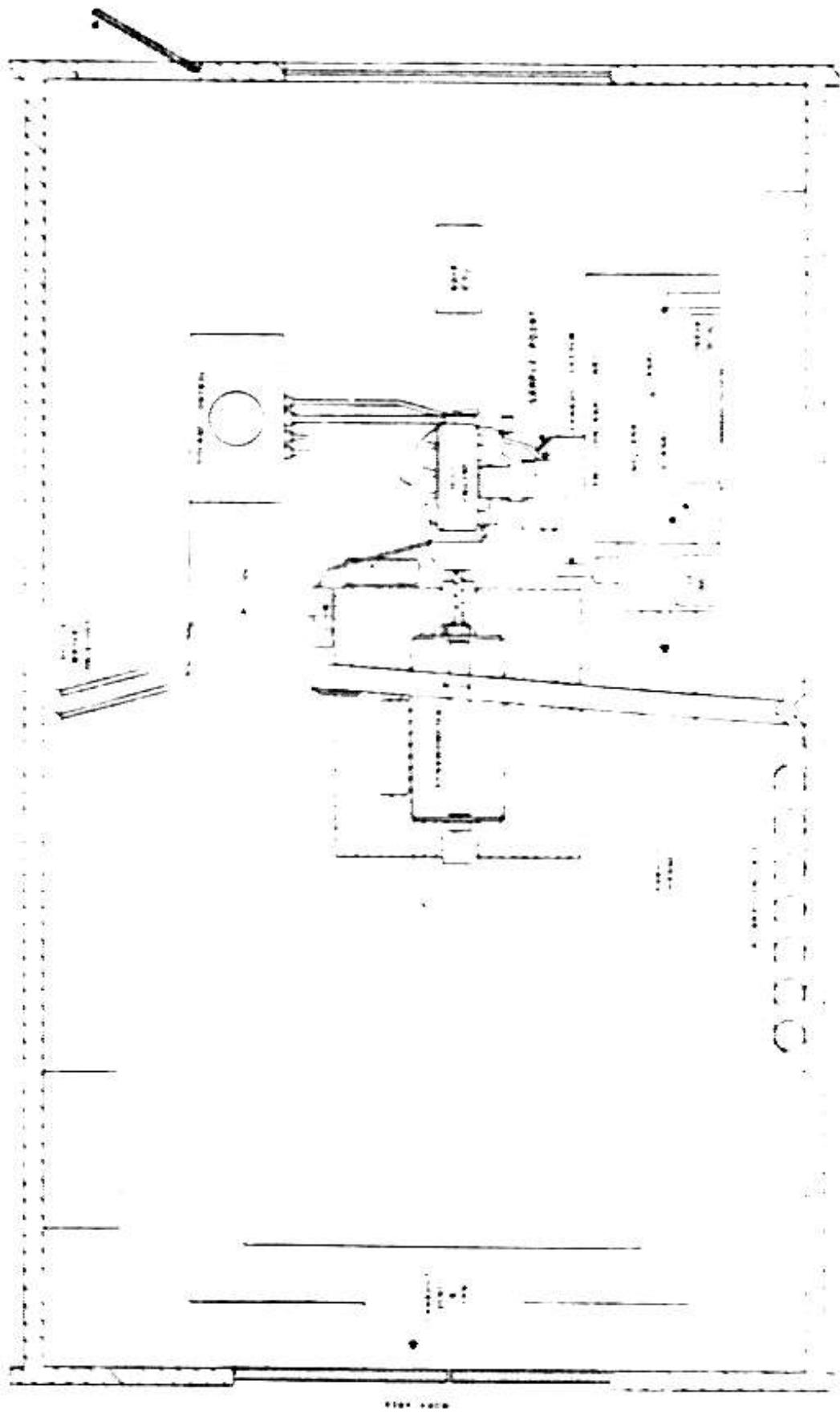


FIGURE 3. TEST CELL NO. 2 LAYOUT

Instead of the standard air cleaner, air induction was directed through a laminar-flow element connected by 4-inch ID plastic tubing to the carburetor. All engine vents (valve cover, carburetor, charcoal canister) were connected to the air-inlet system. Engine coolant and oil temperatures were maintained at $185^{\circ}\text{F} \pm 2^{\circ}\text{F}$ by external heat exchangers. Although all emissions measurements were made before the catalyst, it, as well as a muffler, were installed in the exhaust system in order to duplicate the normal engine back-pressure.

2.2.1 Instrumentation

Real-time measurements of emissions and engine performance were accomplished by the analog instrumentation shown in Table 4.

2.2.2 Data Acquisition

The data acquisition system was Hewlett-Packard 21MX mini-computer with a 45kHz A/D converter and multiplexer. Analog signals from the instrumentation shown in Table 4 were routed, along with digital speed and torque signals, to this computer. The data was continuously updated on a video display and line printer. Data was stored on a disc for subsequent reduction.

2.3 BASELINE TEST MATRIX

The baseline data for this engine was collected by operating the engine in the steady-state mode with the factory calibrations of air-to-fuel ratio (A-F), spark advance (SA), and exhaust gas recirculation (EGR). The spark timing, distributor advance characteristics are shown in Figure 4. Figure 5 gives the EGR valve displacement versus actuation pressure. During these baseline tests the A-F varied between 13.2 and 19.8, spark advance from 10° to 24° BTDC, and EGR from 0.6t to 16.8t.

The test matrix consisted of 57 speed/load points from 750 rpm to 3600 rpm, as shown in Table 5. In general, these points correspond to 0%, 25%, 50%, 75%, and 100% of maximum positive torque and 25% and 75% of maximum negative torque. (Negative torque is obtained by motoring the powered engine with the dynamometer, thus simulating deceleration.)

TABLE 4. ANALOG INSTRUMENTATION

MEASURED VARIABLE	INSTRUMENT TYPE
Oil temp.	OMEGA type K thermocouple; CJ compensator
Ambient temp.	YSI Model 46T6 Tele-thermometer
Inlet air flow	MERRIAM 50MC2-4 Laminar flow element
Inlet air diff. press.	MKS Baratron type 77
Air inlet temp.	OMEGA type K thermocouple; CJ compensator
Fuel inlet temp.	OMEGA type K thermocouple; CJ compensator
Coolant inlet temp.	OMEGA type K thermocouple; CJ compensator
Relative/Humidity	WEATHERMEASURE Model HM111
Exhaust CO	BECKMAN 864 Infrared Analyzer
Exhaust CO ₂	BECKMAN 864 Infrared Analyzer
Exhaust HC	BECKMAN 402 Hydrocarbon Analyzer
Exhaust NOx	BECKMAN 951 Chemiluminescent Analyzer
Exhaust O ₂	BECKMAN 14330 Paramagnetic Analyzer
Intake CO ₂	BECKMAN 864 Infrared Analyzer
Manifold vacuum	TYCO type AB 15 PSI transducer
Coolant exhaust temp.	OMEGA type K thermocouple; CJ compensator
Spark timing	1.8K precision potentiometer
Peak cylinder press.	KISTLER 538A/601B1 Piezoelectric
Exhaust temp. AC	OMEGA type K thermocouple; CJ compensator
Exhaust temp. BC	OMEGA type K thermocouple; CJ compensator
Exhaust press. AC	TYCO type AB 6 PSI transducer
Exhaust press. BC	TYCO type AB 6 PSI transducer

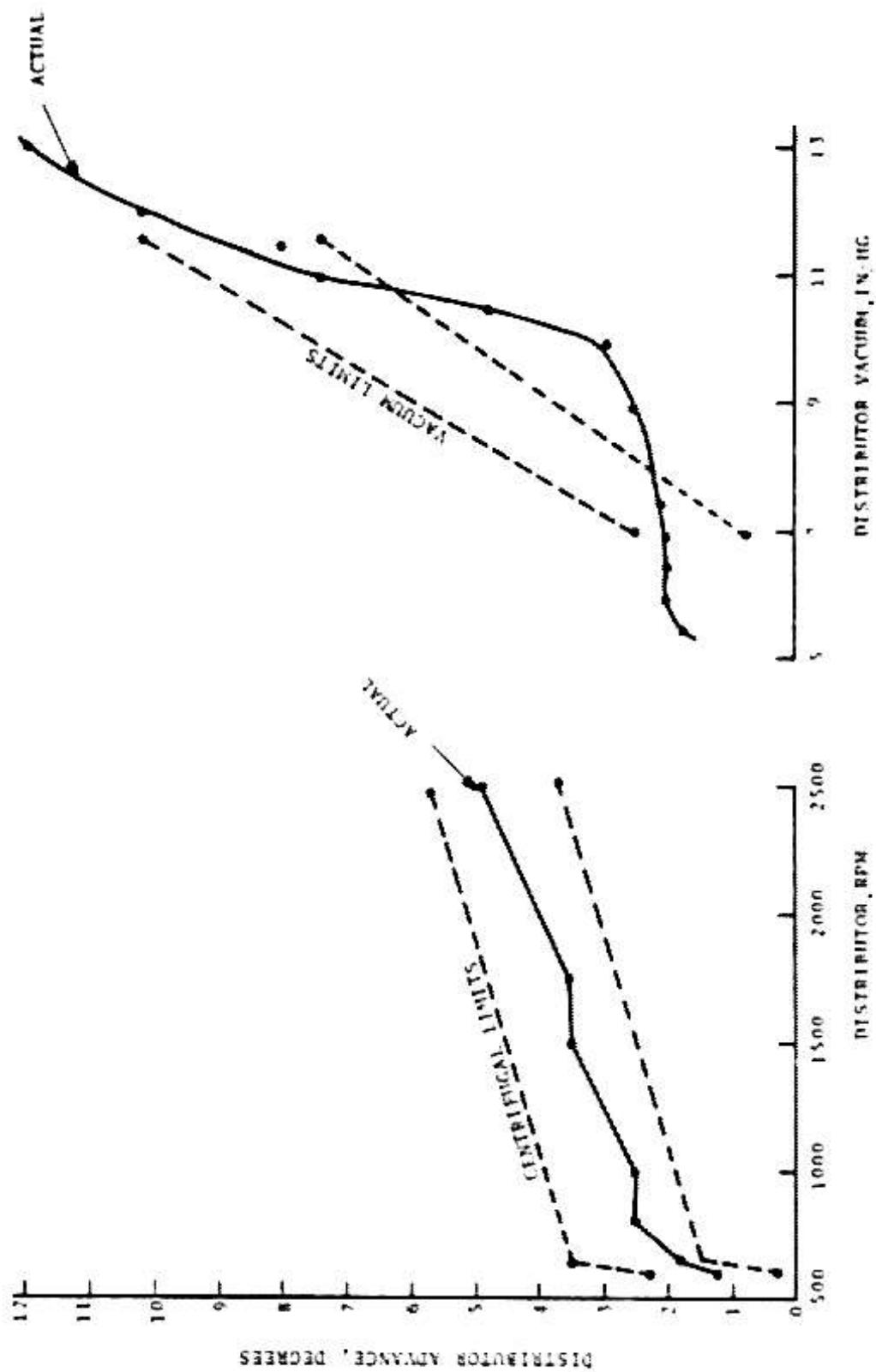


FIGURE 4. DISTRIBUTOR ADVANCE CHARACTERISTICS

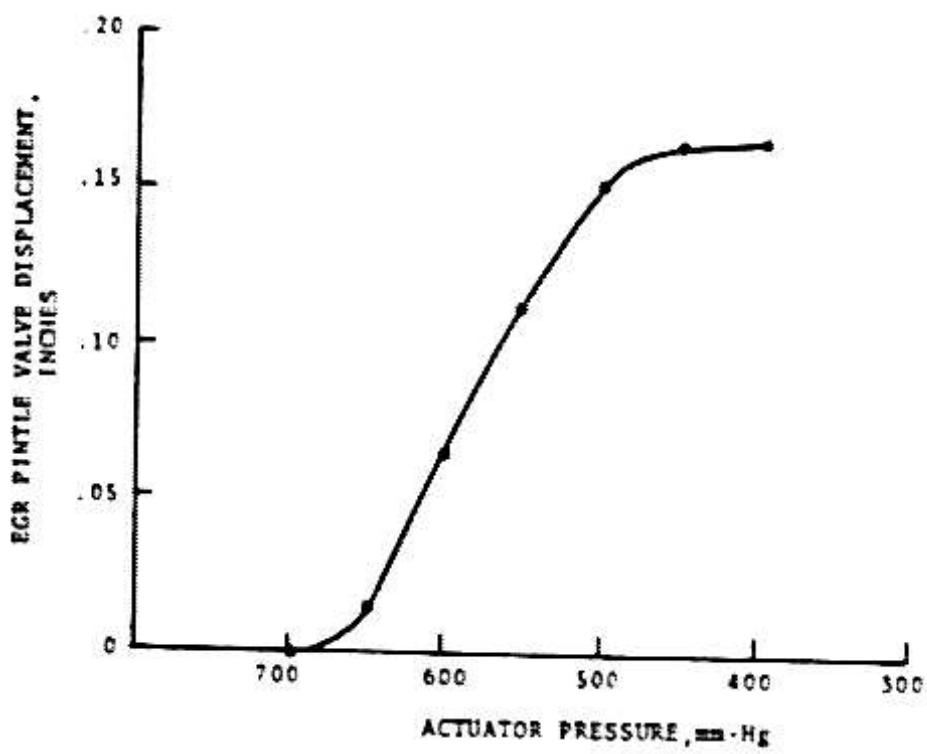


FIGURE 5. MEASURED VACUUM ACTUATOR

TABLE 5. STEADY-STATE BASELINE MAPPING SPEED-TORQUE VALUES

ENGINE SPEED, (RPM)	TORQUE, (LB-FT)						
750	-7,	0,	40				
880	-10,	-2,	0,	40,	80,	120,	160
1000	-15,	-10,	-5,	0,	40,	80,	120,
1500	-25,	-10,	-5,	0,	40,	80,	120,
2000	-32,	-7,	0,	40,	80,	120,	160
2500	-30,	-10,	0,	40,	80,	120,	160
3000	-40,	-30,	-10,	0,	40,	60,	80,
3600	-48,	-36,	-12,	0,	35,	70,	105,
							140

3. TEST PROCEDURES

All instrumentation was periodically checked and calibrated. Emissions instrumentation was calibrated before and after data collection at each power point, or every 30 minutes whichever came first. At each baseline test point, sufficient time was allowed for the operating temperatures and pressures to stabilize (typically 5 to 45 minutes) before data was collected. Each data channel was sampled 30 times in approximately 5 seconds and the data averaged.

4. DATA REDUCTION

The data acquisition system sampled the signals from the analog instrumentation shown in Table 4. All samples in each data set (speed/load point) were averaged and stored on the HP-21MX disc. Test variables were also displayed to the operator by video display and line-printer. The stored disc-data was subsequently transferred to magnetic tape for further reduction on a PDP-10 computer. All corrections to the test data and mass emissions and fuel-flow calculations were performed on the PDP-10 using the equations shown in Appendix A.

All corrected data was printed out in the format of Table B-1. The data includes Engine Speed (RPM), Load (torque) (lb-ft), the EGR, the Air-Fuel Ratio (A/F) from exhaust and fuel-flow calculations, the exhaust concentrations (ppm or %), various temperatures and pressures, measured air flow to the engine (airfio), the humidity (grains/pound), and fuel flow (grams/second).

5. TEST RESULTS

Figure 6 gives the corrected bhp, torque, and BSFC at wide-open throttle (WOT) for this engine. The maximum horsepower and torque are within manufacturer's specifications. The BSFC is at a minimum at 1500 rpm. Figure 7 shows the fuel rate in lb/hr for all speed-load points. Figures 8 and 9 show the A-F ratio and EGR for all positive horsepower points. Distributor advance is shown in Figure 4, while CO, HC, and NOx mass emission rates as a functions of speed and load are given in Figures 10, 11, and 12.

At low power a lean mixture resulted in relatively low levels of CO and HC. At higher speeds and loads, with richer mixtures, CO and HC increased substantially. As expected, NOx increased at higher power points. In general, the results from this engine are typical of those obtained with a modern production engine. Attached as Appendix B is the test and reduced data for all positive and negative speed/load points.

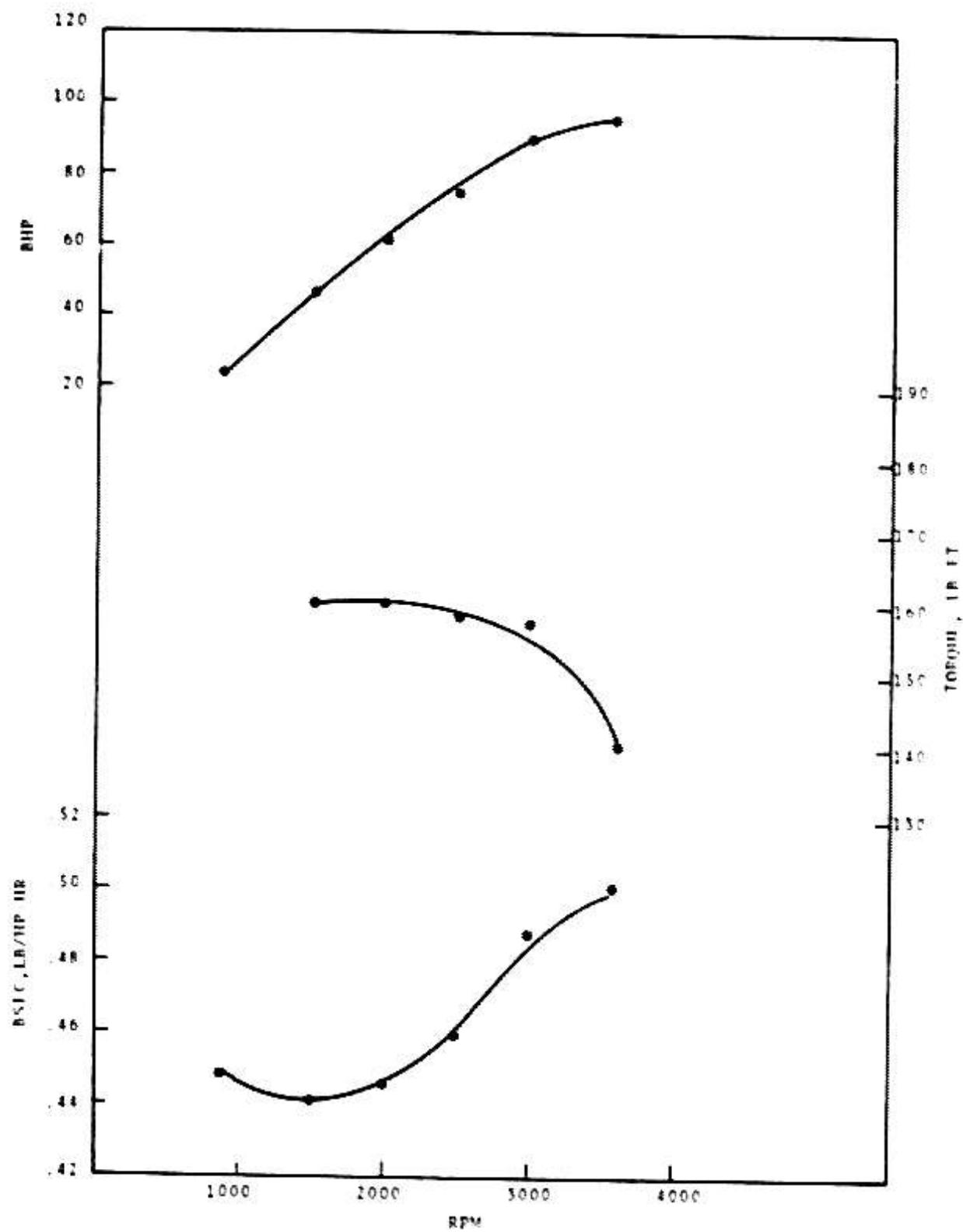


FIGURE 6. CORRECTED BHP, TORQUE, AND BSFC VS WIDE-OPEN THROTTLE

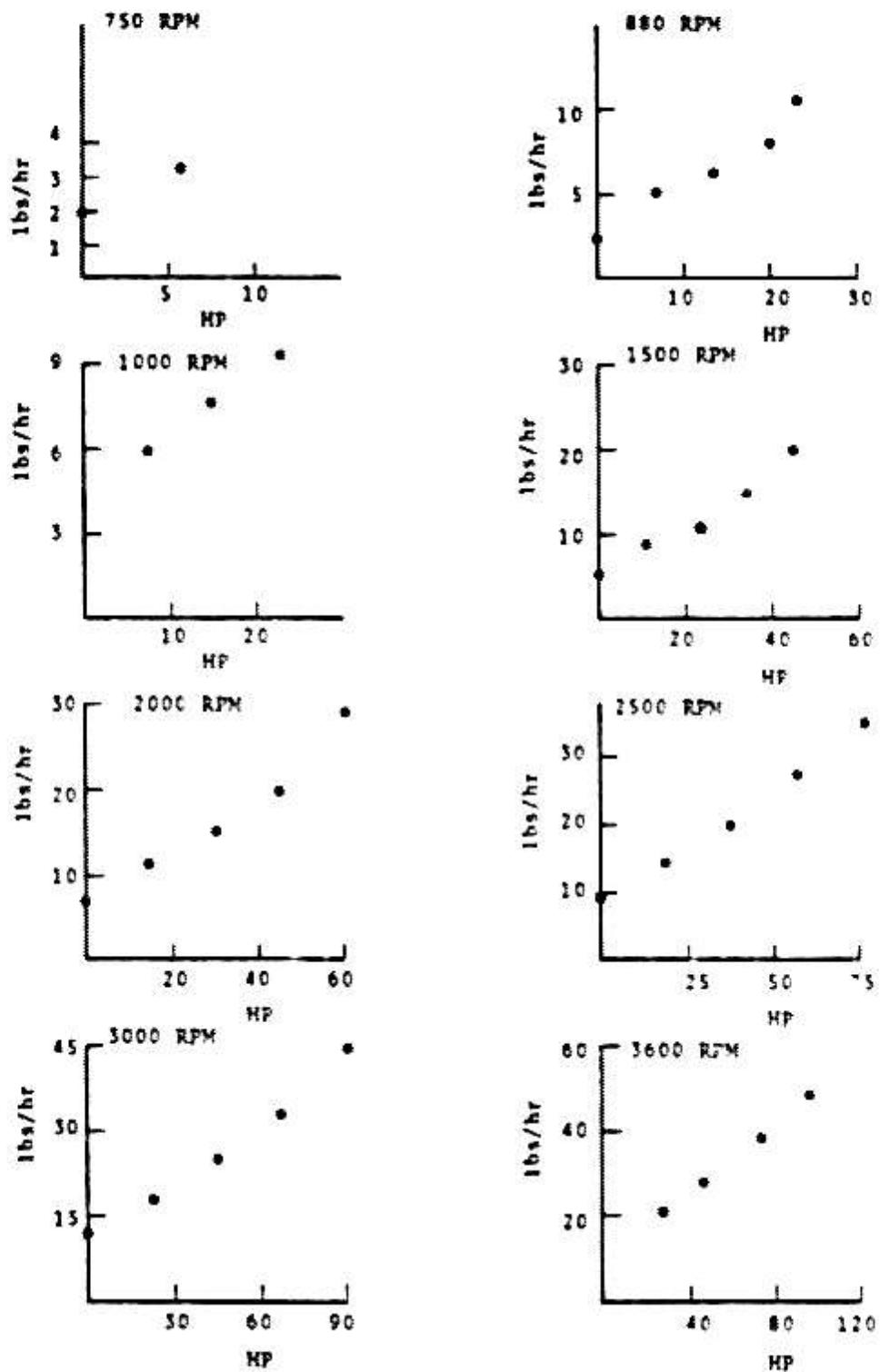


FIGURE 7. FUEL CONSUMPTION VS POWER

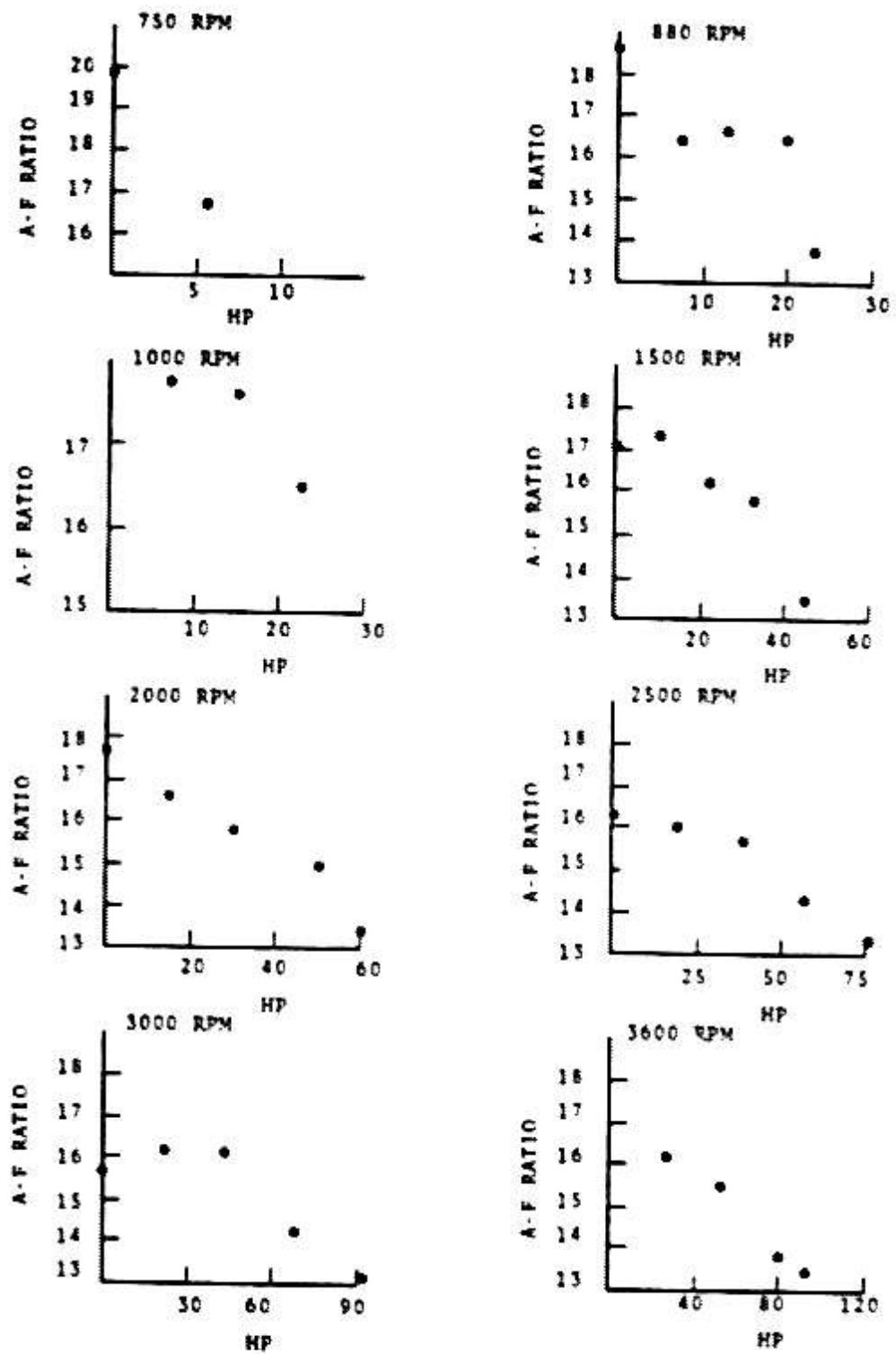


FIGURE 8. AIR-FUEL RATIO VS POWER

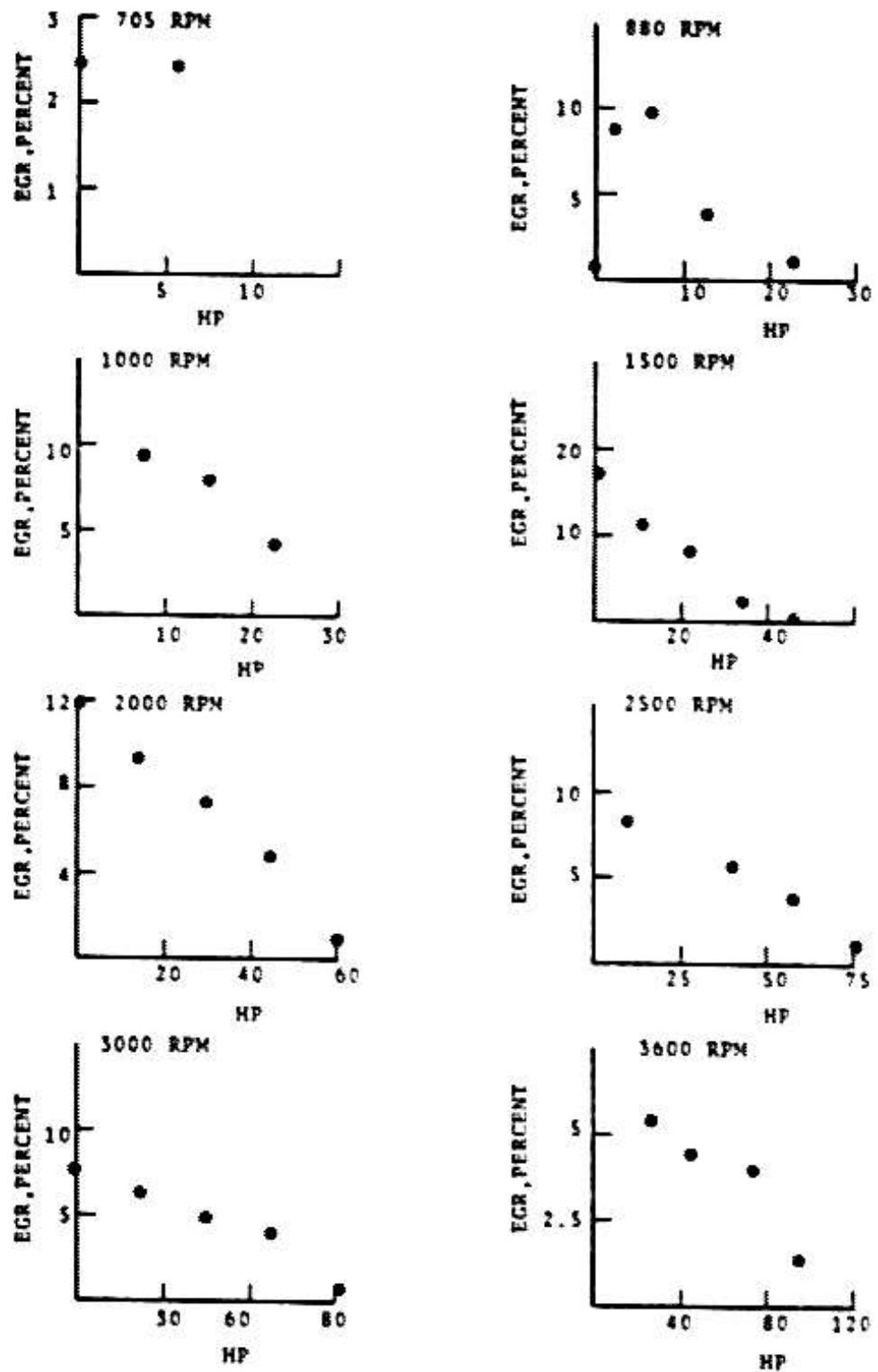


FIGURE 9. EGR VS POWER

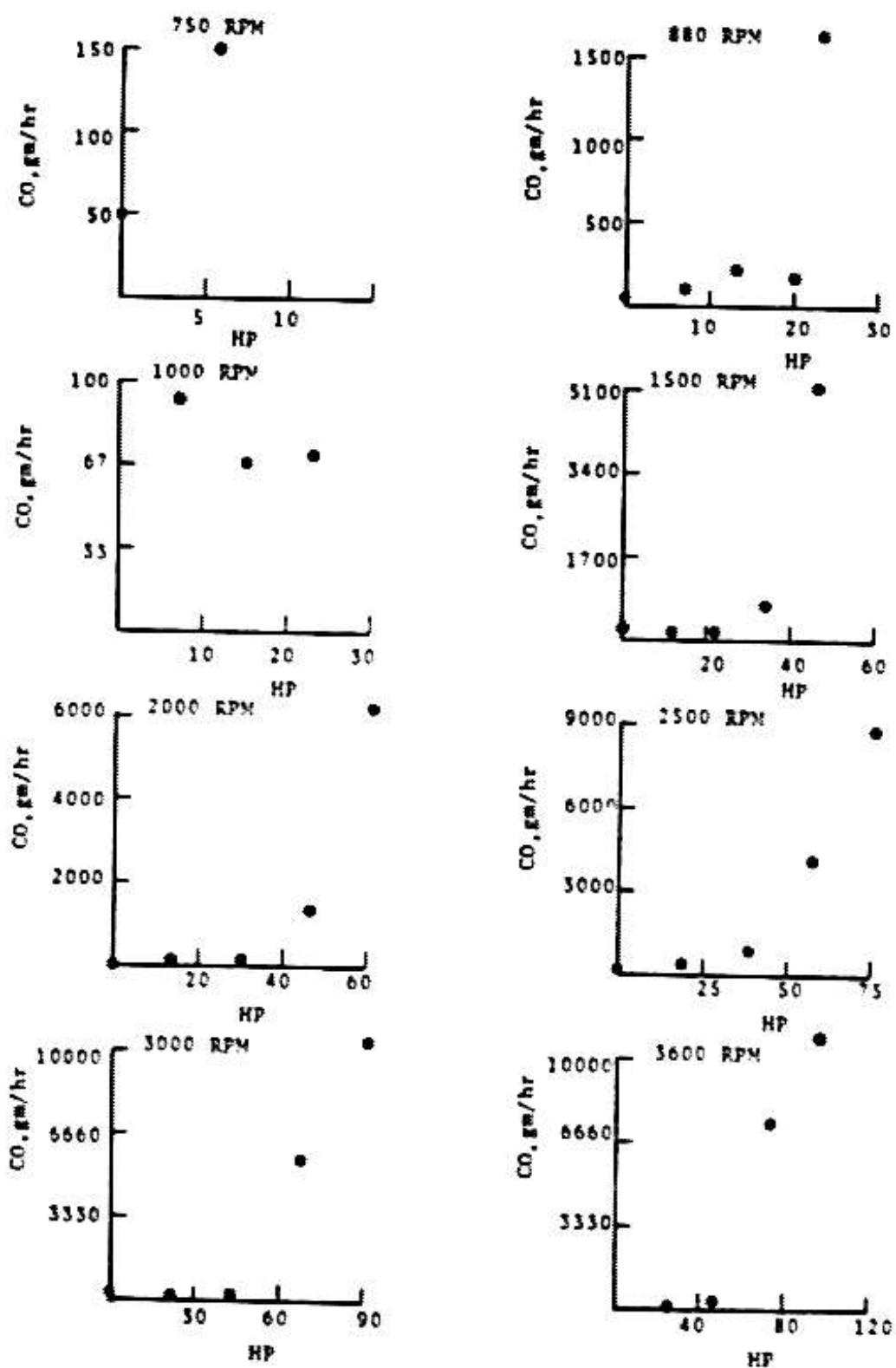


FIGURE 10. CO VS POWER

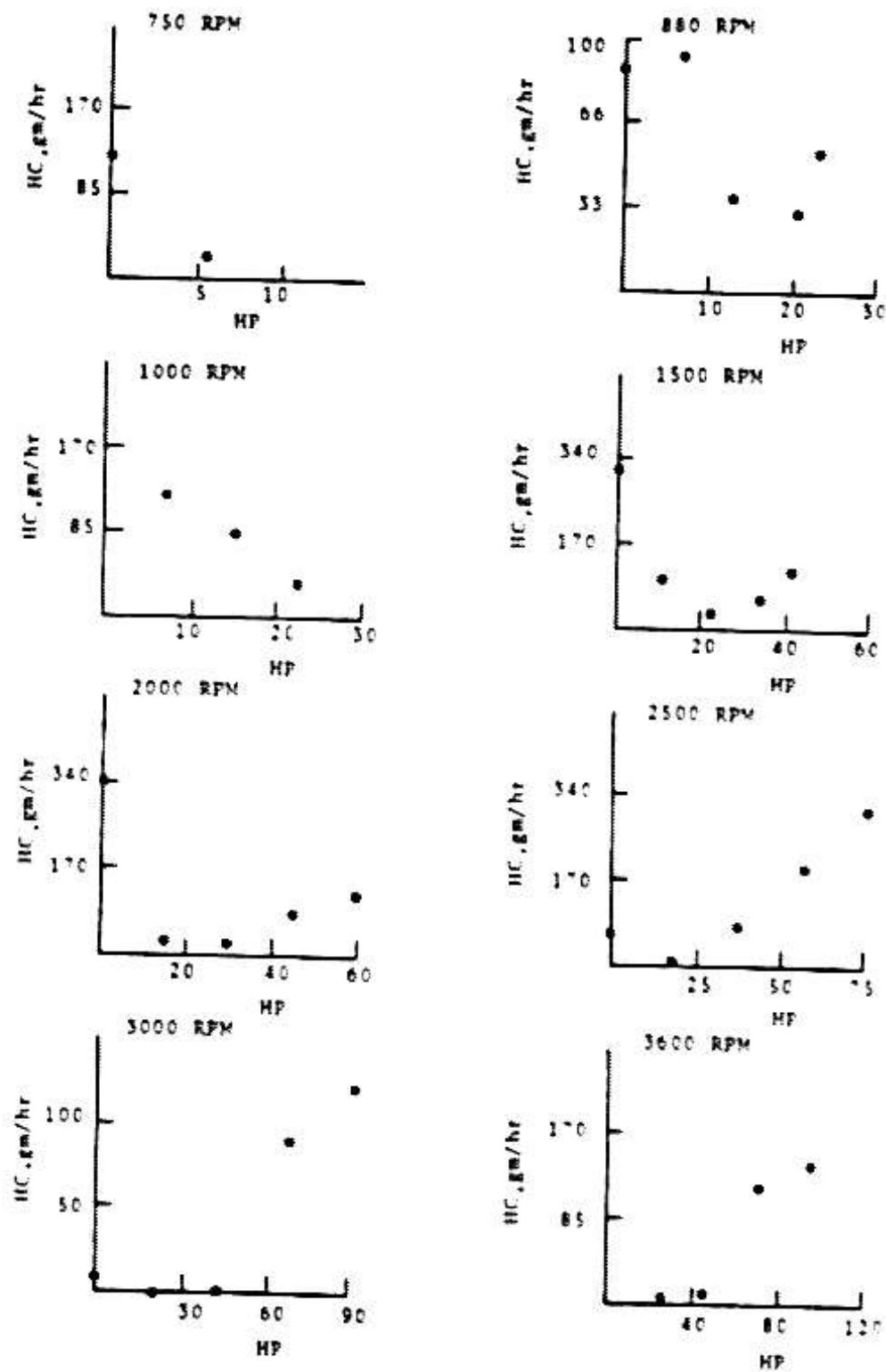


FIGURE 11. HC VS POWER

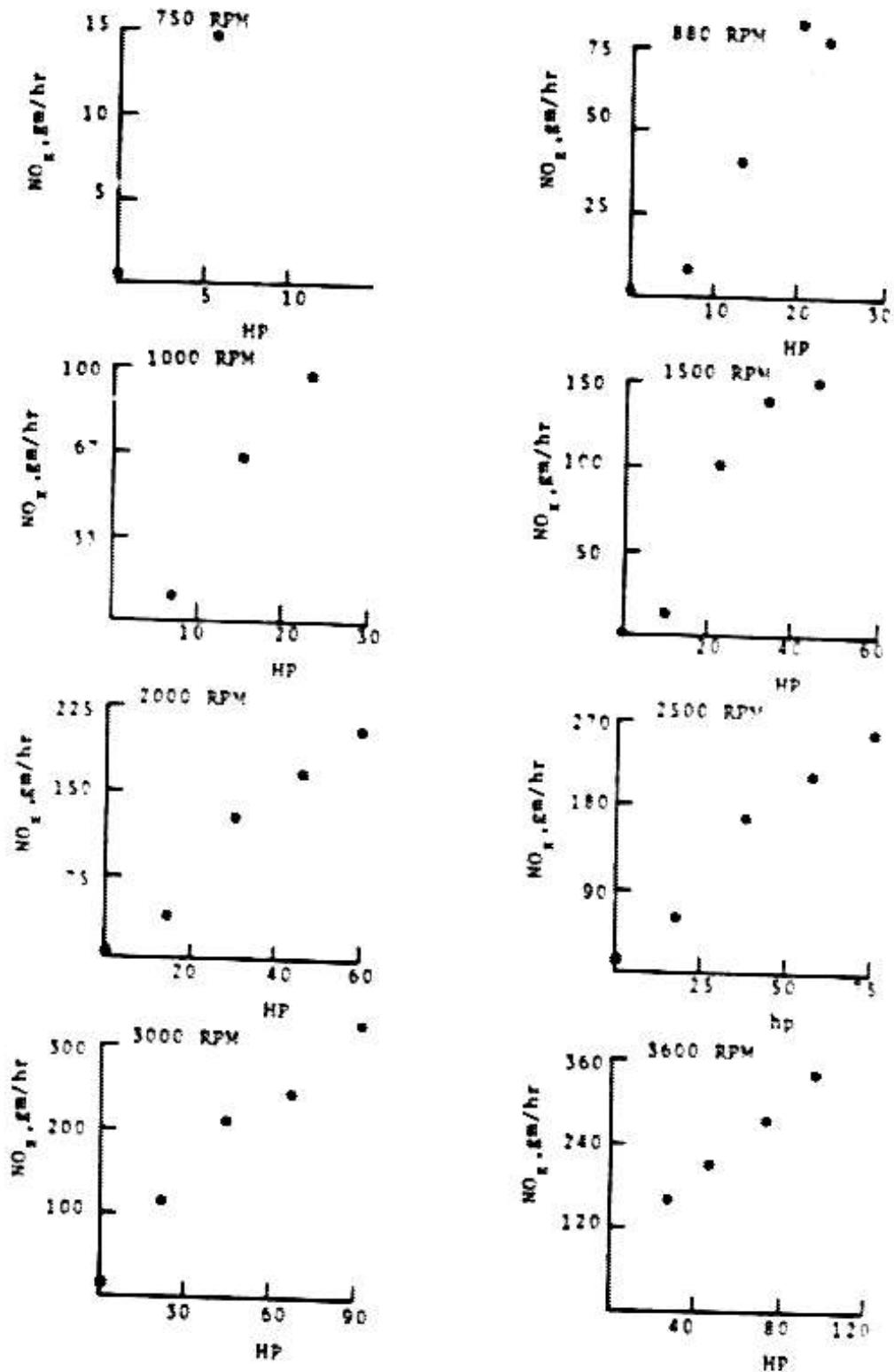


FIGURE 12. NO_x VS POWER

APPENDIX A DATA REDUCTION EQUATIONS

CORRECTED TORQUE, T_c (lb·ft)⁽¹⁾ From SAE J245, Spark Ignition Engine Rating Code, adjusted to standard SAE ambient conditions

$$T_c = \frac{P_d}{P_{dt}} \left(\frac{T_t \cdot A}{T^* \cdot A} \right)^{1/2} T_t$$

where

P_d = Standard Dry Barometric Pressure (29.00 in. Hg, 97.9 kPa)

P_{dt} = Dry Barometric Pressure at Test Conditions

T_t = Ambient Air Temperature at Test Conditions

T^* = Standard Ambient Temperature (85°F, 29.4°C)

A = Absolute Temperature Constant (460°F, 273°K)

T_t = Measured Torque at Test Conditions.

CORRECTED HORSEPOWER, hp_c ⁽¹⁾ From SAE J245, Spark Ignition Engine Rating Code, adjusted to standard SAE ambient conditions:

$$hp_c = \frac{T_c N}{G}$$

where

T_c = Corrected Torque (See above.)

N = Engine Speed (RPM)

G = Power Constant (3252 English, 955 SI).

(1)Engines with manifold preheated air inlet systems are designed to control carburetor air inlet temperature to a specific temperature. Excursions in ambient temperature below this value do not appreciably affect the controlled temperature. The engine performance correction factor as described in SAE J245 Engine Rating Code for Spark Ignition Engines has therefore been updated as follows: If ambient temperature is less than or equal to the manufacturer's stated controlled temperatures, no correction component involving carburetor inlet temperature is made. If ambient temperature exceeds the targeted controlled temperature, the normal J245 correction factor is applied with the targeted controlled temperature used in place of the standard ambient temperature.

MASS FUEL FLOW RATE (lb/hr) From volumetric measurement (corrected to 60°F per ASTM petroleum tables) and fuel specific gravity:

$$\dot{m}_f = \frac{(SpG)_f \left(\frac{lb H_2O}{lb vol} \right) (vol)_f}{\Delta t_T}$$

where

- \dot{m}_f = Fuel Flow Rate lb/hr
- $(SpG)_f$ = Specific Gravity of Fuel
- $(lb H_2O/lb vol)$ = Pounds of Water per Unit Volume
- $(vol)_f$ = Volume of Fuel Measured, corrected to 60°F per ASTM petroleum tables
- Δt_T = Time interval of volume measurement (hrs.)

CORRECTED BRAKE SPECIFIC FUEL CONSUMPTION (BSFC) (lb/HP·hr)

$$BSFC_c = \frac{\dot{m}_f}{HP_c}$$

where

- $BSFC_c$ = Corrected Brake Specific Fuel Consumption
- HP_c = Corrected Horsepower
- \dot{m}_f = Mass Fuel Flow Rate (lb/hr).

AIR/FUEL RATIO (A/F) Based on emissions measurements from SPINDT, SAE #650507:

$$A/F = F_b \left[11.492 F_c \left(\frac{1+R/2+Q}{1+R} \right) + \left(\frac{120(1-F_c)}{3.5+R} \right) \right]$$

where

- $R = \frac{\%CO}{\%CO_2}$ = Percent CO Concentration
- $\%CO_2$ = Percent CO_2 Concentration
- F_c = Mass Fraction of Carbon in Fuel
- $F_b = \frac{\%CO + \%CO_2}{\%CO + \%CO_2 + \%CH}$
- $Q = \frac{\%O_2}{\%CO_2}$ = Percent O_2 Concentration
- $\%CO_2$ = Percent CO_2 Concentration.

CARBON MONOXIDE (CO) MASS EMISSION RATE (Grams/Hr)

$$\text{MASS CO} = (4.383) (\dot{m}_f) (A/F+1) (\% \text{CO})$$

$$\left[\frac{1}{1 + 0.03148 (\% \text{CO}_2) \left(\frac{\% \text{CO} + \% \text{CO}_2}{\% \text{CO} - 3\% \text{CO}_2} \right)} \right]$$

where

 \dot{m}_f = Mass Fuel Flow Rate

A/F = Air-to-Fuel Ratio

 $\% \text{CO}$ = Percent CO, Concentration $\% \text{CO}_2$ = Percent CO₂, Concentration.HYDROCARBON (HC) MASS EMISSION RATE (Grams/Hr)

$$\text{Mass HC} = (0.0002207) (\dot{m}_f) (A/F+1) (\text{ppm HC})$$

where

 \dot{m}_f = Mass Fuel Flow Rate

A/F = Air-to-Fuel Ratio

ppm HC = Parts per Million, HC Concentration.

OXIDES OF NITROGEN (NOx) MASS EMISSION RATE (Grams/Hr)

$$\text{Mass NOx} = 0.007201 (\dot{m}_f) (A/F+1) (\text{ppm NOx})$$

$$\left[\frac{1}{1 + .03148 (\% \text{CO}_2) \left(\frac{\% \text{CO} + \% \text{CO}_2}{\% \text{CO} - 3\% \text{CO}_2} \right)} \right]$$

where

 \dot{m}_f = Mass Fuel Flow Rate

A/F = Air-to-Fuel Ratio

ppm NOx = Parts per Million, NOx Concentration

 $\% \text{CO}$ = Percent CO, Concentration $\% \text{CO}_2$ = Percent CO₂, Concentration K_H = Humidity Correction Factor.HUMIDITY CORRECTION FACTOR

$$K_H = \frac{1}{1 - .0047 (\text{Absolute Humidity} - 75)}$$

where absolute humidity is in grains/pound of dry air.

ABSOLUTE HUMIDITY (AH) (Grains/Lb.Dry Air):

$$AH = \frac{(RH) P_{SU}}{1.608 (P_{AMB} - RH \cdot P_{SU})}$$

where

RH = Measured Relative Humidity

P_{SU} = Saturated Vapor Pressure (from Keenan and Keyes Steam Tables)

P_{AMB} = Ambient Barometric Pressure.

APPENDIX B
CHRYSLER 225 CID TEST DATA AND REDUCED TEST DATA

EXPLANATION OF DATA COLUMN HEADINGS

RPM	
CTORQ LB·FT	Torque corrected to standard conditions, SAE, J245
EGR	
A/FEX	Air-to-Fuel ratio Calculated using Spindt method
A/FTC LB/HR	Air-to-Fuel ratio Calculated using total carbon method
CNOX PPM	Oxides of nitrogen corrected for relative humidity
HC PPM	Hydrocarbons, parts per million
CO	Carbon monoxide, percent
CO2	Carbon dioxide, percent
O2	Oxygen, percent
COOLE DEG-F	Exit engine coolant to heat exchanger
EXOIL DEG-F	Exit engine oil to heat exchanger
AIRDP INH2O	Pressure differential across Merriam airflow sensor
AIRFLO LB/MIN	Engine inlet airflow
ABSHUM GR/LB	Grains per pound
FLO G/S	Measured fuel flow, grams per second

Note: Runs 1-85 were taken during warm-up.

